

Suppressing Parametric Instabilities



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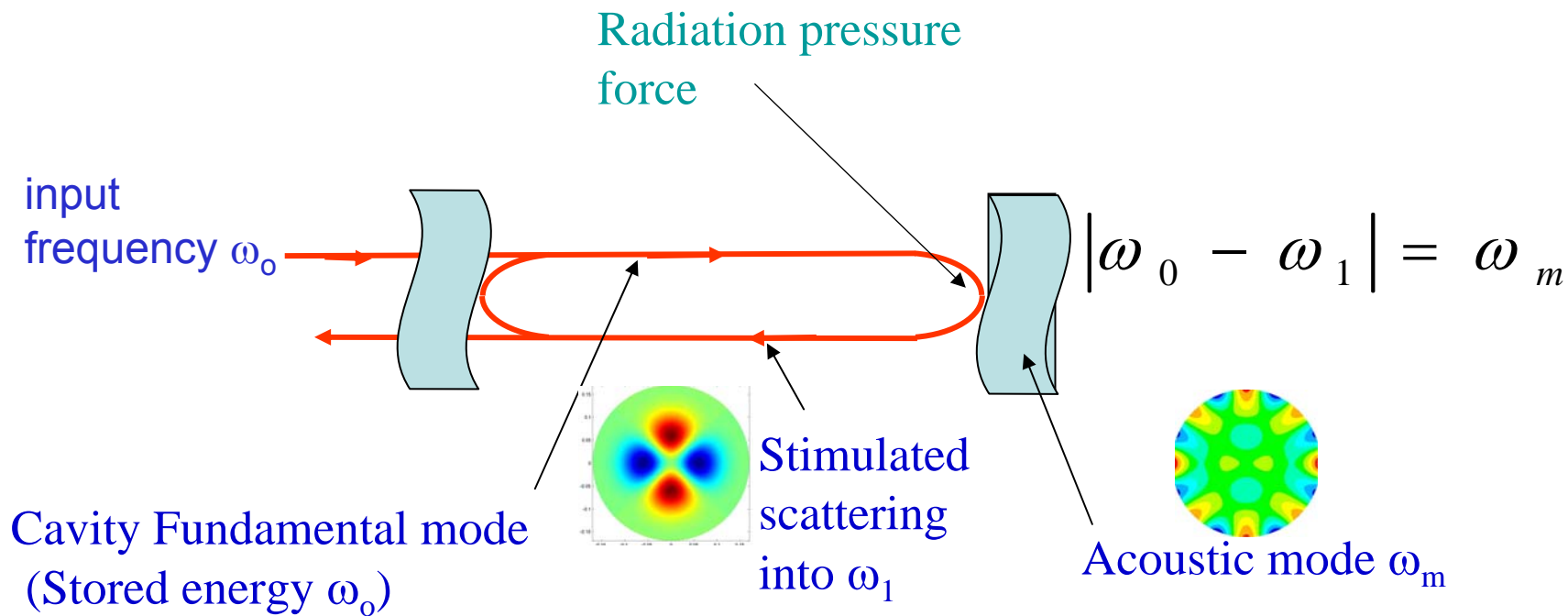
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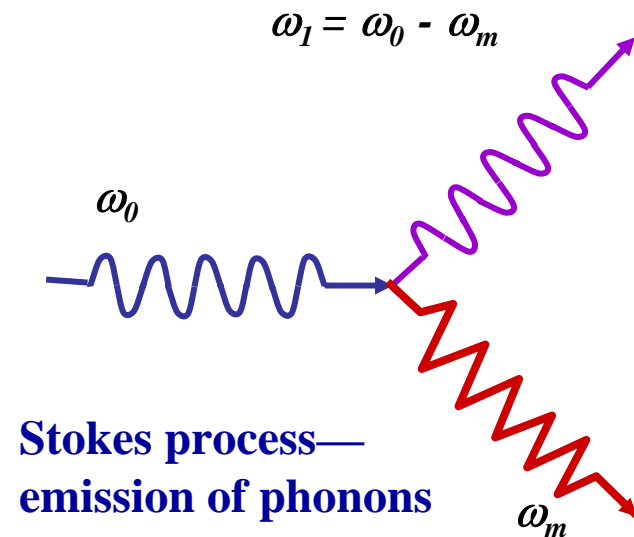
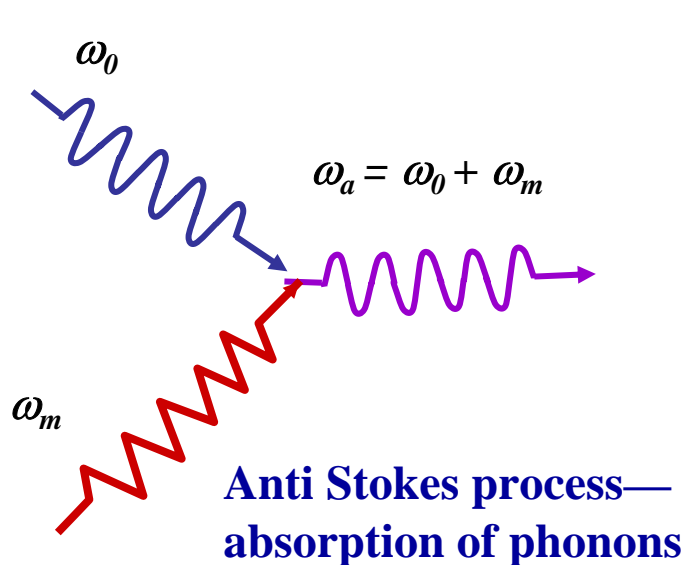
Parametric Instability



Photon-phonon scattering

Instabilities from photon-phonon scattering

- A test mass phonon can be **absorbed** by the photon, increasing the photon energy (**damping**);
- The photon can **emit** the phonon, decreasing the photon energy (**potential acoustic instability**).



