

CE/ET BEAMTUBE WORKSHOP III

Mechanical/Optical Design Options for Baffles

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24/08/2025

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INTRODUCTION

OVERVIEW OF THE TALK

INTRODUCTION



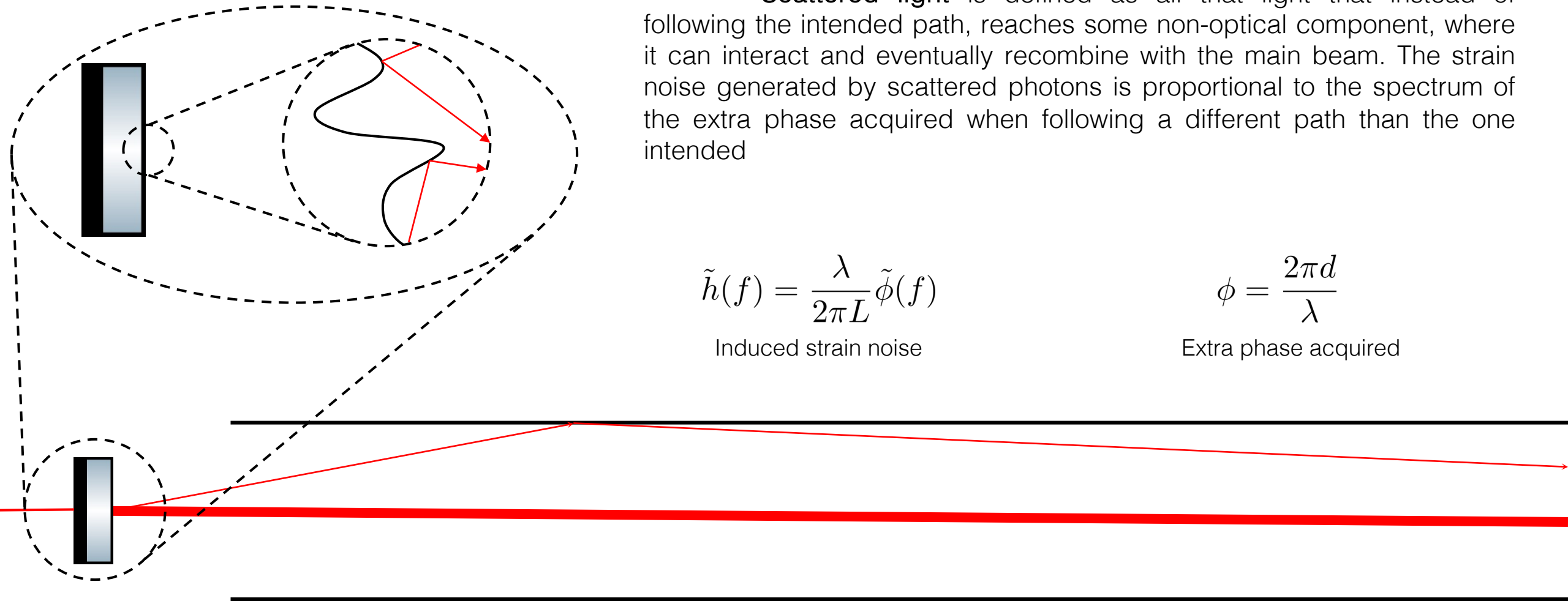
Scattered light is defined as all that light that instead of following the intended path, reaches some non-optical component, where it can interact and eventually recombine with the main beam. The strain noise generated by scattered photons is proportional to the spectrum of the extra phase acquired when following a different path than the one intended

$$\tilde{h}(f) = \frac{\lambda}{2\pi L} \tilde{\phi}(f)$$

Induced strain noise

$$\phi = \frac{2\pi d}{\lambda}$$

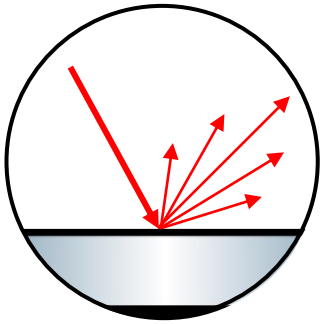
Extra phase acquired



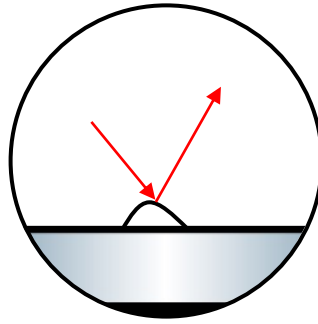
INTRODUCTION



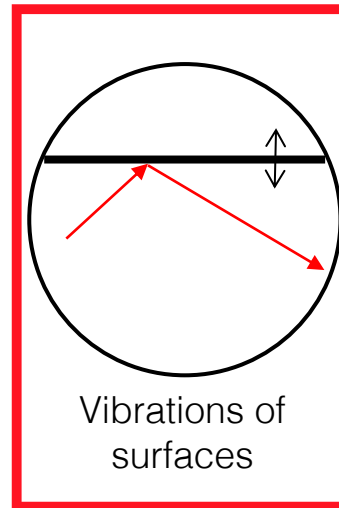
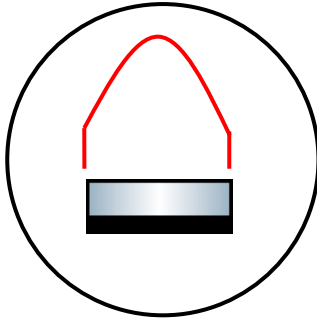
Surface imperfections



Thermal deformations



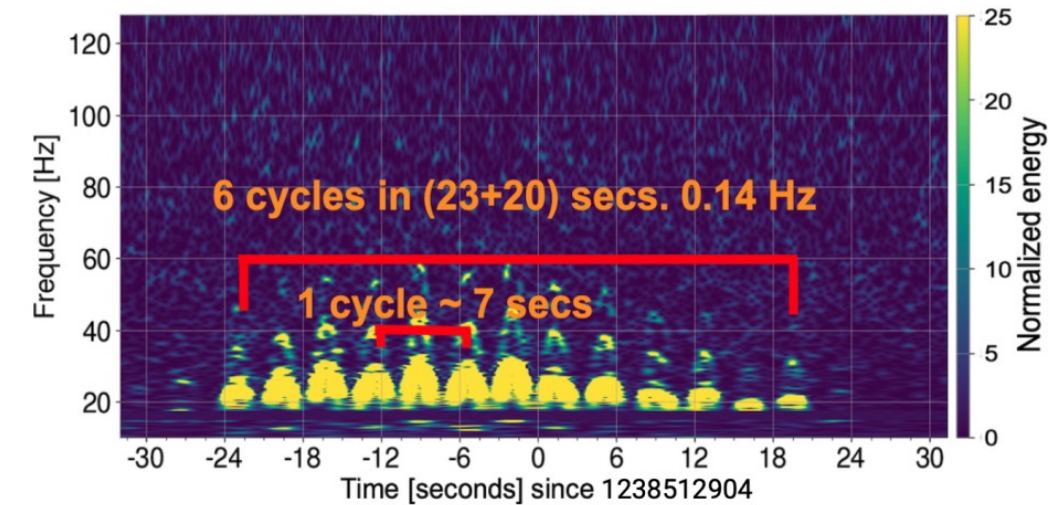
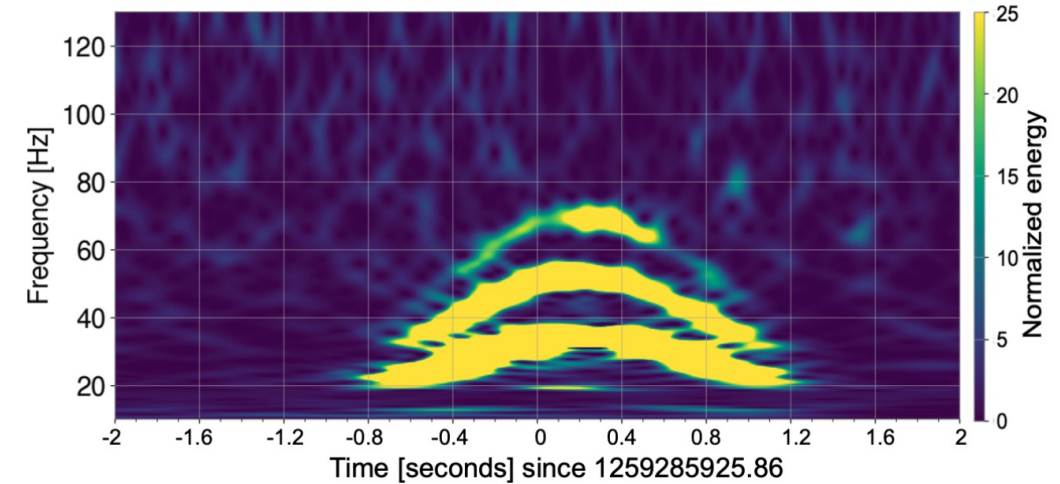
Finite mirror size



