

# 引力波

LIGO-G1601928-v1

## Control system in Gravitational Wave Detectors

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GW mini-school: Beijing Normal University 2016/9/15~18

# Introduction ~ Control?

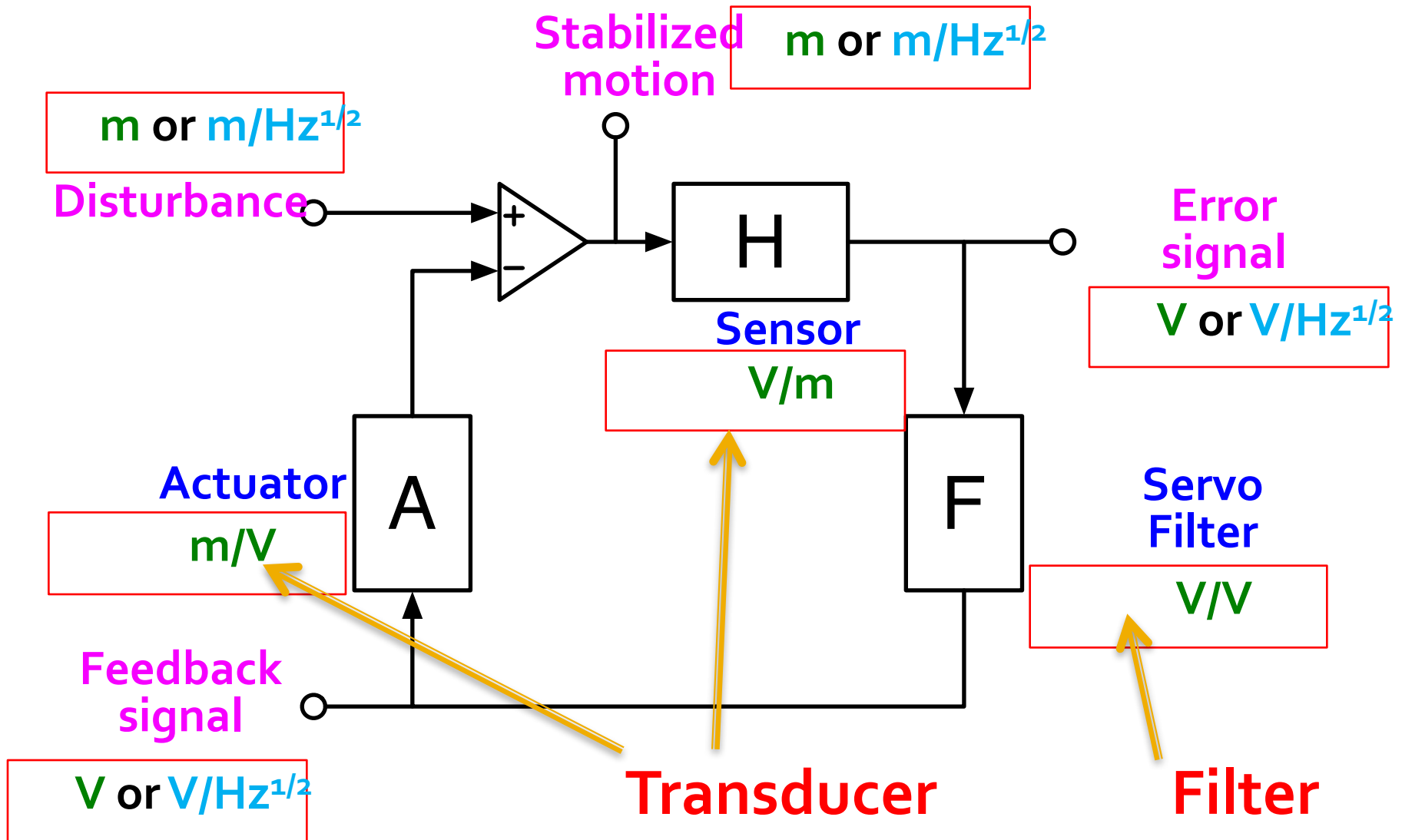
- **Gravitational wave detection**  
**Laser displacement sensor**  
**Requires linear displacement detection**
  
- **Control for measurement**  
**Laser interferometer = nonlinear device**  
**Feedback control => linearization**

# Introduction ~ Control?

- What is the feedback control?
  - A scheme to monitor and modify output(s) of a system by changing the input(s) depending on the output(s)
- Examples
  - Shower temperature
  - Car driving
  - Tight rope walking
  - Air conditioning
  - Bike riding
  - Inverted bar on a hand
- Imagine what happens
  - If the response is too slow?
  - If the response is too fast?

# Introduction ~ Control?

- Block diagram: Elements of a feedback loop



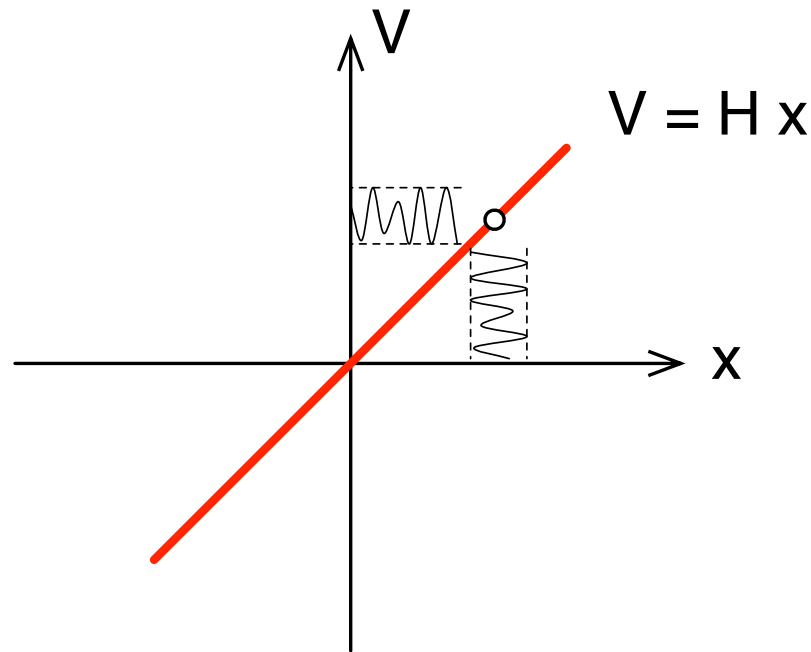
# Introduction ~ Control?

- Sensor:

Transducer for displacement-to-voltage conversion

- If the sensor is completely linear

(and has or no frequency dependence)

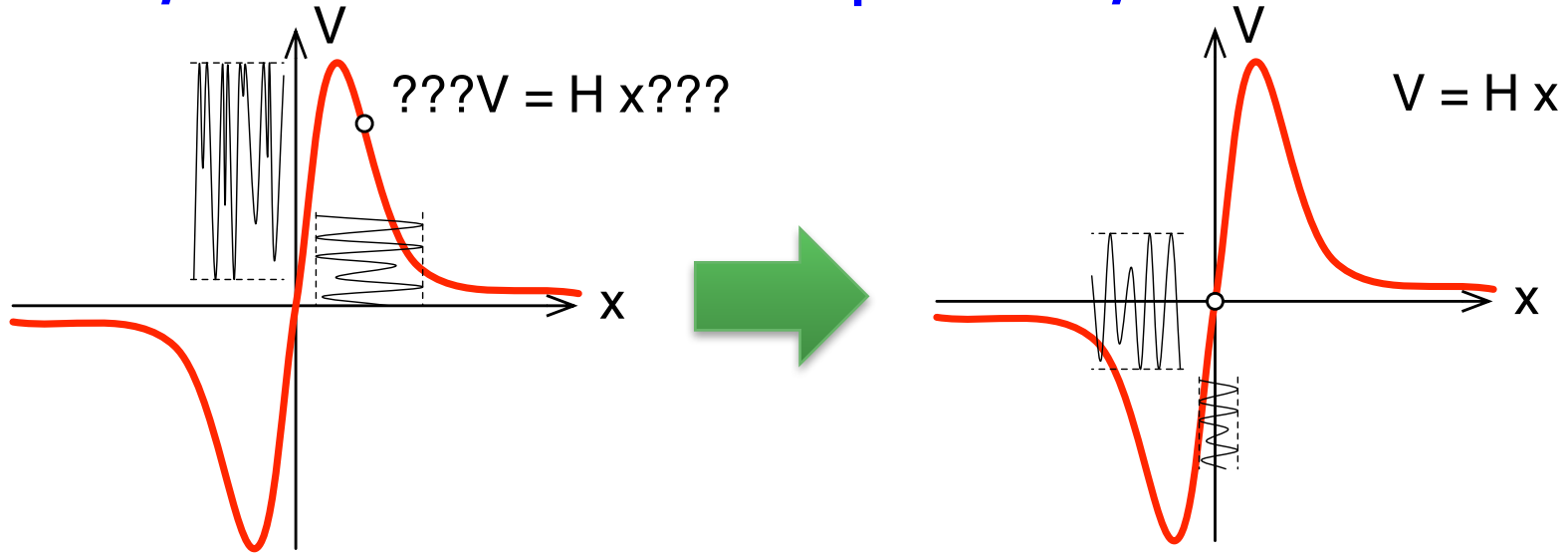


We don't need feedback control!