Instability Condition

Parametric gain^[1]

$$R \approx \frac{2PQ_m}{McL\omega_m^2} \left(\frac{Q_1\Lambda_1}{1 + \Delta\omega_1^2 / \delta_1^2} - \frac{Q_{1a}\Lambda_{1a}}{1 + \Delta\omega_{1a}^2 / \delta_{1a}^2} \right) > 1$$

Cavity Power
 Mechanical Q
 Stokes mode contribution
 Anti-Stokes mode contribution

$$\Delta\omega_{1(a)} = \left| \omega_0 - \omega_{1(a)} - \omega_m \right|$$

Λ —overlap factor

$$\delta_{1(a)} = \frac{\omega_{1(a)}}{2Q_{1(a)}}$$

Fundamental mode frequency

High order transverse mode frequency

Acoustic mode frequency

[1] V. B. Braginsky, S.E. Strigin, S.P. Vyatchanin, *Phys. Lett. A*, 305, 111, (2002)

Parametric Instability Condition

$$R = \sum \frac{4PQ_{mj}}{McL\omega_{mj}^2} \left(\frac{Q_{1i}\Lambda_{1i}}{1 + \Delta\omega_{1i}^2 / \delta_{1i}^2} \times \Omega_{1i} - \frac{Q_{1ai}\Lambda_{1ai}}{1 + \Delta\omega_{1ai}^2 / \delta_{1ai}^2} \times \Omega_{1ai} \right)$$

Stokes contribution

Anti-Stokes contribution

related to the power recycling cavity

- Stokes and Anti-Stokes modes usually do not compensation,
- $R \propto Q_{\text{mech}}, Q_{\text{opt}}$
- $\Delta\omega$ is a function of RoC
- Total parametric gain is the summation over all the unstable modes

Parametric instability is a reality

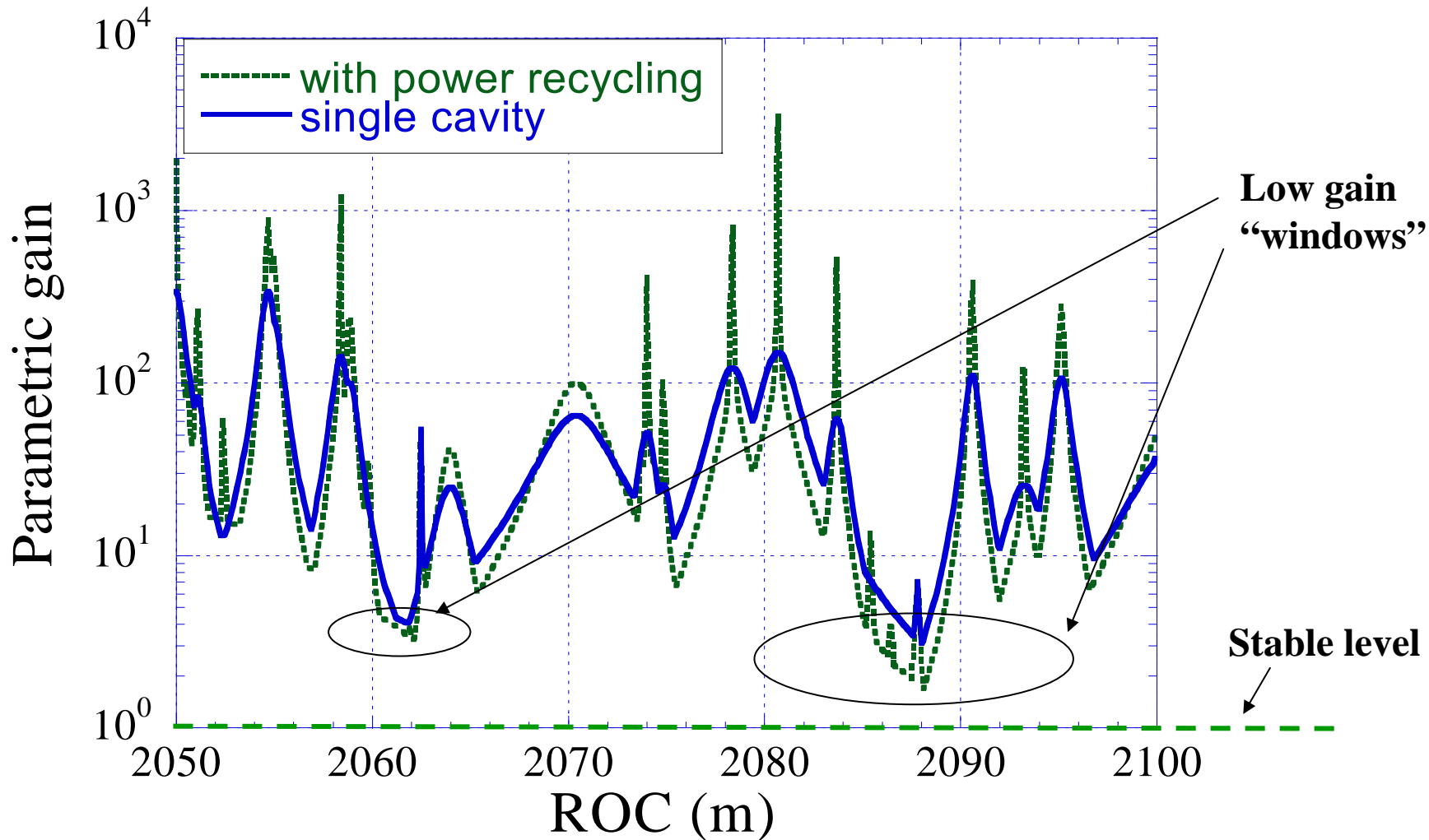
Low frequency parametric instability observed ($f_{\text{mech}} < \text{FSR}$)

