

# Control system in Gravitational Wave Detectors

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# Introduction ~ Control?

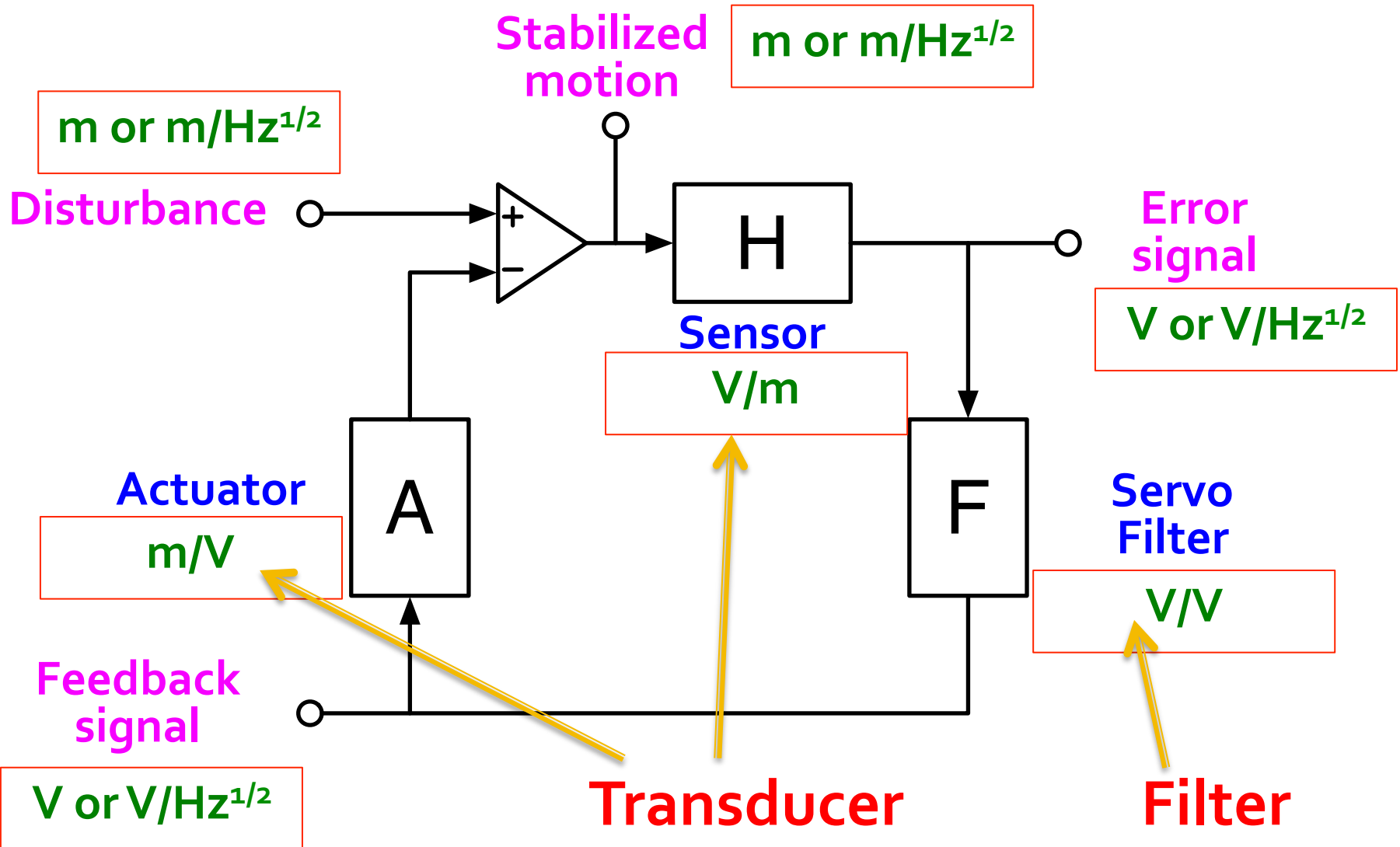
- **Gravitational wave detection**  
**Laser displacement sensor**  
**Requires linear displacement detection**
  
- **Control for measurement**  
**Laser interferometer = nonlinear device**  
**Feedback control => linearization**

# Introduction ~ Control?

- **What is the feedback control?**
  - A scheme to monitor and modify output(s) of a system by changing the input(s) depending on the output(s)
- **Examples**
  - Shower temperature
  - Car driving
  - Tight rope walking
  - Air conditioning
  - Bike riding
  - Inverted bar on a hand
- **Imagine what happens**
  - If the response is too slow?
  - If the response is too fast?

# Introduction ~ Control?

- Elements of a feedback loop





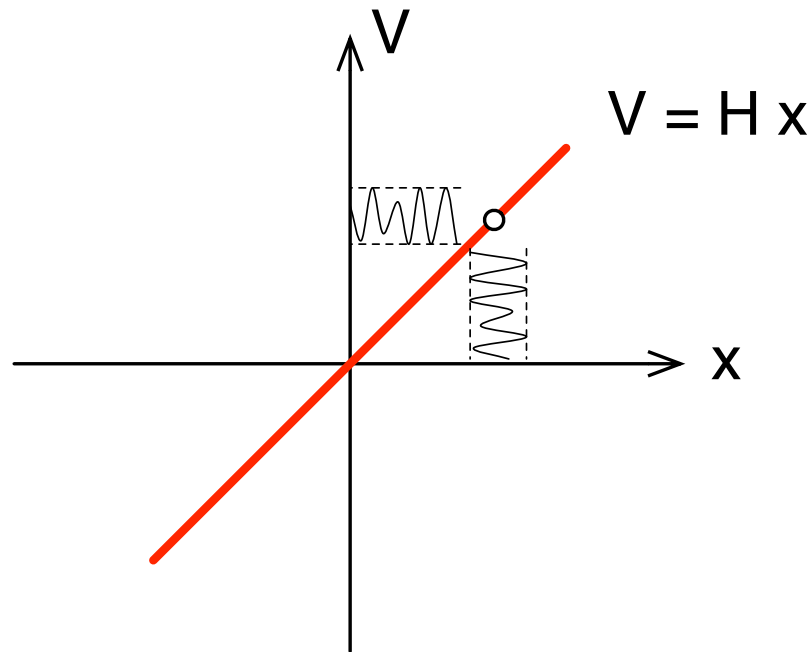
# Introduction ~ Control?

- Sensor:

Transducer for displacement-to-voltage conversion

- If the sensor is completely linear

(and has or no frequency dependence)

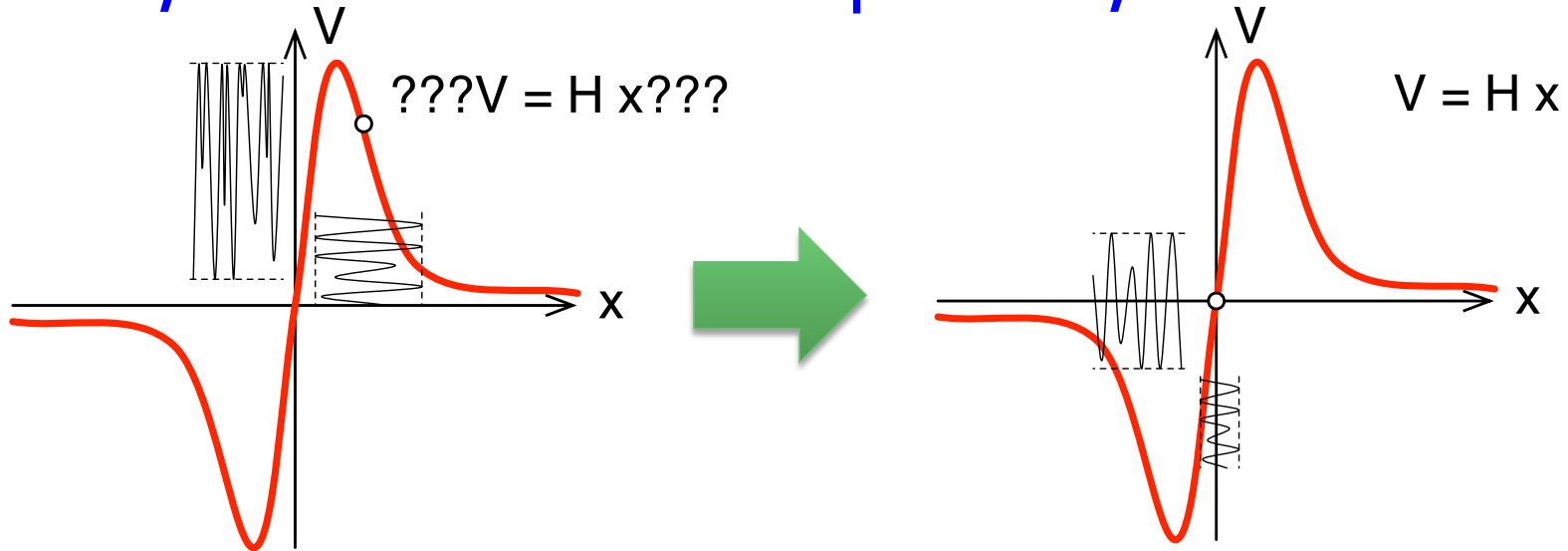


**We don't need feedback control!**

# Introduction ~ Control?

- In reality:

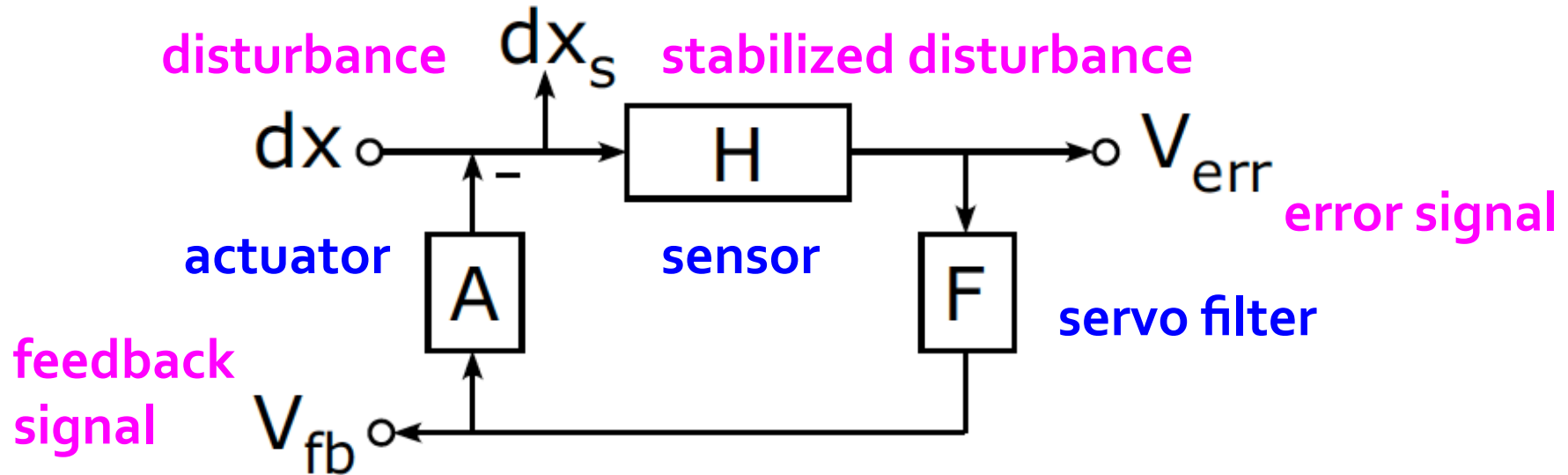
Sensors, laser interferometers in particular, are **nonlinear!**



- Enclose the operating point in the linear region  
=> The system recovers linearity
- Was the displacement modified by the feedback?  
=> Precise knowledge of the control system  
for signal reconstruction