Vibrations of surfaces

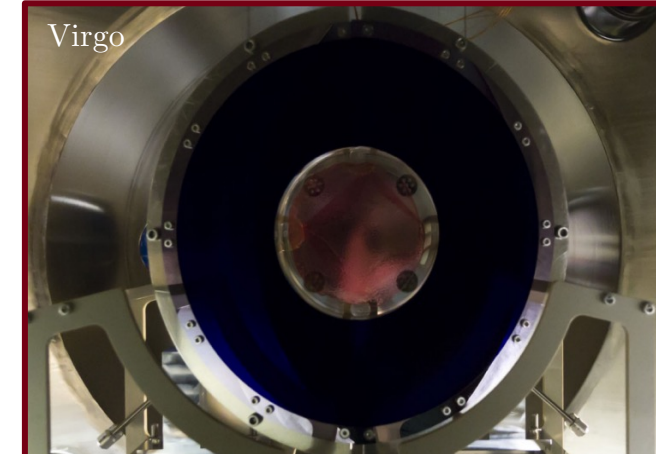
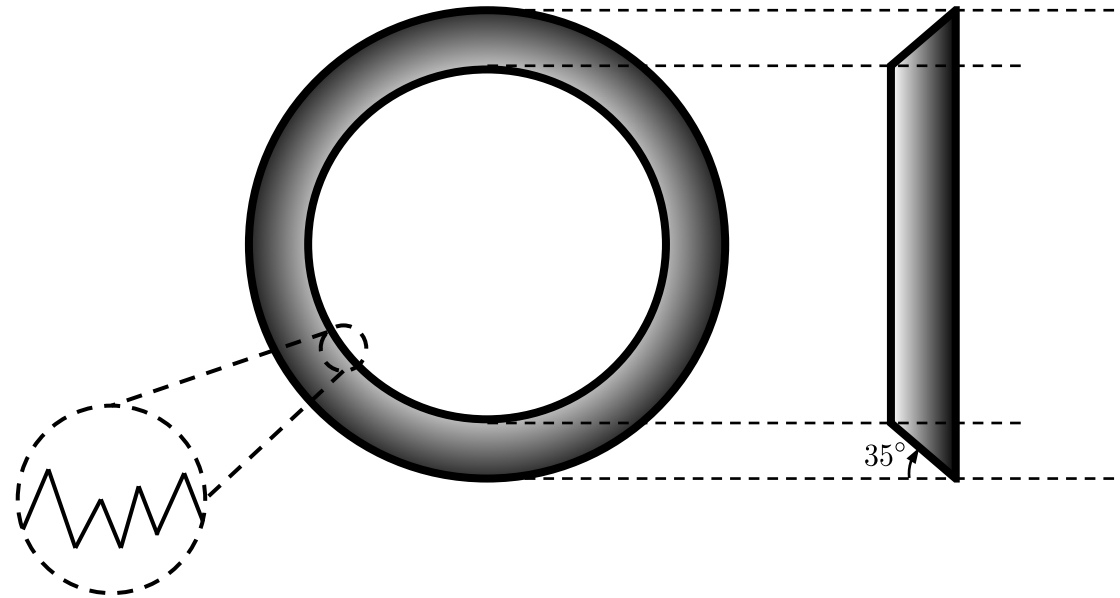
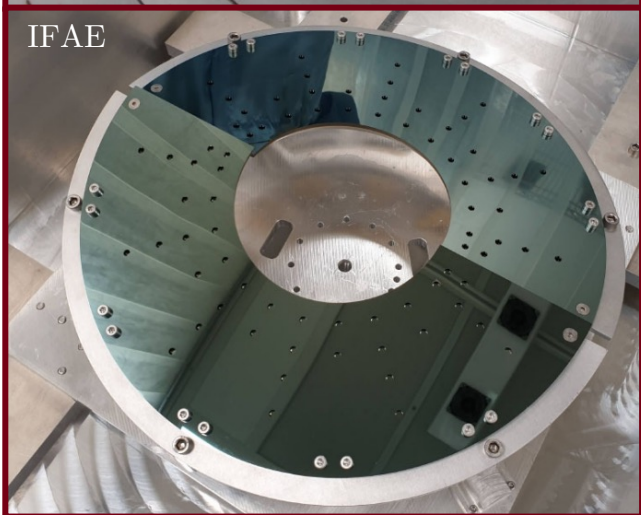
Some effective solutions include:

- ◆ Add baffles to absorb or redirect stray light.
- ◆ Improve the mirror coating and baffle/beampipe surface.



[S. Soni, et al., CQG 38 025016](#)

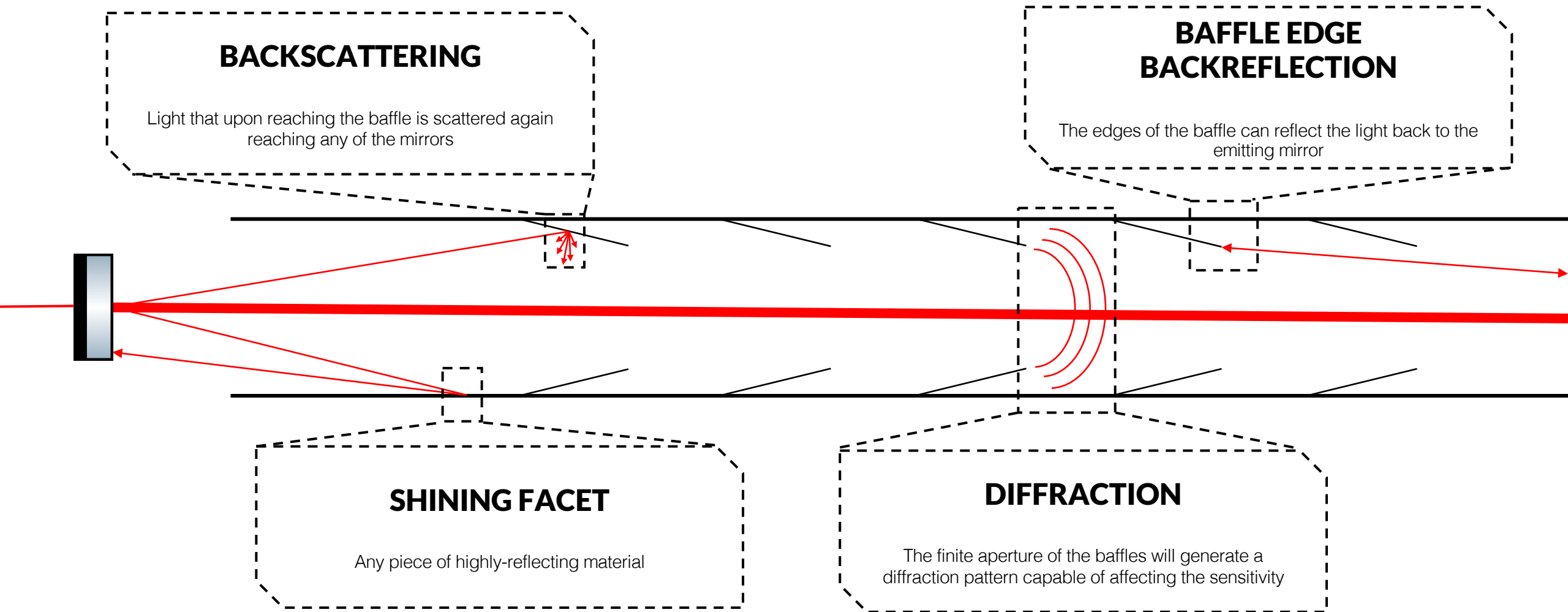
INTRODUCTION



SCATTERED LIGHT

SOME MORE DETAILS

SCATTERED LIGHT



SCATTERED LIGHT



BAFFLE BACKSCATTERING

This noise is mainly a function of:

- Amount of light that reaches the baffle:
 - Geometry/design
 - Mirror quality
 - Cavity conditions
- Baffle material
- **Baffle displacement/vibration**

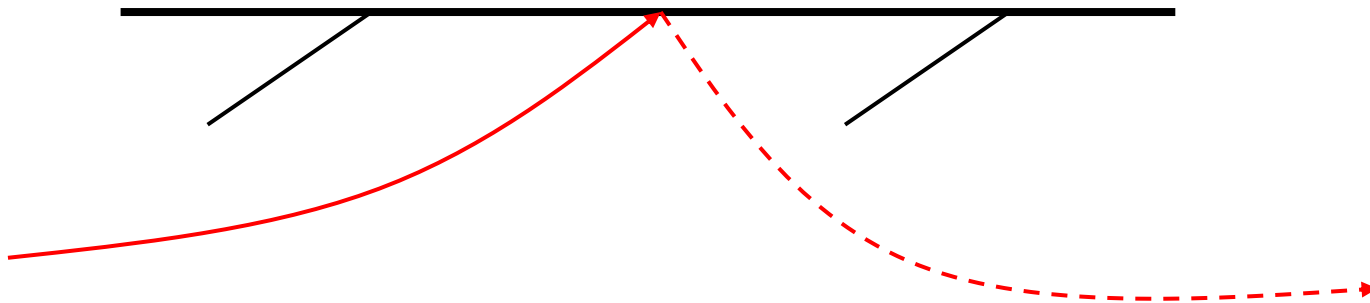
BAFFLE DIFFRACTION

This noise is mainly a function of:

- Overlap of the diffracted field with the resonating one:
 - Geometry/design
 - Mirror quality
 - Cavity conditions
- Baffle serration
- **Baffle displacement/vibration**

For the details, there is a dedicated talk about the methods in the parallel session of scattered light

SCATTERED LIGHT

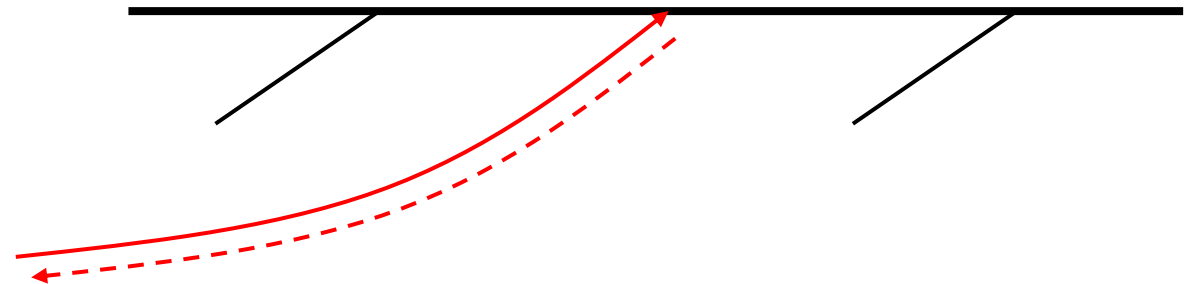


FORWARD TUBE SCATTERING

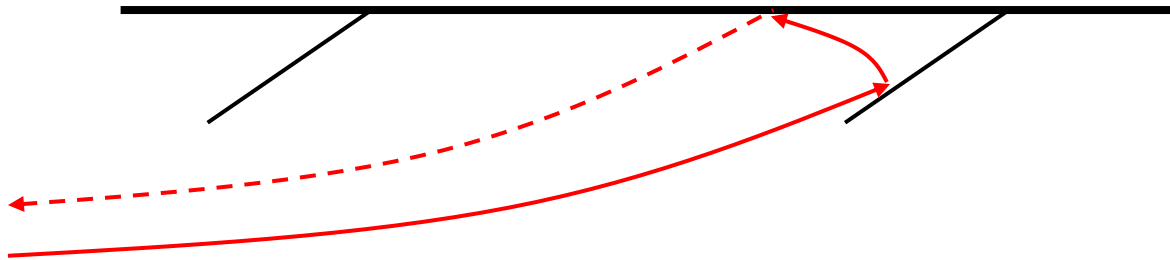
This represents the noise generated by light that hits the tube and reflects (scatters) and reaches the non-emitting mirror