- MIT experiment (reported by Tomas Corbitt on Sunday)
- LIGO recent observation of mechanical Q change of 37.8kHz mode (H1 at Hanford, A.C. Melissinos and S. Giampanis, February 27, 2006)

High frequency parametric instability

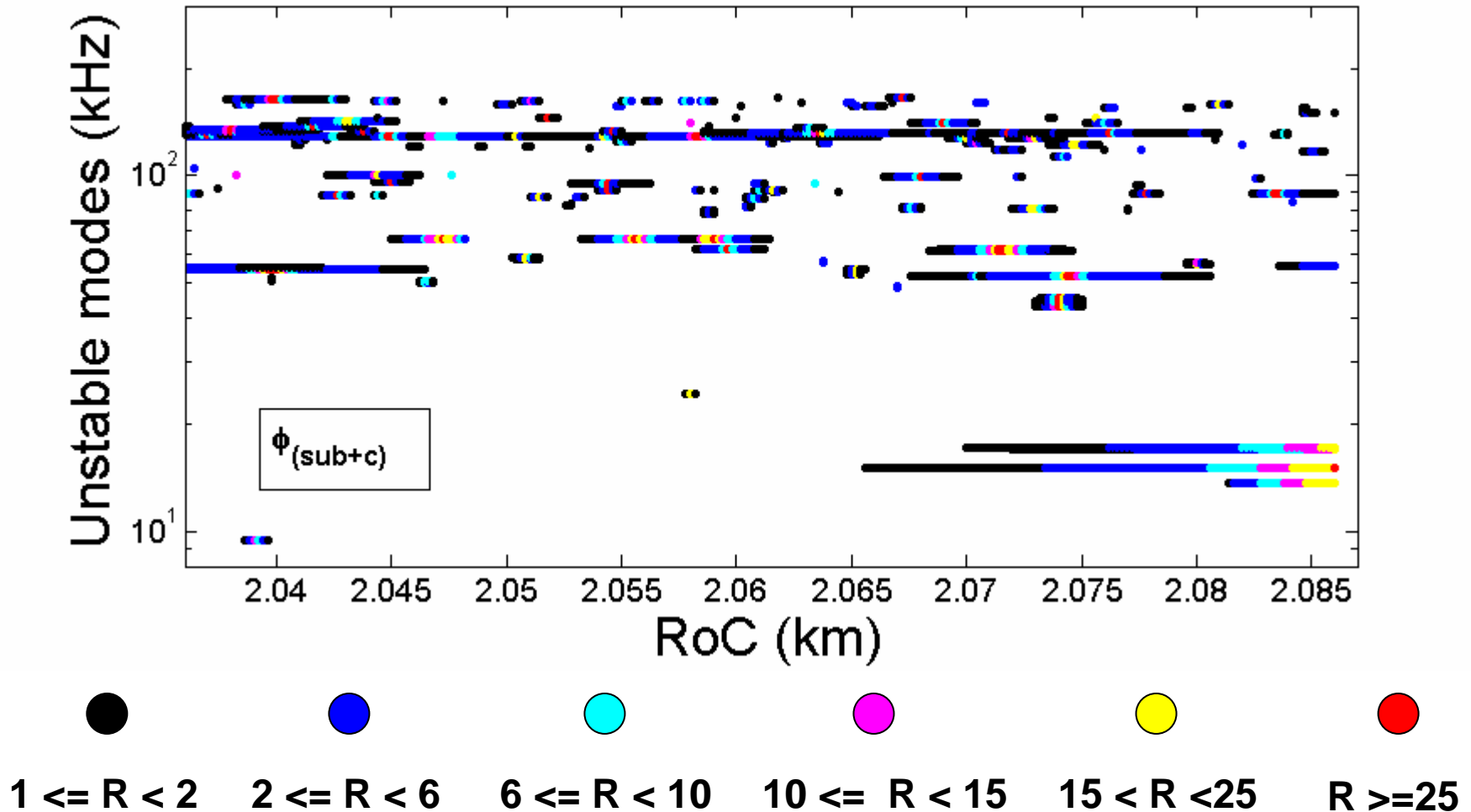
- Stokes & anti-Stokes are not balanced

Parametric Gain Changes with RoC

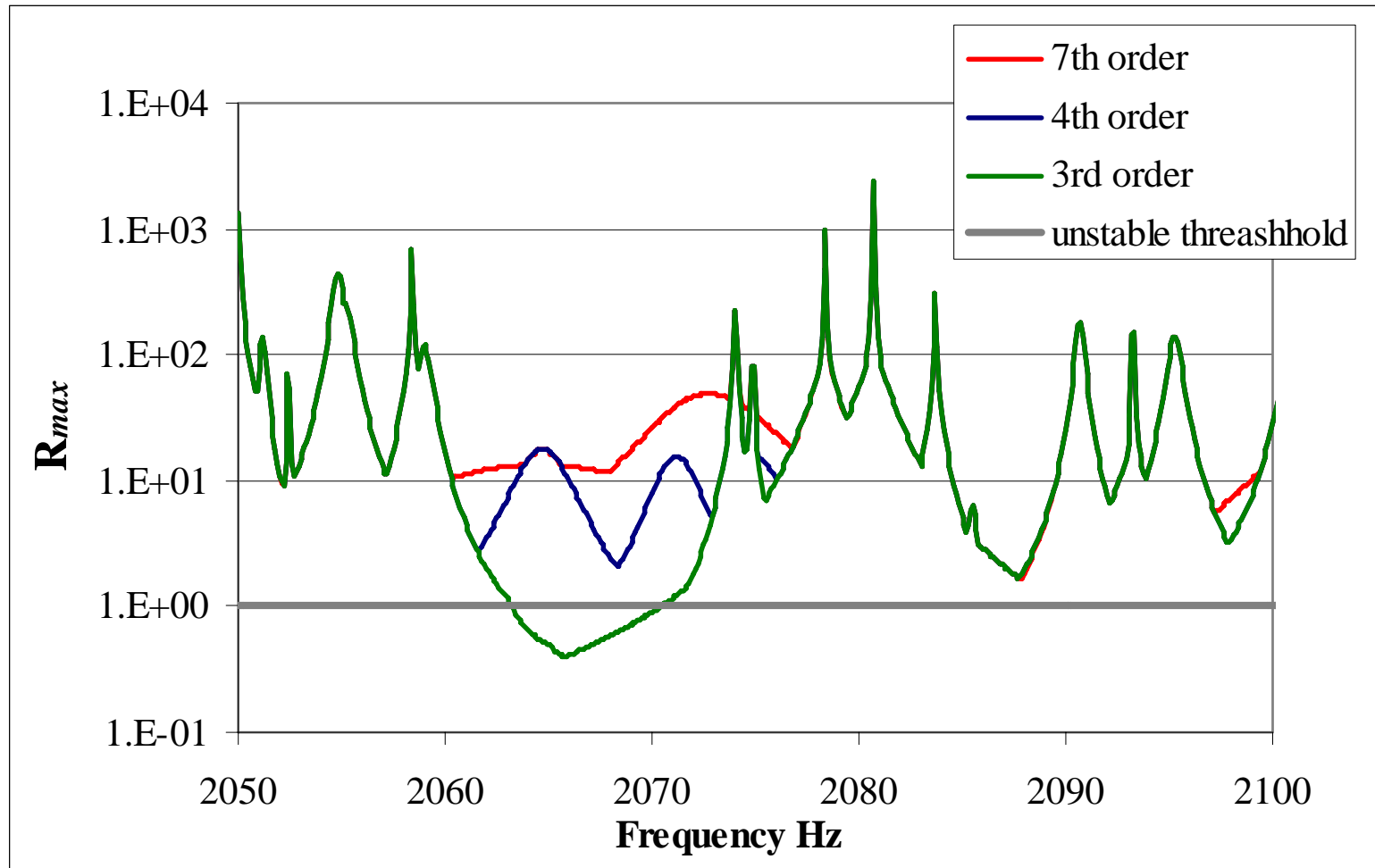


Unstable modes

Selected from 1000 acoustic modes. For the test mass with a substrate and coating loss source there are 317 of unstable modes in the range of the RoC 2.039km – 2.086km.



Higher order optical modes contribution



Explore the low gain “window”

- High order optical mode loss
 - Correct calculation of diffraction loss
 - Can we increase higher order mode loss by non-uniform coating?