# Introduction ~ Control?

- Elements of a feedback loop



$$dx_s = dx - G dx_s$$

$$\Rightarrow dx_s = dx / (1+G)$$

$$\Rightarrow dx = V_{err} (1+G) / H$$

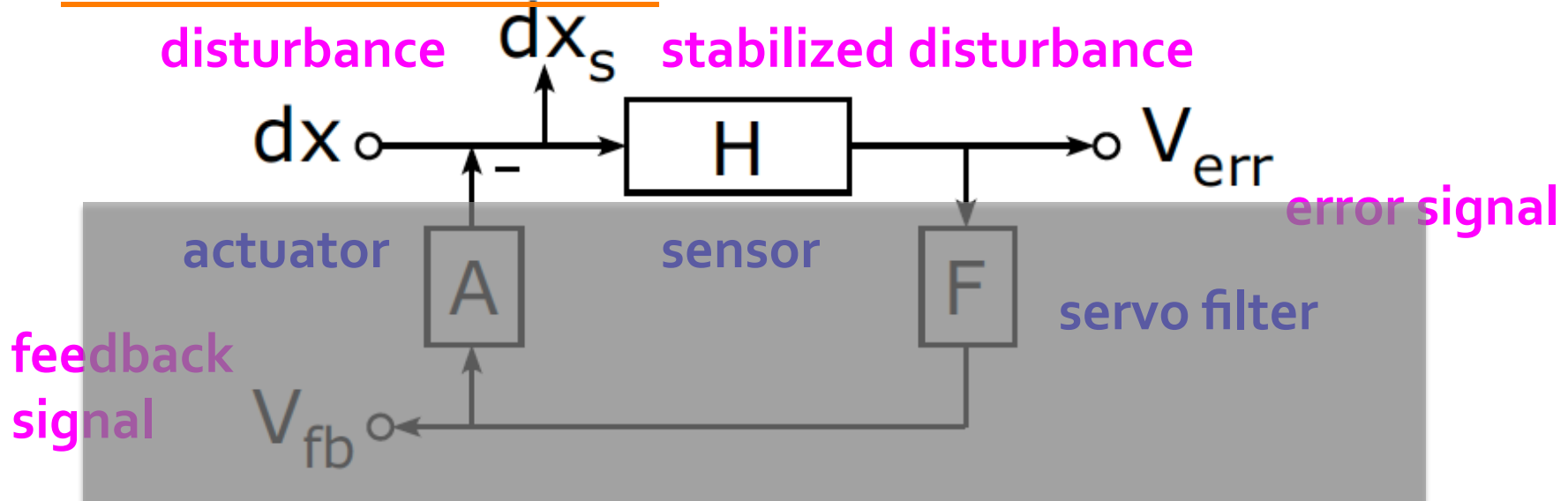
$$dx = V_{fb} A (1+G) / G$$

Open loop transfer function

$$G \stackrel{\text{def}}{=} H F A$$

# Introduction ~ Control?

## ■ When G is small:



$$dx_s = dx - G dx_s$$

$$\Rightarrow dx_s = dx / (1 + G)$$

$$\Rightarrow dx = V_{err} (1 + G) / H$$

$$dx = V_{fb} A (1 + G) / G$$

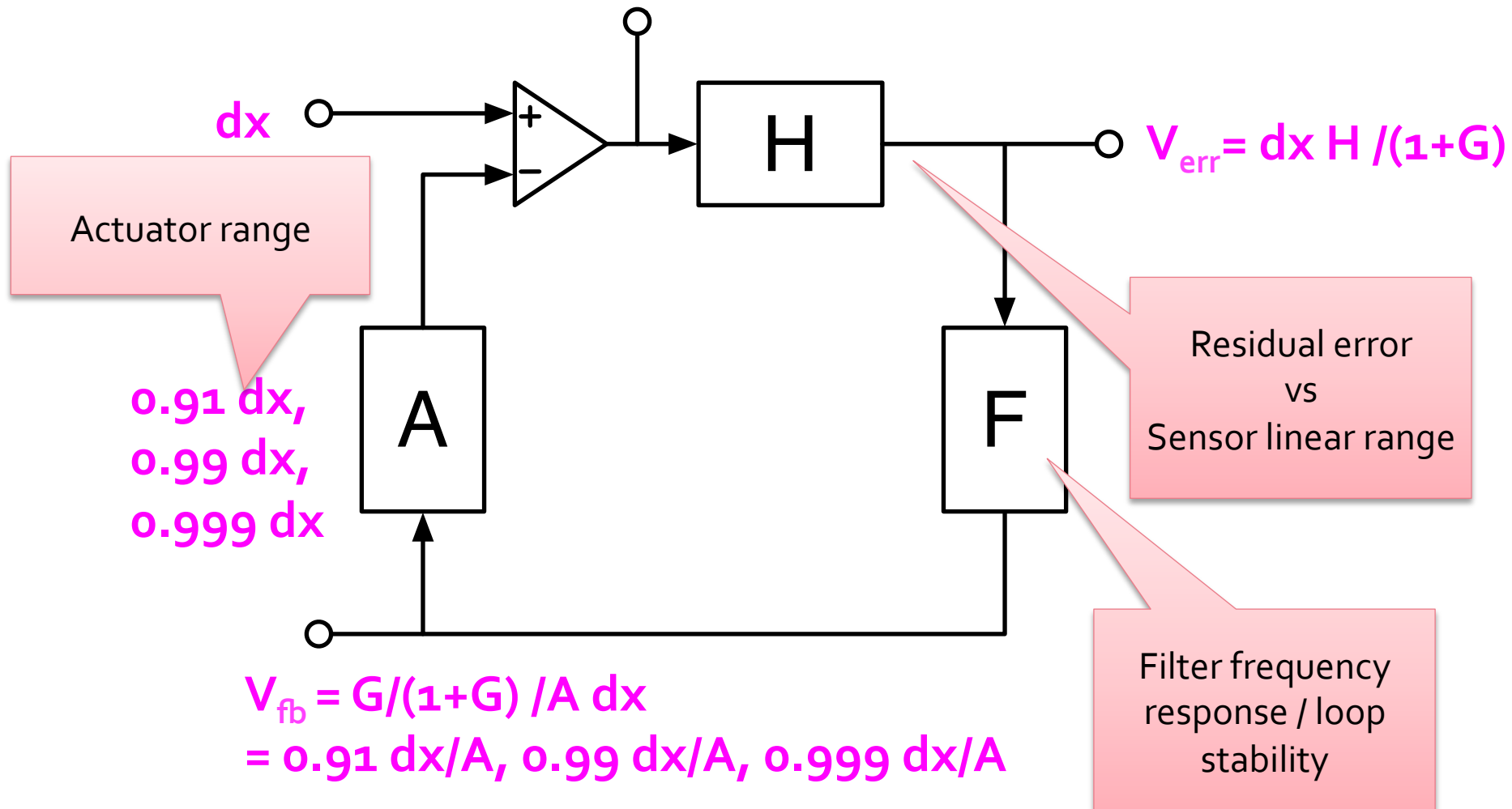
Open loop transfer function

$$G \stackrel{\text{def}}{=} H F A$$

# Introduction ~ Control?

- When  $G$  is big: e.g.  $G = 10, 100, \text{ or } 1000$

$$dx_s = dx / (1 + G) = 0.09 dx, 0.01 dx, 0.001 dx$$



# Introduction ~ Control?

- When the openloop gain  $G$  is  $\gg 1$ , the error signal gets suppressed
- “Wow! our sensor signal became smaller!”

Is our system more sensitive now?

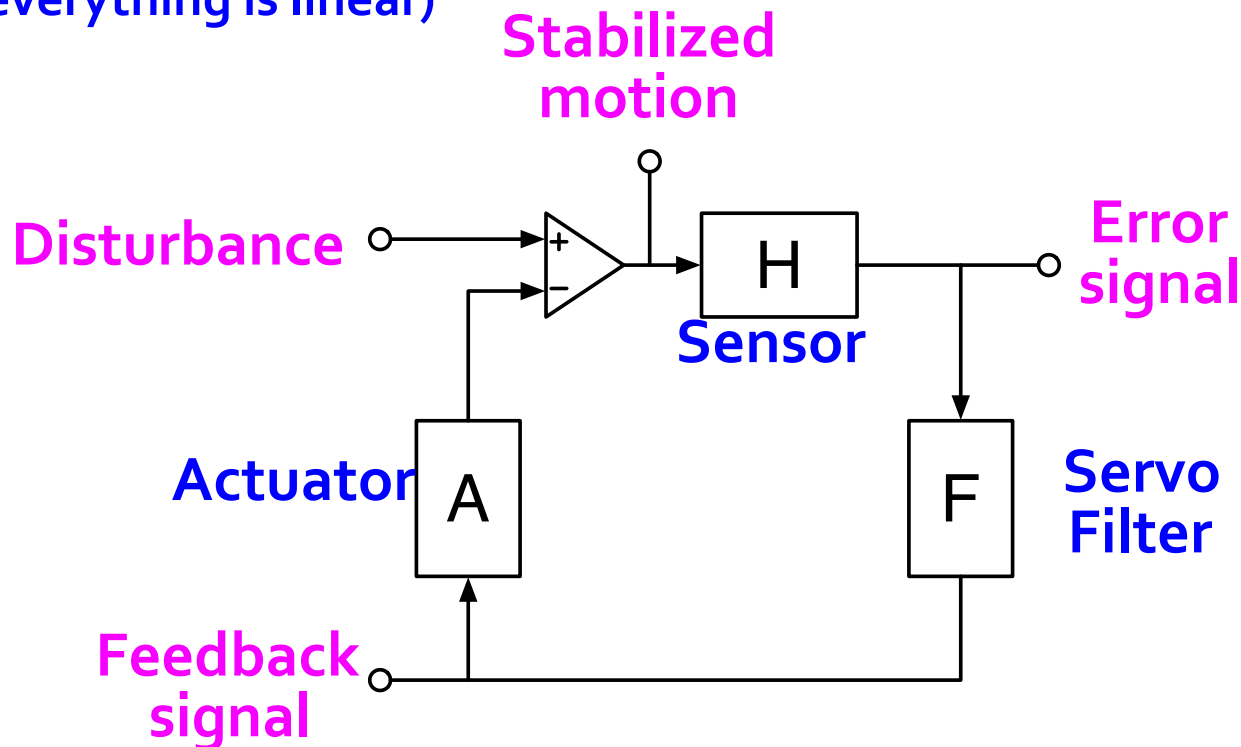
- No. We are just moving the actuator so that the error signal looks smaller. The signal and noise are equally suppressed in the error signal. Thus the SNR does not change.

- OK... So can we still measure gravitational waves even if the error signal is almost zero?

- Yes. We should be able to recover the original signal by compensating the effect of the control i.e.  $(1+G)$
- And we can also use the feedback signal in order to reconstruct the original signal with appropriate compensation i.e.  $(1+G)/G$

# Introduction ~ Control?

- **Important difference between**
  - “Feedback control for stabilization”  
and “Feedback control for measurement”
  - Feedback control changes **the stabilized motion**  
but **reconstructed Disturbance** is not modified by the loop\*  
(\*if everything is linear)



# Linear systems and their stability

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# Linear systems and their stability

- A deterministic and time-invariant system: H



- The system H is LTI (linear & time-invariant) when

$$y_1(t) = H \{x_1(t)\}$$

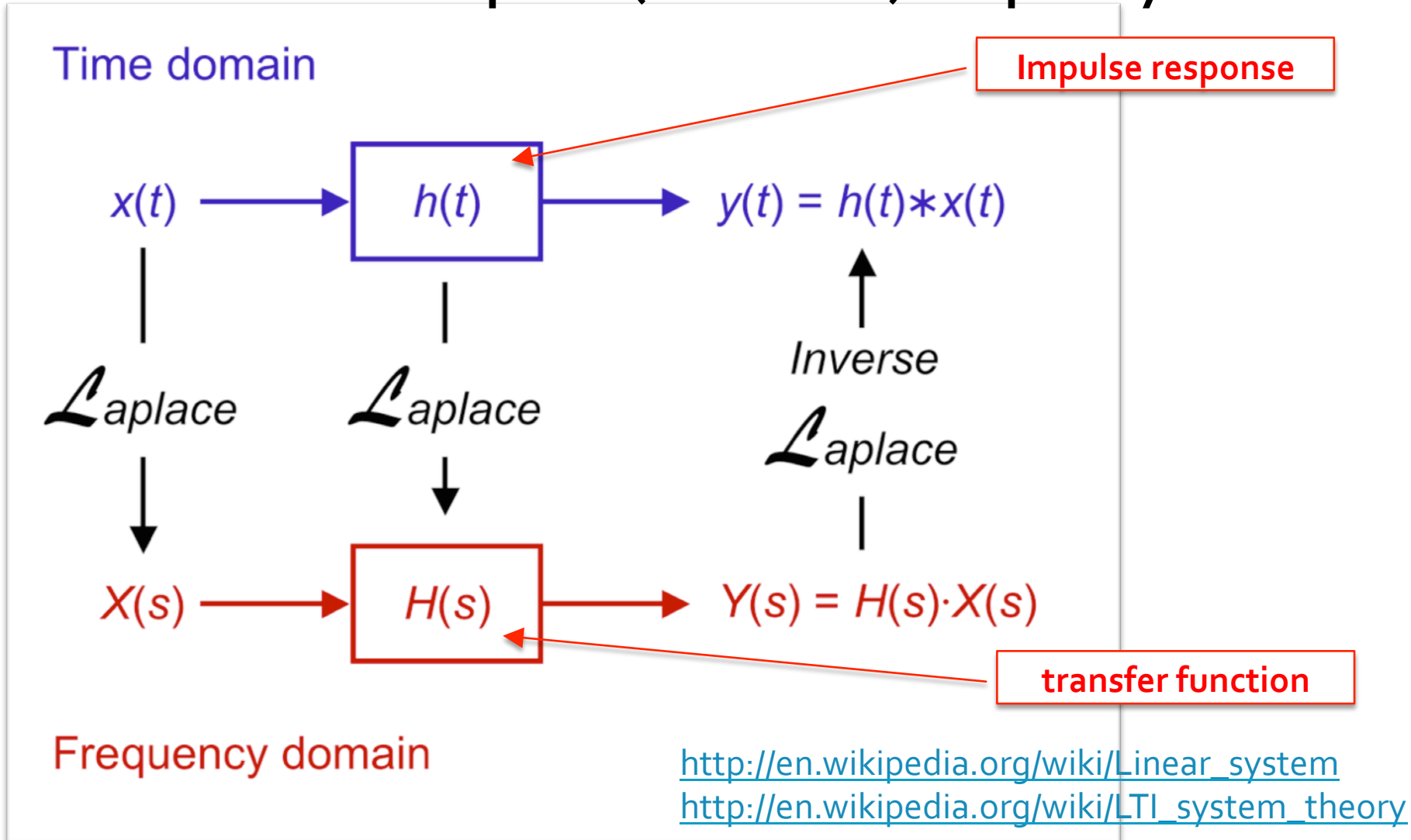
$$y_2(t) = H \{x_2(t)\}$$

$$\implies \alpha y_1(t) + \beta y_2(t) = H \{\alpha x_1(t) + \beta x_2(t)\}$$

- We can deal with such a system using Laplace transform (or almost equivalently Fourier Transform)

