



# Cosmic Explorer Beamtube Experiment (CEBEX)



## Optimized Corrugated Tube Design

Beamtube Workshop #3 (29 Sep – 2 Oct 2025)



Artist: Eddie Anaya (Cal State Fullerton)

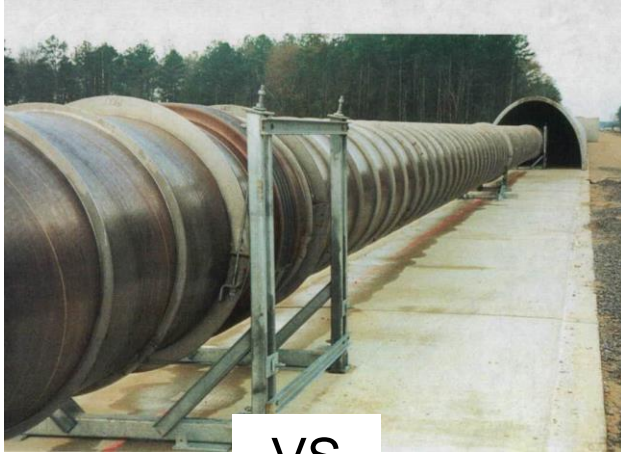
LIGO-G2502143-v1

Caltech  
30 Sep 2025 v1  
Alberto Franco-Ordovas

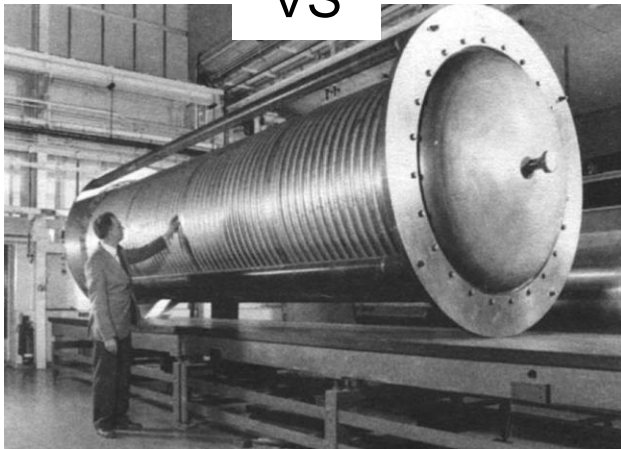
# Scope

- ☐ Corrugated design: Why?
- ☐ Convolution geometry: Parameters & Selection
- ☐ FEA Model:
  - ☐ Geometry & Python Script
  - ☐ BC's + Set Up
  - ☐ Results
- ☐ Parametric runs
- ☐ Design Explorer (DX)
  - ☐ Results + Response Surfaces
- ☐ Optimization

# Corrugated Tube: WHY!?



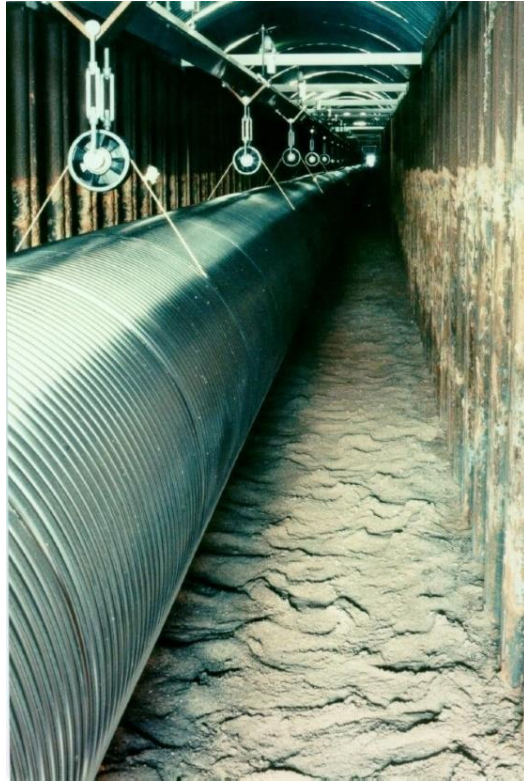
VS



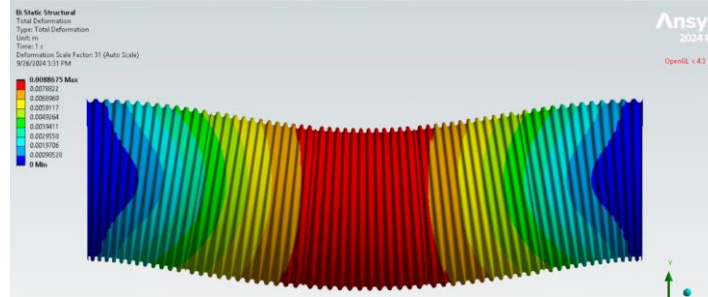
- ❑ Original LIGO:
  - ❑ 3.2 [mm] wall thickness x 20 [m] x 1.2 [m]
  - ❑ Stiffener rings (welded)
  - ❑ Expansion joints (40 [m])
- ❑ Corrugations could help with:
  - ❑ Improving buckling response (NO stiffener rings)
  - ❑ Absorbing thermal expansion (NO/Less exp. Joints)
  - ❑ Thinner wall
- ❑ 90's vs 2020's: better computational & manufacturing processes to help exploring



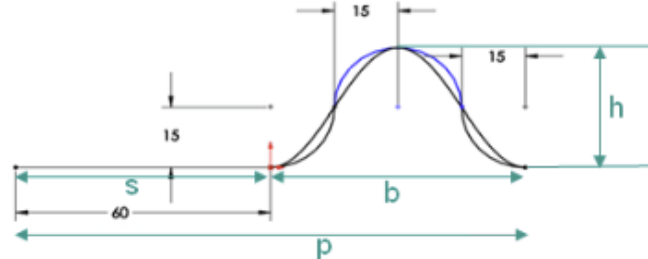
# Previous Corrugated Tubes



GEO 600



- ☐ GEO 600: Hung from the ceiling → For 40kms?!?!
- ☐ Supports are also to be taken into account
- ☐ Continued corrugation: too slinky if supports are not placed close enough
- ☐ Trade-off: singular convolution modules: Flat span + Corrugation