Diffraction loss investigation

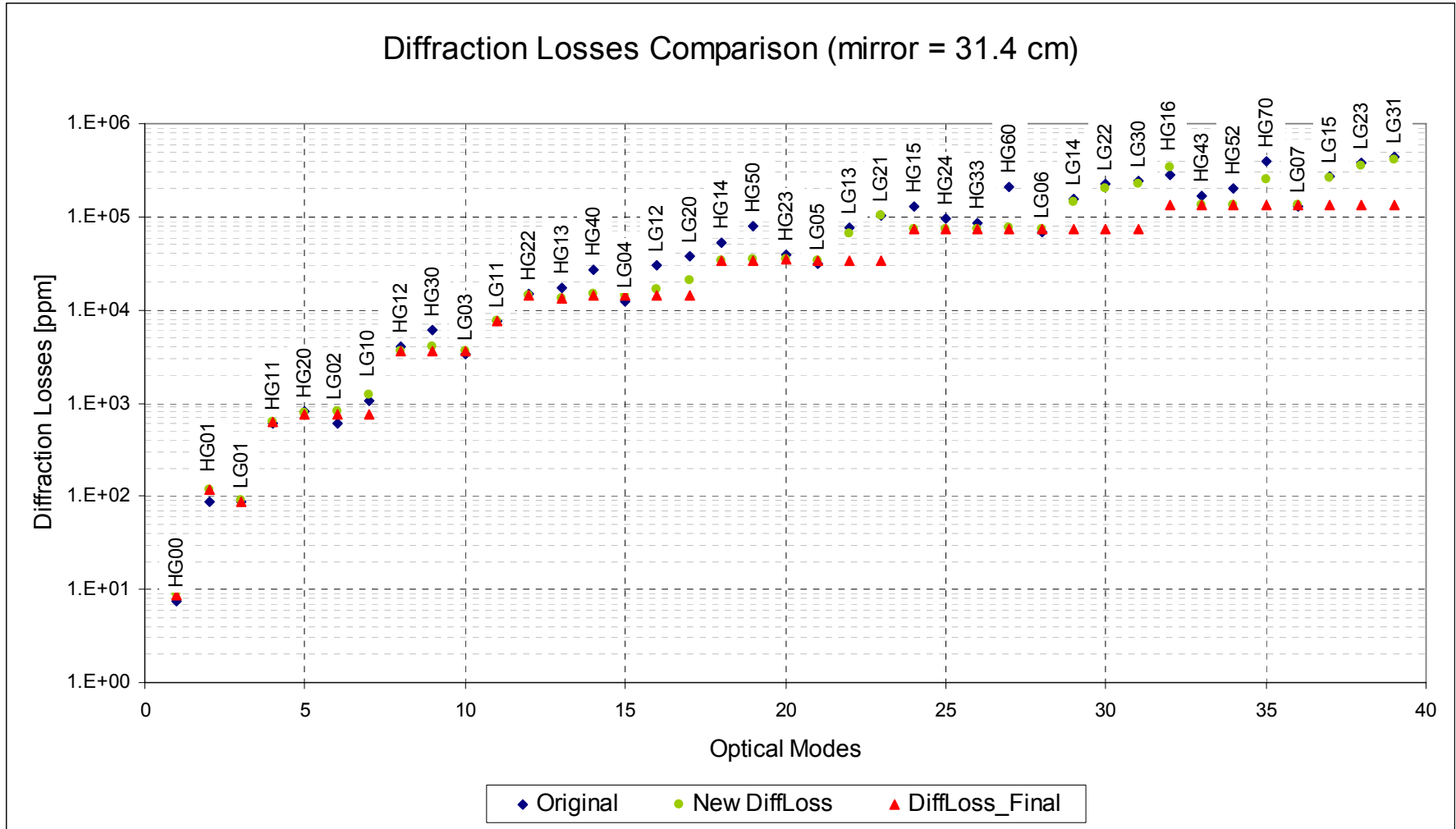


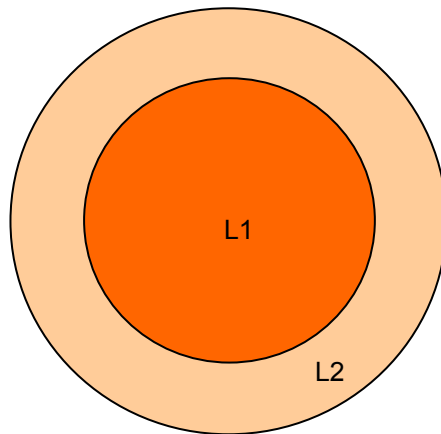
Figure 1: Comparison of the diffraction losses results for an Advanced LIGO type cavity with test masses of diameter 31.4 cm.