BACKWARD TUBE SCATTERING (TUBE BACKSCATTERING)

This represents the noise generated by light that hits the tube and scatters back to the emitting mirror



SCATTERED LIGHT

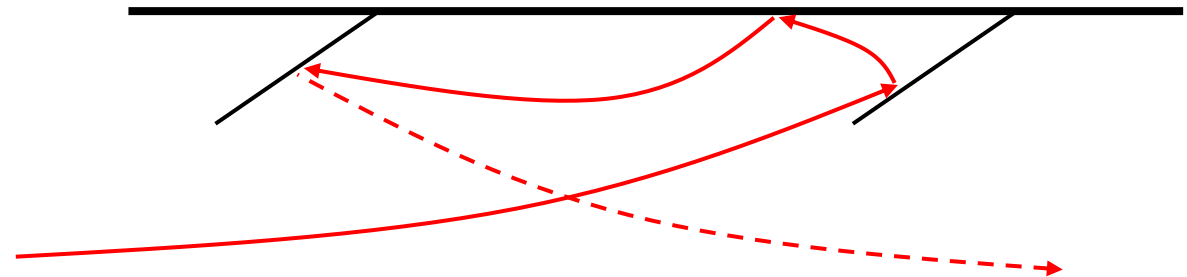


DOUBLE BOUNCE TUBE SCATTERING

This represents the noise generated by light that hits the tube, then the baffle (or first the baffle and then the tube) and then reaches the emitting mirror

TRIPLE BOUNCE TUBE SCATTERING

This represents the noise generated by light that hits a baffle, then the tube, then the other baffle and scatters back to the emitting mirror



SCATTERED LIGHT

BAFFLING STRATEGIES



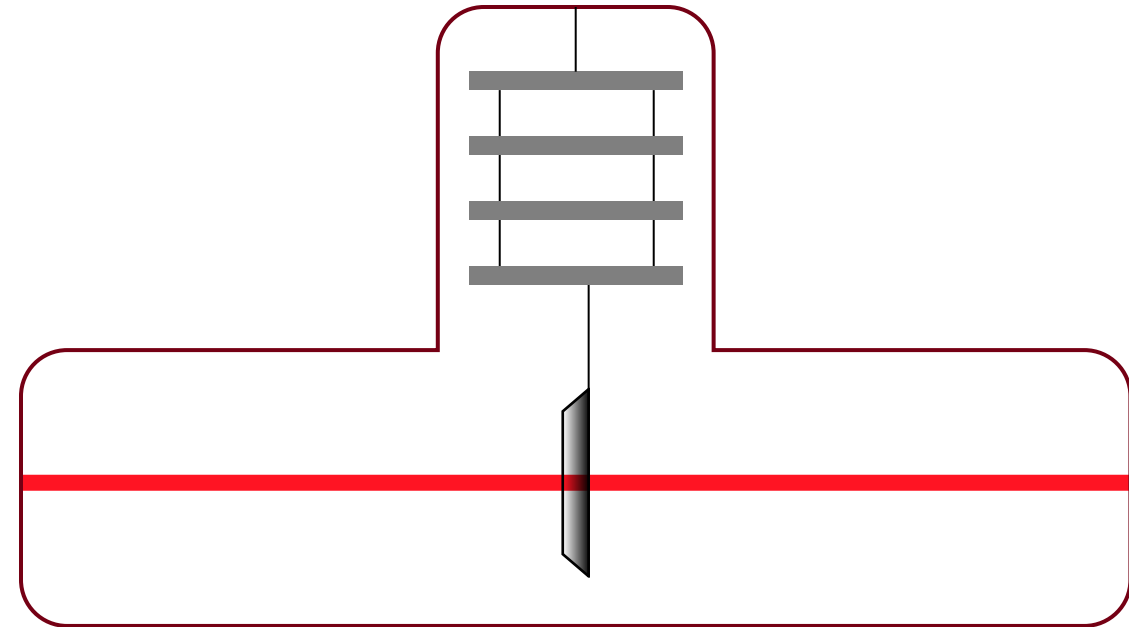
RAY OPTICS WAVE OPTICS

Regular baffle design but different strategies to decide its placement



SUSPENDED BAFFLES

New concept to minimize the number of baffles



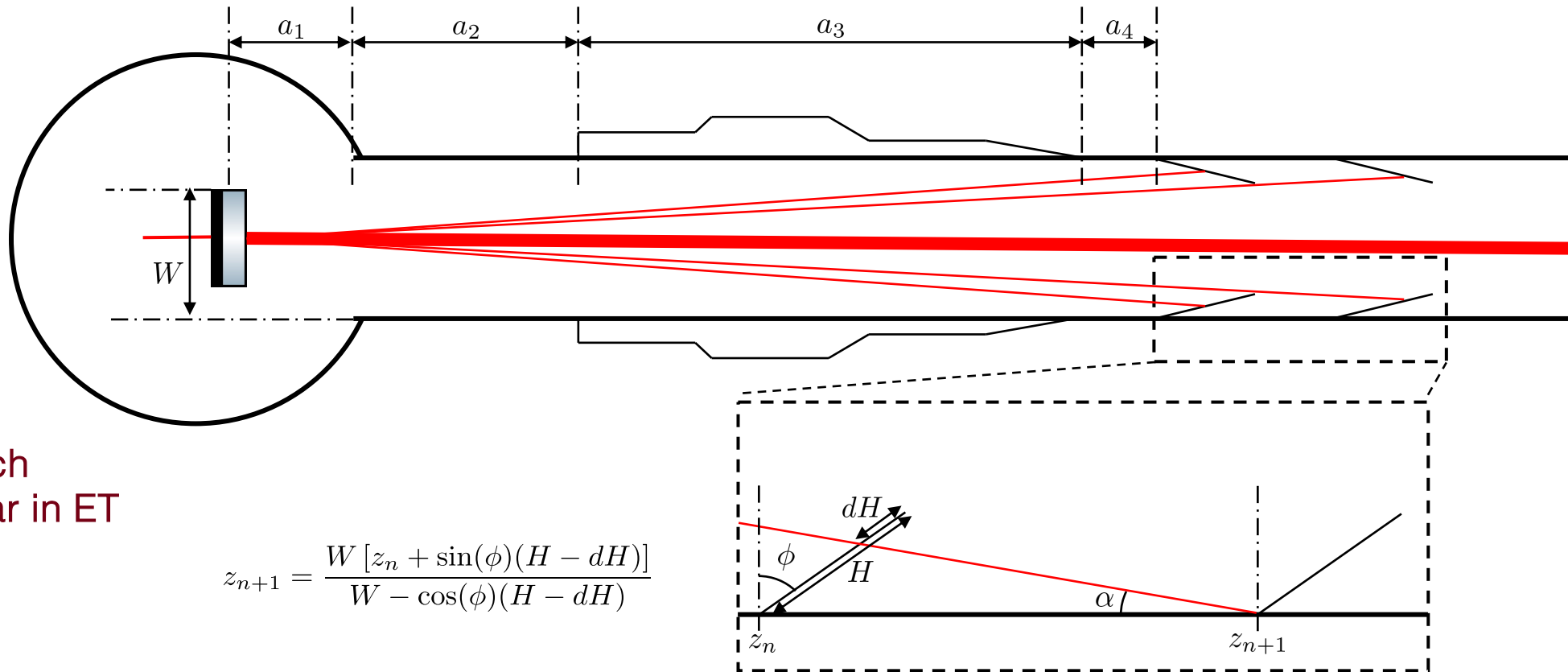
SCATTERED LIGHT



BAFFLING STRATEGIES

RAY OPTICS

Placing baffles shielding the tube according the direct line of sight from the emitting mirrors.



Main approach
followed so far in ET

$$z_{n+1} = \frac{W [z_n + \sin(\phi)(H - dH)]}{W - \cos(\phi)(H - dH)}$$

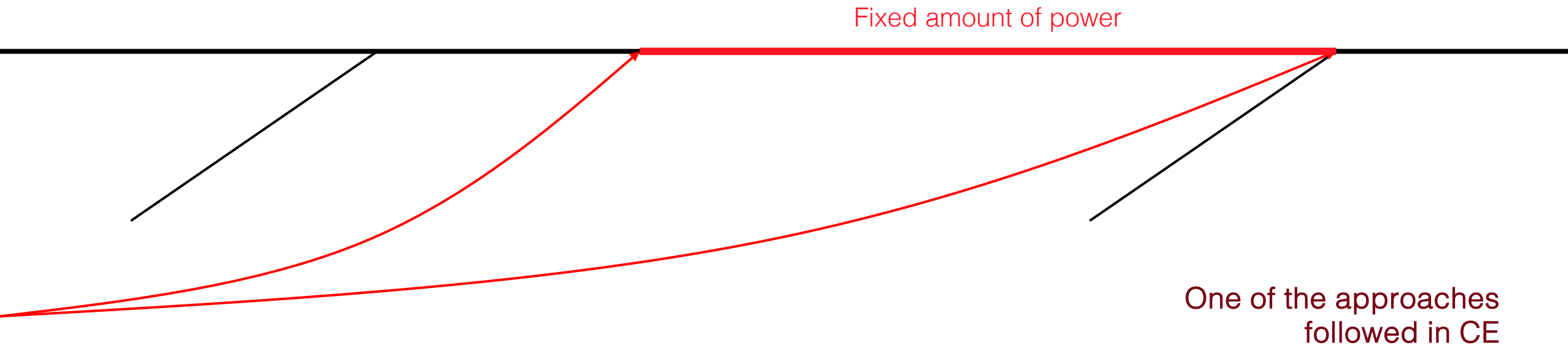
SCATTERED LIGHT



BAFFLING STRATEGIES

WAVE OPTICS

Placing baffles such that a fixed and controlled amount of light reaches the tube



SCATTERED LIGHT



BAFFLING STRATEGIES

SUSPENDED BAFFLES

Placing way less baffles (~6), but suspended to avoid the vibration and with a much tighter aperture