# Convolution Geometry: Shape?

□ Deflection (Sag):  $y = -\frac{FL^3}{48EI}$

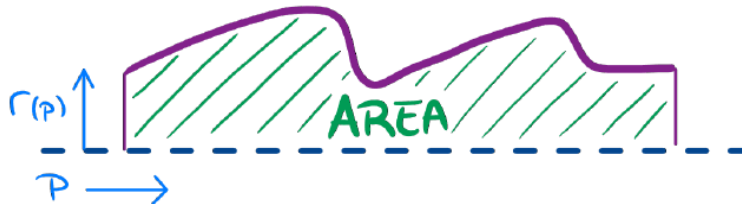
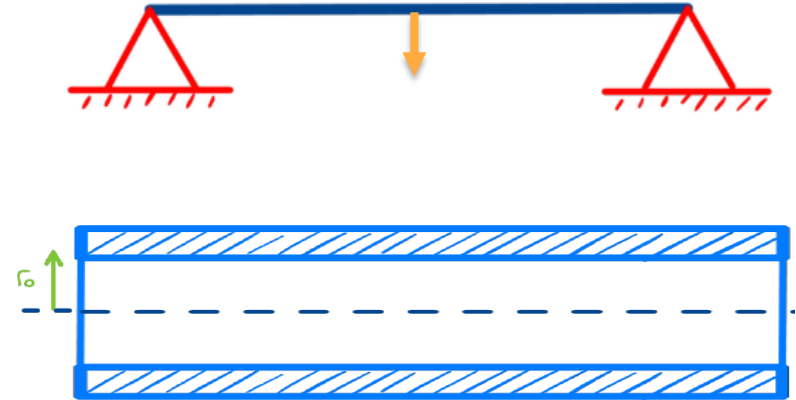
□ Global Buckling (Euler):  $P_{cr} = \frac{\pi^2 EI}{(KL)^2}$

□ For Thin Wall Members:

$$I = \pi R_m^3 t$$

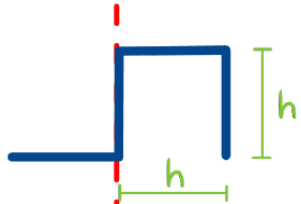
·  $R_m$  : Mean radius

·  $t$  : Thickness

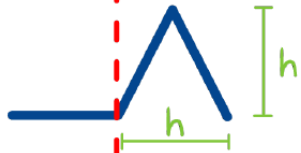


$$\bar{r} = \frac{\text{AREA}}{\text{LENGTH}} = \frac{\int_0^P r(p) \cdot dp}{L_{\text{Total}}}$$

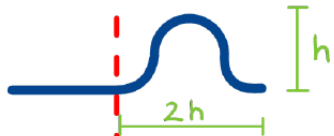
# Convolution Geometry: Sinusoidal



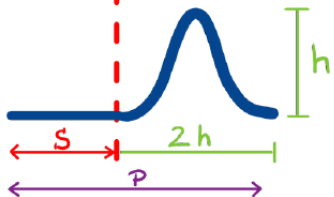
$$\bar{r}_{Sq} = \frac{6 \cdot P + h^2}{S + 3h} = 445 \text{ [mm]} \rightarrow \xrightarrow{-22\%} \xrightarrow{-80\%} \frac{R}{I (\propto R^3)}$$



$$\bar{r}_{Tri} = \frac{6 \cdot P + \frac{h^2}{2}}{S + \sqrt{5} \cdot h} = 507 \text{ [mm]} \rightarrow -7\% \rightarrow -22\%$$



$$\bar{r}_{Cir} = \frac{6P + h^2}{S + \pi \cdot h} = 519 \text{ [mm]} \rightarrow -4\% \rightarrow -14\%$$



$$\bar{r}_{Sin} = \frac{6 \cdot P + h}{87.82^*} = 542 \text{ [mm]} \#$$

□ Numerical values for:

- $h = 30 \text{ [mm]}$
- $P = 120 \text{ [mm]}$

□ The sinusoidal corrugation provides a better performance

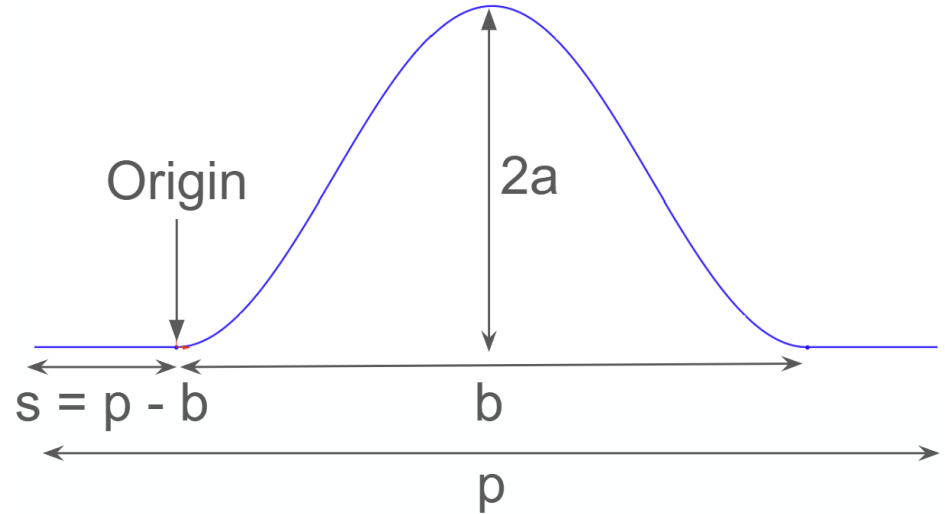
# Convolution Geometry

## ❑ Sinusoidal Corrugation Parameters:

- ❑ Amplitude:  $a$
- ❑ Period:  $b$
- ❑ Pitch:  $p$

## ❑ Beamtube Parameters:

- ❑ Thickness:  $t$
- ❑ Length:  $L$  (\*)