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- Cannot ignore higher order modes contributions

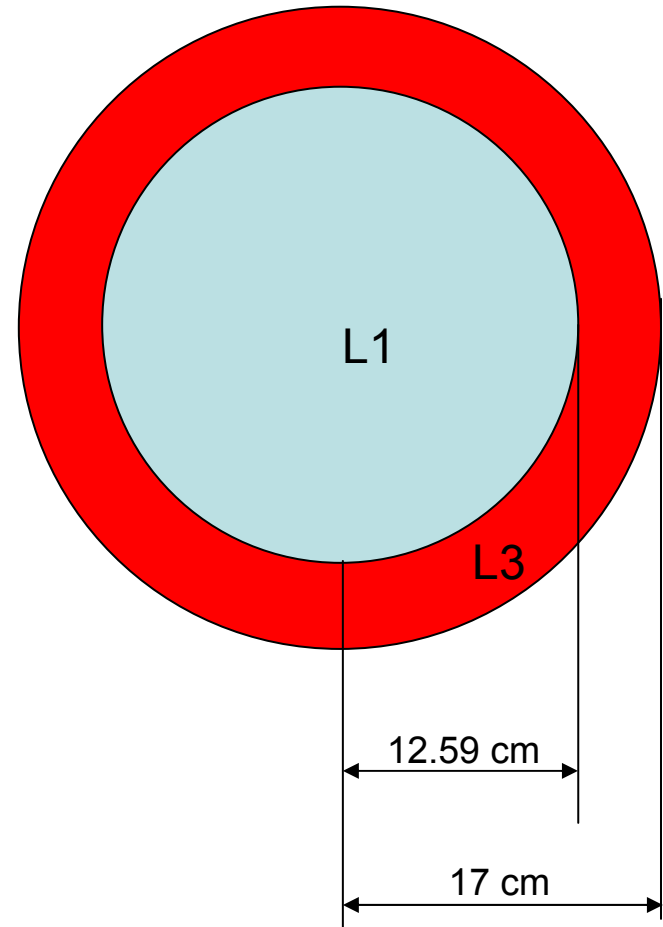
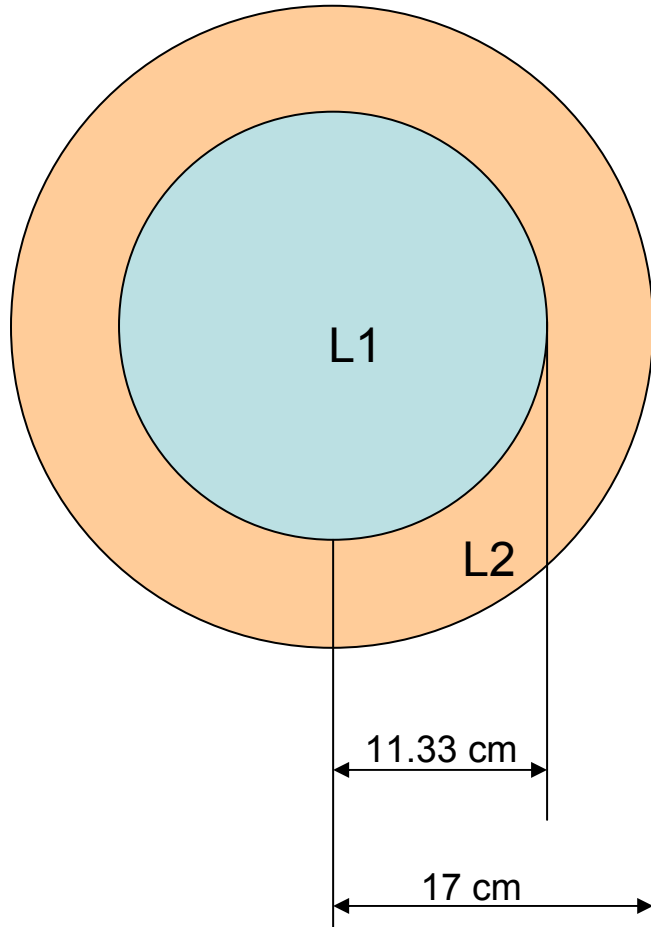
Increase high order modes loss by differential coating?