One of the approaches
considered for CE

SCATTERED LIGHT

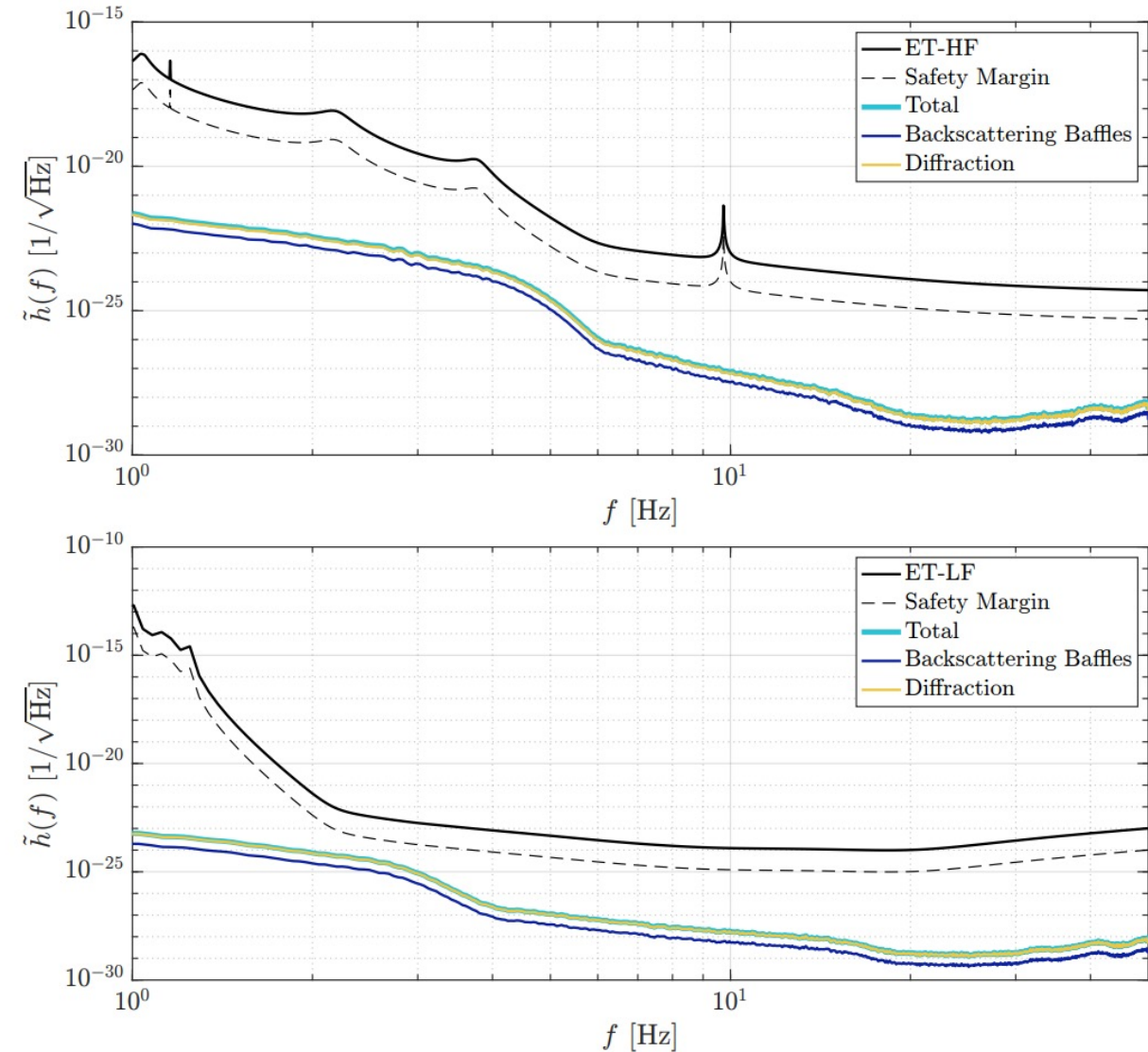
EXAMPLE NOISE ESTIMATION FOR ET



RAY OPTICS BAFFLING

Example calculation for the **baffle backscattering** noise and **diffraction** noises for the ET detector. These calculations ignore the noise from the beampipe as the baffle placing and design should cover it completely.

We observe how the **diffraction noise dominates over the backscattering**, implying that the number of baffles is big enough so as to pile up enough noise over them. This result calls for a baffle optimization.



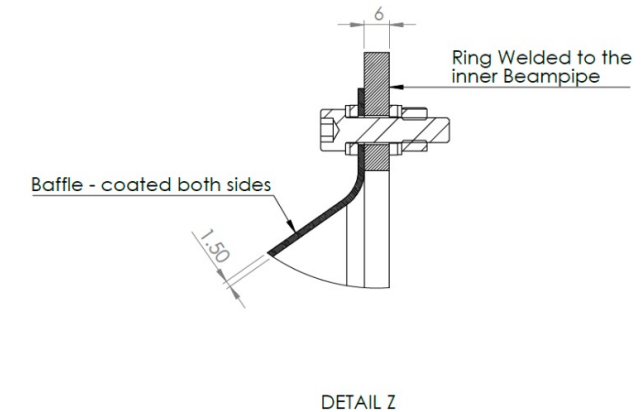
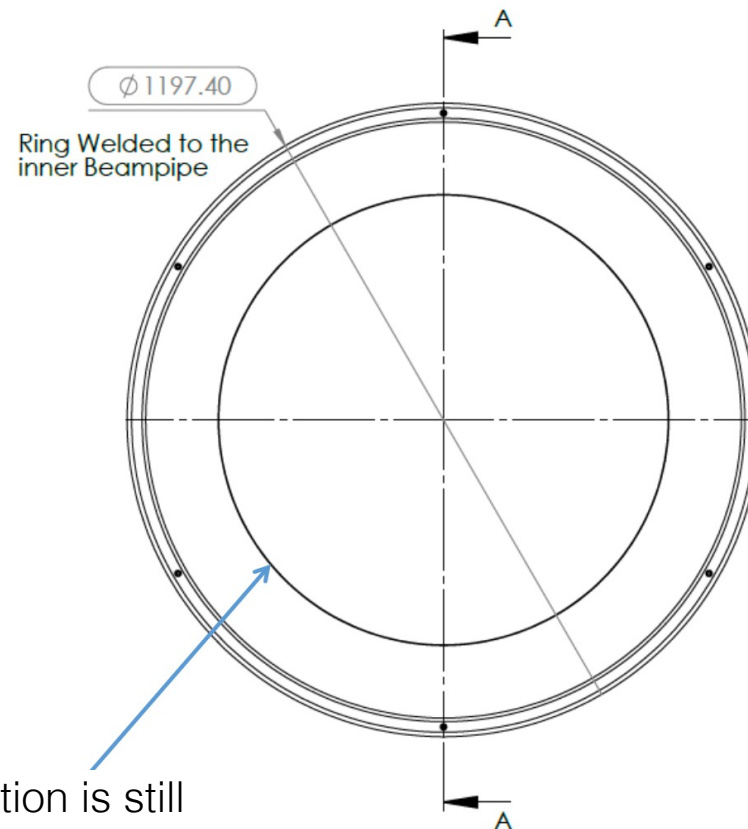
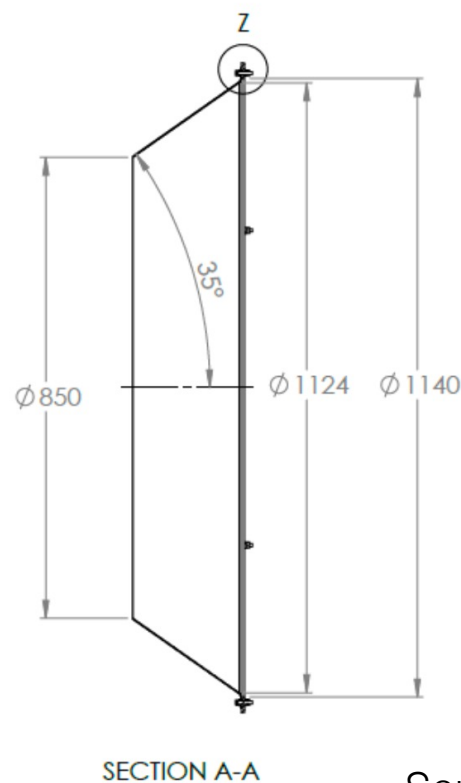
MECHANICAL DESIGN

WHAT DOES THIS IMPLY FOR THE MECHANICS?

MECHANICAL DESIGN



Example of the mechanical design considered for ET's beampipe baffles



ITEM NO.	PART NUMBER	QTY.
1	ET-Beampipe-Baffle	1
2	ET-Beampipe-Baffle fixation ring	1
3	ET-Beampipe-DIN 7349 M6 customized	12
4	Hexagon Nut ISO 4034 - M6 - N	6
5	DIN 912 M6 x 25 --- 25N	6

Parameter	Value	Unit
Thickness	1.5	mm
Material	304L SS	-
Flange outer \varnothing	118	cm
Inner \varnothing	84	cm
Cone angle	35	deg
Mass (incl. ring)	10	kg

Credit of some values to engineers J. Mundet (IFAE) and R. García (IFAE)