$$a * \{\sin[2 * \pi * x / b - \pi / 2] + 1\} \text{ from } 0 \text{ to } b$$

**4 Parameters:** How to find an “optimal” space? Through Parameter Correlation!

FEA Model

Input/Output Param

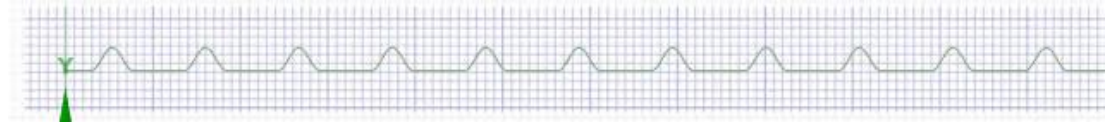
Design Points

Response Surface

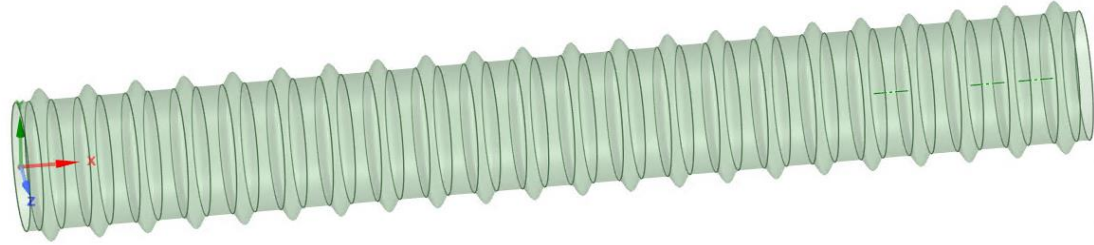
Optimization

# FEA Model: Geometry Definition

- Initial geometry: 2D Sketch



- Final geometry: 3D Shell

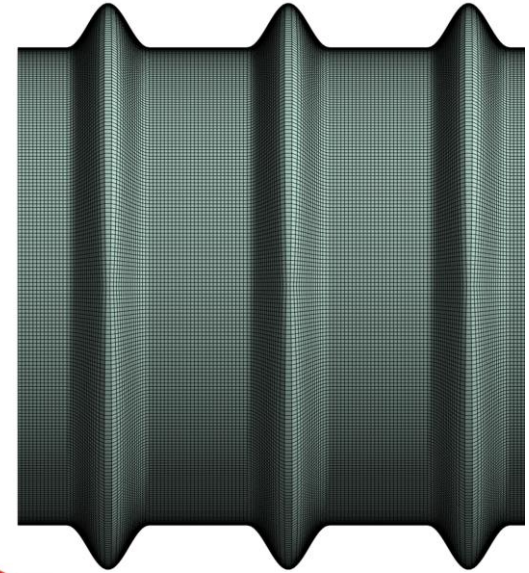


- A Python Script is needed to generate the geometry from scratch every run
- Otherwise, the input parameters ( $a$ ,  $b$ ,  $p$ ) cannot be modified once the 2D sketch has been revolved and became a 3D Shell body.



# FEA Model: BC's & Configuration

- ❑ Boundary conditions implemented:
  - ❑ Gravity + Bakeout (@ 150°C) + Inner Vacuum
  - ❑ All through “Named Selections”: automatization
- ❑ Mesh defined: 10x10 [mm]



## B: Static Structural

Static Structural

Time: 1. s

9/15/2025 3:13:54 PM

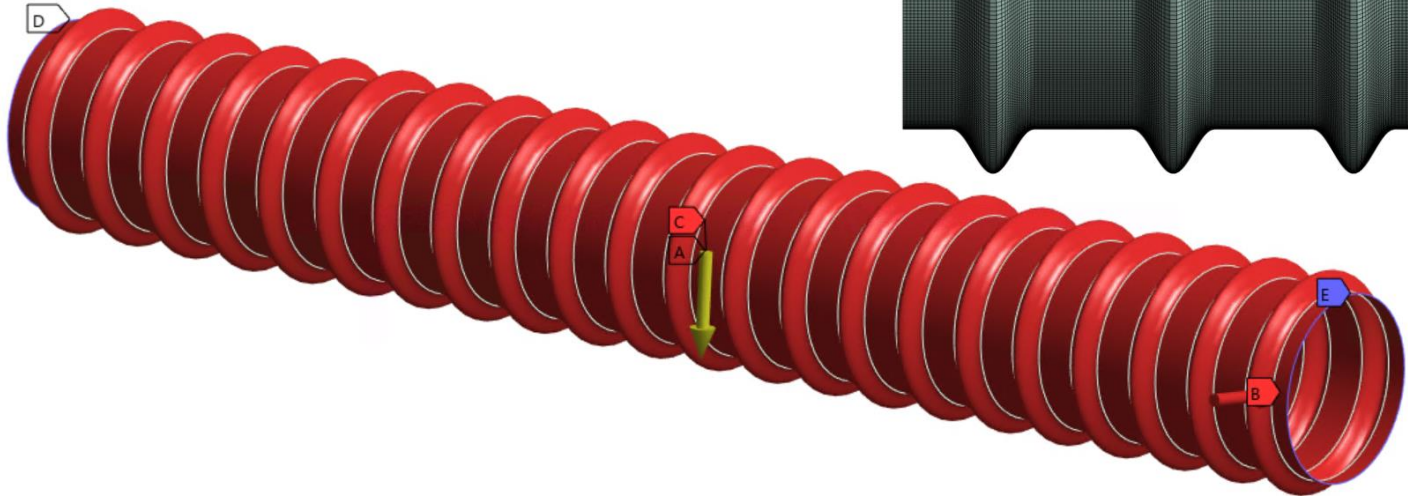
**A** Standard Earth Gravity: 9806.6 mm/s<sup>2</sup>

**B** Pressure: -0.1013 MPa

**C** Thermal Condition: Both: 150. °C

**D** Simply Supported X=0: 0. mm

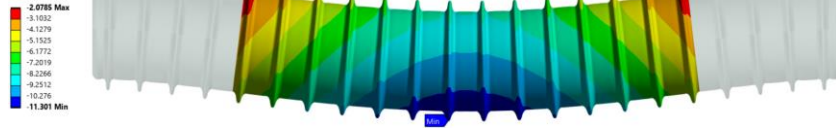
**E** Simply Supported X=END: 0. mm



# FEA Model: Results to observe

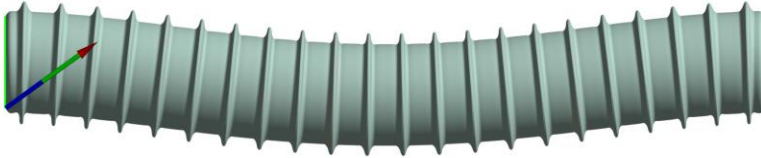
The following results were set as Output Parameters

B: Static Structural  
Sag (Y)  
Type: Directional Deformation(Y Axis)  
Unit: mm  
Global Coordinate System  
Time: 1 s  
9/15/2023 3:36:42 PM



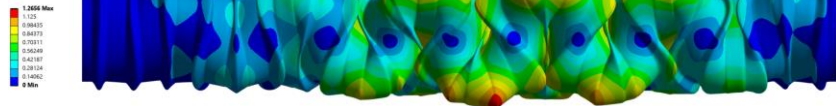
☐ Sag: Deflection at the beamtube center

B: Static Structural  
Force Reaction  
9/15/2023 3:37:44 PM



☐ Reaction Forces: X & Y axis at Supports

C: Eigenvalue Buckling  
Total Deformation  
Type: Total Deformation  
Load Multiplier (linear): 3.7193  
Unit: mm  
9/15/2023 4:02:23 PM