Suggested by Reccardo DeSalvo, analyzed by Pablo Barriga

- Reduce the parametric gain by increasing the higher order mode loss while maintaining the fundamental mode loss < 1 ppm

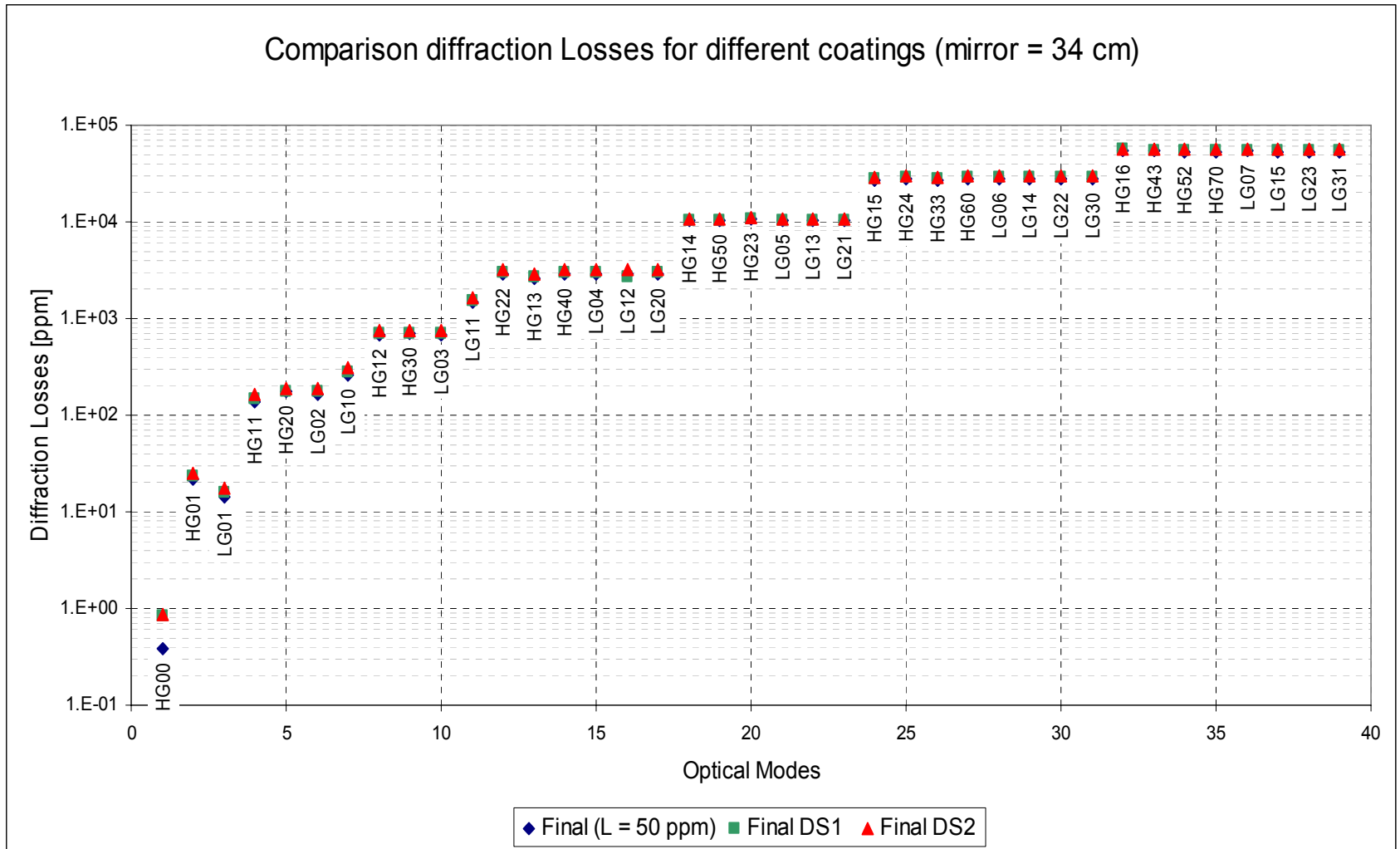


Differential Coating



L1=50ppm, L2=25,000ppm, L3=100,000ppm

Differential Coating



L1=50ppm, L2=25,000ppm, L3=100,000ppm

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- No significant difference of diffraction loss between the homogeneous and the differential coatings