# Introduction ~ Control?

- In reality:

Sensors, laser interferometers in particular, are **nonlinear!**



- Enclose the operating point in the linear region

=> The system recovers linearity

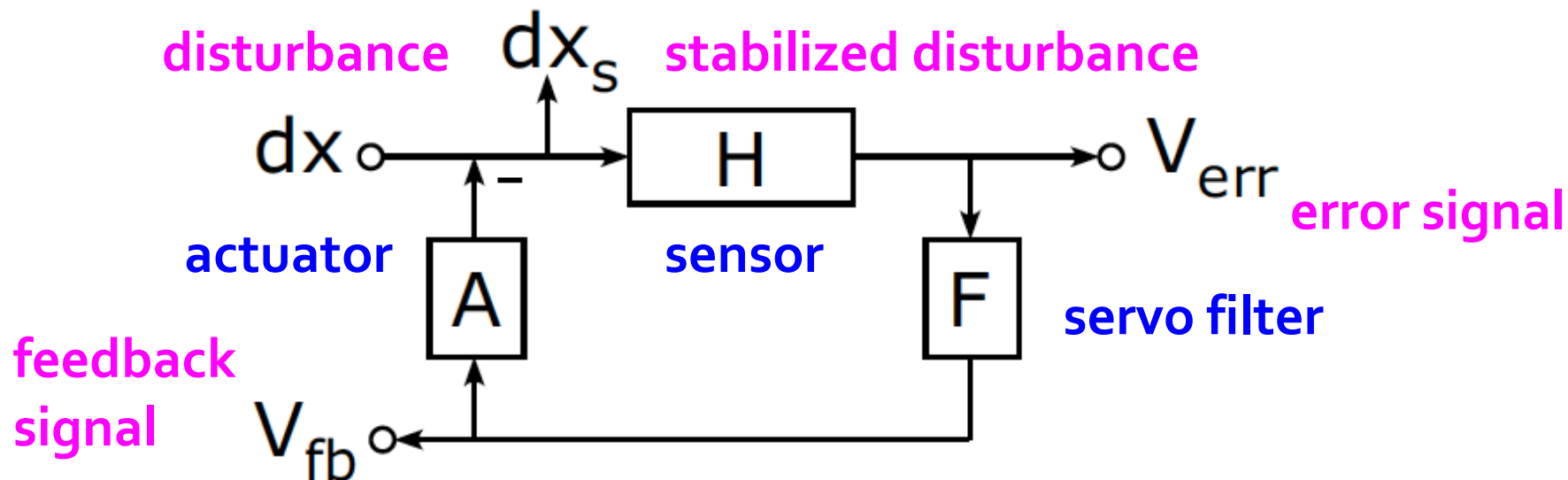
- Was the displacement modified by the feedback?