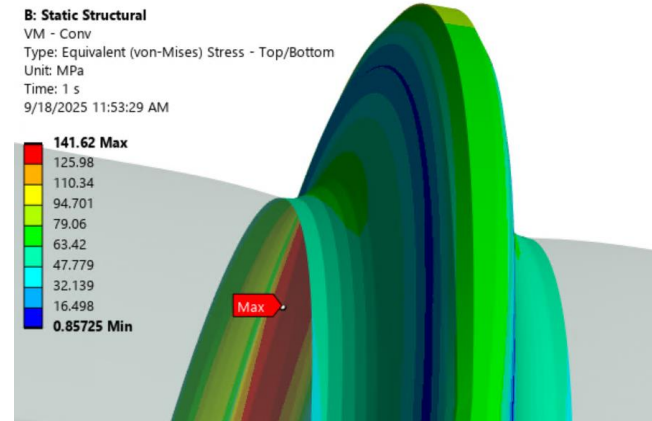
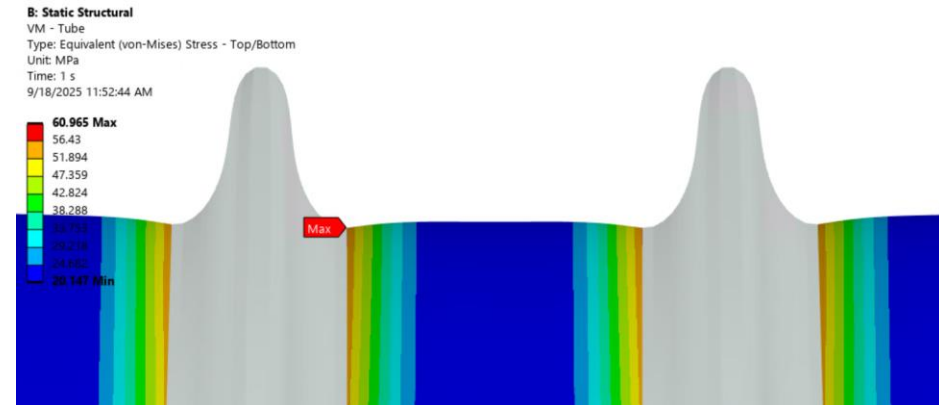
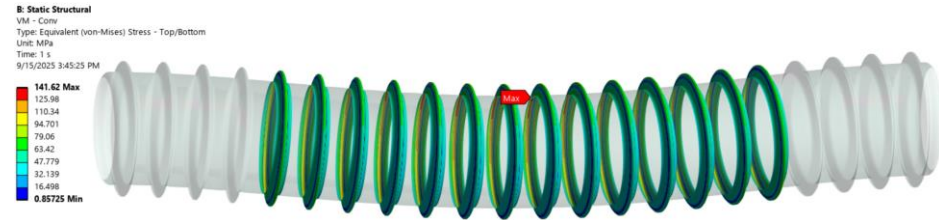
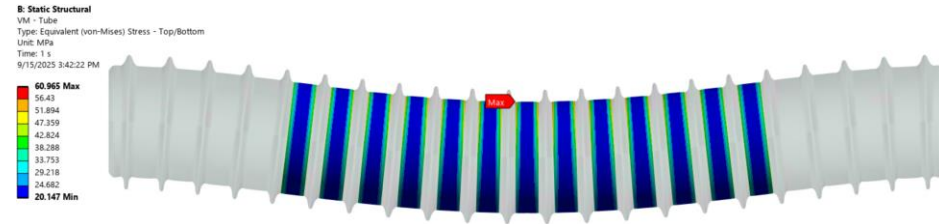


☐ Buckling Factor: Global buckling

# FEA Model: Results to observe

The following results were set as Output Parameters

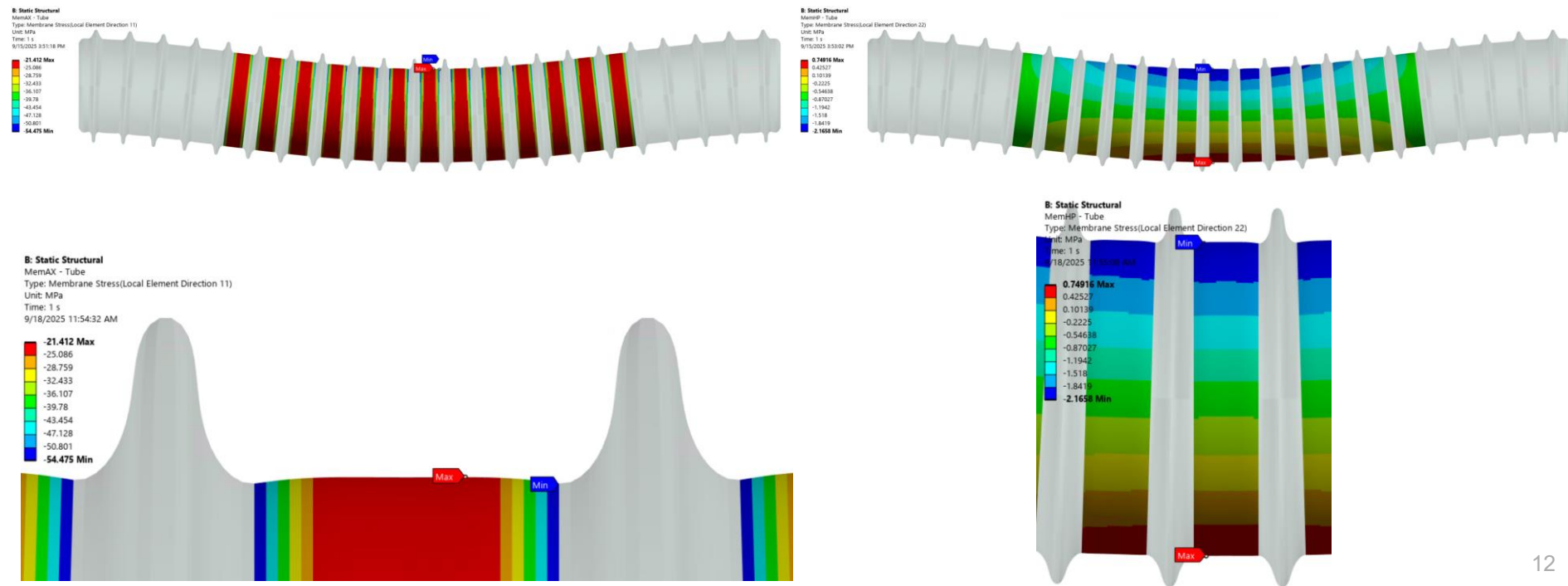
- Von-Mises: Max Stress at flats & convolutions, for a better understanding



# FEA Model: Results to observe

The following results were set as Output Parameters

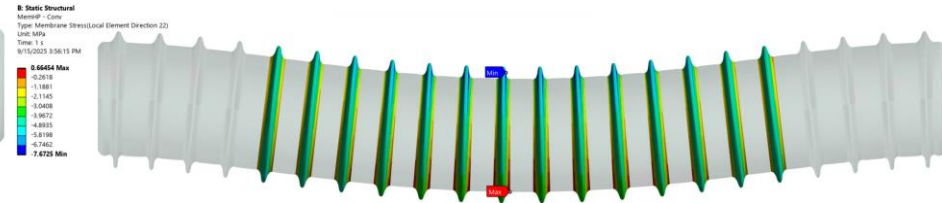
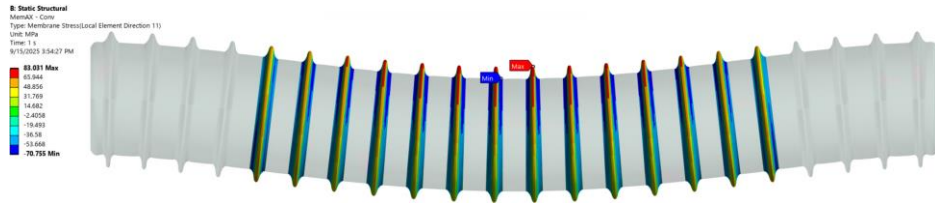
- Membrane: Max/Min at flats for Axial (left) and Hoop (right)