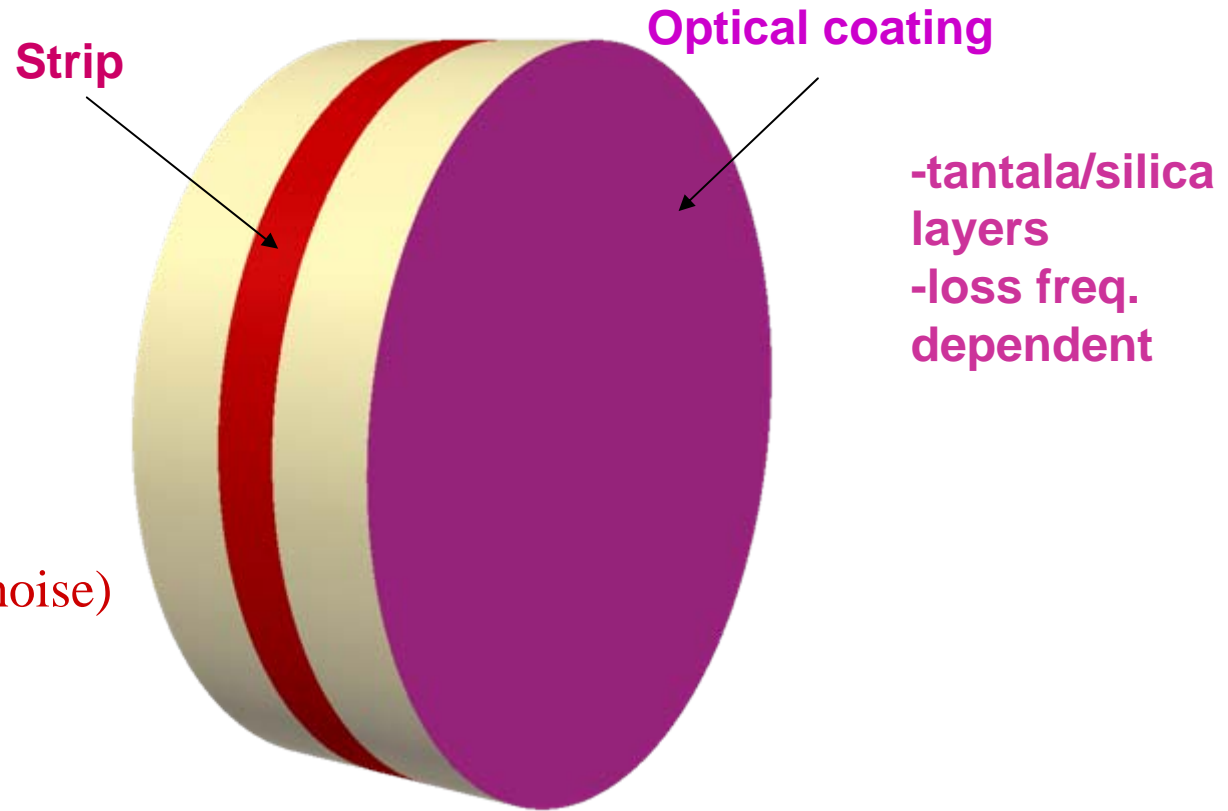
Ring damper—reducing the mechanical Q

- Introduce local damping (rim of the test masses) far away from the centre of the mirror
 - reduce mechanical mode the mechanical modes Q of the test mass without degrading thermal noise (much)

(Reccardo, Gretarsson, UWA)

 - Tests with a rubber o-ring and tape on a test mass at Caltech thermal noise facility

Test mass with ring damper model

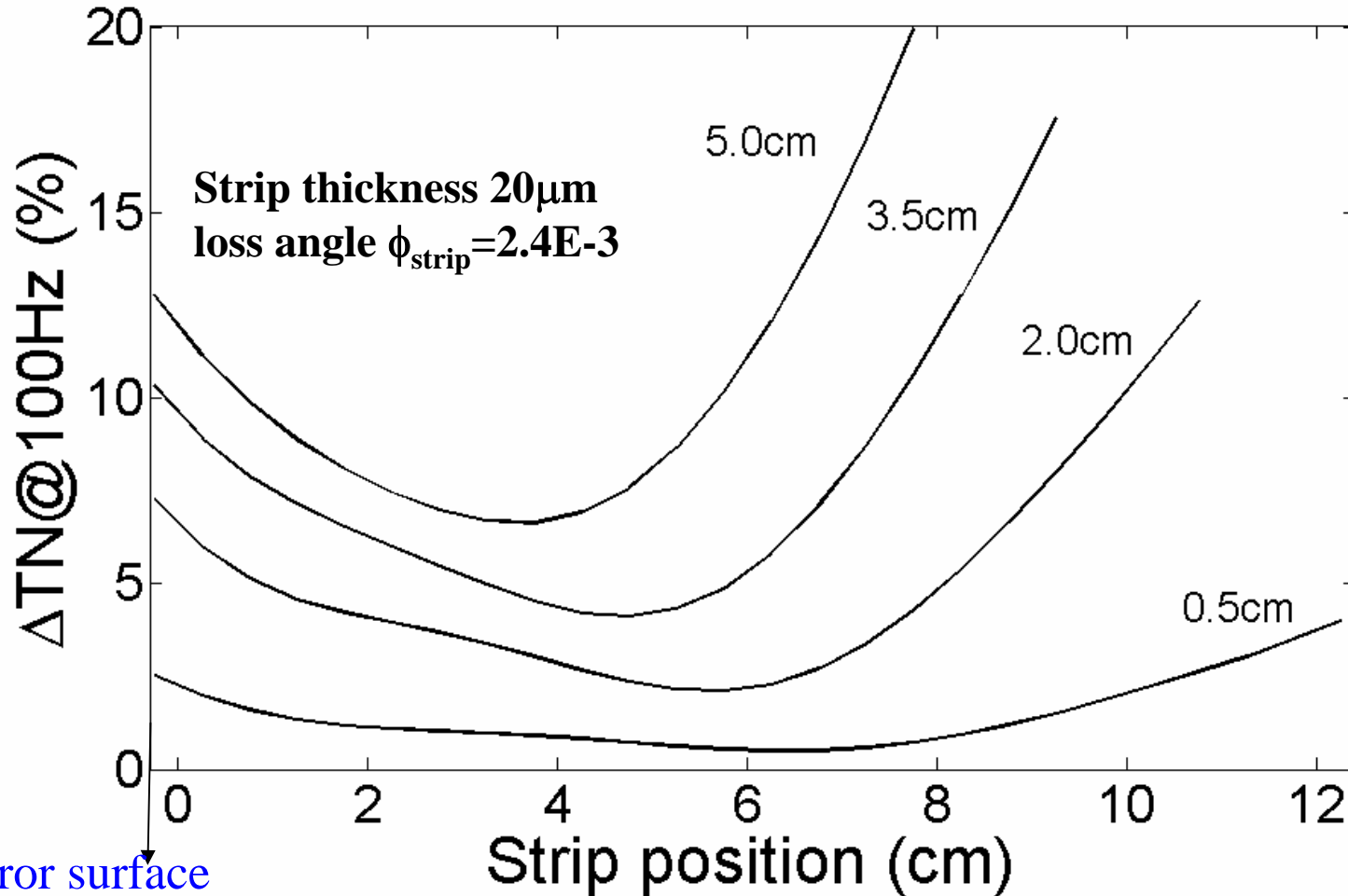


Strip variables:

- position
- dimensions:
 - thickness
 - width
- loss angle (thermal noise)

Test mass radius $r = 0.157\text{m}$
Thickness $d = 0.13\text{m}$

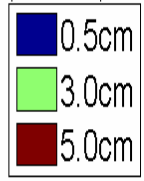
Ring damper position and width vs thermal noise degradation