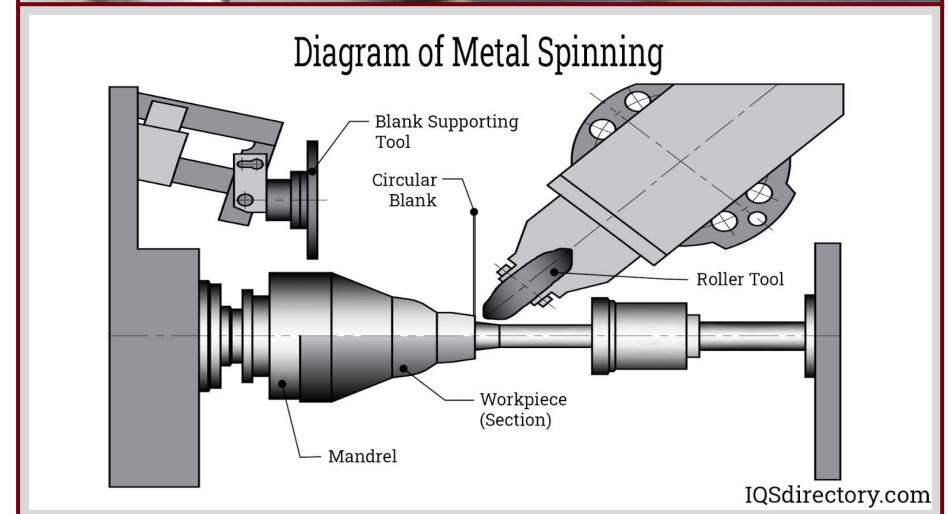
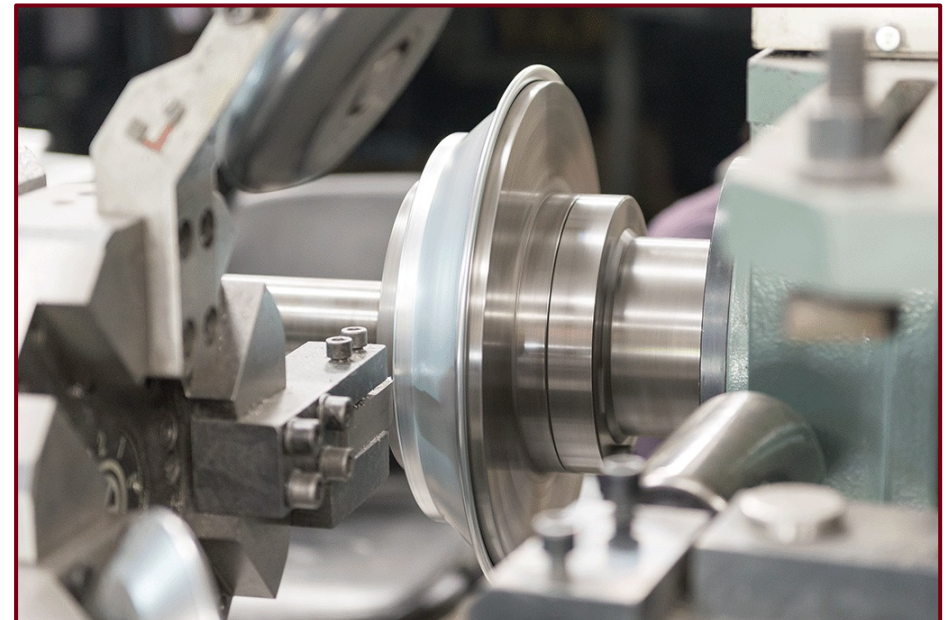
This mechanical design under consideration of the baffle assume a **“Bolted To a Welded Ring” (BTWR)** attachment to the beampipe. This method can allow for some decoupling of the tube fabrication and the baffle installation, since if the ring is already in placed, the baffle only needs to be screwed.

Ideally, the presence of the ring can allow a better alignment and placement of the baffle than directly welding it to the beampipe. Important if the tolerances/requirements are very strict.

These parameters have interfaces with the beampipe: the mass, the size, the position and welding of the ring,

A manufacturing procedure that has been proposed by the IFAE engineers (although still preliminary) is the following:

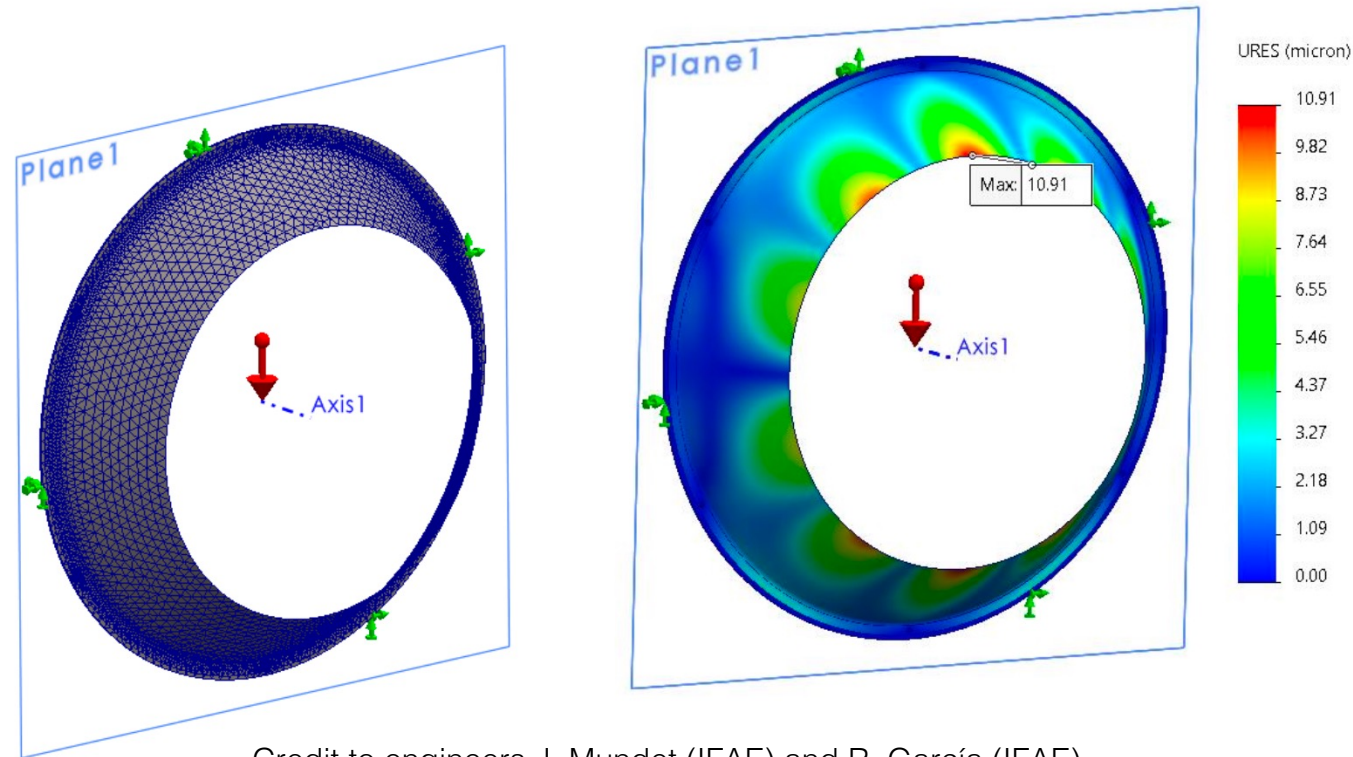
- ❑ New material: ASME specification SA-240, type 304L with mill finish (to be certified)
- ❑ Metal spinning forming (for the conical shape and the flange)
- ❑ Bead blasting with AlO_2
- ❑ Cleaning
- ❑ Laser cutting with 5 axis CNC for flange holes & serration
- ❑ Final cleaning
- ❑ Coating



MECHANICAL DESIGN



A static baffle analysis shows that the maximum displacement under only its own weight is of the order of $10.91\ \mu\text{m}$. From the optics perspective, this should not be extremely problematic.



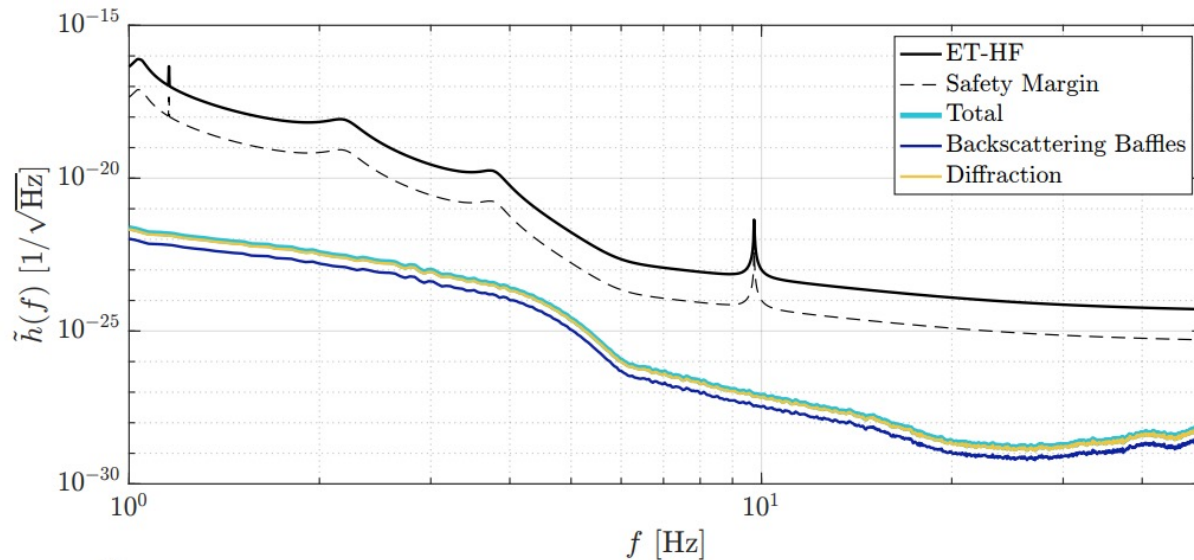
Credit to engineers J. Mundet (IFAE) and R. García (IFAE)

MECHANICAL DESIGN



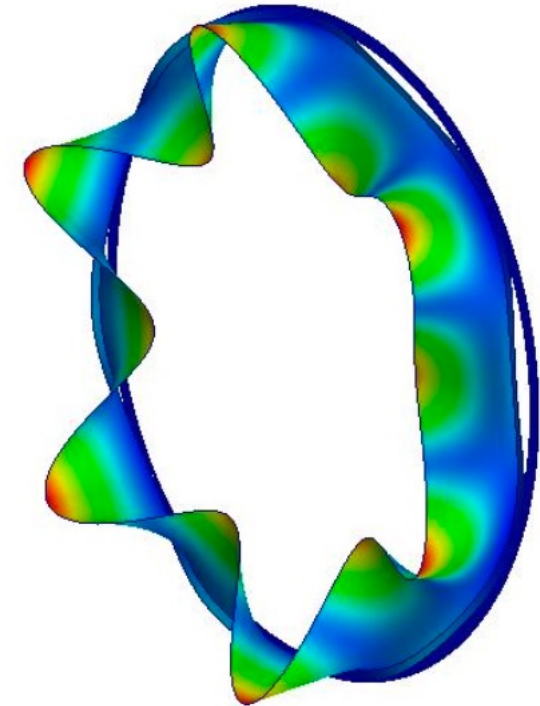
A dynamical analysis of the baffle and the ring can reveal the resonances of the system and the response to vibrations of the baffles.

In this case, the first resonant mode is observed to occur at **136.1 Hz** with a mass participation in the z-direction of about an 8%.