# Linear systems and their stability

- Time domain vs Laplace (or Fourier) frequency domain



# Linear systems and their stability

- It is easy to convert from an ordinary differential equation to a transfer function

$$\frac{d}{dt} \Longrightarrow s$$

Laplace Transform

$$\Longrightarrow i\omega = i2\pi f$$

Fourier Transform

- e.g. Damped oscillator

$$m\ddot{x}(t) = -kx(t) - \gamma\dot{x}(t) + F(t)$$

$$ms^2 X(s) = -kX(s) - \gamma sX(s) + F(s)$$

$$H(s) \equiv \frac{X(s)}{F(s)} = \frac{1}{ms^2 + \gamma s + k}$$

# Linear systems and their stability

- e.g. Damped oscillator

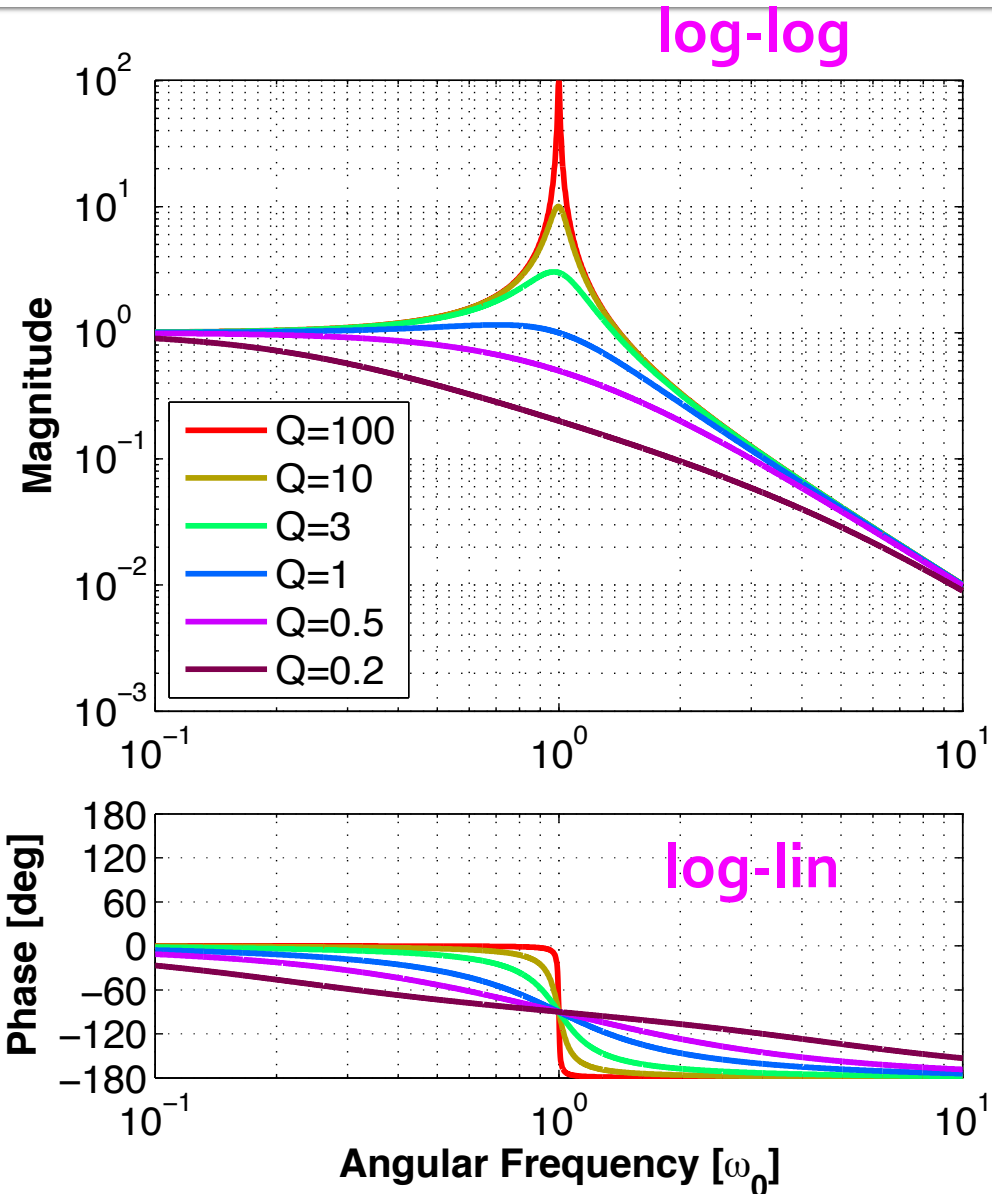
$$H(s) = \frac{1}{ms^2 + \gamma s + k}$$

$$H(s) = \frac{1}{m} \frac{1}{s^2 + \frac{\omega_0}{Q} s + \omega_0^2}$$

$$H(\omega) = \frac{1}{m} \frac{1}{-\omega^2 + i\frac{\omega_0}{Q}\omega + \omega_0^2}$$

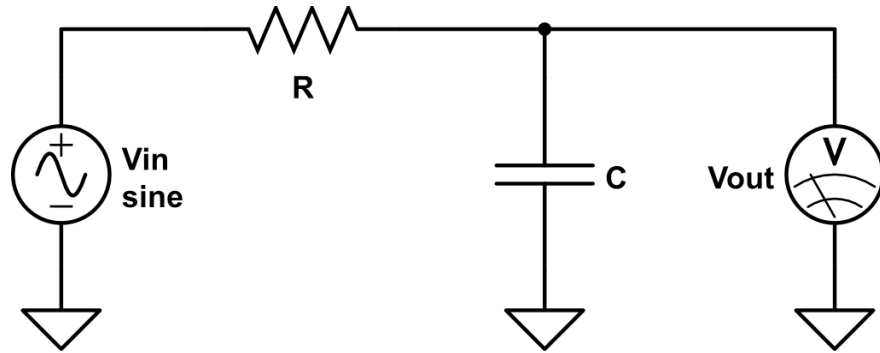
$$\omega_0 = \sqrt{k/m}, \quad \gamma = m\omega_0/Q$$

**Bode diagram**



# Linear systems and their stability

- e.g. RC filter

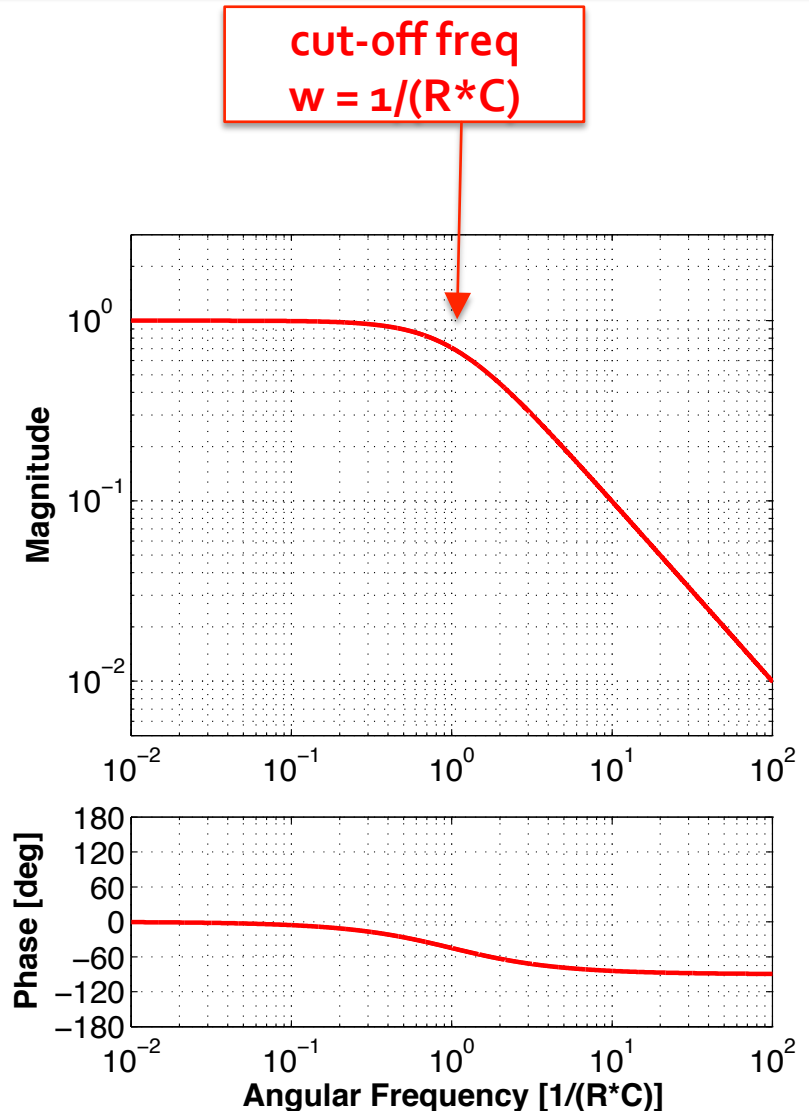


$$V_{out} = q/C$$

$$\dot{q} = (V_{in} - V_{out})/R$$

$$\Rightarrow i\omega C V_{out}(\omega) = (V_{in}(\omega) - V_{out}(\omega))/R$$

$$\Rightarrow \frac{V_{out}(\omega)}{V_{in}} = \frac{1}{1 + i\omega RC}$$



# Linear systems and their stability

- In most cases, a system TF can be expressed as:

$$H(s) = \frac{b_0 + b_1s + b_2s^2 + \dots + b_ms^m}{a_0 + a_1s + a_2s^2 + \dots + a_ns^n}$$

- The roots of the numerator are called as “**zeros**” and the roots of the denominator are called as “**poles**”

$$H(s) = \frac{b_m \prod_{i=1}^m (s - s_{zi})}{a_n \prod_{j=1}^n (s - s_{pj})}$$

- Zeros ( $s_{zi}$ ) and poles ( $s_{pi}$ ) are

**real numbers (single zeros/poles)**

or

**pairs of complex conjugates (complex zeros/poles)**

# Linear systems and their stability

- Poles rule the stability of the system!

H(s) can be rewritten as

$$H(s) = \sum_{j=1}^n \frac{K_j}{(s - s_{pj})}$$

- Each term imposes exponential time impulse response

$$\text{T.F.}: H_j(s) = \frac{1}{s - s_{pj}} \iff \text{I.R.}: h_j(t) = e^{s_{pj}t}$$

- Therefore, if there is ANY pole with  $\text{Re}(s_{pj}) > 0$   
the response of the system diverges

# Linear systems and their stability

- Poles rule the stability of the system!

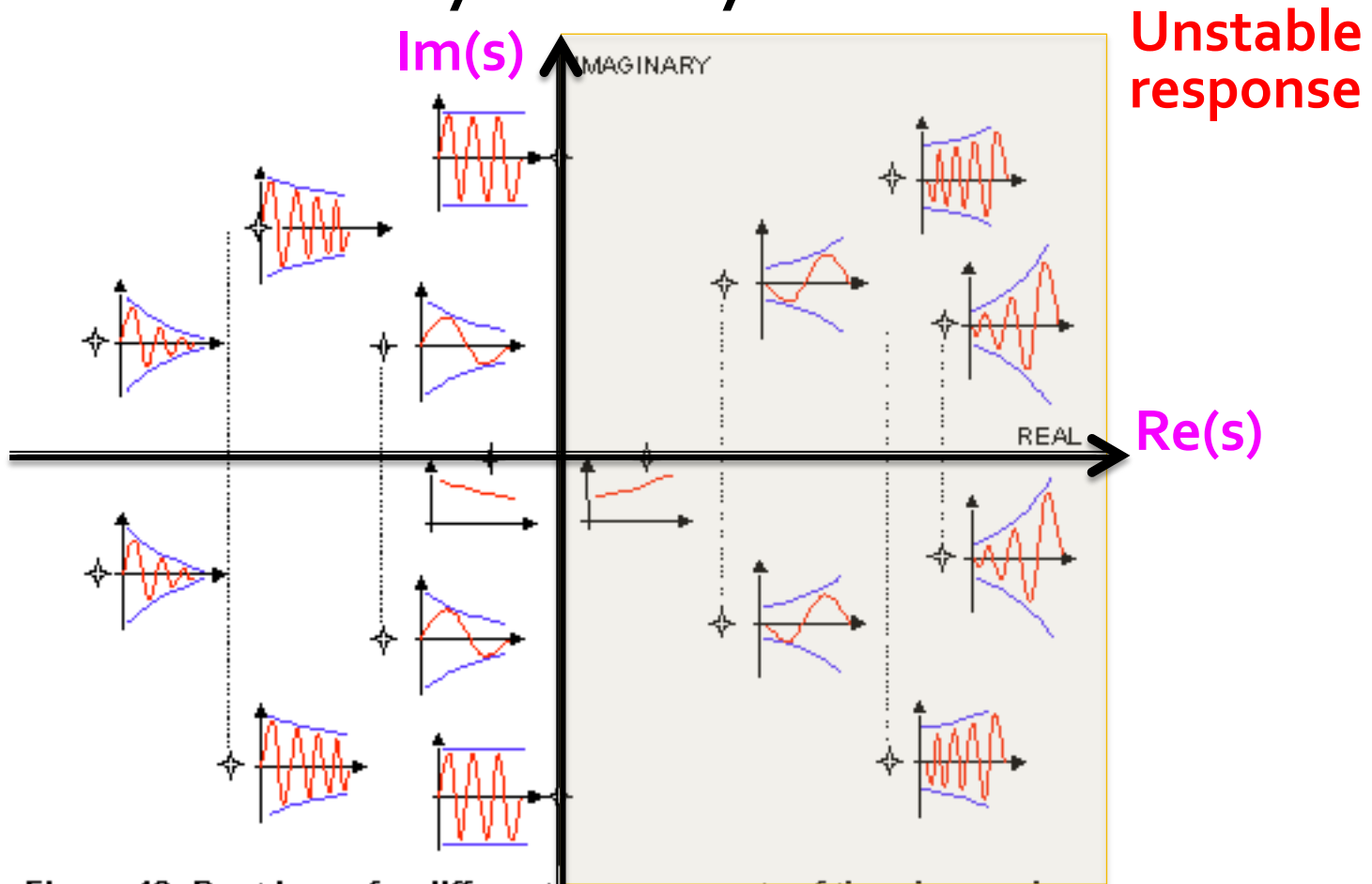
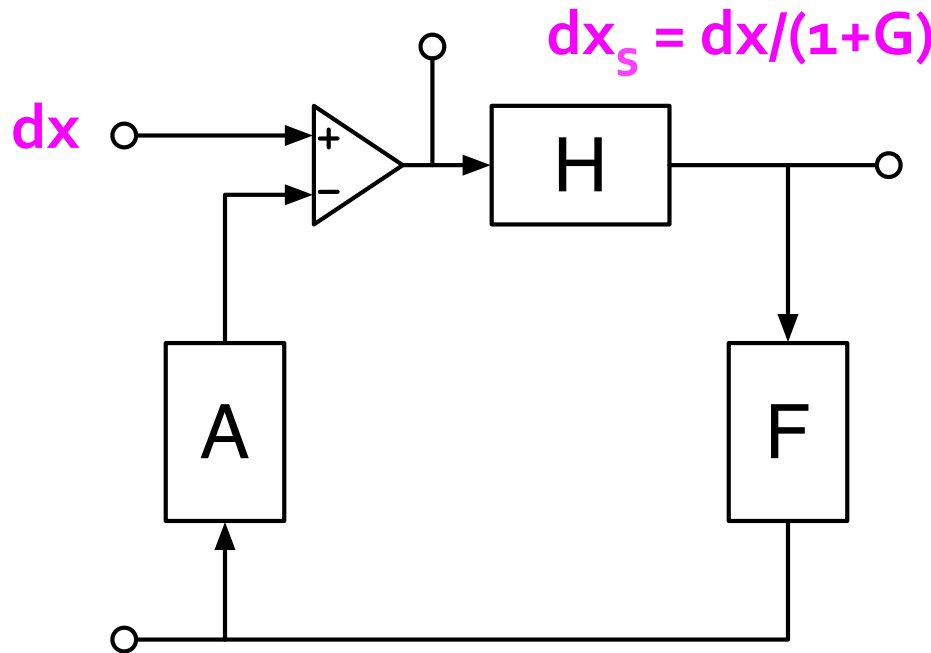


Figure 12: Root locus for different arrangements of the eigen values



# Linear systems and their stability

- Now we eventually came back to this diagram



Open loop TF:  
 $G = H F A$

Closed loop TF:  
 $G_{CL} = 1/(1+G)$

Requirement:

All the roots for  $1+G$  should be  
in the left hand side of Laplace plane

(!)