=> Precise knowledge of the control system

for signal reconstruction

# Introduction ~ Control?

- Elements of a feedback loop



Open loop transfer function

$$G \stackrel{\text{def}}{=} H F A$$

$$dx_s = dx - G dx_s$$

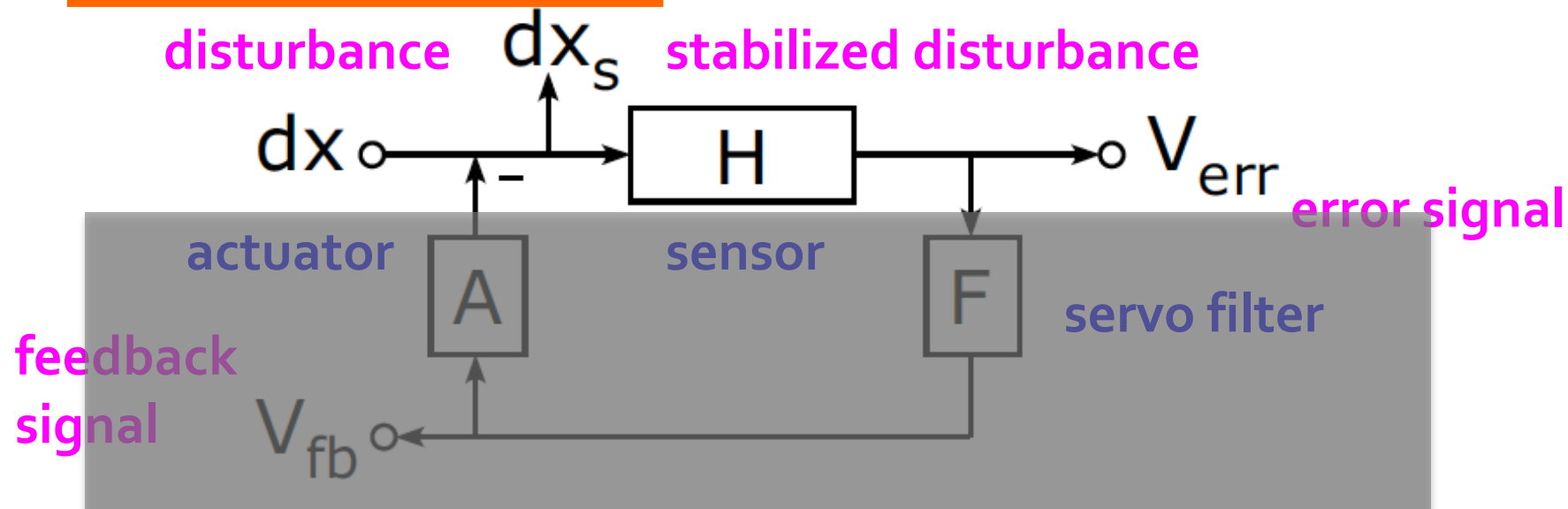
$$\Rightarrow dx_s = dx / (1+G)$$

$$\Rightarrow dx = V_{err} (1+G) / H$$

$$dx = V_{fb} A (1+G) / G$$

# Introduction ~ Control?

## ■ When G is small:



Open loop transfer function

$$G \stackrel{\text{def}}{=} H F A$$

$$dx_s = dx - G dx_s$$

$$\Rightarrow dx_s = dx / (1 + G)$$

$$\Rightarrow dx = V_{err} (1 + G) / H$$

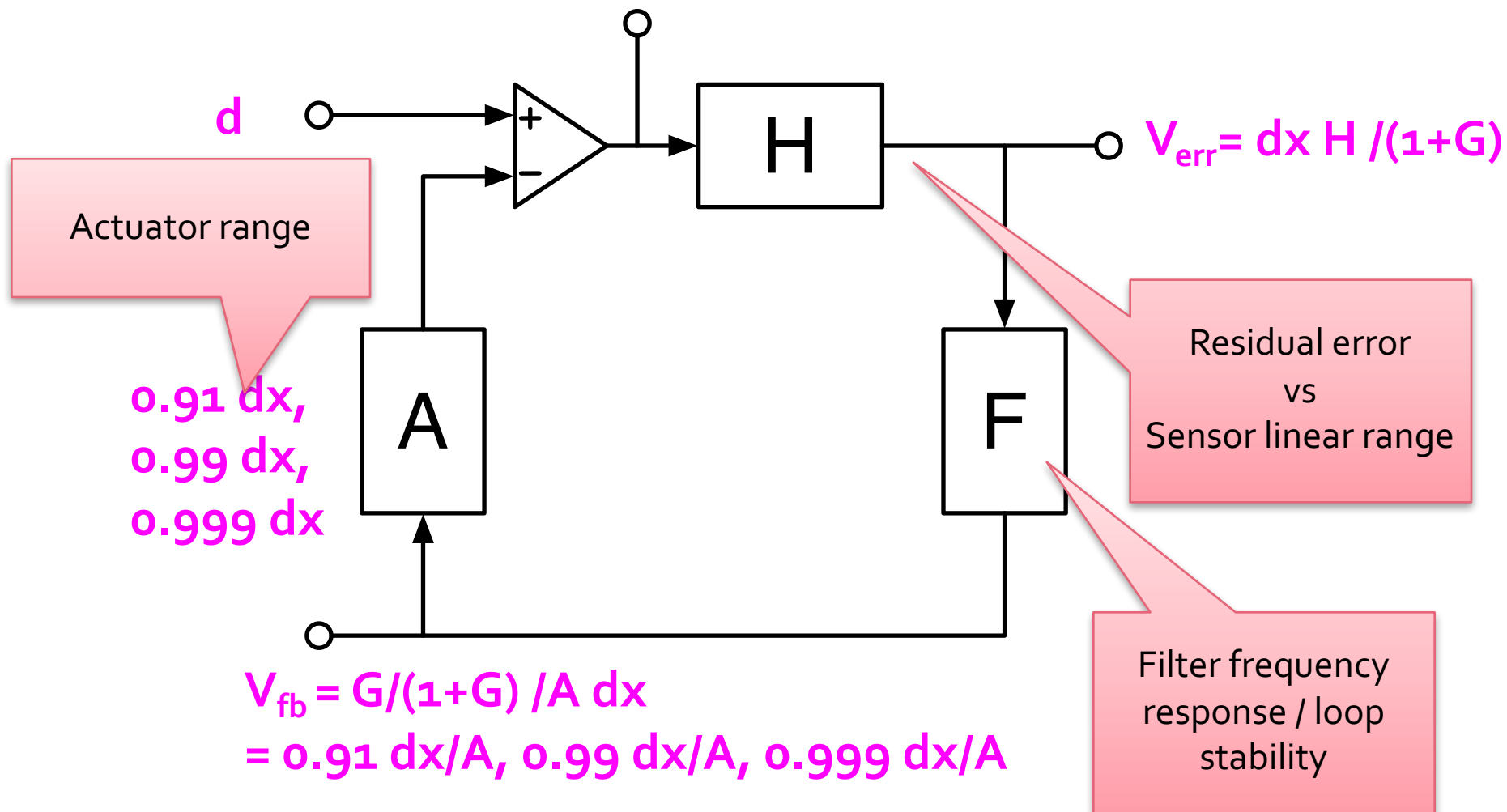
$$dx = V_{fb} A (1 + G) / G$$



# Introduction ~ Control?

- When  $G$  is big: e.g.  $G = 10, 100, \text{ or } 1000$

$$dx_s = dx / (1 + G) = 0.09 dx, 0.01 dx, 0.001 dx$$

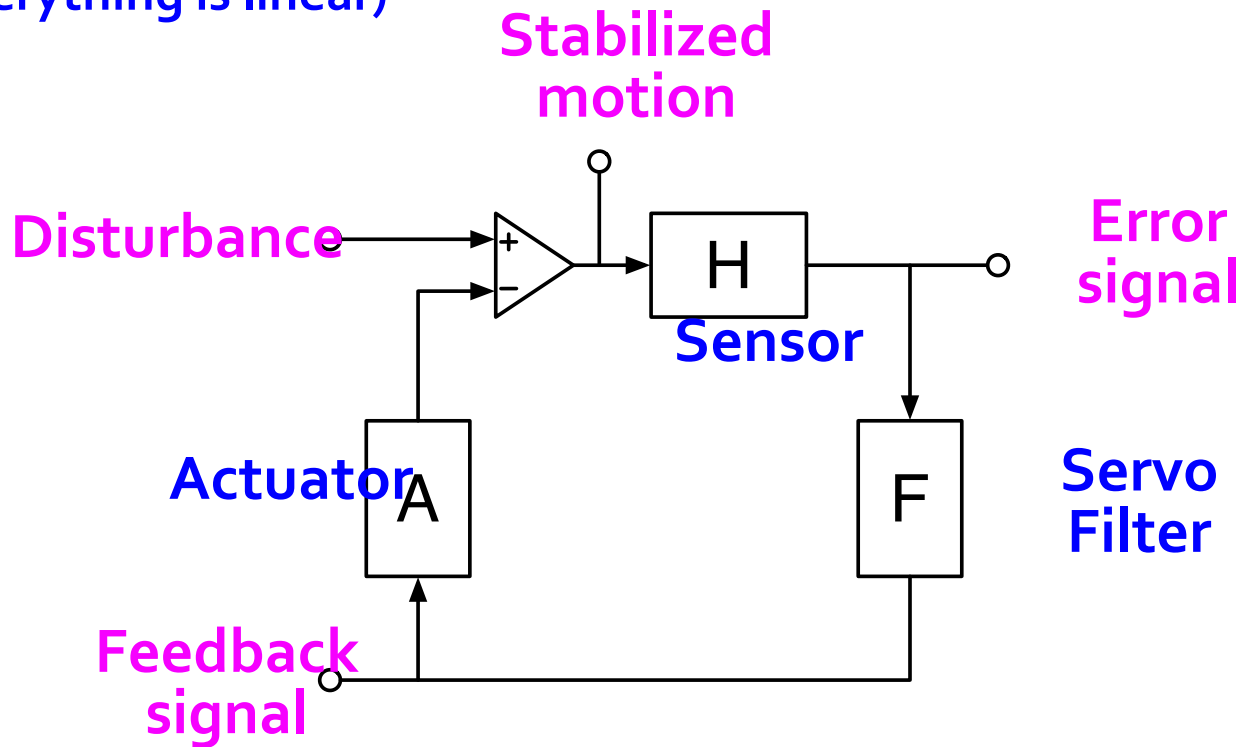


# Introduction ~ Control?

- When the openloop gain  $G$  is  $\gg 1$ , the error signal gets suppressed
- **“Wow! our sensor signal became smaller!”**
  - Is our system more sensitive now? => **No**
  - Then, can we still measure gravitational waves even if the error signal is almost zero? => **Yes**

# Introduction ~ Control?

- Important difference between
  - “Feedback control for stabilization”  
and “Feedback control for measurement”
  - Feedback control changes **the stabilized motion**  
but **reconstructed Disturbance** is not modified by the loop\*  
(\*if everything is linear)



# Linear systems and their stability

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# Linear systems and their stability

- A deterministic and time-invariant system:  $H$



- The system  $H$  is LTI (linear & time-invariant) when

$$y_1(t) = H \{x_1(t)\}$$

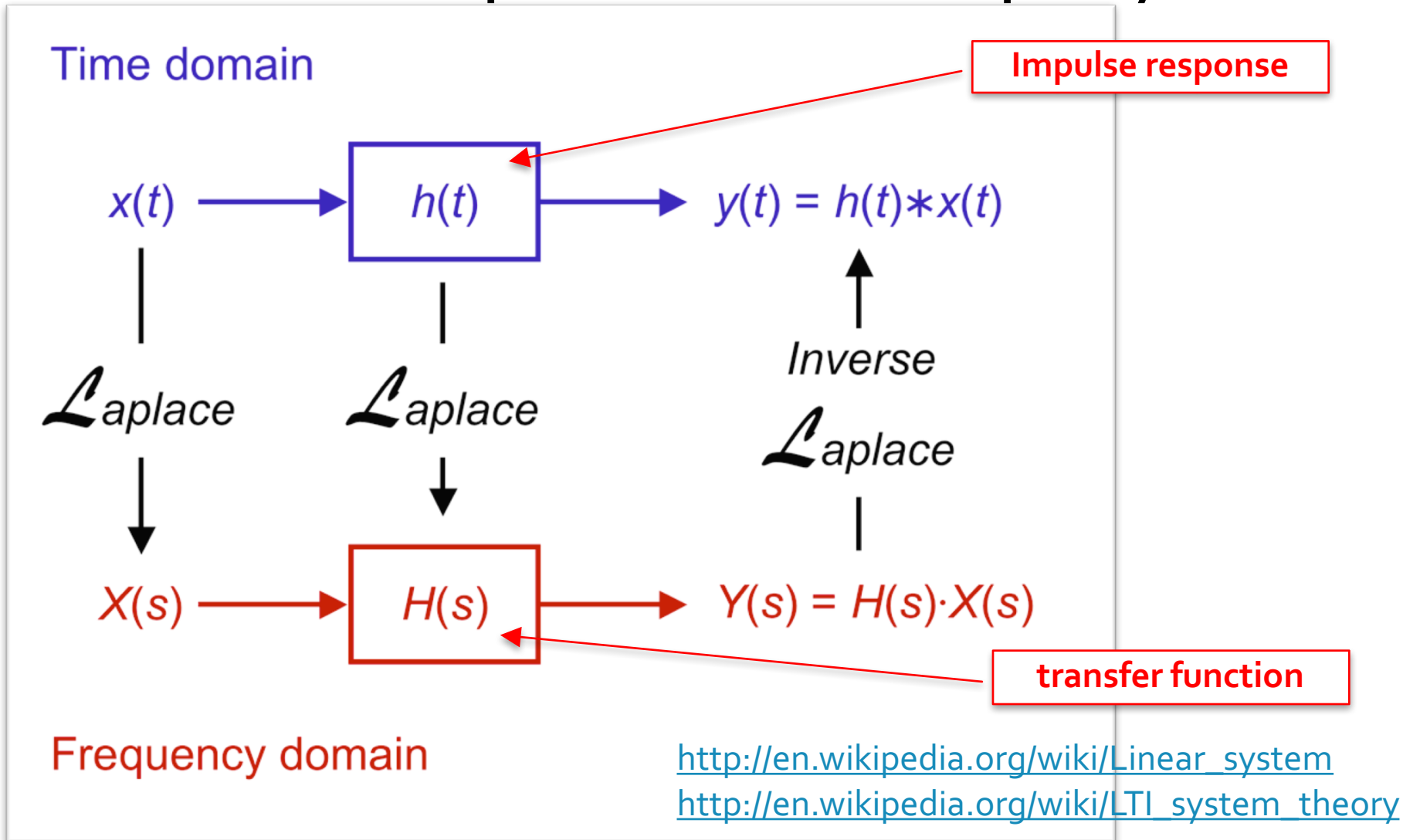
$$y_2(t) = H \{x_2(t)\}$$

$$\implies \alpha y_1(t) + \beta y_2(t) = H \{\alpha x_1(t) + \beta x_2(t)\}$$

- We can deal with such a system using Laplace transform (or almost equivalently Fourier Transform)

# Linear systems and their stability

- Time domain vs Laplace (or Fourier) frequency domain



# Linear systems and their stability

- It is easy to convert from an ordinary differential equation to a transfer function

$$\frac{d}{dt} \implies s$$

Laplace Transform

$$\implies i\omega = i2\pi f$$

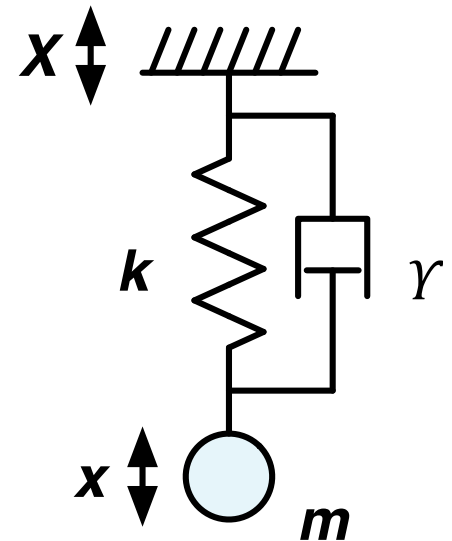
Fourier Transform

- e.g. Damped oscillator

$$m\ddot{x}(t) = -kx(t) - \gamma\dot{x}(t) + F(t)$$

$$ms^2 X(s) = -kX(s) - \gamma sX(s) + F(s)$$

$$H(s) \equiv \frac{X(s)}{F(s)} = \frac{1}{ms^2 + \gamma s + k}$$



# Linear systems and their stability

- e.g. Damped oscillator

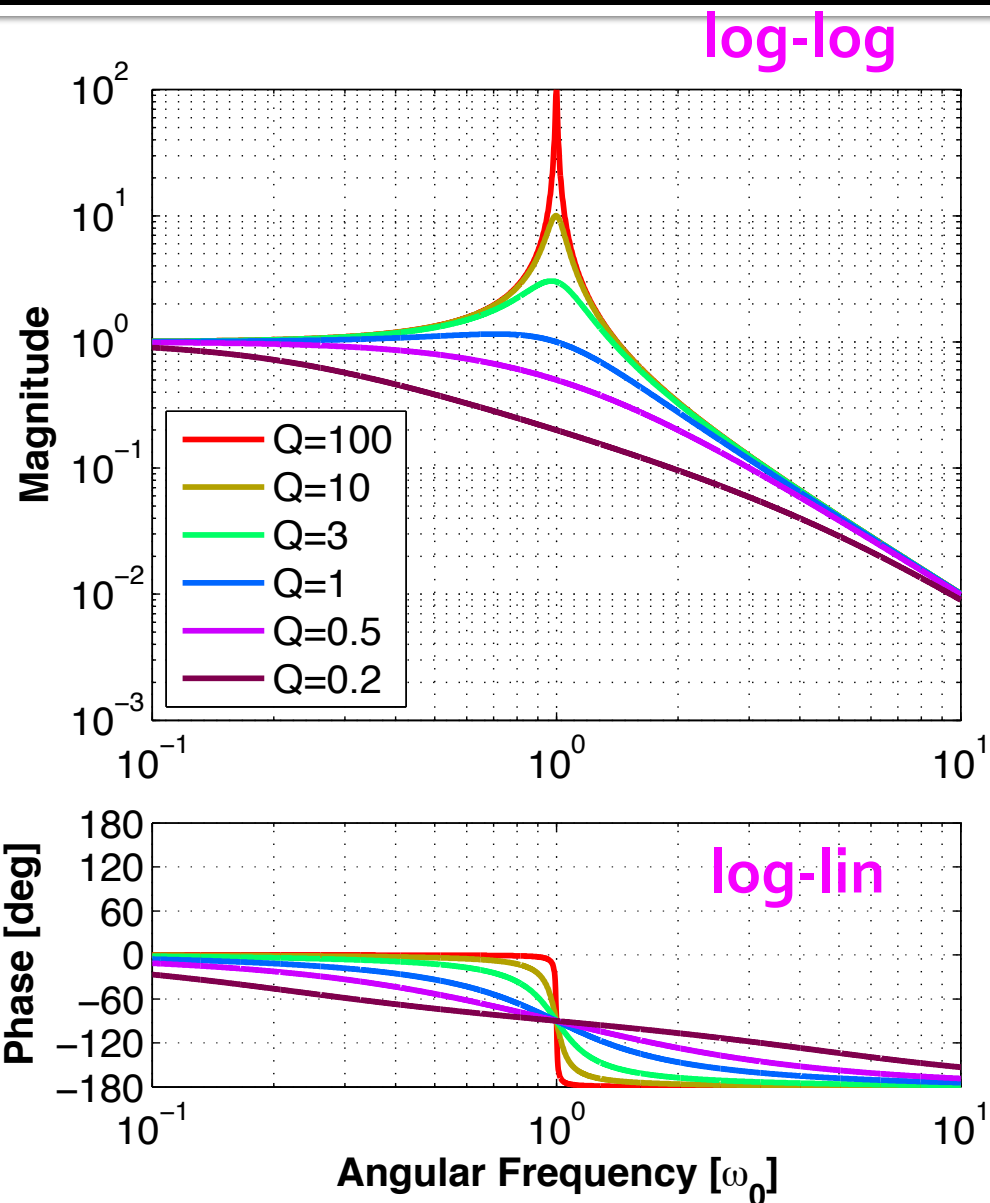
$$H(s) = \frac{1}{ms^2 + \gamma s + k}$$