Remember that the seismic motion scales as $1/f^2$

The first resonant frequency is **well above the relevant range of frequencies** that can pose a problem for scattered light

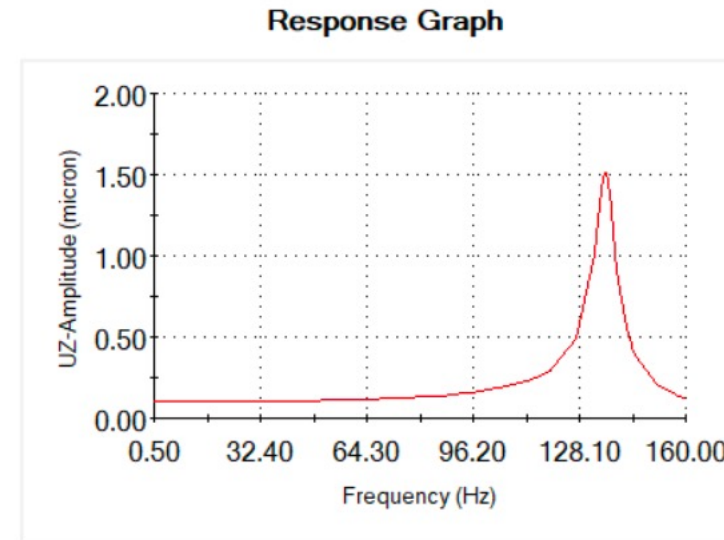
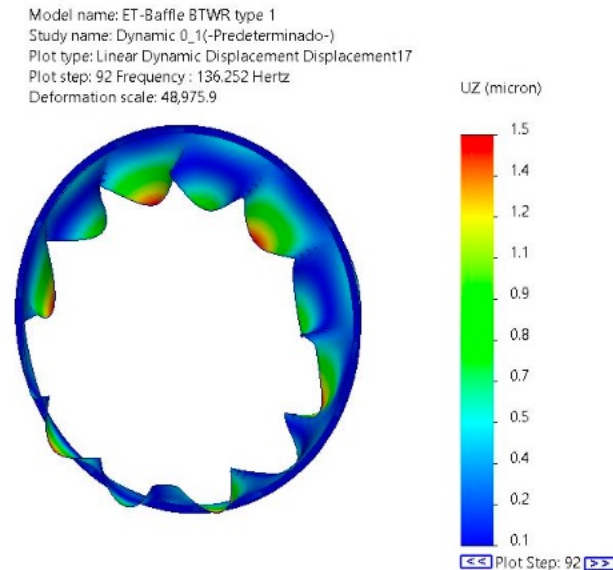


Credit to engineers J. Mundet (IFAE) and R. García (IFAE)

MECHANICAL DESIGN

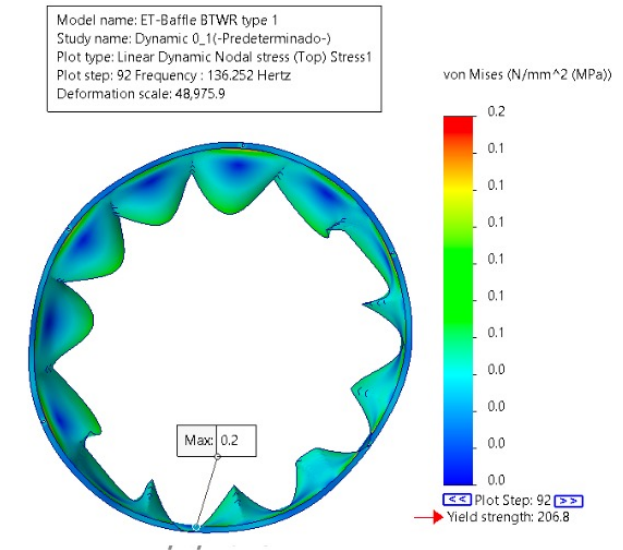


Similarly, we can study the frequency response of the baffle design by adding an excitation in the z-axis (horizontal direction) with a displacement of 0.1, 1, 10, & 100 μm of amplitude between a range of 0.5 and 100/200 Hz.



Credit to engineers J. Mundet (IFAE)
and R. García (IFAE)

Amplification:
1.51 microns / 0.1 microns
15.1

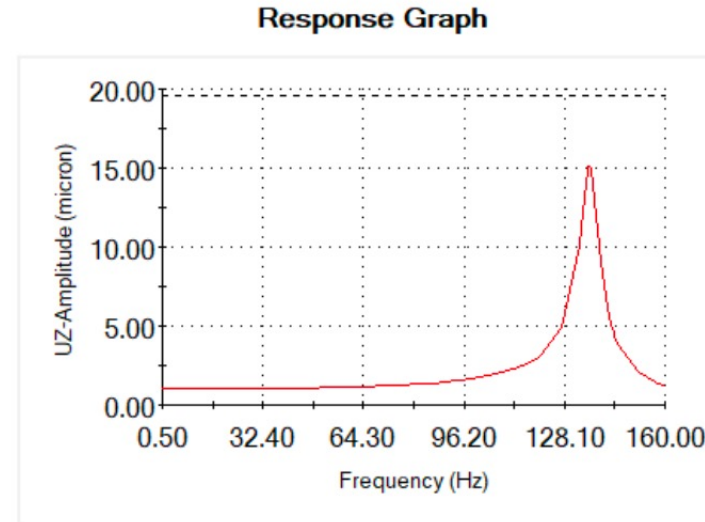
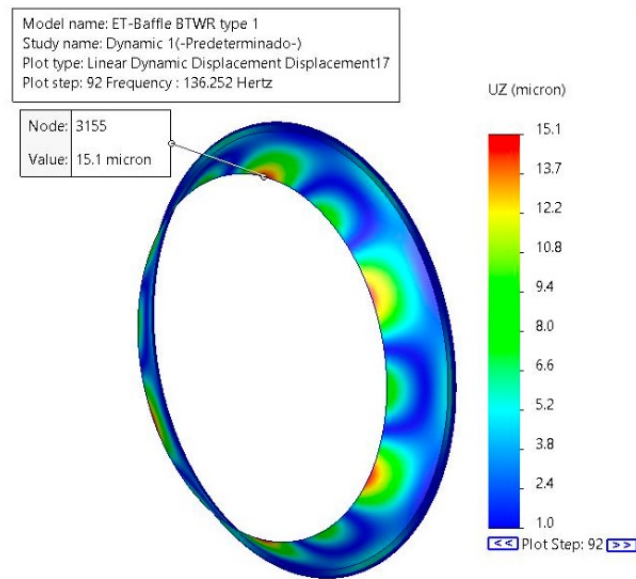


Max. yield below yield strength of SS

MECHANICAL DESIGN

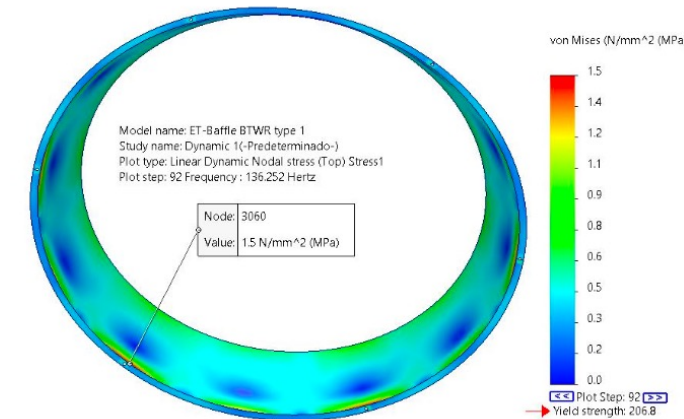


Similarly, we can study the frequency response of the baffle design by adding an excitation in the z-axis (horizontal direction) with a displacement of 0.1, 1, 10, & 100 μm of amplitude between a range of 0.5 and 100/200 Hz.



Credit to engineers J. Mundet (IFAE)
and R. García (IFAE)

Amplification:
15.1 microns / 1 microns
15.1



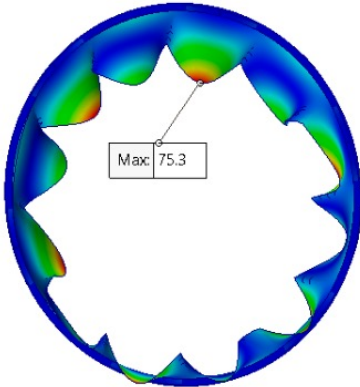
Max. yield below yield strength of SS

MECHANICAL DESIGN

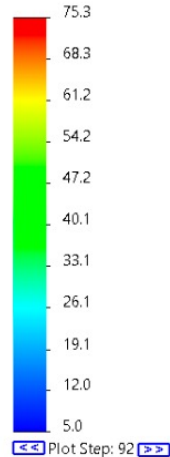


Similarly, we can study the frequency response of the baffle design by adding an excitation in the z-axis (horizontal direction) with a displacement of 0.1, 1, 5, & 100 μm of amplitude between a range of 0.5 and 100/200 Hz.

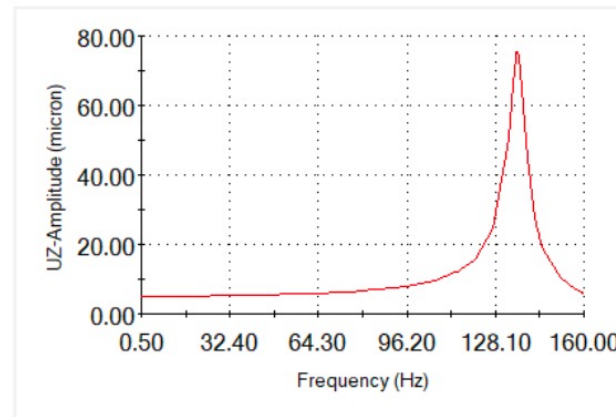
Model name: ET-Baffle BTWR type 1
Study name: Dynamic 5(-Predeterminado-)
Plot type: Linear Dynamic Displacement Displacement17
Plot step: 92 Frequency: 136.252 Hertz
Deformation scale: 979.517



UZ (micron)



Response Graph

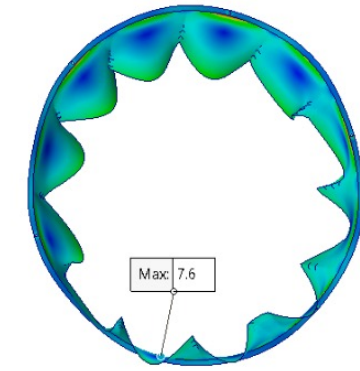


Node 3155

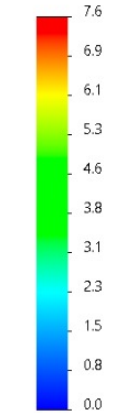
Credit to engineers J. Mundet (IFAE)
and R. García (IFAE)