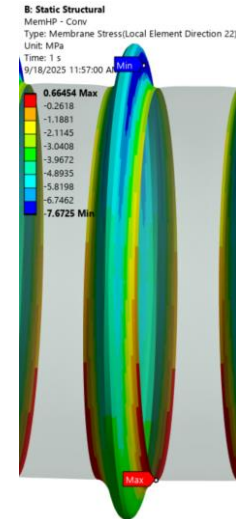
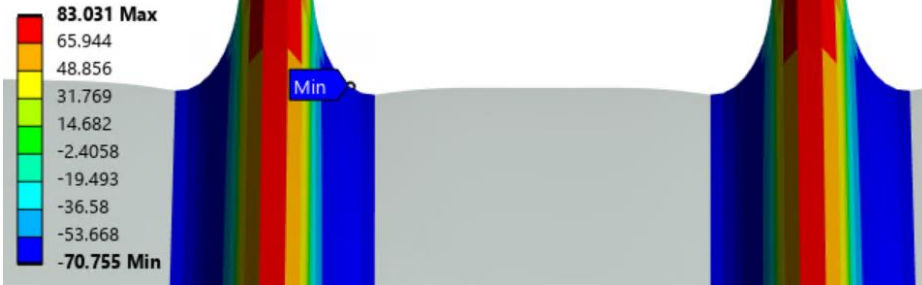
# FEA Model: Results to observe

The following results were set as Output Parameters

- Membrane: Max/Min at corrugations for Axial (left) and Hoop (right)



**B: Static Structural**  
MemAX - Conv  
Type: Membrane Stress(Local Element Direction 11)  
Unit: MPa  
Time: 1 s  
9/18/2025 11:56:27 AM





# FEA Model: Results to observe

The following results were set as Output Parameters

- ❑ To wrap-up: 14 output parameters total

RESULT	PARAMETERS	LOCATION
Sag (Y)	Min	Mid beamtube
Force Reaction	X, Y	Ends
Buckling Factor	Load Multiplier	Global
Von Mises – Tube	Max	Central flat: boundary w conv
Von Mises – Convolution	Max	Central conv: peak, inner face
Tube – Membrane Axial	Min, Max	Central flat
Tube – Membrane Hoop	Min, Max	Central flat
Conv – Membrane Axial	Min, Max	Central conv: peak & valley
Conv – Membrane Hoop	Min, Max	Central conv: valleys

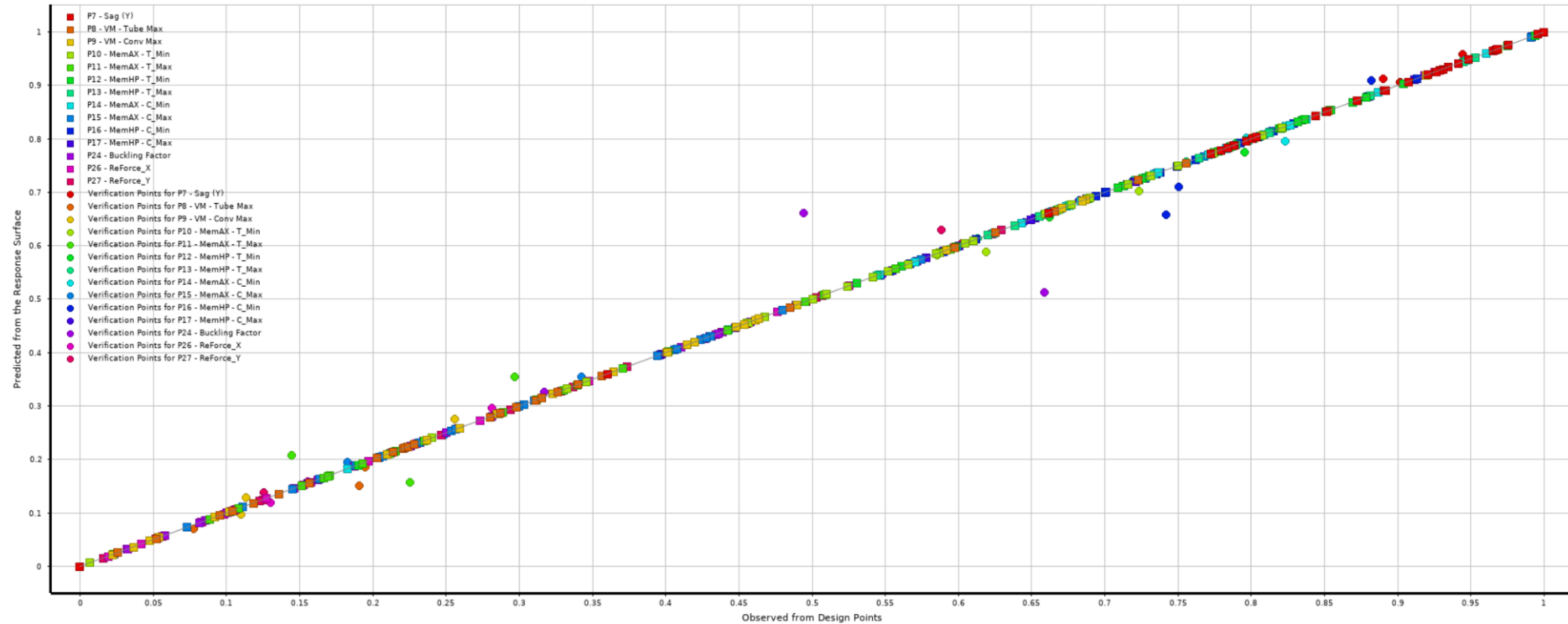
Outline of All Parameters				
	A	B	C	D
1	ID	Parameter Name	Value	Unit
2	Input Parameters			
3	Geometry (A1)			
4	P20	a	58	mm
5	P21	b	215	mm
6	P22	p	470	mm
7	Static Structural (B1)			
8	P25	t	2.7	mm
*	New input parameter	New name	New expression	
10	Output Parameters			
11	Static Structural (B1)			
12	P7	Sag (Y)	-11.301	mm
13	P8	VM - Tube Max	60.965	MPa
14	P9	VM - Conv Max	141.62	MPa
15	P10	MemAX - T_Min	-54.475	MPa
16	P11	MemAX - T_Max	-21.412	MPa
17	P12	MemHP - T_Min	-2.1658	MPa
18	P13	MemHP - T_Max	0.74916	MPa
19	P14	MemAX - C_Min	-70.755	MPa
20	P15	MemAX - C_Max	83.031	MPa
21	P16	MemHP - C_Min	-7.6725	MPa
22	P17	MemHP - C_Max	0.66454	MPa
23	P26	ReForce_X	7781.5	N
24	P27	ReForce_Y	5577.4	N
25	Eigenvalue Buckling (C1)			
26	P24	Buckling Factor	3.7195	