$$H(s) = \frac{1}{m} \frac{1}{s^2 + \frac{\omega_0}{Q} s + \omega_0^2}$$

$$H(\omega) = \frac{1}{m} \frac{1}{-\omega^2 + i\frac{\omega_0}{Q}\omega + \omega_0^2}$$

$$\omega_0 = \sqrt{k/m}, \quad \gamma = m\omega_0/Q$$

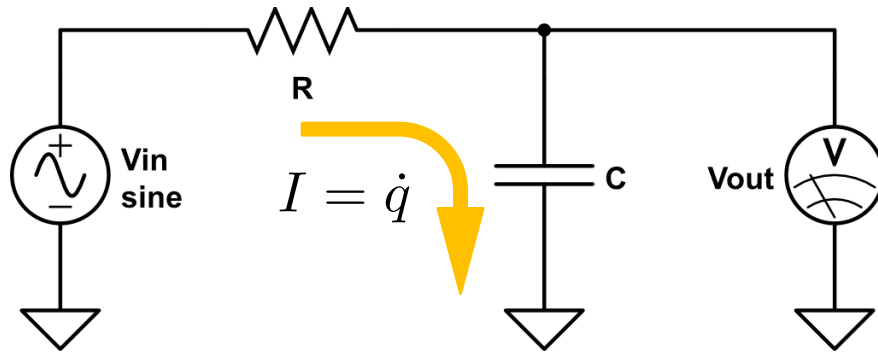
**Bode diagram**





# Linear systems and their stability

- e.g. RC filter

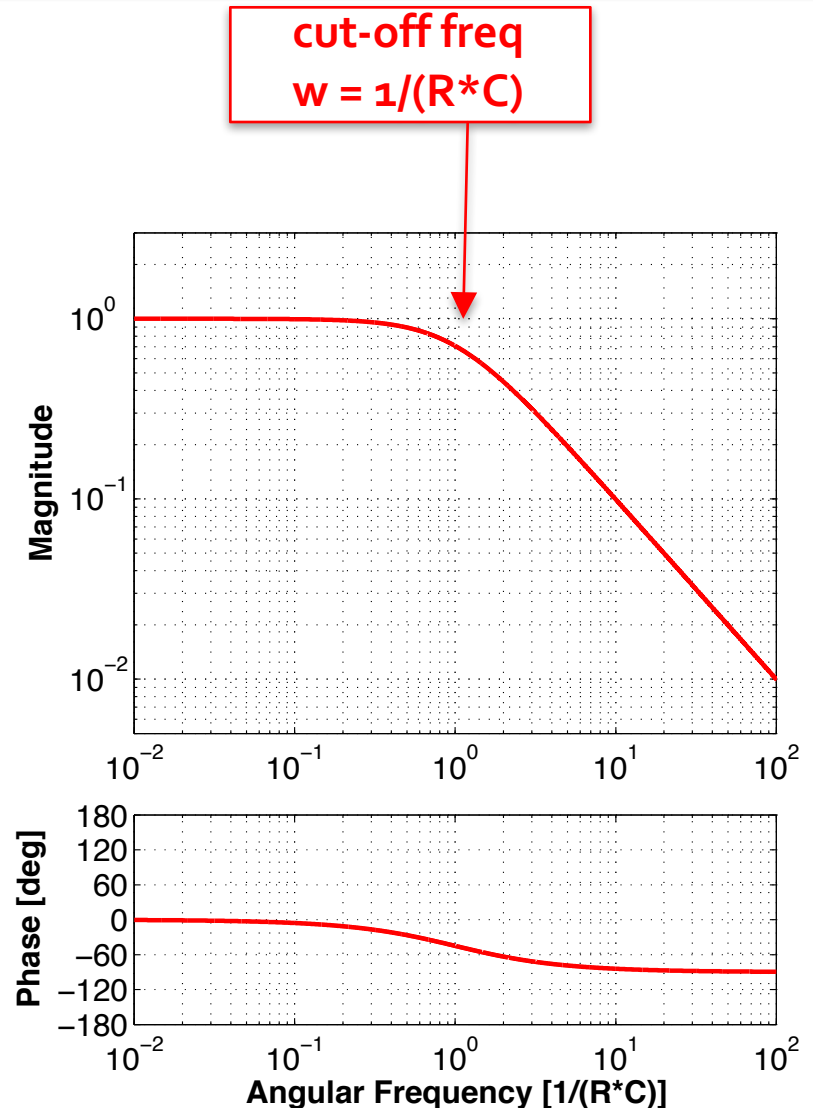


$$V_{\text{out}} = q/C$$

$$\dot{q} = (V_{\text{in}} - V_{\text{out}})/R$$

$$\Rightarrow i\omega C V_{\text{out}}(\omega) = (V_{\text{in}}(\omega) - V_{\text{out}}(\omega))/R$$

$$\Rightarrow \frac{V_{\text{out}}(\omega)}{V_{\text{in}}} = \frac{1}{1 + i\omega RC}$$



# Linear systems and their stability

- In most cases, a system TF can be expressed as:

$$H(s) = \frac{b_0 + b_1s + b_2s^2 + \dots + b_ms^m}{a_0 + a_1s + a_2s^2 + \dots + a_ns^n}$$

- The roots of the numerator are called as “zeros” and the roots of the denominator are called as “poles”

$$H(s) = \frac{b_m \prod_{i=1}^m (s - s_{zi})}{a_n \prod_{j=1}^n (s - s_{pj})}$$

- Zeros ( $s_{zi}$ ) and poles ( $s_{pi}$ ) are

real numbers (single zeros/poles)

or

pairs of complex conjugates (complex zeros/poles)

# Linear systems and their stability

- Poles rule the stability of the system!

H(s) can be rewritten as

$$H(s) = \sum_{j=1}^n \frac{K_j}{(s - s_{pj})}$$

- Each term imposes exponential time impulse response

$$\text{T.F.}: H_j(s) = \frac{1}{s - s_{pj}} \iff \text{I.R.}: h_j(t) = e^{s_{pj}t}$$

- **Therefore, if there is ANY pole with  $\text{Re}(s_{pj}) > 0$  the response of the system diverges**

# Linear systems and their stability

- Poles rule the stability of the system!

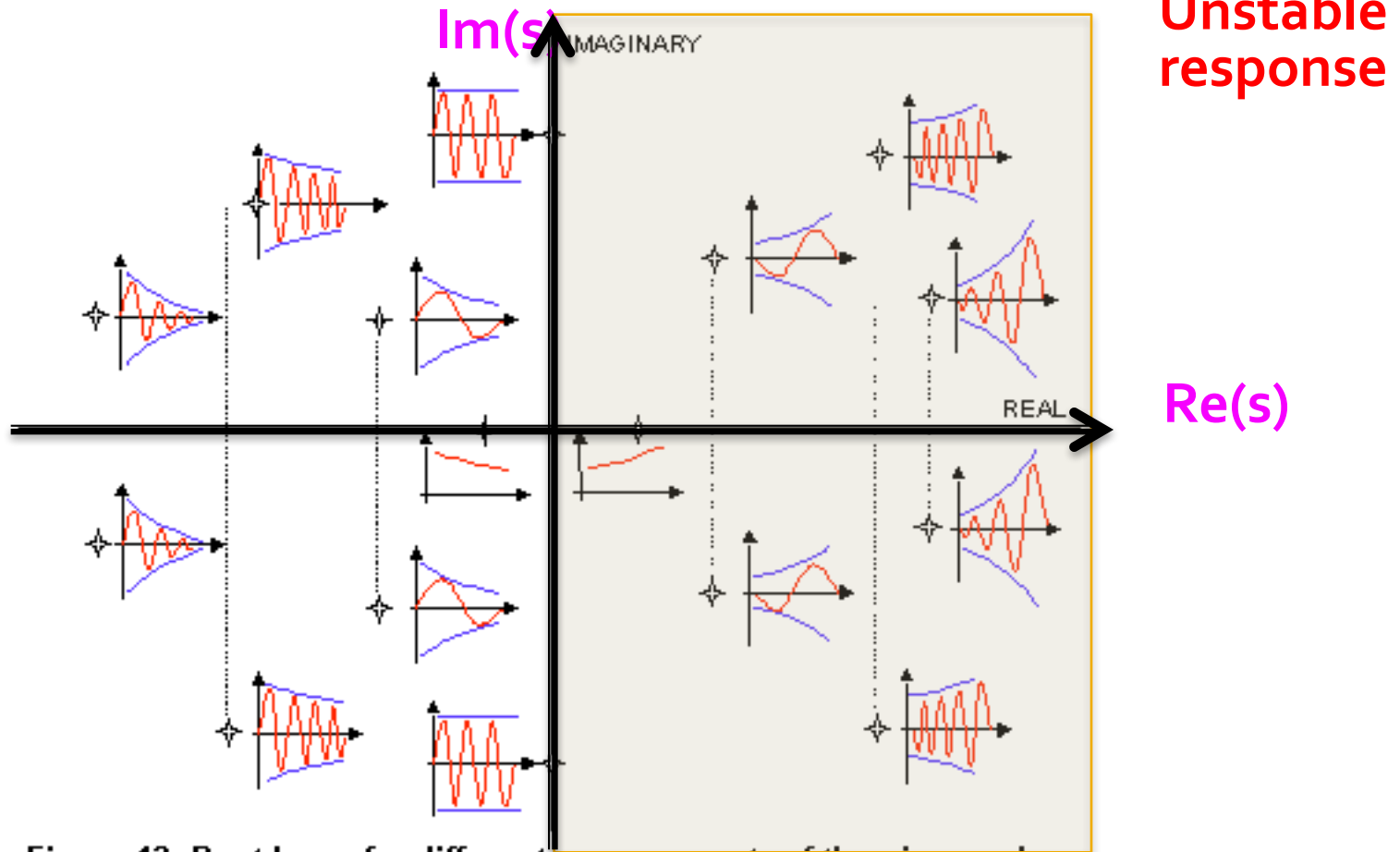
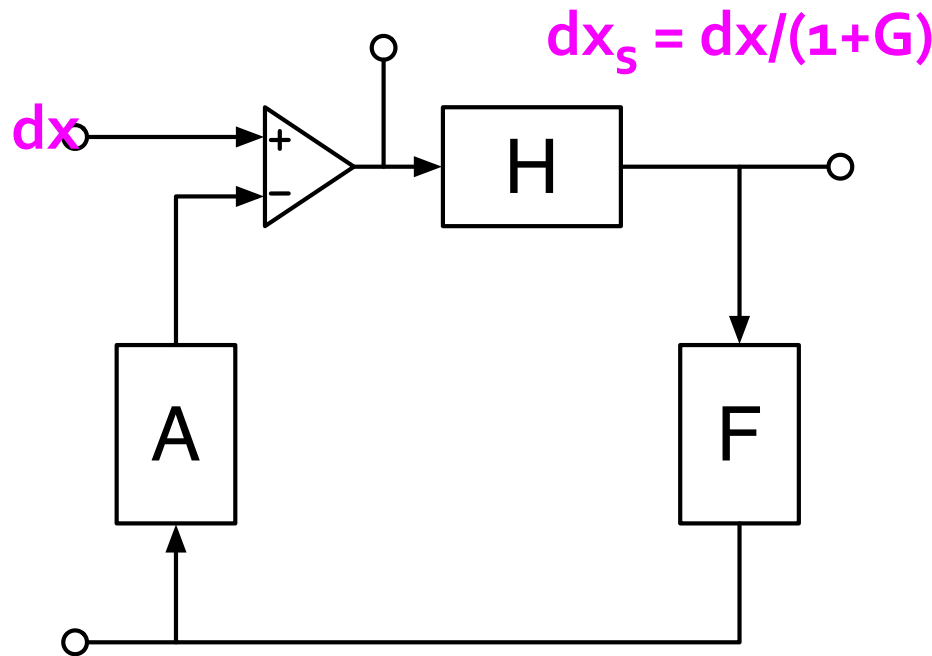