Amplification:
75.3 microns / 5 microns
15.1

Model name: ET-Baffle BTWR type 1
Study name: Dynamic 5(-Predeterminado-)
Plot type: Linear Dynamic Nodal stress (Top) Stress1
Plot step: 92 Frequency: 136.252 Hertz
Deformation scale: 979.517



von Mises (N/mm² (MPa))



<< Plot Step: 92 >>
→ Yield strength: 206.8

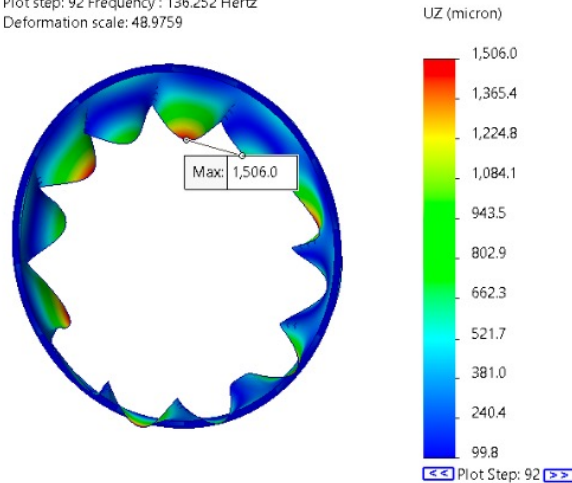
Max. yield below yield strength of SS

MECHANICAL DESIGN

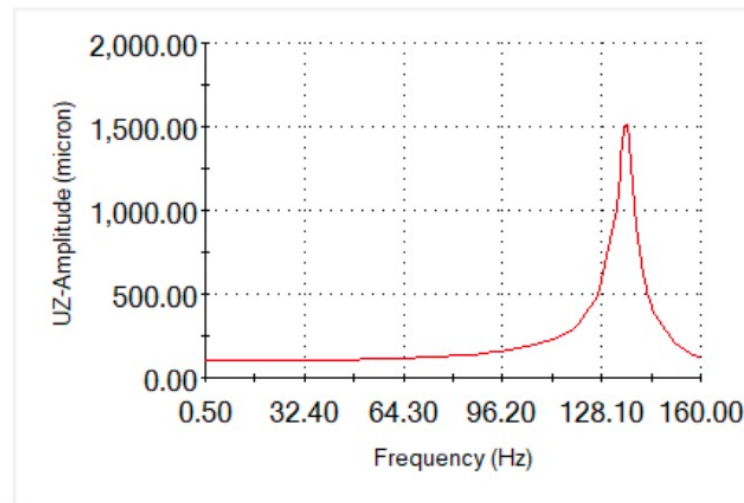


Similarly, we can study the frequency response of the baffle design by adding an excitation in the z-axis (horizontal direction) with a displacement of 0.1, 1, 5, & 100 μm of amplitude between a range of 0.5 and 100/200 Hz.

Model name: ET-Baffle BTWR type 1
Study name: Dynamic 100(-Predeterminado-)
Plot type: Linear Dynamic Displacement Displacement17
Plot step: 92 Frequency: 136.252 Hertz
Deformation scale: 48.9759



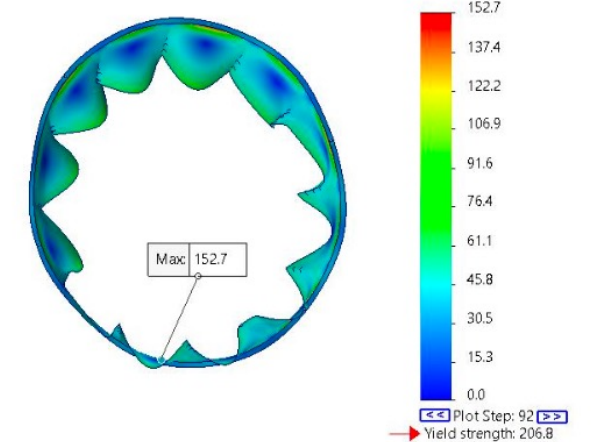
Response Graph



Credit to engineers J. Mundet (IFAE)
and R. García (IFAE)

Amplification:
1506 microns / 100 microns
15.06

Model name: ET-Baffle BTWR type 1
Study name: Dynamic 100(-Predeterminado-)
Plot type: Linear Dynamic Nodal stress (Top) Stress1
Plot step: 92 Frequency: 136.252 Hertz
Deformation scale: 48.9759



Max. yield below yield strength of SS

MECHANICAL DESIGN

CONCLUSIONS



1

The first resonant frequency is well far from the range of frequencies where scattered light is relevant

2

The amplification at the resonance is of about an order of magnitude, which is also well within the margins

3

The material and attachment should easily withstand any reasonable seismic motion from the tube

4

Machining procedure is feasible with current technology and the technical requirements fulfilled

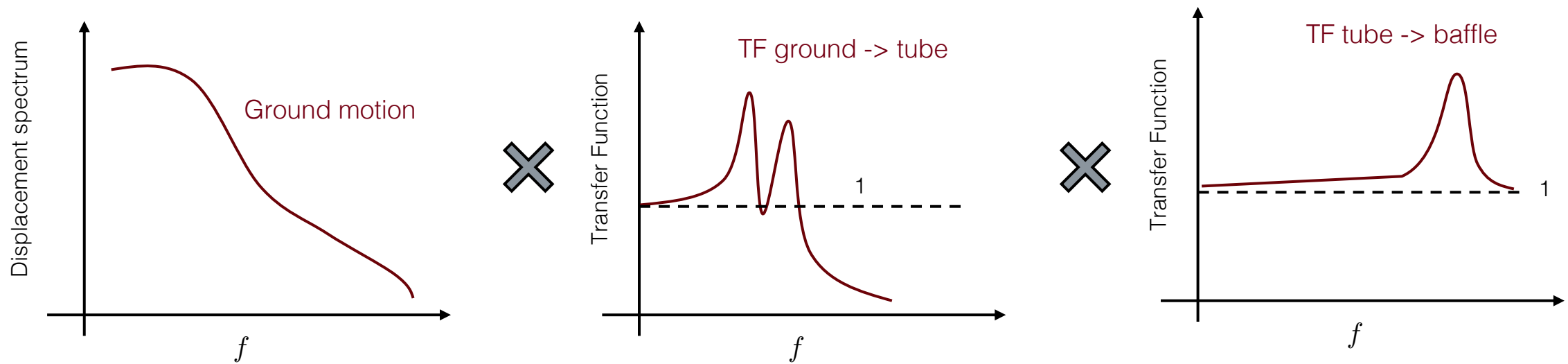
TUBE VIBRATIONS

WHICH ARE THE LIMITS TO TUBE VIBRATIONS?

VIBRATIONS



An important question to be addressed during the design of the vacuum beampipe is the maximum allowable level of vibrations, as an increase can pose a problem from the scattered light perspective.



The ground motion depends on the site, the transfer factor of the tube of its design and the one from the baffle due to its mechanical attachment.

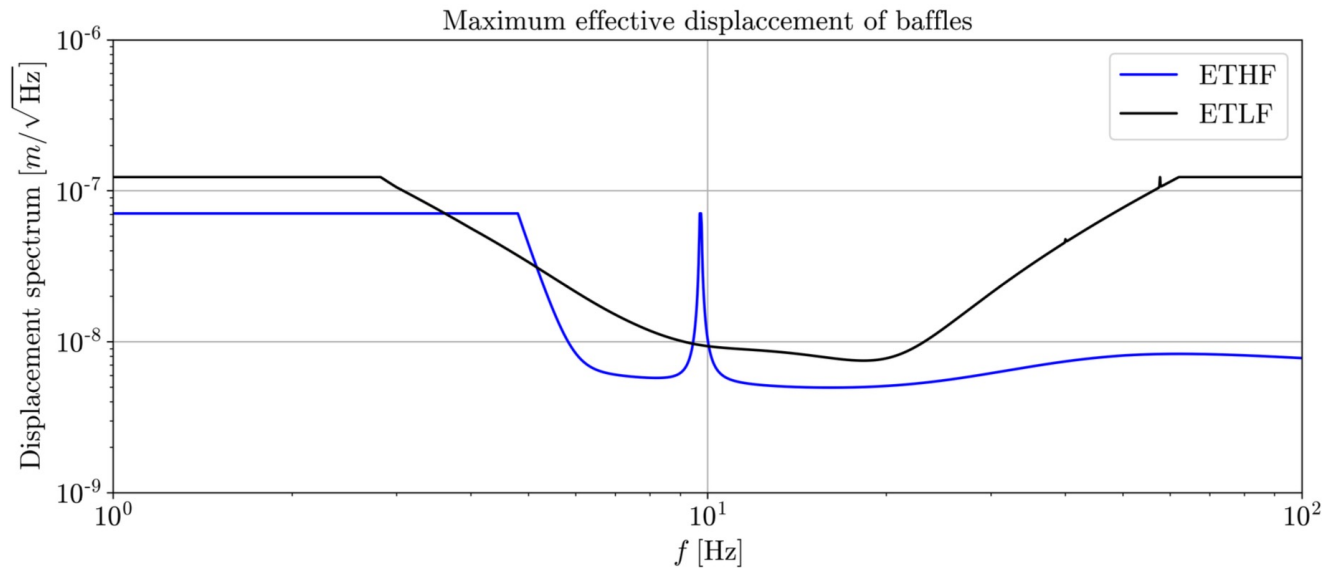
We need a very good communication between teams to ensure that the product remains below acceptable levels

VIBRATIONS



Procedure to compute if a design of the tube fulfills the vibration requirements:

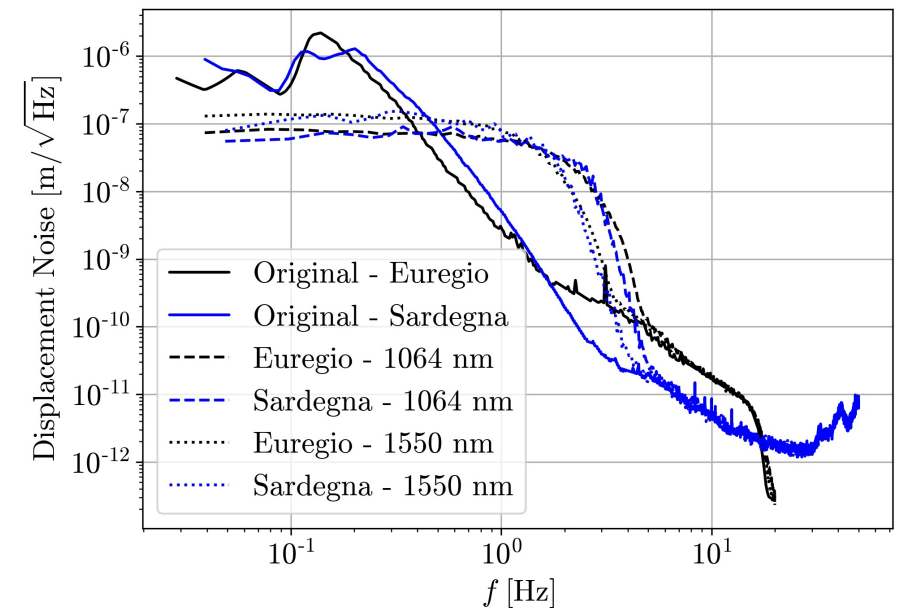
- Compute TF from the ground to the tube
- Compute the total baffle displacement (ground motion x TF tube x TF baffle)
- Upconvert the displacement
- Compare to the upper limit