# Linear systems and their stability

## ■ Remarks

### Requirement:

All the roots for  $1+G$  should be in the left hand side of Laplace plane

- This does not mean all  $H, F, A$  needs to be stable.  
e.g. Unstable mechanical system  $A$  can be stabilized by a control loop. (cf. An inverted Rod)

- We usually play with  $F$  to tune the result.

$$\text{Open loop TF:} \\ G = H F A$$

It is awkward to evaluate the stability of  $1/(1+G)$  every time.

$$\text{Closed loop TF:} \\ G_{CL} = 1/(1+G)$$

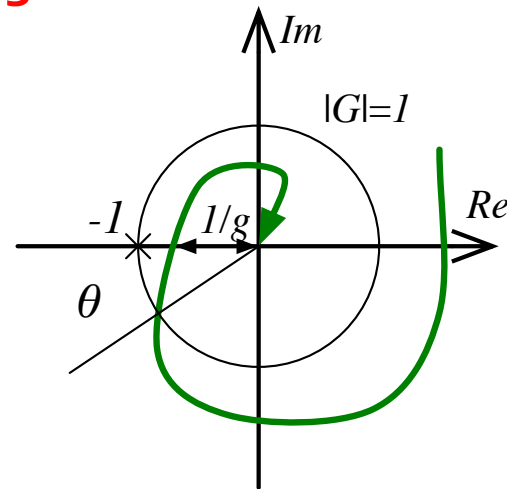
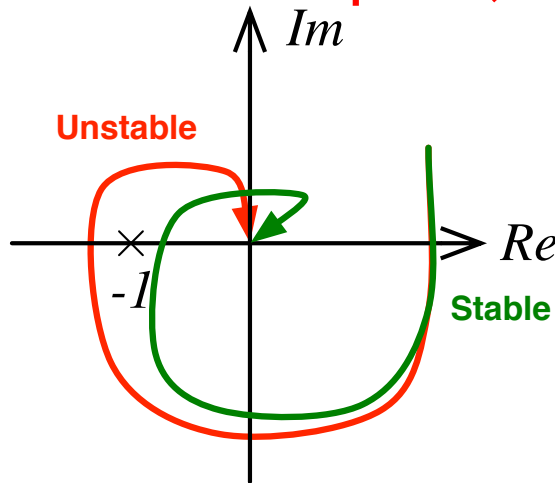
There is a way to tell the stability only from  $G$

**Nyquist's stability criterion**

# Linear systems and their stability

## ■ Nyquist stability criterion

- Plot openloop gain  $G$  in a complex plane (i.e. Nyquist diagram)
- If the locus of  $G(f)$  from  $f=0$  to  $\infty$ , goes to 0 looking at the point  $(-1 + 0 i)$  at the left side => Stable
- If the locus sees the point  $(-1+0 i)$  at the right side => Unstable



- Unity gain frequency  $f_{UGF}$  :

for  $|G(f_{UGF})| = 1$

- Phase margin  $\vartheta$  :

$\vartheta = \text{Arg}(G(f_{UGF}))$

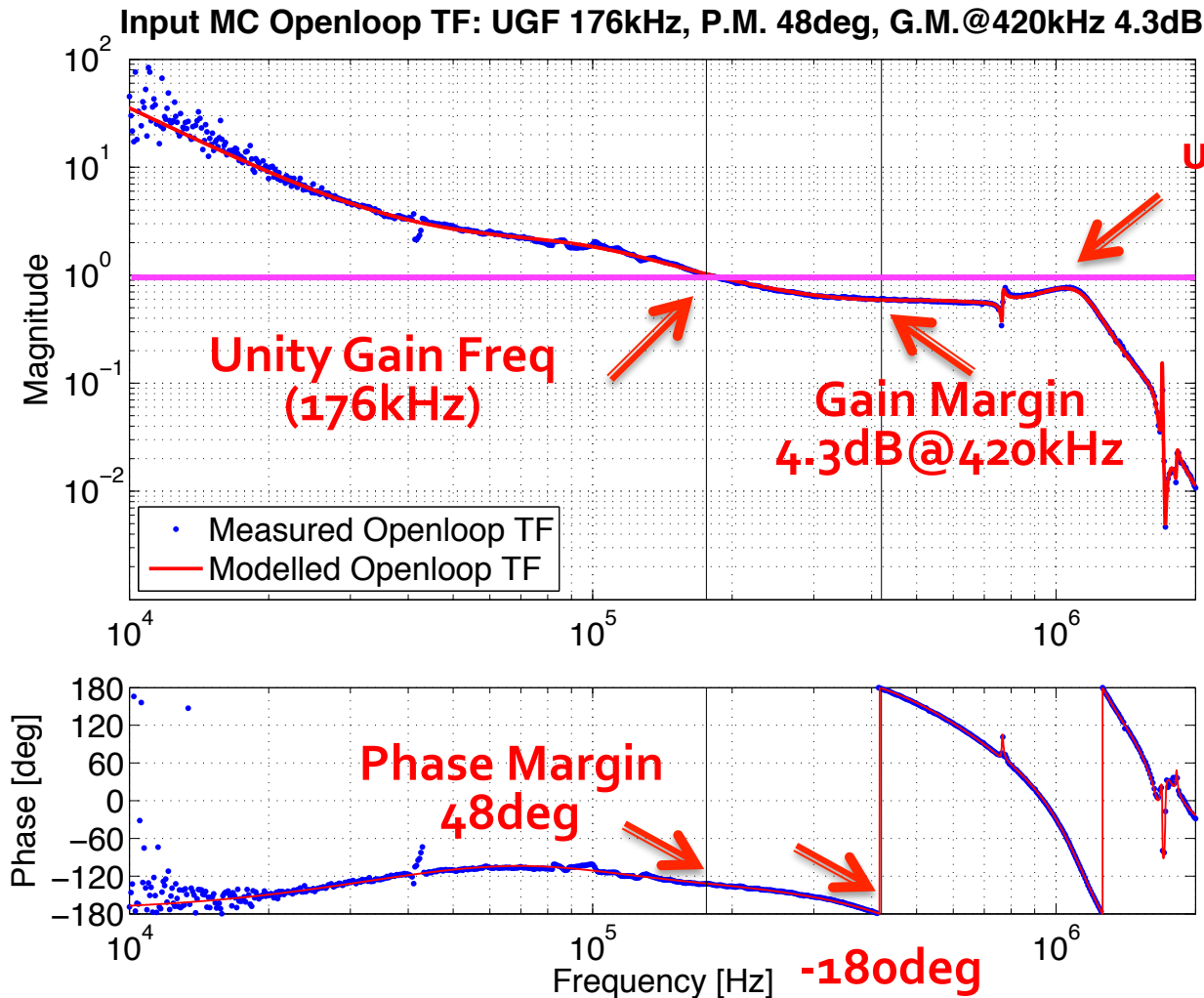
- Gain margin  $g$ :

$g = 1/|G(f_o)|$  where  $\text{Arg}(G(f_o)) = -\pi$

# Linear systems and their stability

## Phase Margin / Gain Margin in Bode diagram

- Most of the case, a bode diagram of  $G$  is enough to see the stability



Nearly unstable

A rough standard of a stable servo loop:  
Phase Margin > 40deg  
Gain Margin > 10dB

# Linear systems and their stability

## ■ Building blocks (“zpk” representation)

### ■ Single pole

$$H(s) = \frac{s_p}{s + s_p} \quad (s_p \in \mathbb{R}, s_p > 0)$$

### ■ Single zero

$$H(s) = \frac{s + s_z}{s_z} \quad (s_z \in \mathbb{R}, s_z > 0)$$

### ■ A pair of complex poles

$$H(s) = \frac{s_p s_p^*}{(s + s_p)(s + s_p^*)} \quad (s_p \in \mathbb{C}, \Re(s_p) > 0)$$

### ■ A pair of complex zeros

$$H(s) = \frac{(s + s_z)(s + s_z^*)}{s_z s_z^*} \quad (s_z \in \mathbb{C}, \Re(s_z) > 0)$$

### ■ Gain

$$H(s) = K \quad (K \in \mathbb{R})$$

# Linear systems and their stability

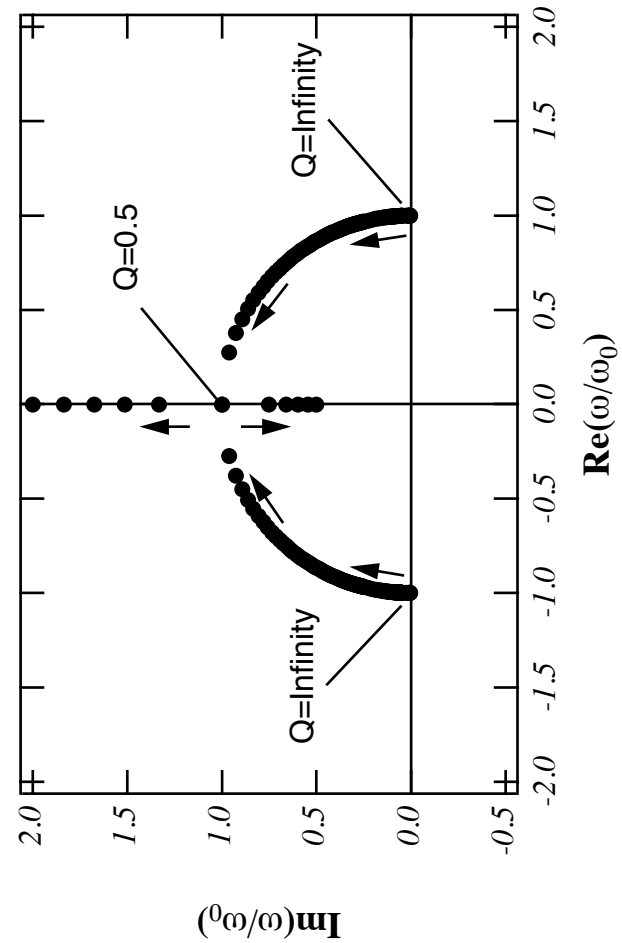
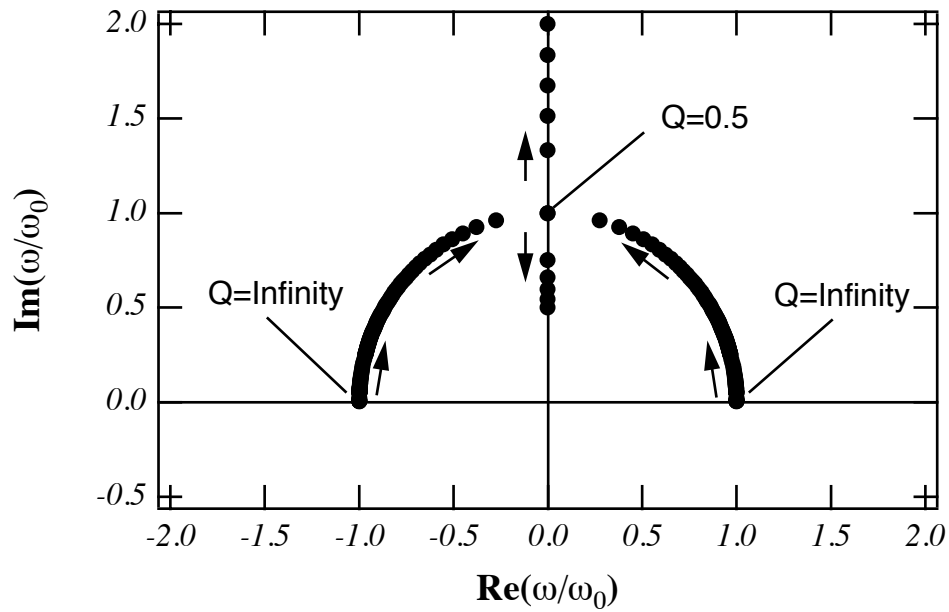
- Relationship between pole/zero locations and  $\omega_0$  &  $Q$

$$\begin{aligned} H(s) &= \frac{s_p s_p^*}{(s + s_p)(s + s_p^*)} \\ &= \frac{|s_p|^2}{s^2 + 2\Re(s_p)s + |s_p|^2} \end{aligned}$$

- To be compared with

$$\begin{aligned} H(\omega) &= \frac{\omega_0^2}{-\omega^2 + i\omega_0\omega/Q + \omega_0^2} \\ \implies \omega_0 &= |s_p|, \quad Q = \frac{|s_p|}{2\Re(s_p)} \end{aligned}$$