- ❑ Parameters Set: all parameters displayed
- ❑ Need to define range of study for input Par
  - ❑ a: 40 to 80 [mm], b: 120 to 240 [mm], p: 300 to 700 [mm]
  - ❑ t: 2 to 3.5 [mm]
- ❑ Need to define Design of Experiment Method
  - ❑ Central Composite Design: Face centered
  - ❑ This settles the way DX will select the Design Points to calculate so they properly cover the design space.
- ❑ Once all the DPs are done: Meta Model
  - ❑ Surface type: Kriging, variable.
  - ❑ This defines the algorithm used to study the correlation among DPs to create response surfaces (prediction)

# Design Explorer (DX) Results

## Goodness of Fit

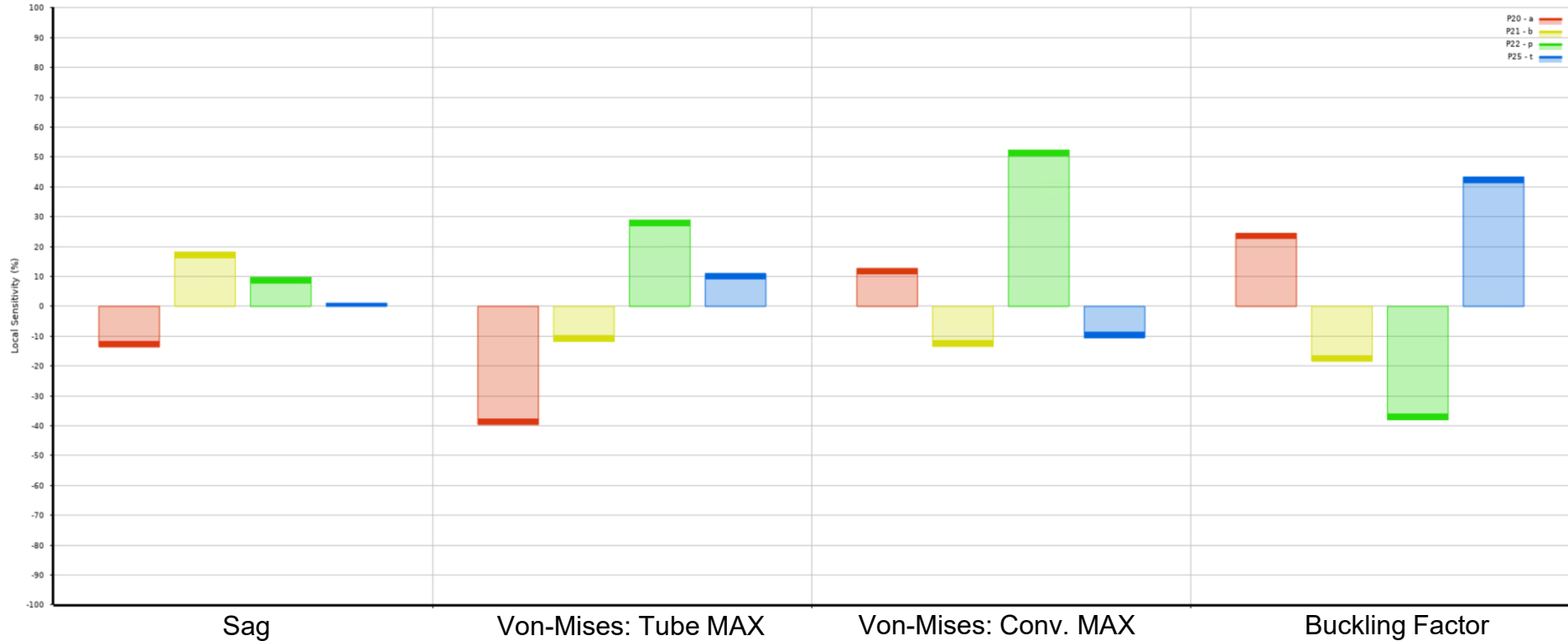
- Explores  $R^2$  in normalized values: Verification Points (X) vs. Predicted Points (Y)
- A few points off-axis: still a good & solid model



# Design Explorer (DX) Results

## Local Sensitivity

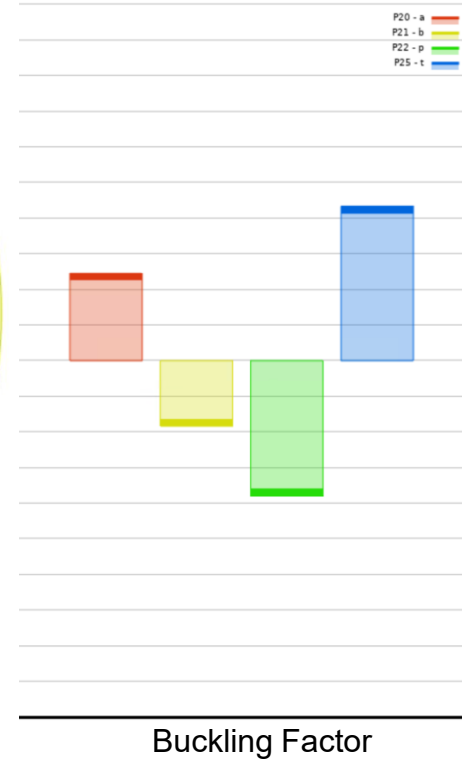
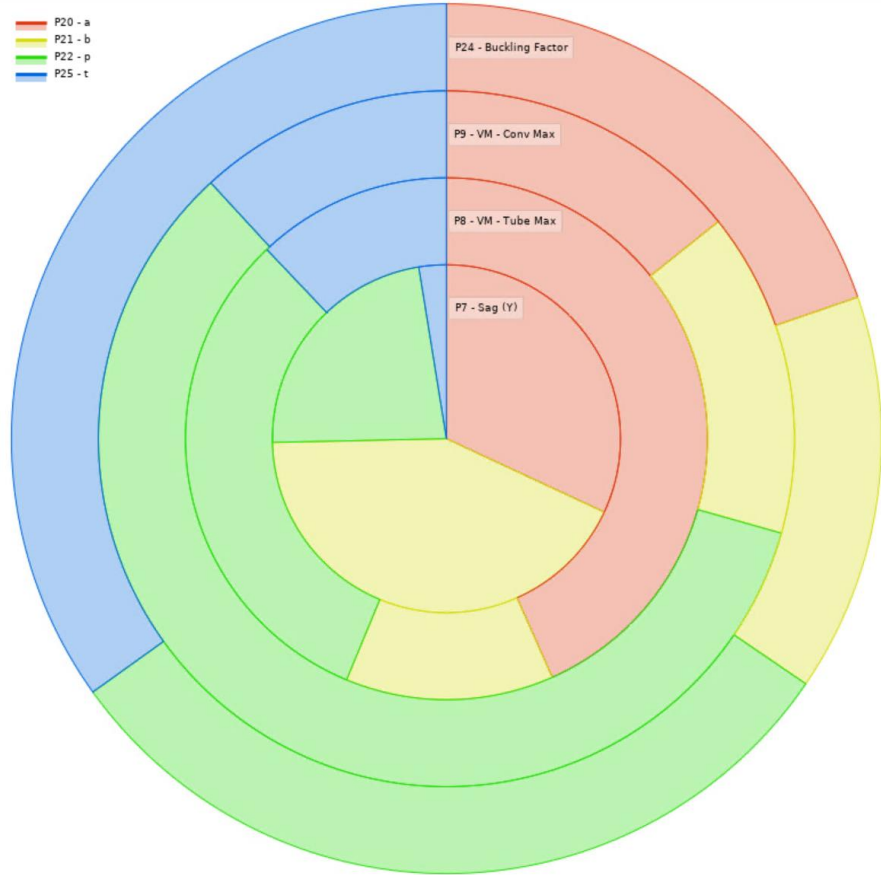
- Measures input parameters impact on output parameters
- Key: a (red), b (yellow), p (green), t (blue)