Figure 12: Root locus for different arrangements of the eigen values

# Linear systems and their stability

- Now we eventually came back to this diagram



Open loop TF:  
 $G = H F A$

Closed loop TF:  
 $G_{CL} = 1/(1+G)$

Requirement:

All the roots for  $1+G$  should be  
in the left hand side of Laplace plane

(!)

# Linear systems and their stability

## ■ Remarks

### Requirement:

All the roots for  $1+G$  should be in the left hand side of Laplace plane

- This does not mean all  $H$ ,  $F$ ,  $A$  needs to be stable.  
e.g. Unstable mechanical system  $A$  can be stabilized by a control loop. (cf. An inverted Rod)

- We usually play with  $F$  to tune the result.

It is awkward to evaluate the stability

of  $1/(1+G)$  every time.

There is a way to tell the stability only from  $G$

Open loop TF:  
 $G = H F A$

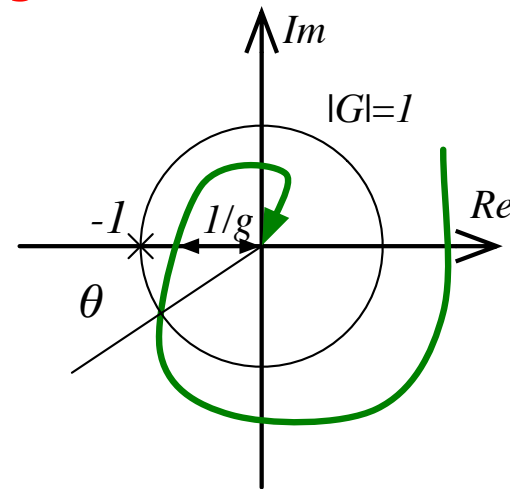
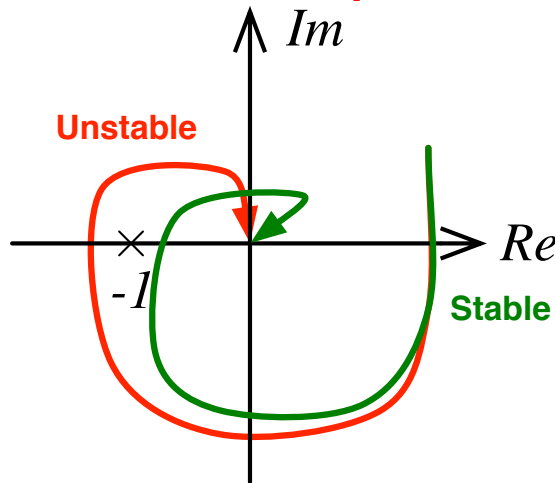
Closed loop TF  
 $G_{CL} = 1/(1+G)$

**Nyquist's stability criterion**

# Linear systems and their stability

## ■ Nyquist stability criterion

- Plot openloop gain  $G$  in a complex plane (i.e. Nyquist diagram)
- If the locus of  $G(f)$  from  $f=0$  to  $\infty$ , goes to 0 looking at the point  $(-1 + 0 i)$  at the left side => Stable
- If the locus sees the point  $(-1+0 i)$  at the right side => Unstable



- Unity gain frequency  $f_{UGF}$  :
- Phase margin  $\vartheta$  :
- Gain margin  $g$ :

$$\text{for } |G(f_{UGF})| = 1$$

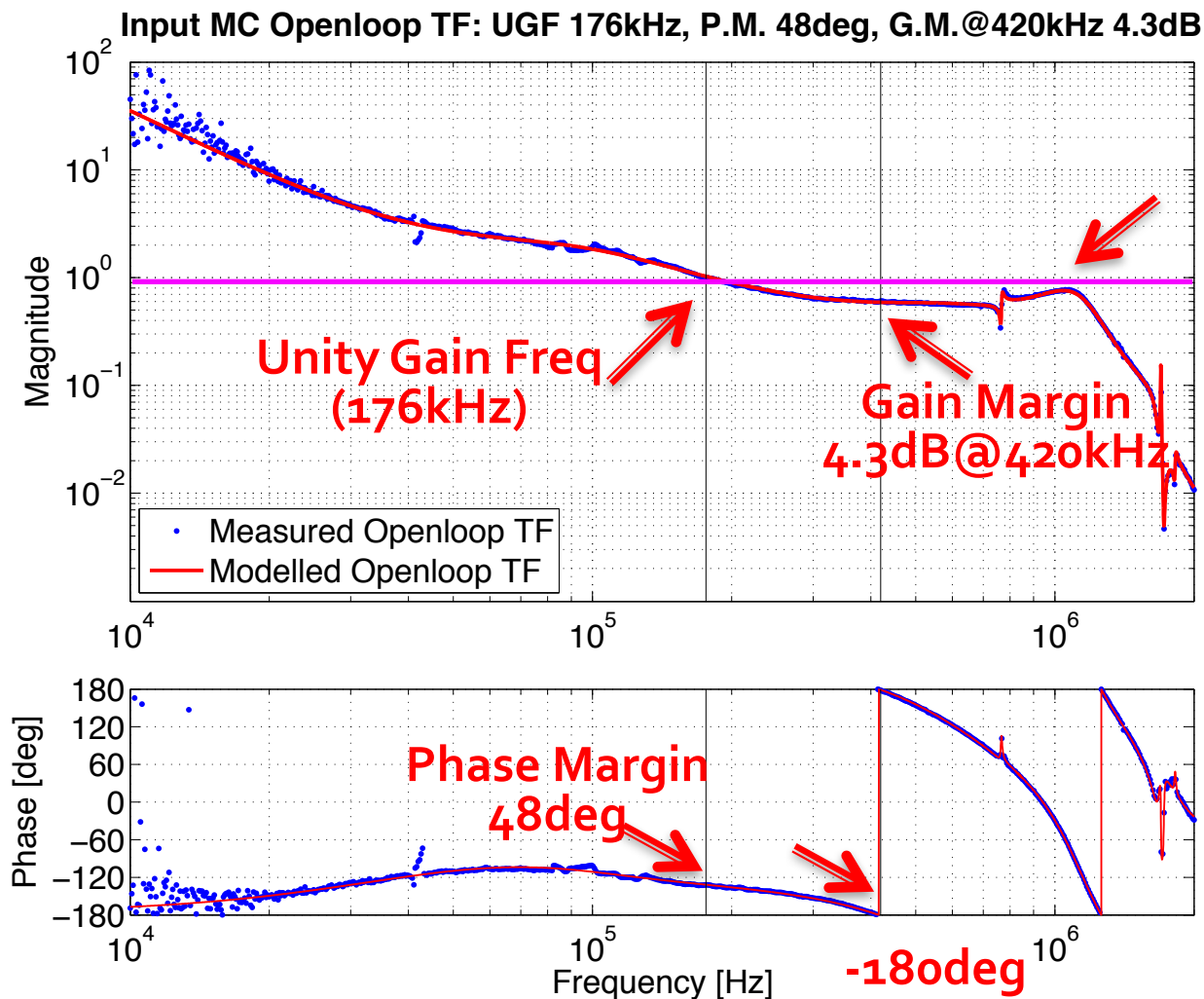
$$\vartheta = \text{Arg}(G(f_{UGF}))$$

$$g = 1/|G(f_o)| \text{ where } \text{Arg}(G(f_o)) = -\pi$$

# Linear systems and their stability

## Phase Margin / Gain Margin in Bode diagram

- Most of the case, a bode diagram of  $G$  is enough to see the stability



Nearly unstable

A rough standard of a stable servo loop:  
Phase Margin > 40deg  
Gain Margin > 10dB