# Linear systems and their stability



# Linear systems and their stability

- **Summary**

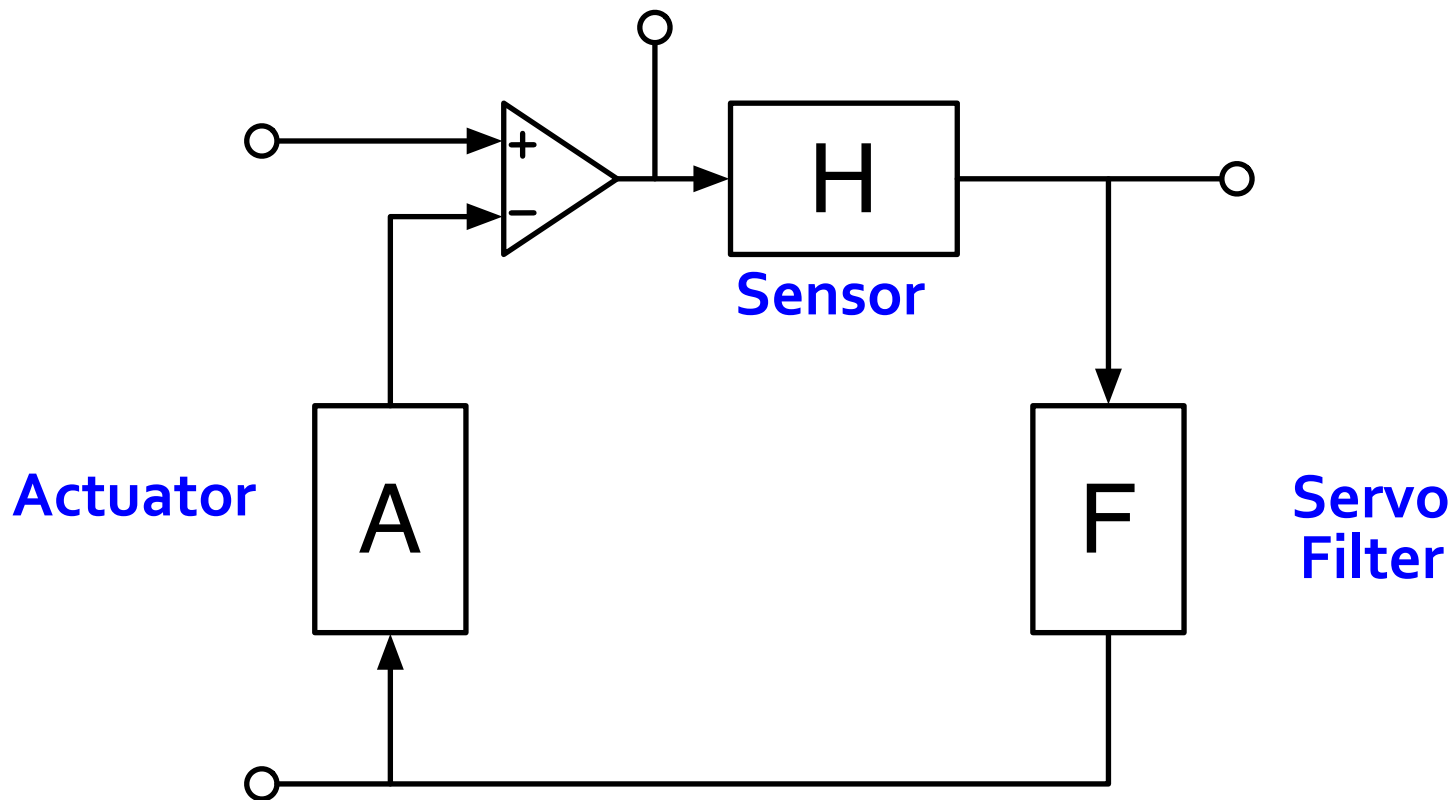
- **Classical control theory**
- **Design locations of poles and zeros**
- **Stability: tuning of open loop transfer function is important**



# Control system components in GW detectors

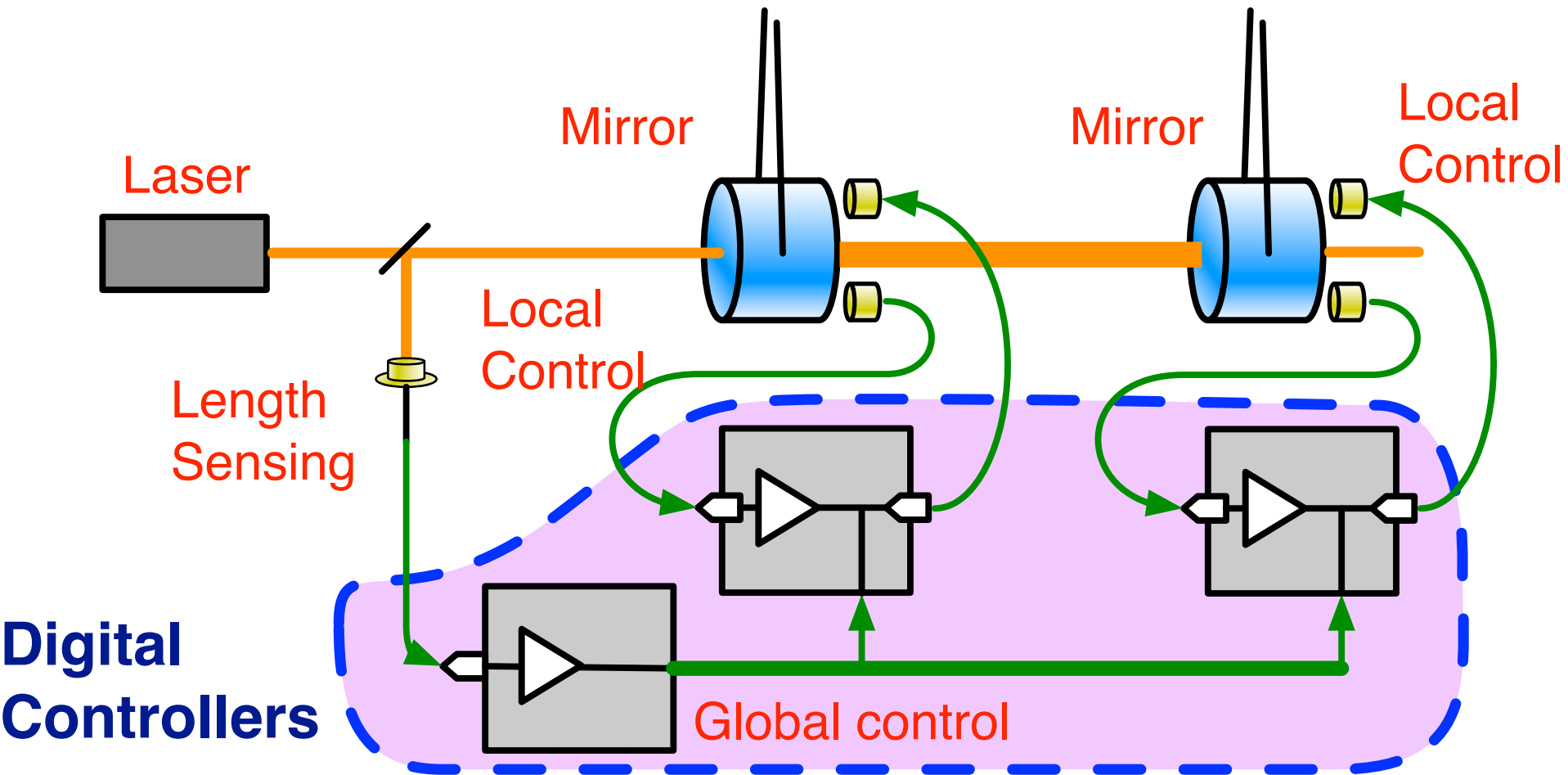
# Control systems

- Elements of a feedback loop (again)



# Interferometer control system

- Local control vs global control

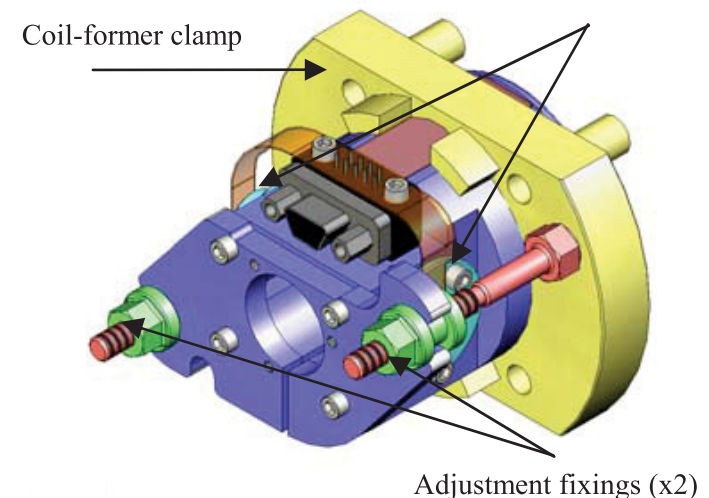
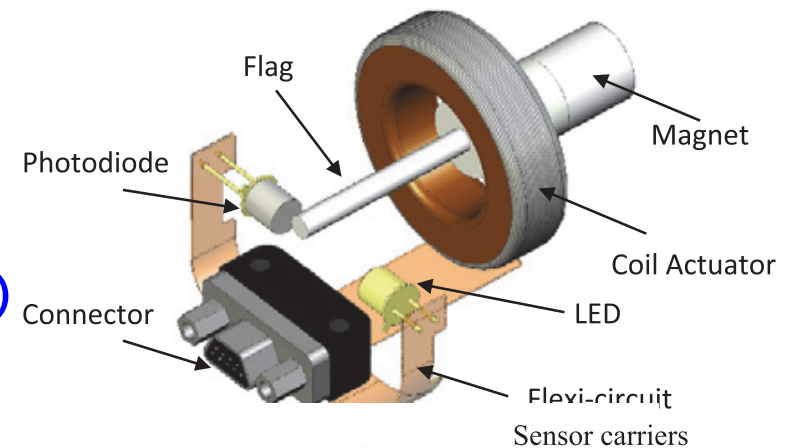
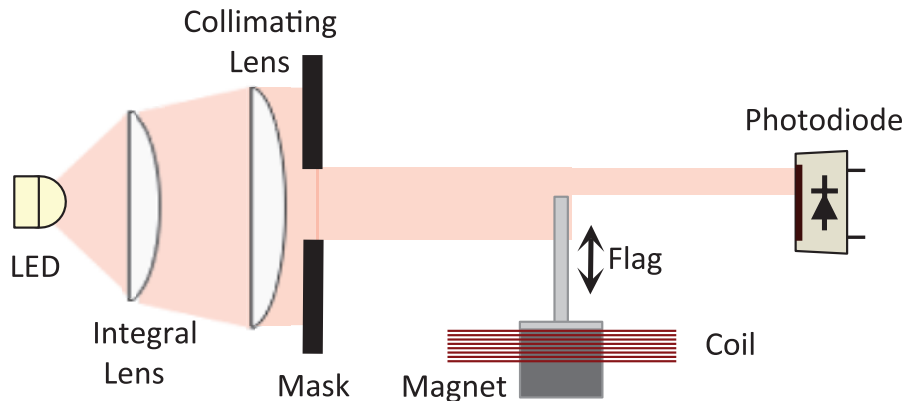


# Local Sensors

## ■ Shadow sensor (relative displacement sensor)

- For suspension damping control, mirror attitude monitor
- Typical linear range  $\sim 1\text{mm}$  for  $0\text{-}10\text{V} \Rightarrow dV/dx = 10 \text{ kV/m}$
- Typical noise level:  $\sim 100 \text{ pm}/\sqrt{\text{Hz}}$

## ■ aLIGO: Birmingham Optical Sensor and Electro-Magnetic actuator (BOSEM)

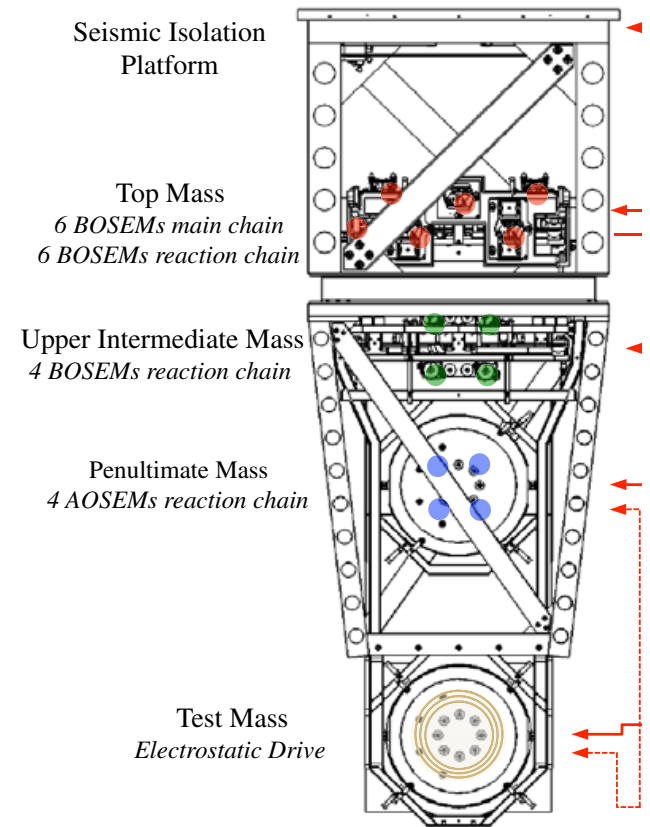
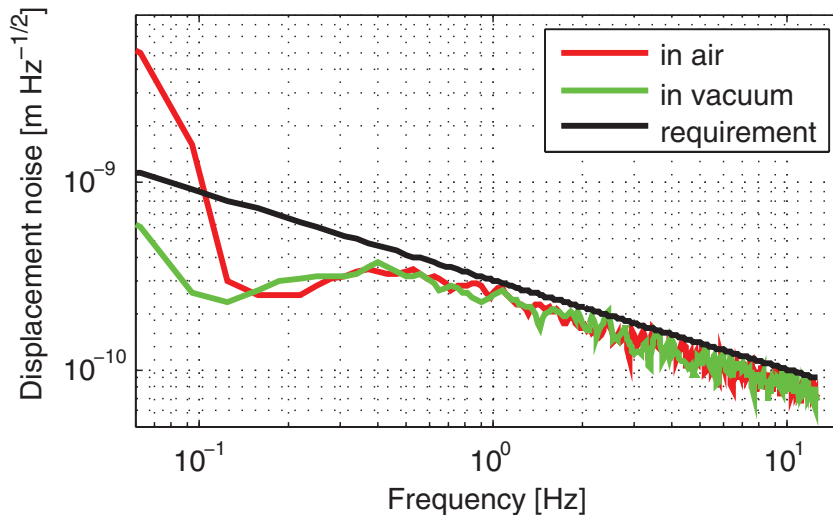
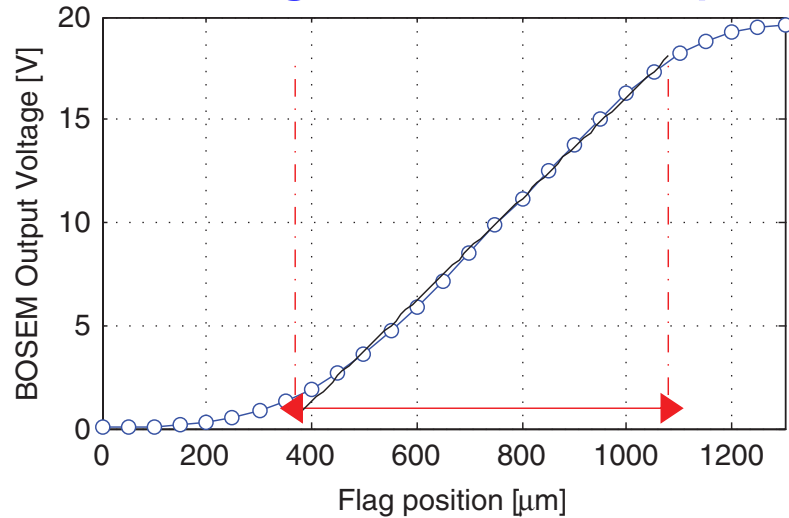