# Design Explorer (DX) Results

## Local Sensitivity

- Measures input pa
- Key: a (red), b (yel)

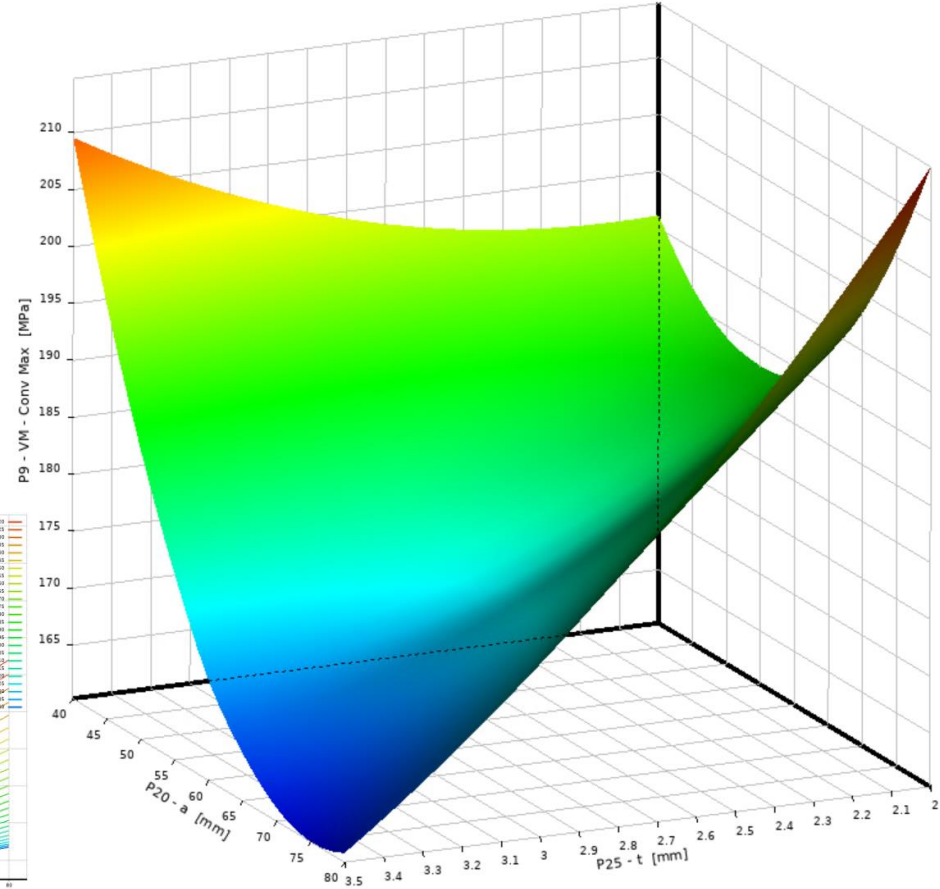
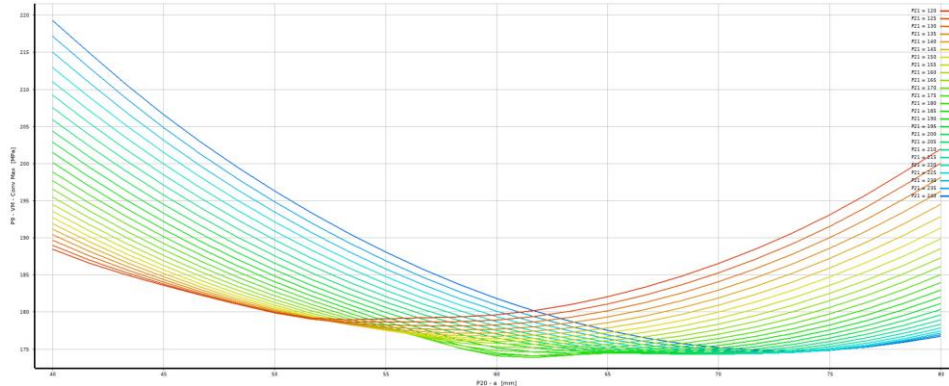