# Linear systems and their stability

## ■ Building blocks (“zpk” representation)

### ■ Single pole

$$H(s) = \frac{s_p}{s + s_p} \quad (s_p \in \mathbb{R}, s_p > 0)$$

### ■ Single zero

$$H(s) = \frac{s + s_z}{s_z} \quad (s_z \in \mathbb{R}, s_z > 0)$$

### ■ A pair of complex poles

$$H(s) = \frac{s_p s_p^*}{(s + s_p)(s + s_p^*)} \quad (s_p \in \mathbb{C}, \Re(s_p) > 0)$$

### ■ A pair of complex zeros

$$H(s) = \frac{(s + s_z)(s + s_z^*)}{s_z s_z^*} \quad (s_z \in \mathbb{C}, \Re(s_z) > 0)$$

### ■ Gain

$$H(s) = K \quad (K \in \mathbb{R})$$

# Linear systems and their stability

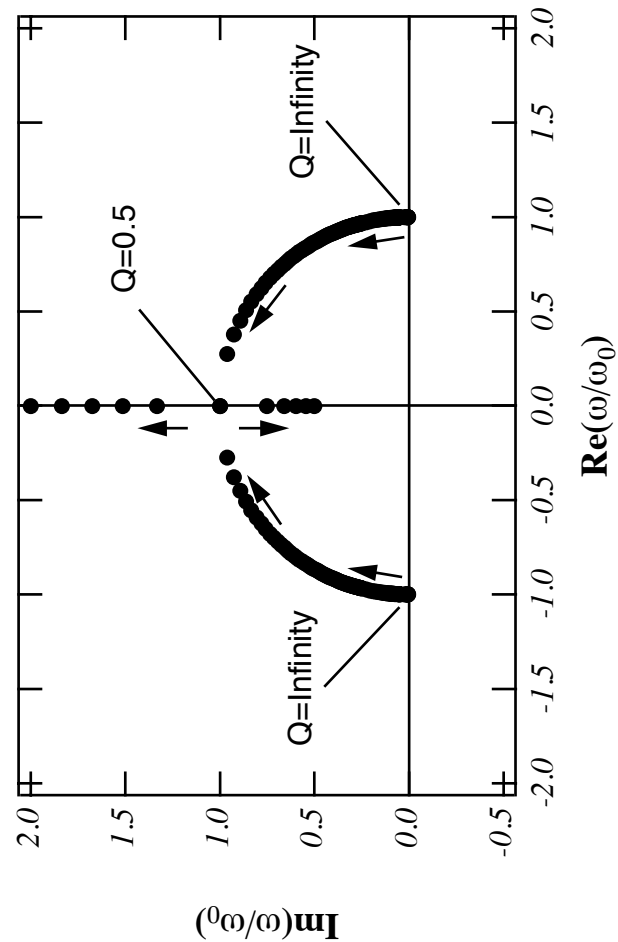
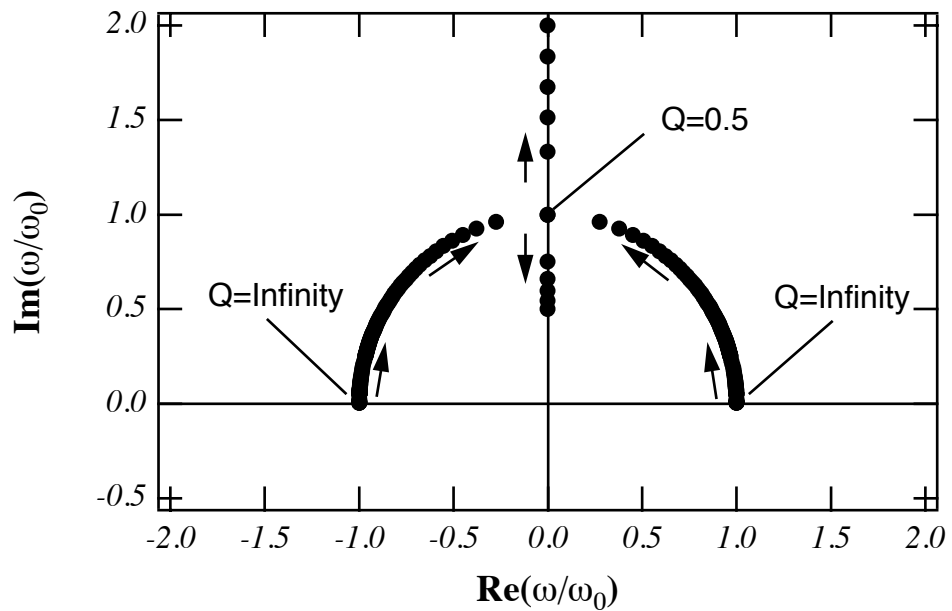
- Relationship between pole/zero locations and  $\omega_0$  &  $Q$

$$\begin{aligned} H(s) &= \frac{s_p s_p^*}{(s + s_p)(s + s_p^*)} \\ &= \frac{|s_p|^2}{s^2 + 2\Re(s_p)s + |s_p|^2} \end{aligned}$$

- To be compared with

$$\begin{aligned} H(\omega) &= \frac{\omega_0^2}{-\omega^2 + i\omega_0\omega/Q + \omega_0^2} \\ \implies \omega_0 &= |s_p|, \quad Q = \frac{|s_p|}{2\Re(s_p)} \end{aligned}$$

# Linear systems and their stability



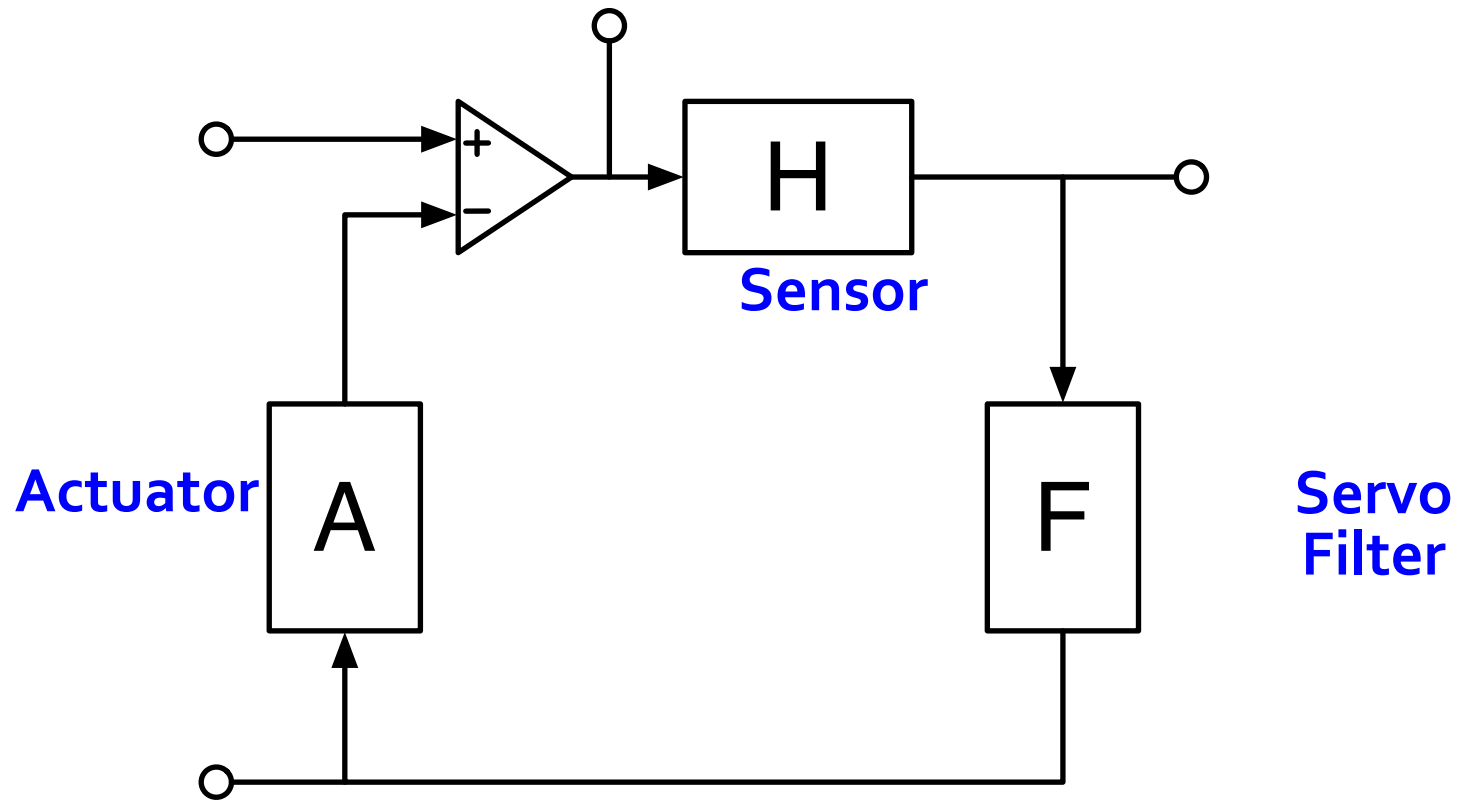
# Linear systems and their stability

- **Summary**
  - Classical control theory
  - Design locations of poles and zeros
  - Stability: tuning of open loop transfer function is important