# Local Sensors

## Shadow sensor (relative displacement sensor)

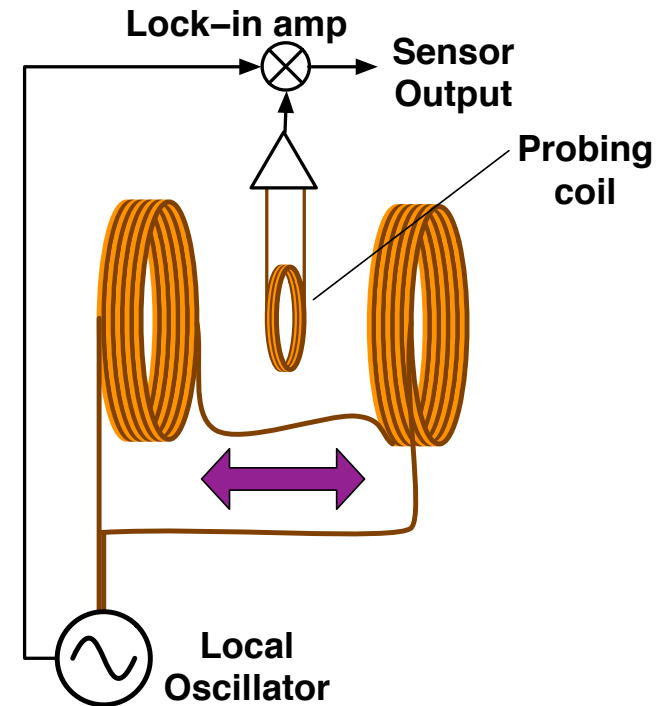
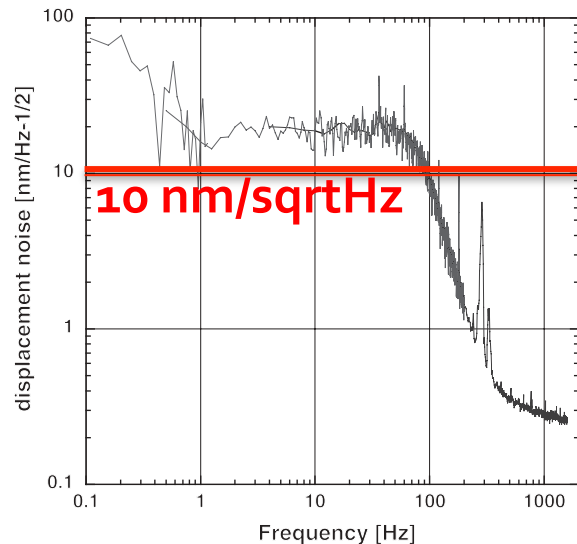
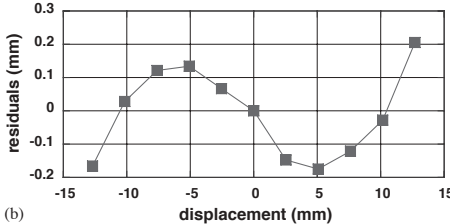
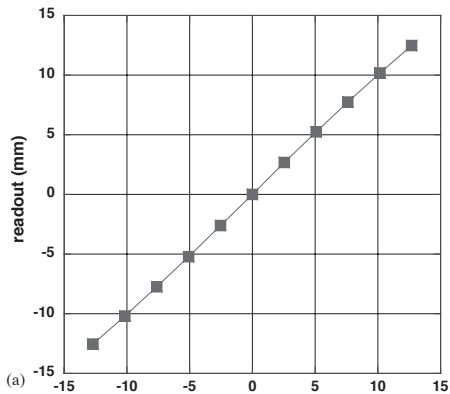
- Linear range (~0.7 mm) / displacement noise

### sensor locations



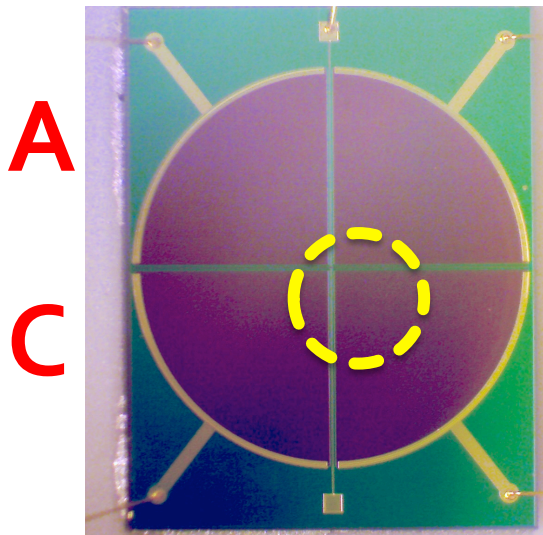
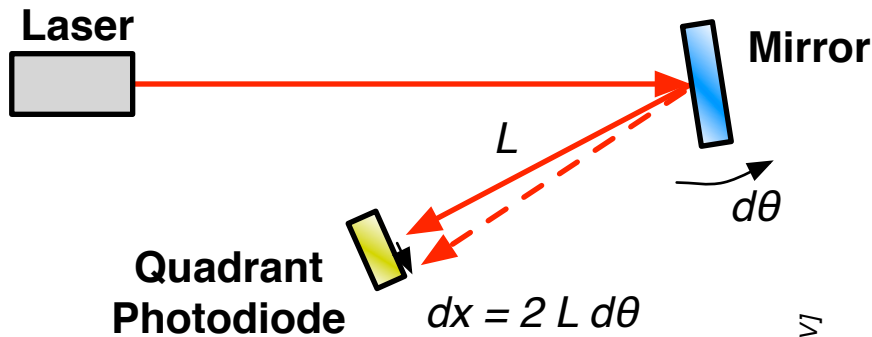
# Local Sensors

- **Linear Variable Differential Transducer (relative disp. sensor)**
  - For low freq pendulum control (inverted pendulum), larger range VIRGO Super attenuator, KAGRA Seismic Attenuation System
  - Typical linear range  $\sim 10\text{mm}$  for  $0\text{-}10\text{V} \Rightarrow dV/dx = 1 \text{ kV/m}$   
Typical noise level:  $10\text{-}100 \text{ nm}/\sqrt{\text{Hz}}$

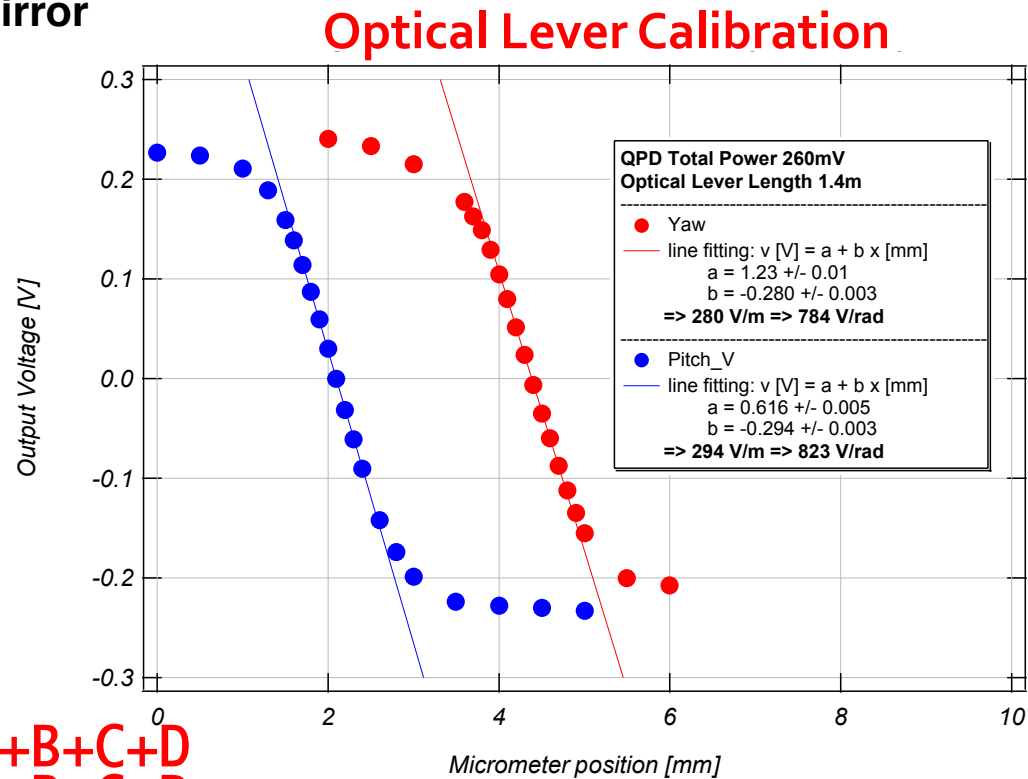


# Local Sensors

- Optical Lever (relative angular sensor)
  - Angle local control
  - Typical linear range ~beam side (0.1~1 mm) =>  $dV/d\theta = 1 \sim 10 \text{ kV/rad}$
  - Typical noise level: 0.01~1 nrad/sqrthz



$$\begin{aligned} \text{SUM} &= A+B+C+D \\ X &= A-B+C-D \\ Y &= A+B-C-D \end{aligned}$$



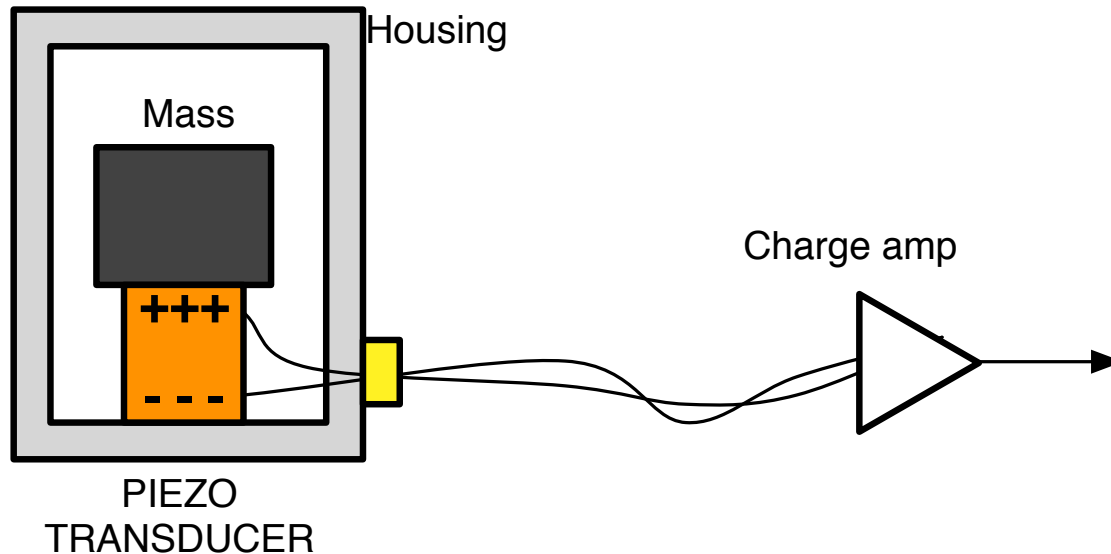
# Local Sensors

- Piezo Accelerometer (Inertial sensor)

- Vibration measurement

- Typical linear range ~ 100~1000 m/s<sup>2</sup>

Typical noise level: 0.5 ~ 50 ( $\mu\text{m/s}^2$ )/sqrtHz

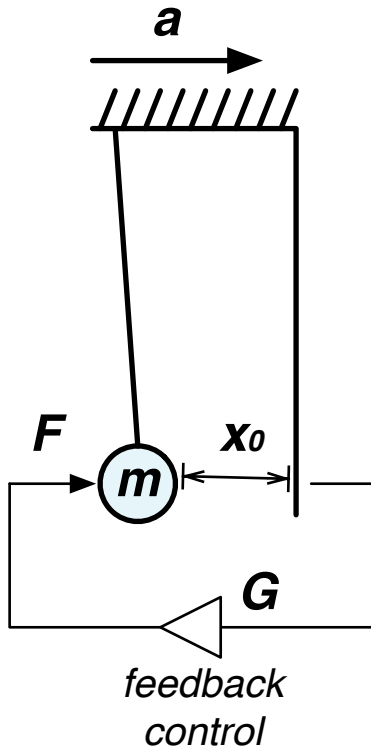




# Local Sensors

## ■ Servo Accelerometer (Inertial sensor)

- Seismic platform control ( $f > 0.1\text{Hz}$ ), Vibration measurement



Apply force to the suspended mass  
 $\Rightarrow$  Keep the distance from a reference

When the control gain  $G \gg 1$   
 $\Rightarrow a = F / m$

# Local Sensors

- Servo Accelerometer (Inertial sensor)
  - Above the resonant freq: Limited by the sensor noise
  - Below the resonant freq: Steep rise of the noise as the mass does not move in relative to the ground  
=> Low resonant freq is beneficial