# Design Explorer (DX) Results

## Response Surfaces – 2D & 3D

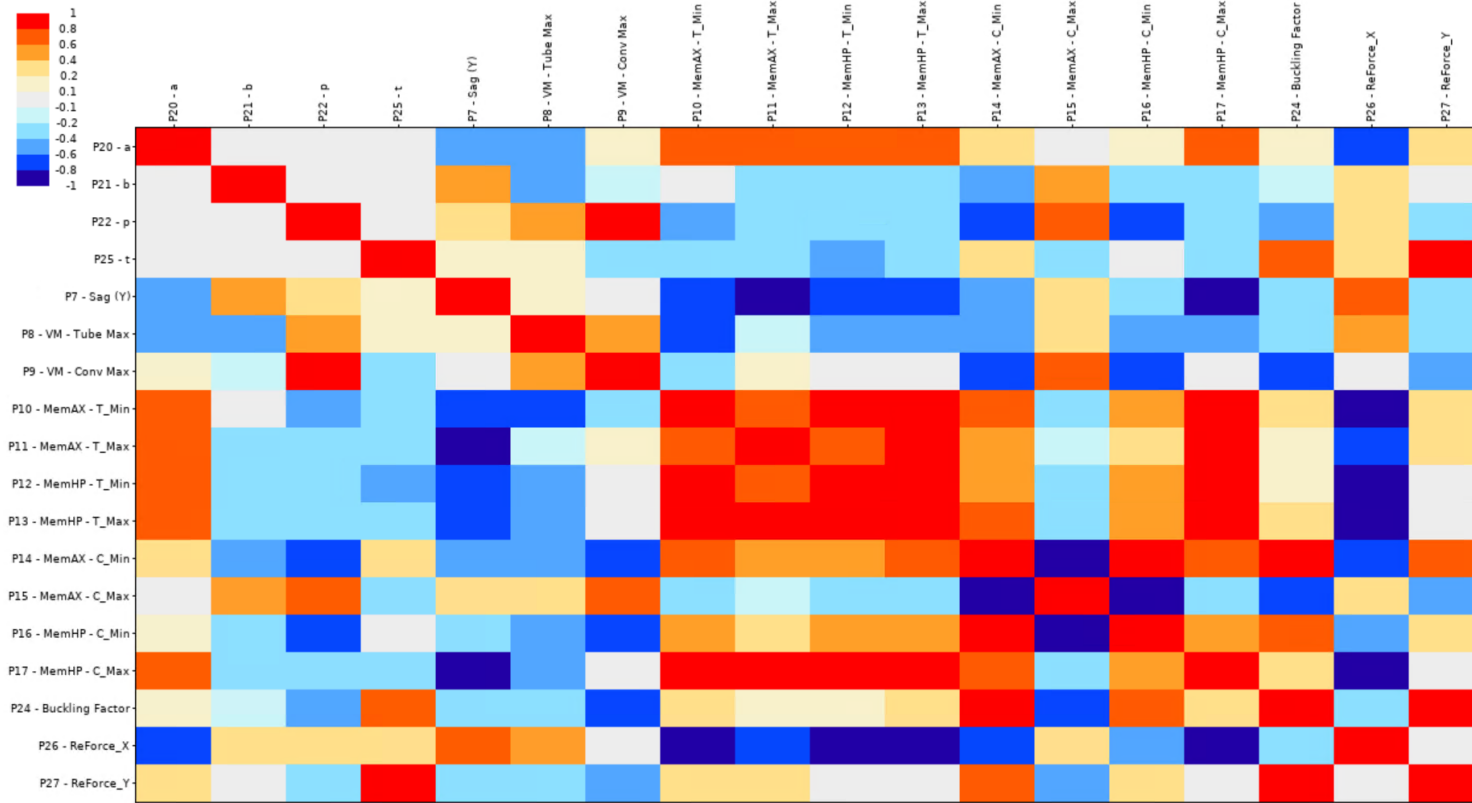
- ☐ Visual display to understand the behavior of the input/output parameters
- ☐ Allows 2 input (x, y) vs. 1 output (z)
- ☐ Rest of inputs: modifiable values
- ☐ Also, 2D graphs w isolines



# Design Explorer (DX) Results

## Correlation Matrix

Interaction between ALL parameters (output/output included! → Trade-offs)



# DX Optimization Tool

- ☐ Design Points & Verification Points calculated + Results checked: OK!
- ☐ DX will cross the Design Space and come with a set of Candidate Points
- ☐ Set the target values for the desired parameters:
  - ☐ Maximize Buckling Factor, consider values only  $> 3$ : Rule of thumb for ASME cod.
  - ☐ Minimize t: set target of 2 [mm] as it is the lowest value for t
  - ☐ Von-Mises  $< 138$  [Mpa]
  - ☐ Minimize Sag, consider values only  $< 10$  [mm]

	A	B	C	D	E	F	G	H	I
1	Name	Parameter	Objective			Constraint			
2			Type	Target	Tolerance	Type	Lower Bound	Upper Bound	Tolerance
3	Maximize P24; P24 $\geq 3$	P24 - Buckling Factor	Maximize ▾	10		Values $\geq$ Lower Bound ▾	3		0.001
4	Minimize P25	P25 - t	Minimize ▾	2		No Constraint			
5	Minimize P8; P8 $\leq 138$ MPa	P8 - VM - Tube Max	Minimize ▾	0		Values $\leq$ Upper Bound ▾		138	0.001
6	Minimize P9; P9 $\leq 138$ MPa	P9 - VM - Conv Max	Minimize ▾	0		Values $\leq$ Upper Bound ▾		138	0.001
7	Minimize P7; P7 $\geq -10$ mm	P7 - Sag (Y)	Minimize ▾	0		Values $\geq$ Lower Bound ▾	-10		0.001
*		Select a Parameter ▾							

# DX Optimization Tool

- 3 candidate points are chosen: Verification can be requested as part of the process
- In this case, all 3 pivot around the same Design Space area (very close values)

7	☐ Optimization Method			
8	MOGA	The MOGA method (Multi-Objective Genetic Algorithm) is a variant of the popular NSGA-II (Non-dominated Sorted Genetic Algorithm-II) based on controlled elitism concepts. It supports multiple objectives and constraints and aims at finding the global optimum.		
9	Configuration	Generate 100 samples initially, 100 samples per iteration and find 3 candidates in a maximum of 20 iterations.		
10	Status	Converged after 718 evaluations.		
11	☐ Candidate Points			
12		Candidate Point 1	Candidate Point 2	Candidate Point 3
13	P20 - a (mm)	46.841	46.967	47.253
14	P21 - b (mm)	199.3	199.3	199.3
15	P22 - p (mm)	350.99	353.71	366.27
16	P25 - t (mm)	★ 2.5468	★ 2.5469	★ 2.5481
17	P7 - Sag (Y) (mm)	★★★ -9.8403	★★★ -9.8484	★★★ -9.7225
18	P8 - VM - Tube Max (MPa)	★★★ 56.898	★★★ 57.002	★★★ 57.916
19	P9 - VM - Conv Max (MPa)	== 120.67	== 121.35	== 124.44
20	P24 - Buckling Factor	== 4.0941	== 4.0696	== 3.9314

- Verification Point:
- Sag: -9.87 [mm]
- VM Tube: 56.92 [MPa]
- VM Conv: 120.8 [MPa]
- Buckling: 4.4

# Corrugated Summary

- ☐ On paper: a reasonable solution for a corrugated tube is possible
  - ☐ Most likely through ASME Method B: elastoplastic qualification
  
- ☐ 4 or 5 parameters are still manageable to get a good parametric model, suitable for optimization
  
- ☐ Final definition of values depends on engineering & tolerancing
  - ☐  $t = 2.5468$  [mm] is not a suitable value for manufacturing...
  - ☐ Post-processing needed after optimization
  
- ☐ Manufacturability of the corrugated beamtube to be studied by vendors as per RFI launched last month.