# Control system components in GW detectors

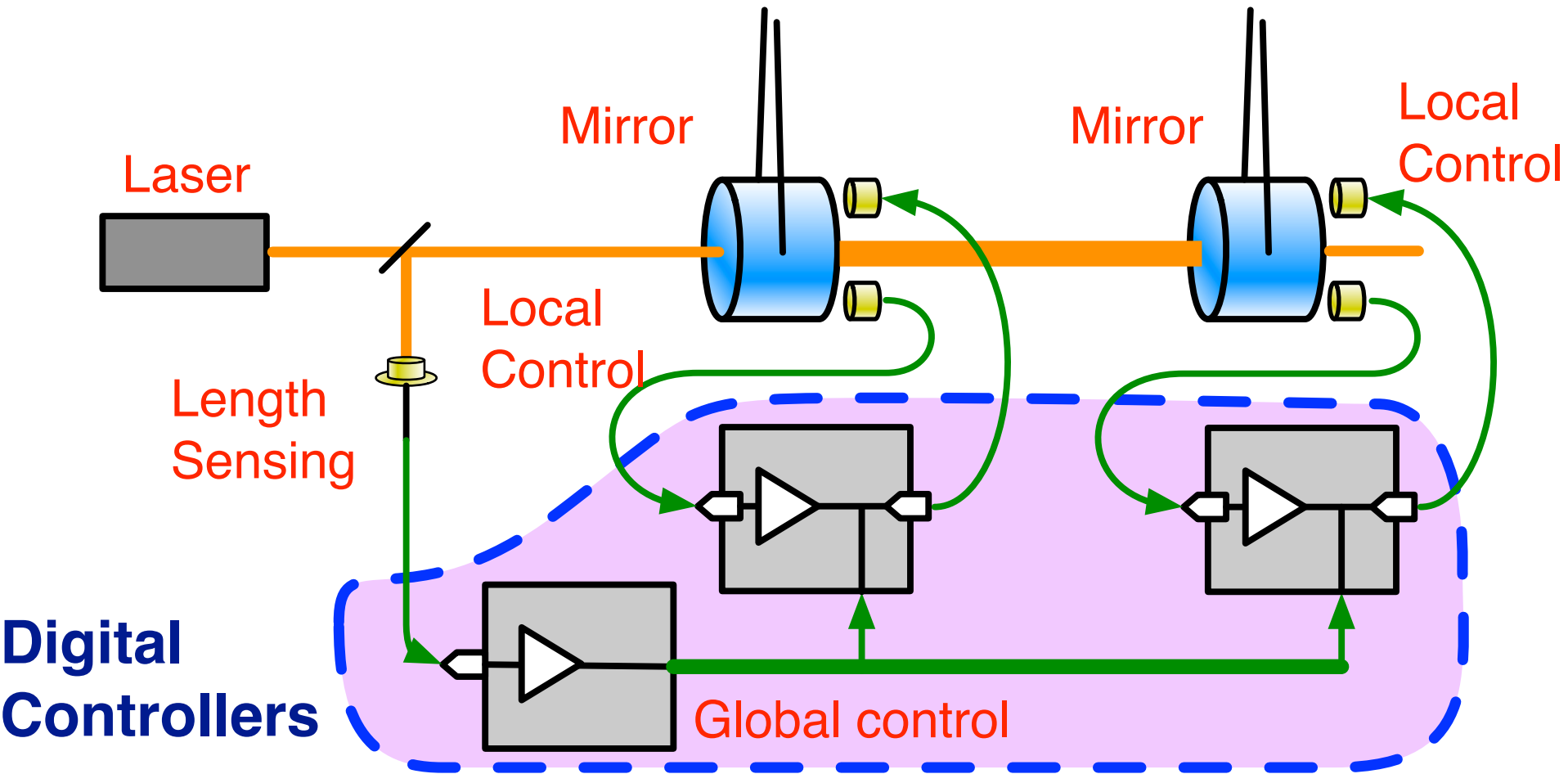
# Control systems

- Elements of a feedback loop (again)



# Interferometer control system

- Local control vs global control



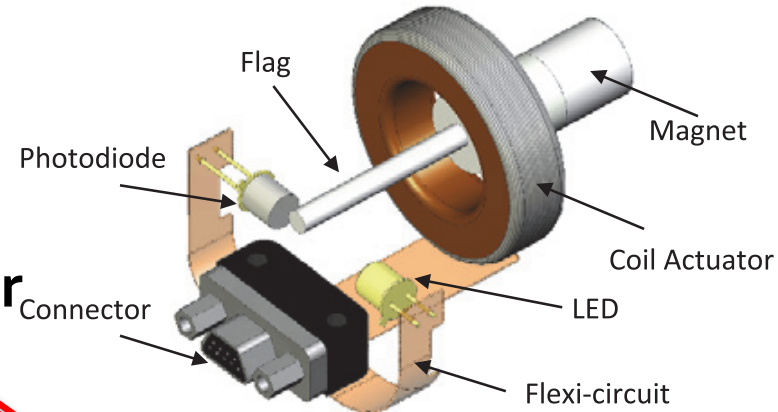
# Local Sensors

- Shadow sensor (relative displacement sensor)

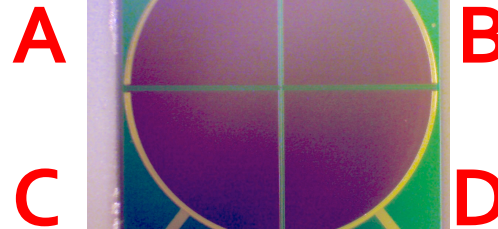
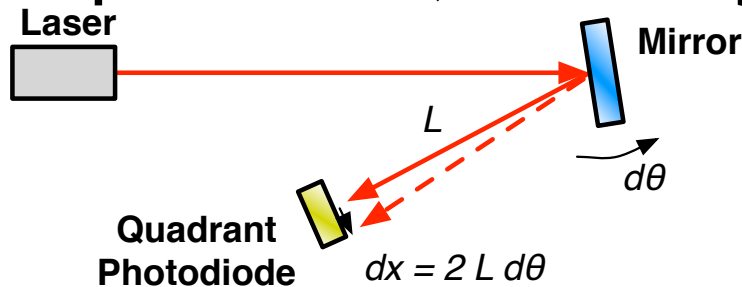
- aLIGO: Birmingham Optical Sensor and Electro-Magnetic actuator (BOSEM)

L Carbone, Class. Quantum Grav. 29 (2012) 115005

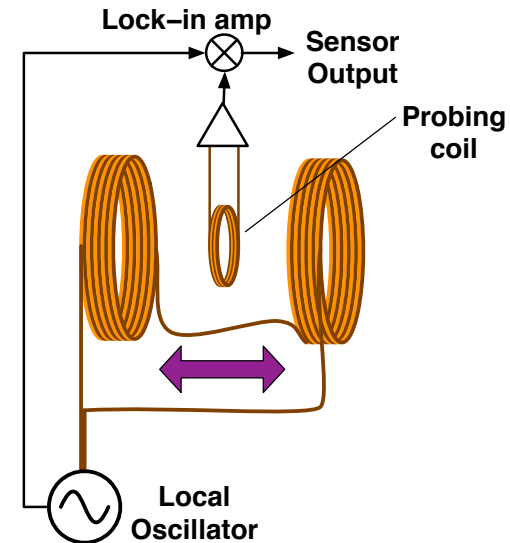
- Linear Variable Differential Transducer (relative disp. sensor)



- Optical Lever (relative angular sensor)



$$\begin{aligned}
 \text{SUM} &= A+B+C+D \\
 X &= A-B+C-D \\
 Y &= A+B-C-D
 \end{aligned}$$



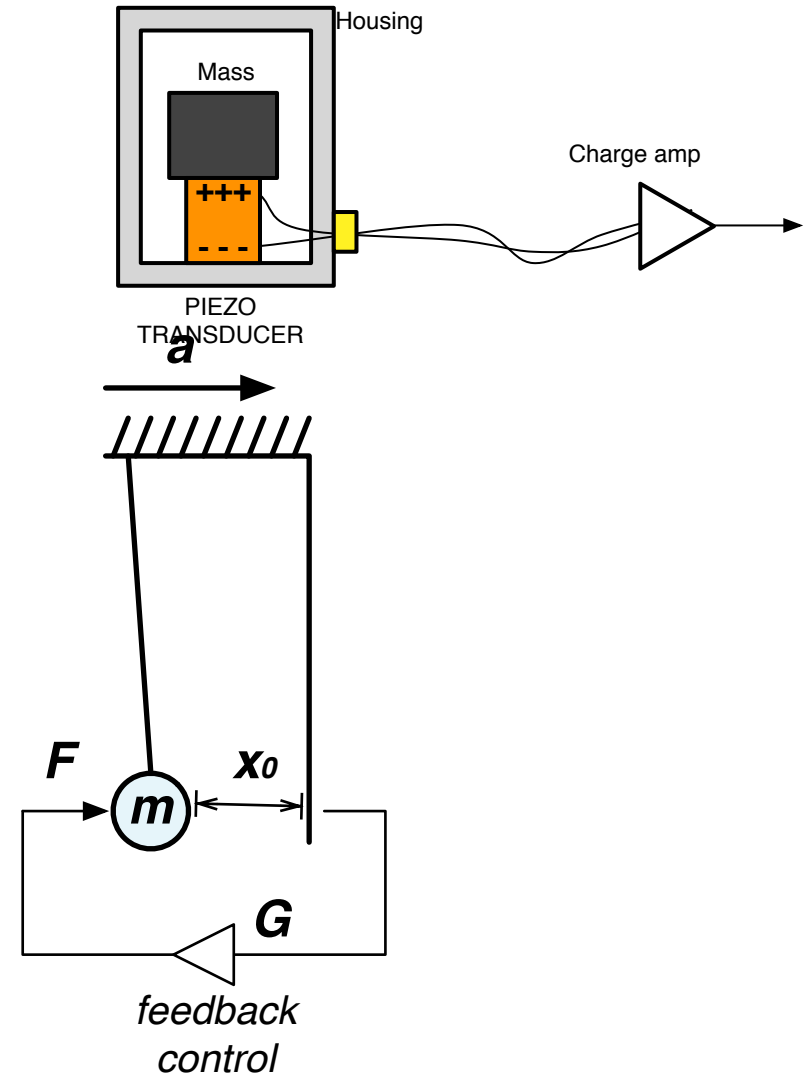


# Local Sensors

- Accelerometers (Inertial sensor)

- Piezo Accelerometer

- Servo Accelerometer



# Local Sensors

- Servo Accelerometer (Inertial sensor)
  - Above the resonant freq: Limited by the sensor noise
  - Below the resonant freq: Steep rise of the noise as the mass does not move in relative to the ground  
=> Low resonant freq is beneficial