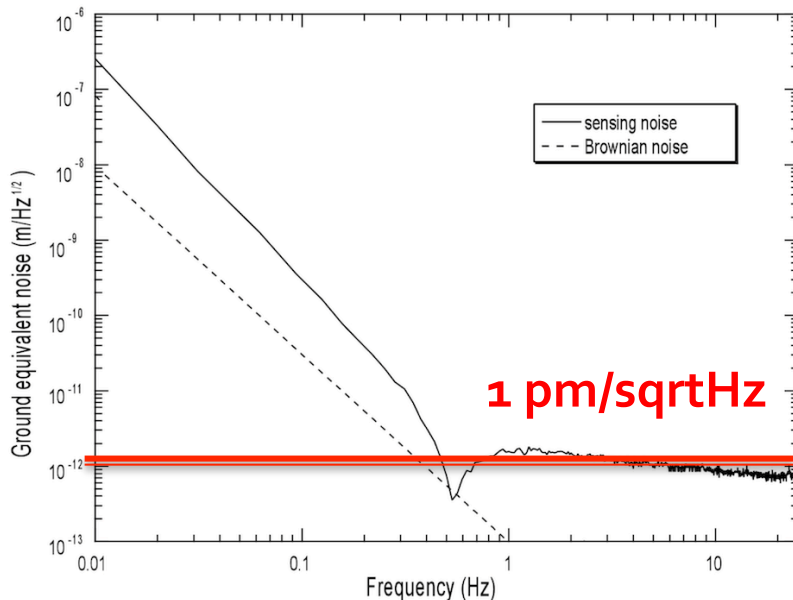
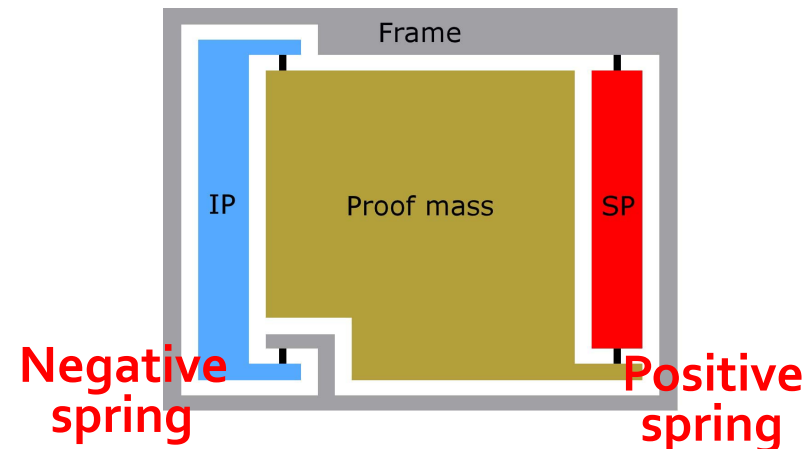
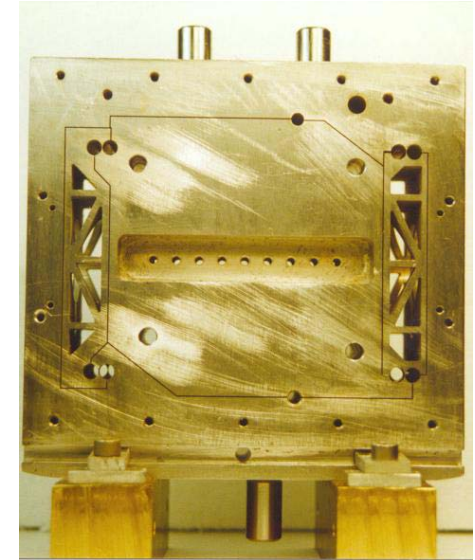


Fig.7. Equivalent frame displacement noise.



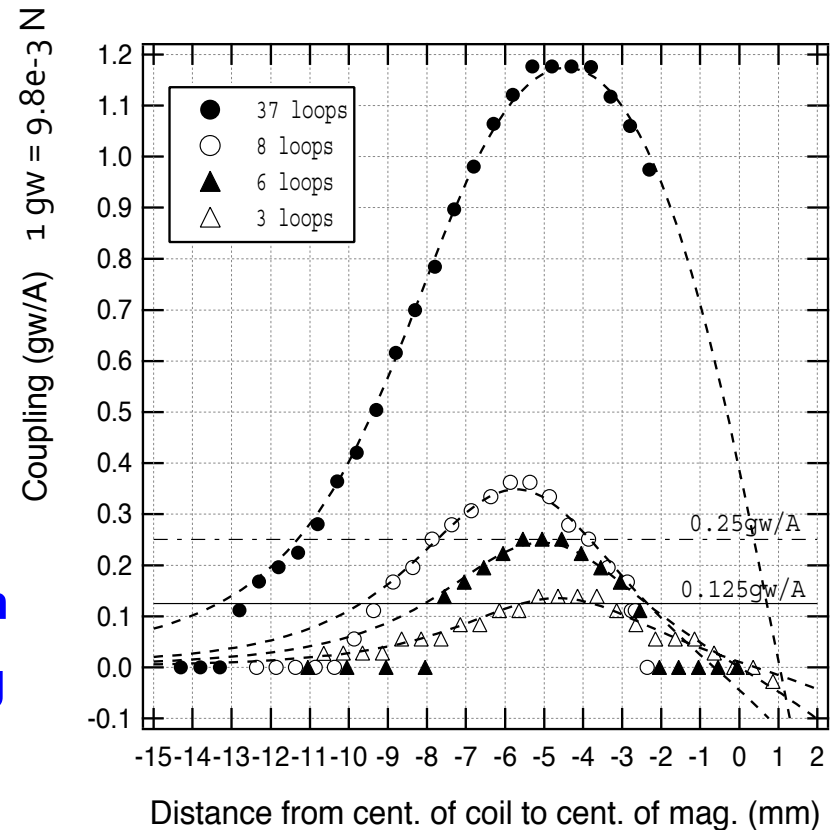
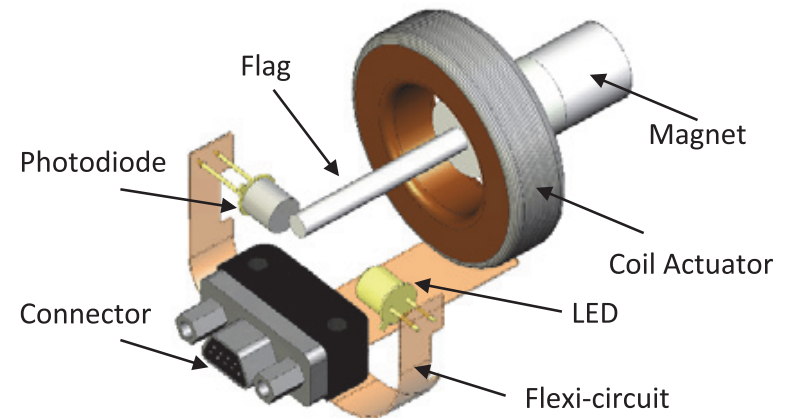
# Actuators

- **Mechanical actuators**
  - Coil Magnet actuator
  - Electro Static Driver (ESD)
  - Piezo (PZT) actuator
  
- **Optical actuators**
  - Acousto-Optic Modulator
  - Electro-Optic Modulator
  - Laser Frequency

# Actuators (Mechanical)

## ■ Coil-magnet actuator

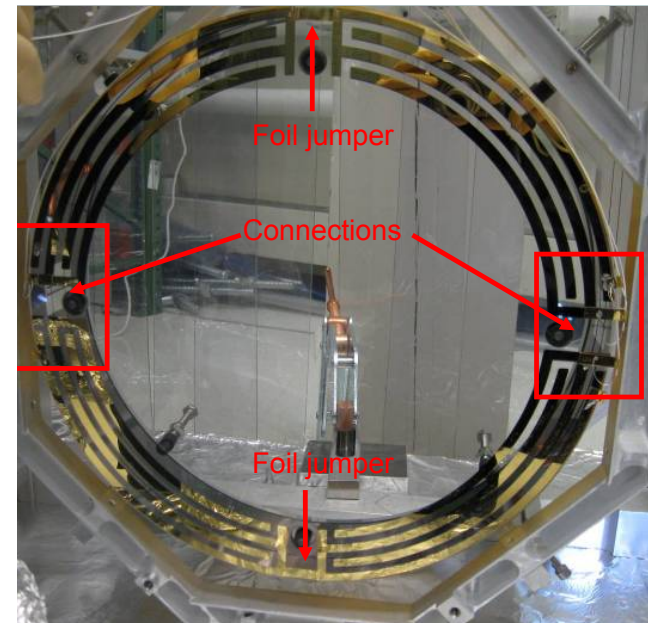
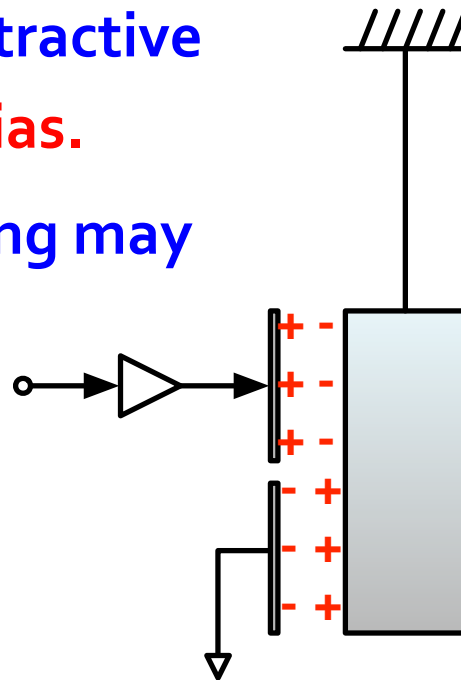
- Coil current induces force on a magnet attached to a mass
- Contactless
- aLIGO coil-magnet actuator is integrated in BOSEM
- Actuator response (coupling) has position dependence. Preferable to use it at its maximum in order to avoid vibration coupling



# Actuators (Mechanical)

## ■ Electro Static Driver (ESD)

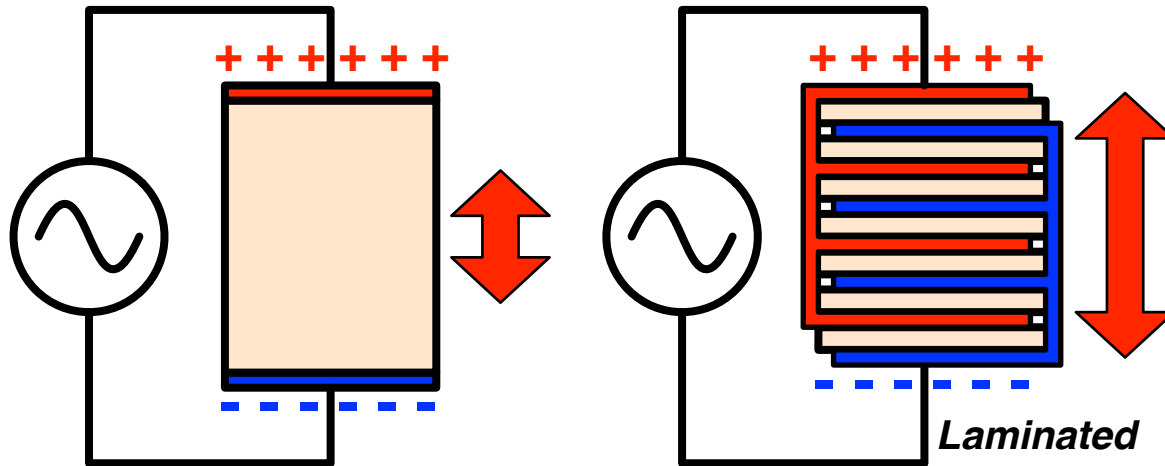
- Apply potential close to the mirror  
=> induces surface charge (or polarization) and attractive force
- In practice, comb patterns are used  
=> strengthen the electric field, but less force range
- Can produce only attractive force. => Need DC Bias.
- Stray surface charging may cause problems.



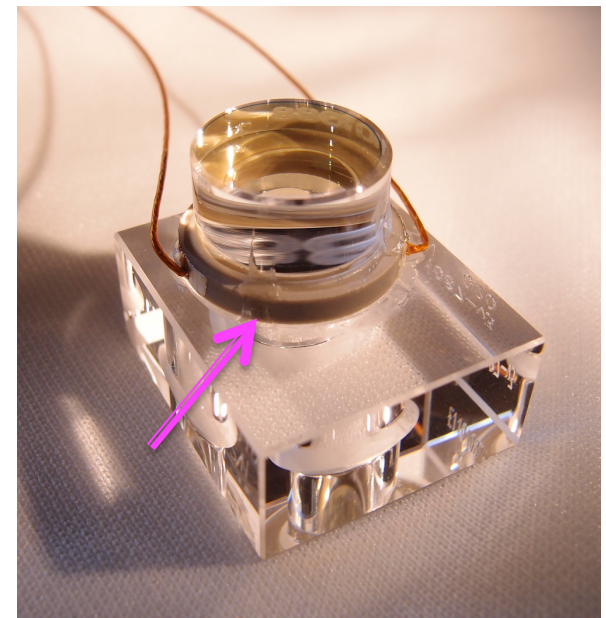
# Actuators (Mechanical)

## ■ Piezo (PZT) actuator

- Apply potential to a ferroelectric material  
=> cause internal polarization and induces strain
- To increase displacement, laminated piezo is often used  
=> displacement 3~10  $\mu\text{m}$
- Requires a bias voltage and HV amplifier, but has wide applications