Example for ET from the Beampipe Requirements (ET-0385A-24)

The seismic noise must be upconverted or phase-wrapped to obtain the effective displacement. This can be done applying the following formula ([LIGO-T1900854](#))

$$X(f) = \frac{\lambda_L}{4\pi} \sqrt{\text{PSD} \left[\sin \left(\frac{4\pi}{\lambda_L} X(t) \right) \right] + \text{PSD} \left[\cos \left(\frac{4\pi}{\lambda_L} X(t) \right) \right]}$$

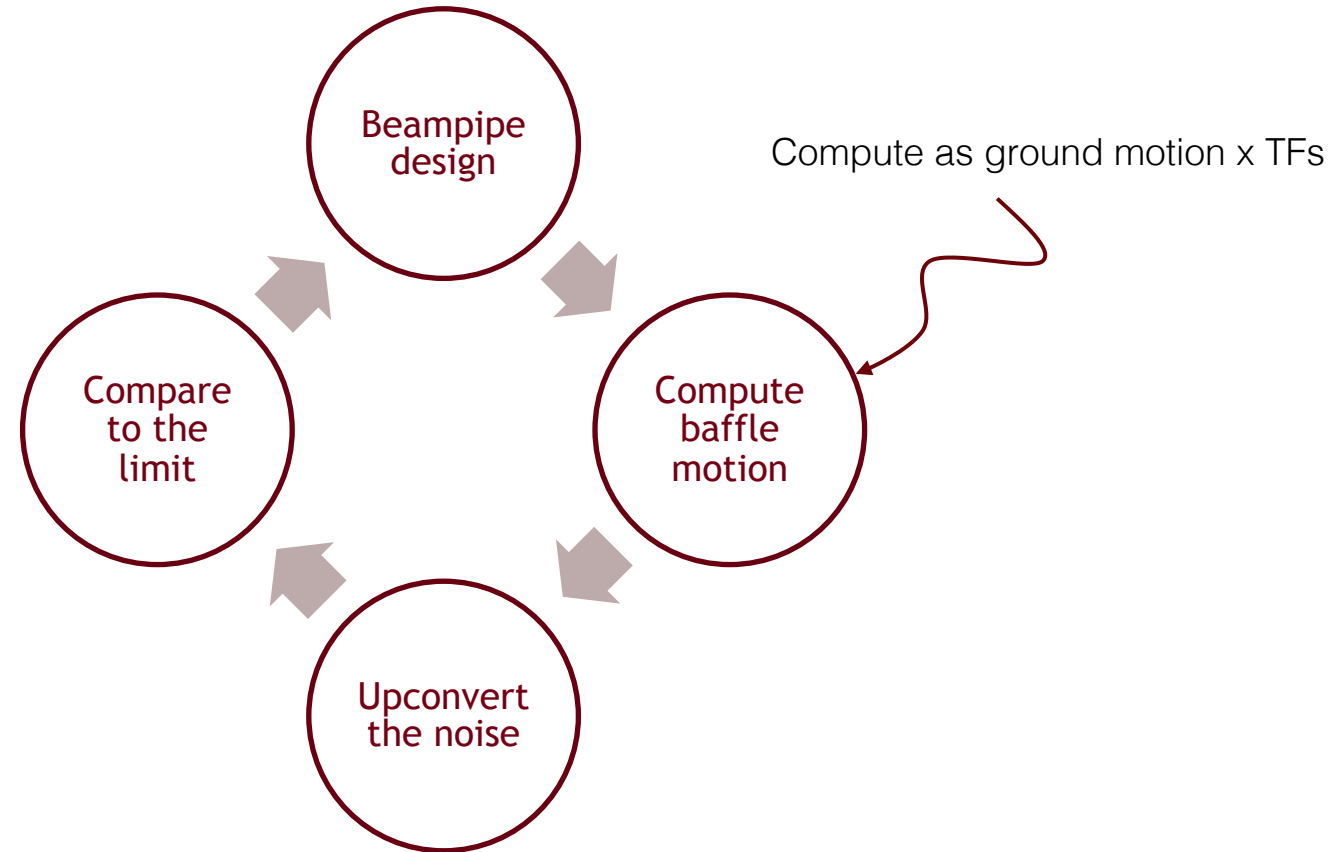


[M. Andrés-Carcasona, et al., Phys. Rev. D 108, 102001](#)

VIBRATIONS



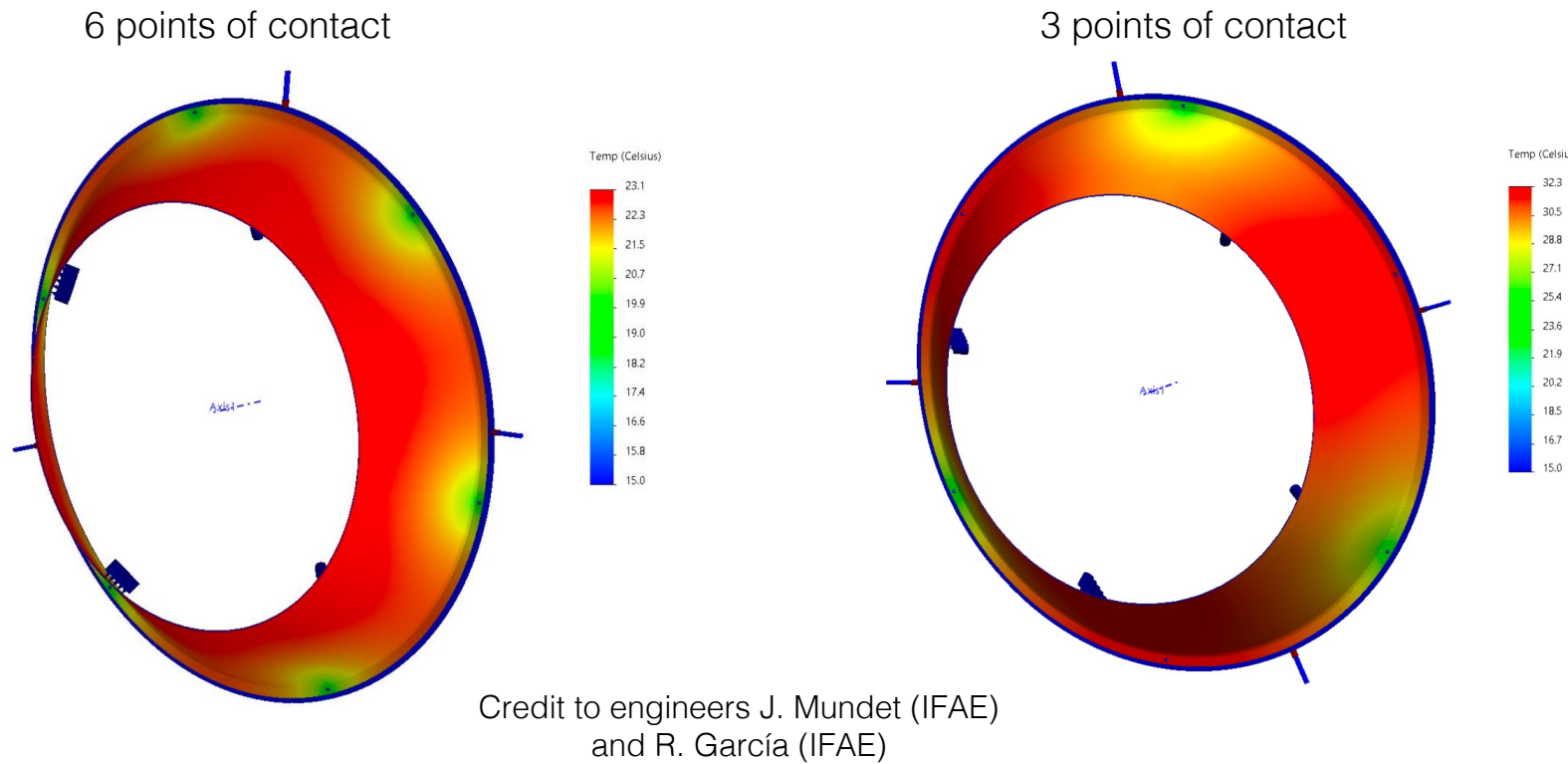
To properly aid the beampipe mechanical design we (the people from scattered light) can provide you with: upconversion code and the maximum tolerable displacement. The workflow can be



THERMAL



We also need to study the thermal effects of a baffle being hit by a continuous amount of light, as thermal expansion can become a problem for the material. Not only that, but the material can emit thermal radiation that can affect the rest of the components of the interferometer. Of special importance can this be for the cryogenic areas.



If the baffle receives 0.5 W of power (input from a simulation) and that the initial baffle temperature of the baffle is 15°C, the thermal simulation indicates a temperature of the baffle of:

- Assuming an interfacial conductance between baffle and ring of 300 W/m²K:
 - 3 points: **32.2°C** (dT = 17.3°C)
 - 6 points: **23.1°C** (dT = 8.1°C)
- Assuming an interfacial conductance between baffle and ring of 3700 W/m²K:
 - 3 points: **25.4°C** (dT = 10.4°C)
 - 6 points: **19.7°C** (dT = 4.7°C)

CONCLUSIONS

CONCLUSIONS



01.

The baffle and tube designs are highly tied and requires a lot of **interaction**

02.

We are considering two baffle designs with different impacts in the vacuum system

03.

Our current loop of the design seems to be feasible both from the scientific point of view and fabrication

04.

If the suspended baffle approach not only the optics are affected: vacuum, suspensions, thermal,...

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Mechanical/Optical Design Options for Baffles

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