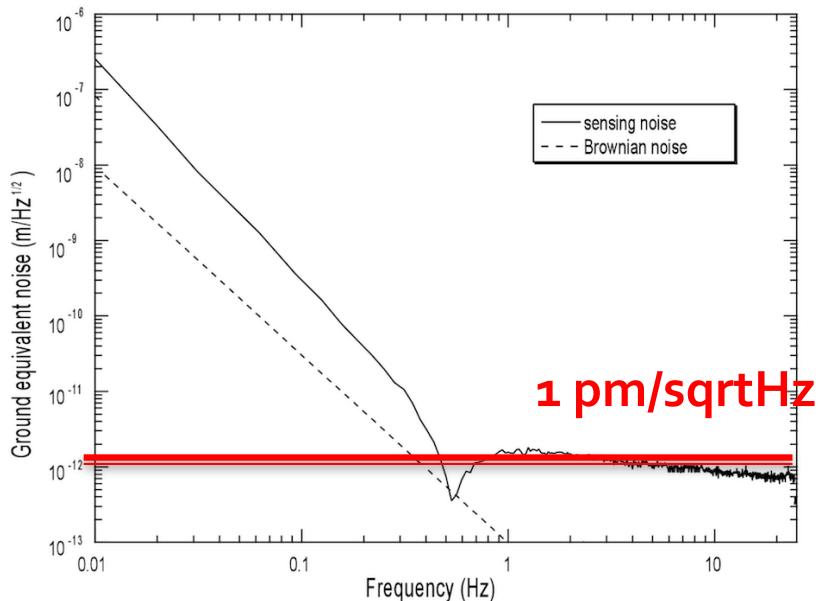
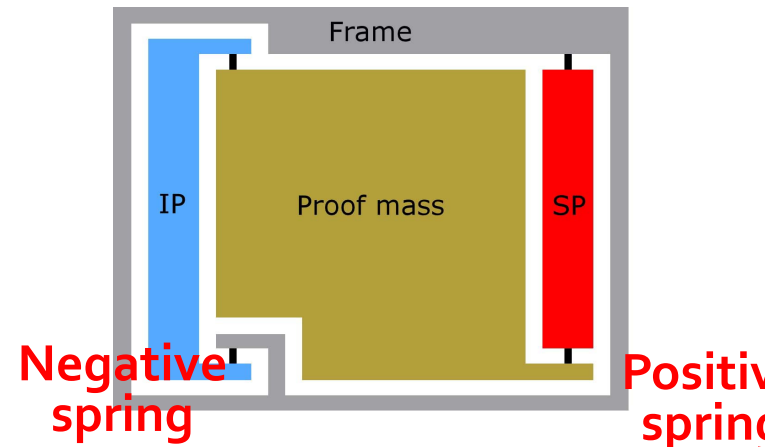
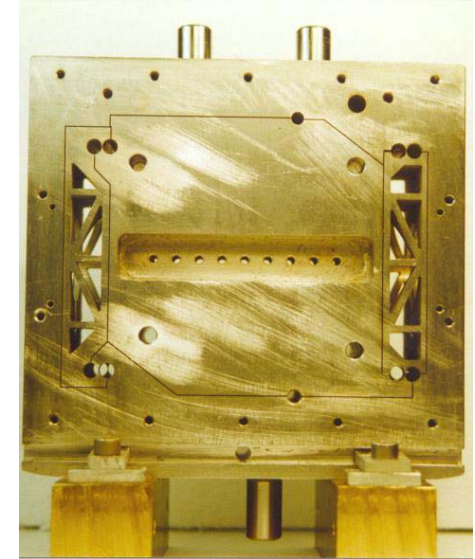


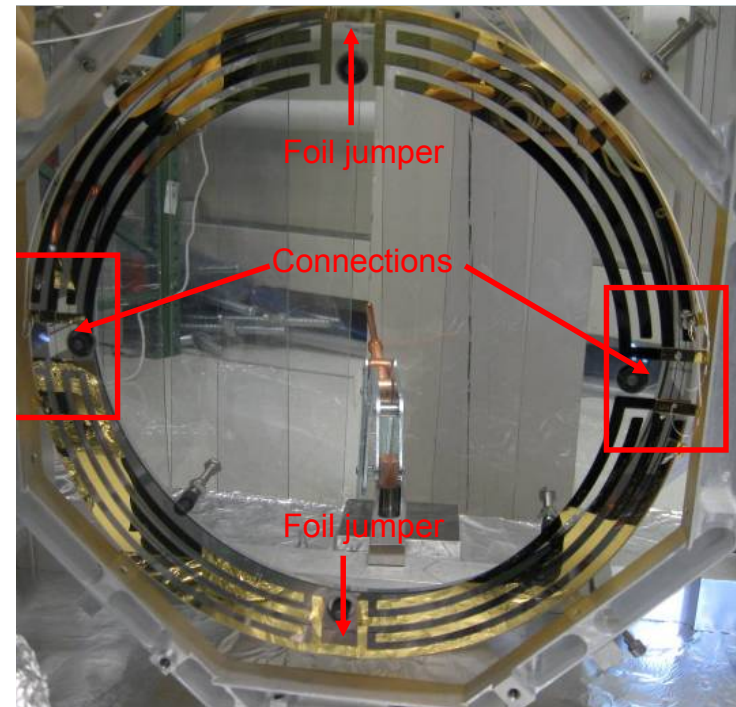
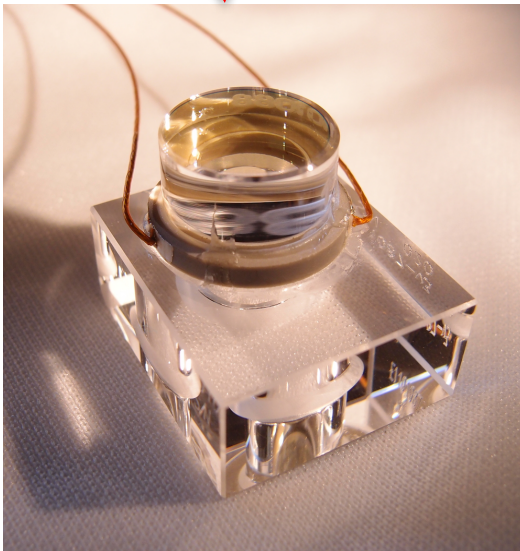
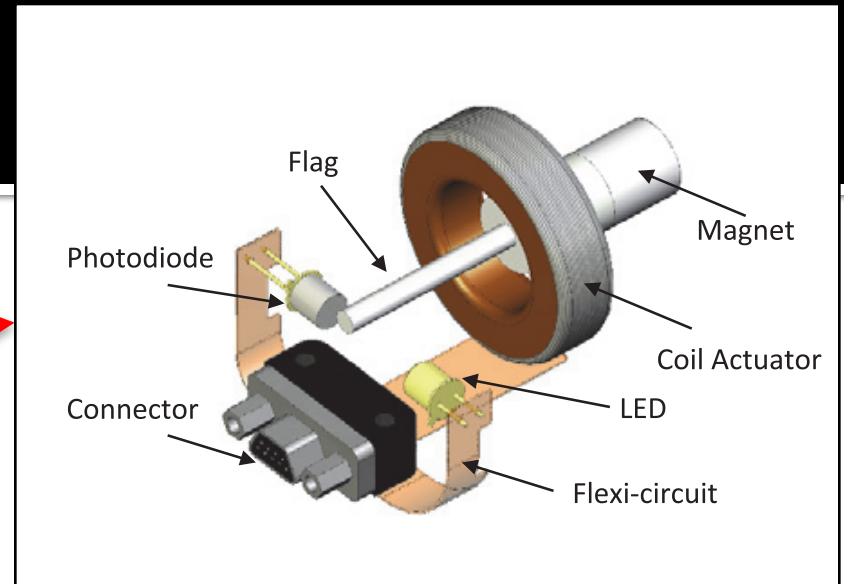
Fig.7. Equivalent frame displacement noise.



# Actuators

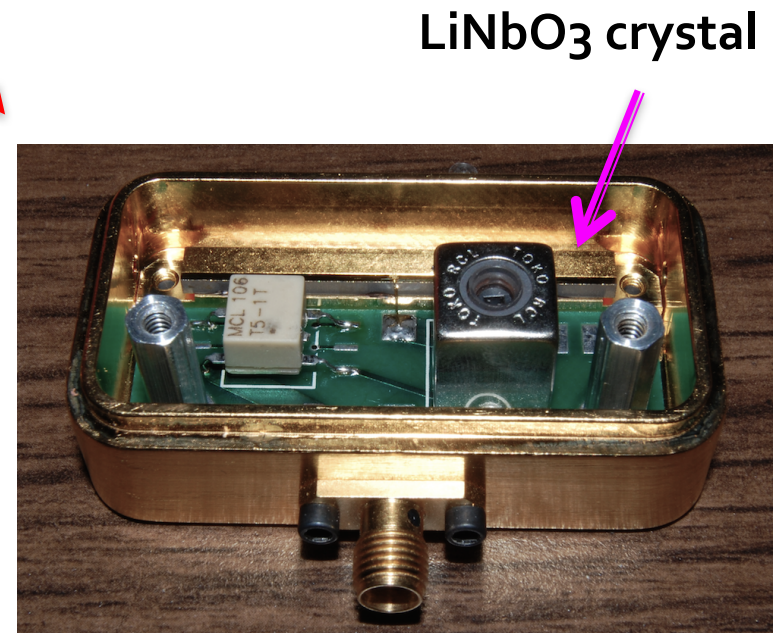
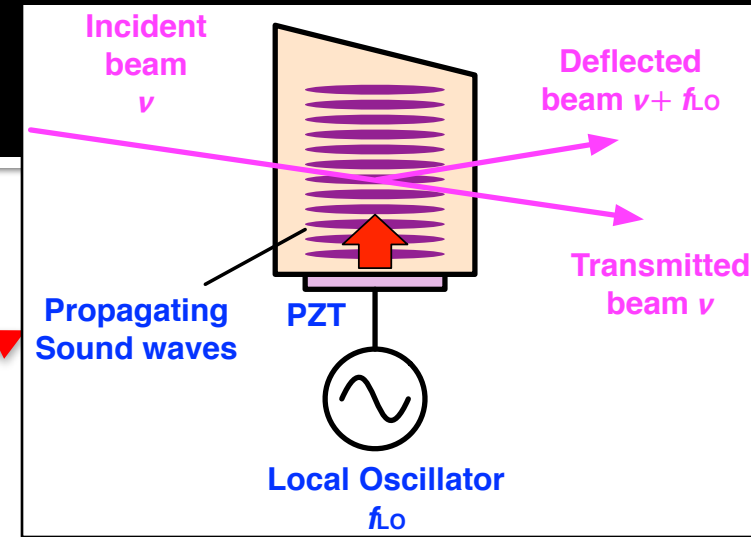
## ■ Mechanical actuators

- Coil Magnet actuator
- Electro Static Driver (ESD)
- Piezo (PZT) actuator



# Actuators

- Optical actuators
  - Acousto-Optic Modulator
  - Electro-Optic Modulator
  - Laser Frequency actuator
    1. Thermal actuator
    2. Fast piezo actuator
    3. External EOM



# Analog servo vs digital servo

- Comparison of the control room in the analog and digital eras



aLIGO (2014)



TAMA300 (2001)