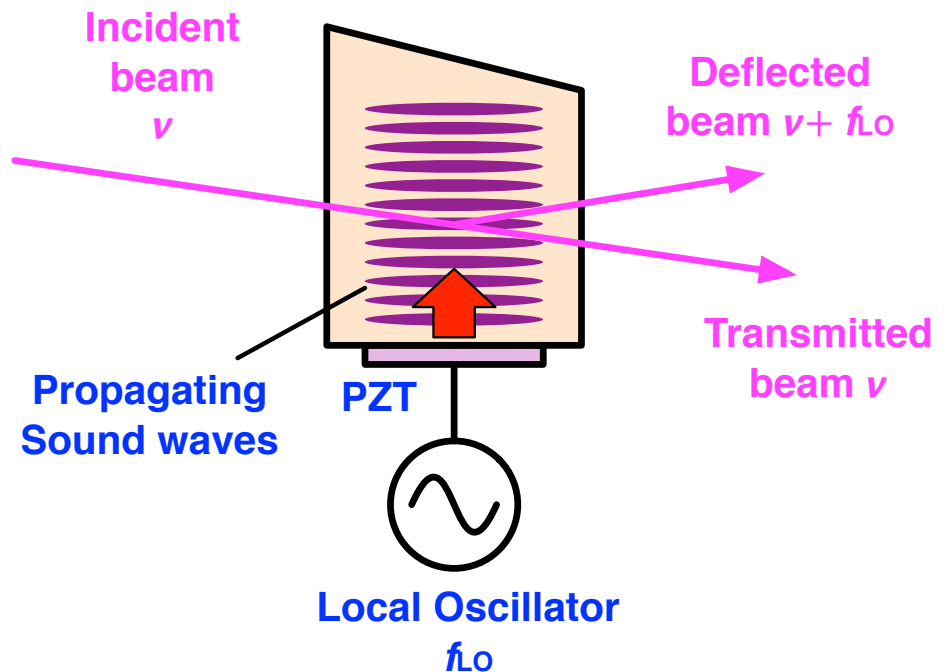
OMC cavity mirror



# Actuators (Optical)

## ■ Acousto-Optic Modulator

- Phonon-Photon scattering (or bragg diffraction) in AOM crystal
- Effect: **Beam deflection / Frequency shift**
- Application: **Laser frequency actuator, Laser intensity actuator**  
**Beam angle scanner**



# Actuators (Optical)

- **Electro-Optic Modulator**

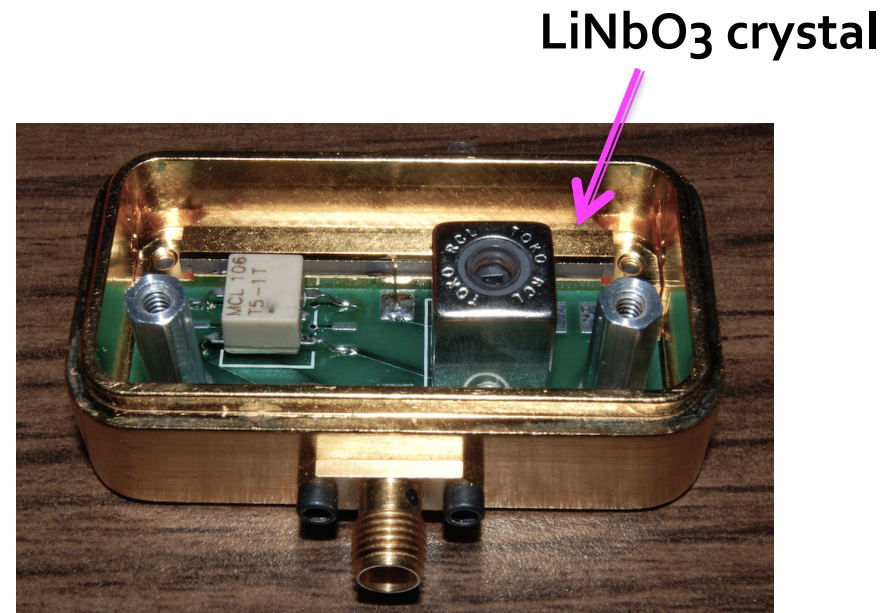
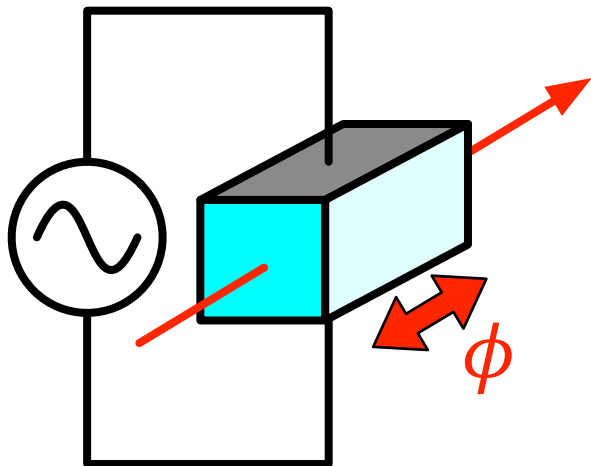
- **Pockels Cell effect:**

**Refractive index changes linearly to the applied E-field**

- **Application:**

**Laser phase modulation**

**Phase actuation (= frequency actuation)**





# Actuators (Optical)

- **Laser frequency actuation (YAG NPRO laser)**
  - We often control laser frequency with multiple actuators
  - **1) Thermal actuator**

Thermo-Electric Cooler attached to the laser crystal.  
Huge response ( $1\text{GHz/K}$  or  $1\text{GHz/V}$ ) but slow ( $f < 0.1\text{Hz}$ )
  - **2) Fast piezo actuator**

A piezo attached on the laser crystal induces stress induced refractive index change.  
Response ( $\sim 1\text{MHz/V}$ ). Bandwidth  $10\sim 100\text{kHz}$
  - **3) External EOM**

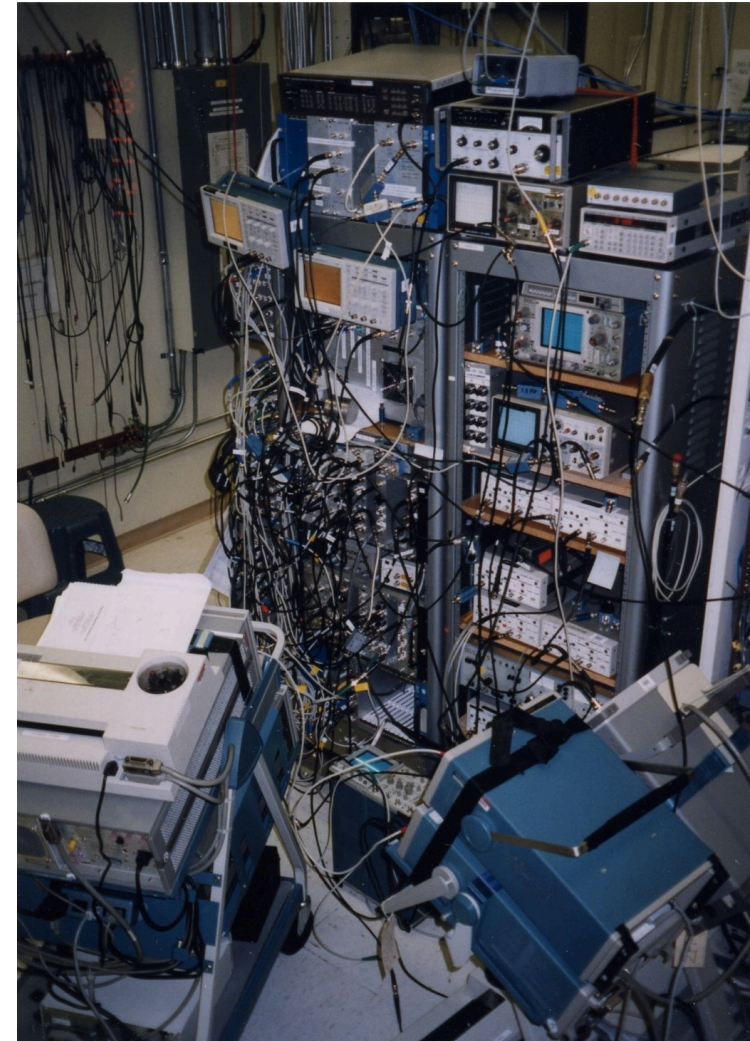
Response ( $\sim 10\text{ mrad/V}$ ), Bandwidth  $\sim 1\text{MHz}$

# Servo Controller

- Analog servo filters

- High dynamic range ( $\sim 1\text{nV}/\sqrt{\text{Hz}}$ ,  $\pm 10\text{V}$ ), High bandwidth
- Pole/zero placement with active op-amp filters
- Until the end of the 20<sup>th</sup> century, analog filters have been commonly used for servo filters in our field
- Analog servos are still in action for the feedback loops with bandwidth  $> 1\text{kHz}$ . (cf. frequency stabilization, intensity stabilization)

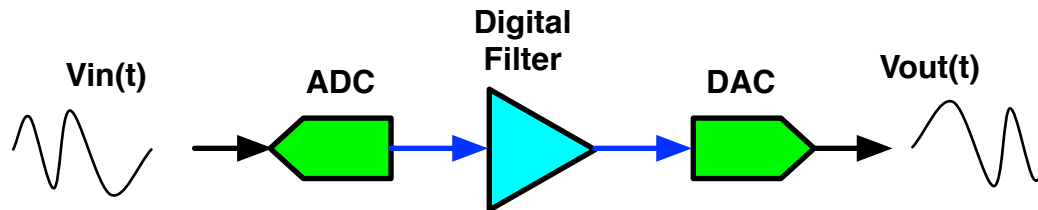
LIGO 40m  
prototype (1998)



# Servo Controller

## ■ Digital servo filters

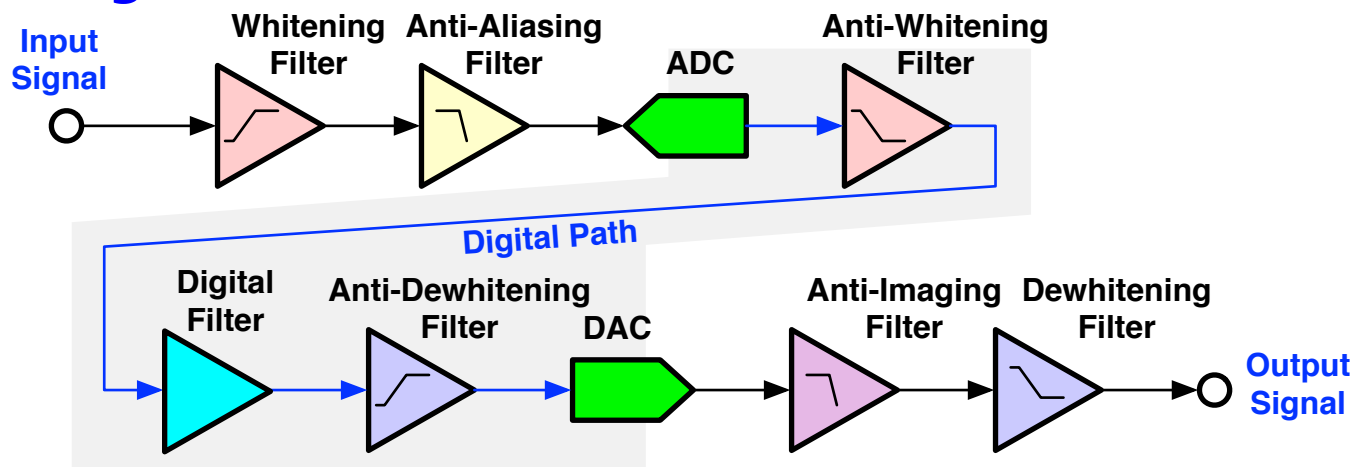
- Process digitized signals in a computer



- Large flexibility  
High compatibility with detector automation and management
- Limited dynamic range ( $\sim 0.1\text{mV}/\sqrt{\text{Hz}}$ ,  $\pm 10\text{V}$  for 16bit)
- Limited bandwidth
  - Each sample needs to be processed before the next sampled data comes
  - Inevitable sampling delay
  - Additional phase delays due to analog filters for analog-digital interface
  - e.g. 16kHz sampling, control bandwidth  $\sim 200\text{Hz}$

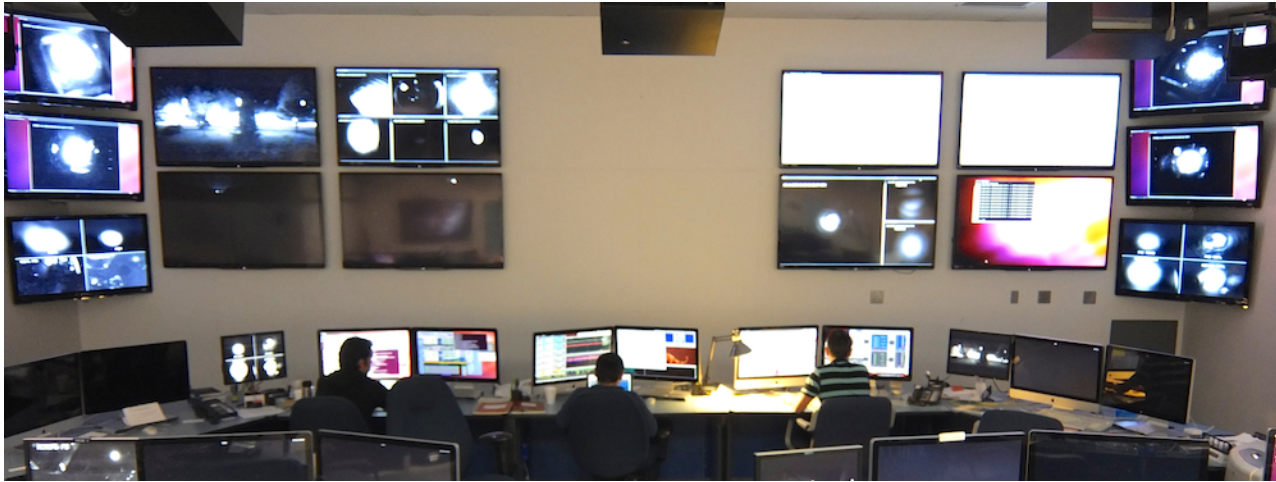
# Analog/Digital interface

- **Restriction of signal digitization**
  - **Voltage quantization: quantization noise**
    - => **limited dynamic range**
    - => **Requires whitening/dewhiting filters**
  - **Temporally discrete sampling: aliasing problem**
    - => **limited signal bandwidth**
    - => **Requires anti-aliasing (AA) / anti-imaging (AI) filters**
  - **Typical signal chain**



# Control room

- Comparison of the control room in the analog and digital eras



aLIGO (2014)



TAMA300 (2001)