There exist a strip position where the thermal noise change is minimal

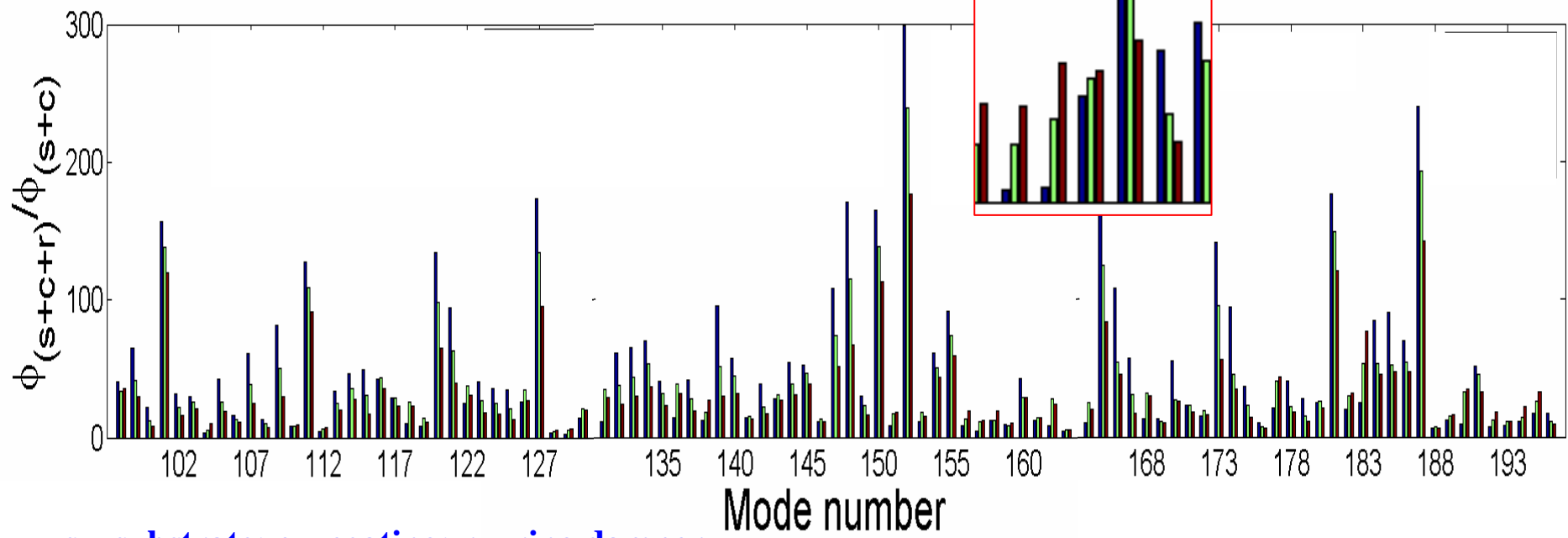
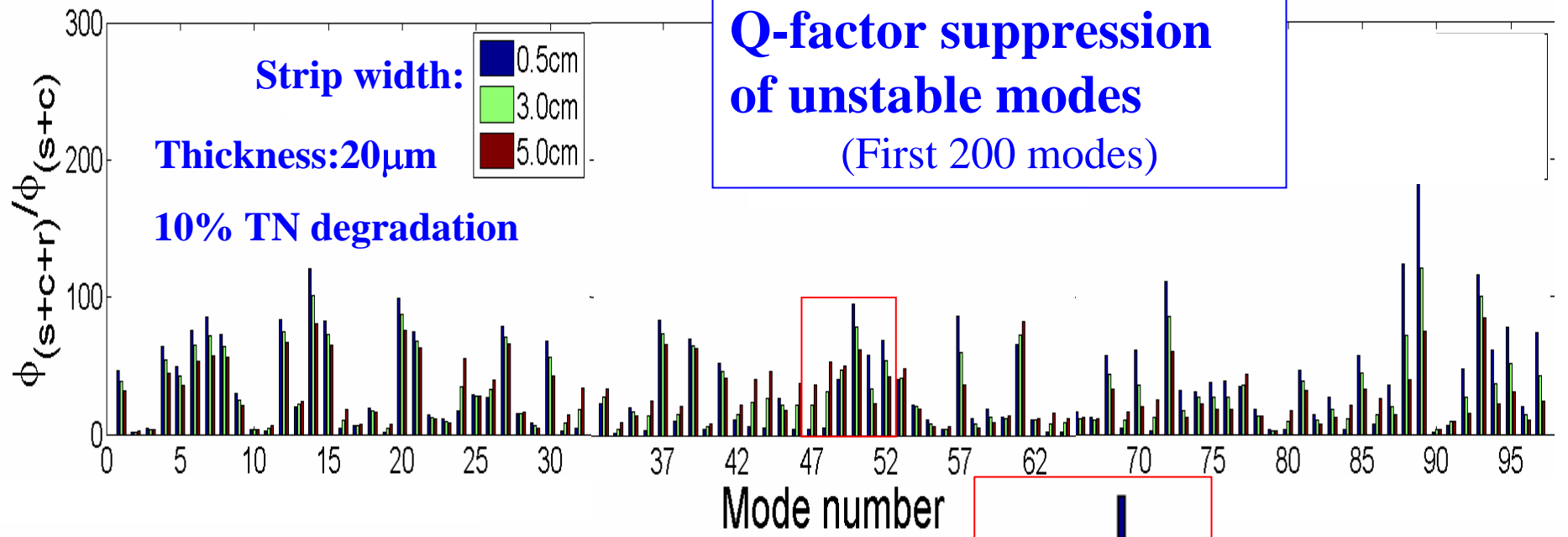
**Q-factor suppression
of unstable modes**
(First 200 modes)

Strip width:



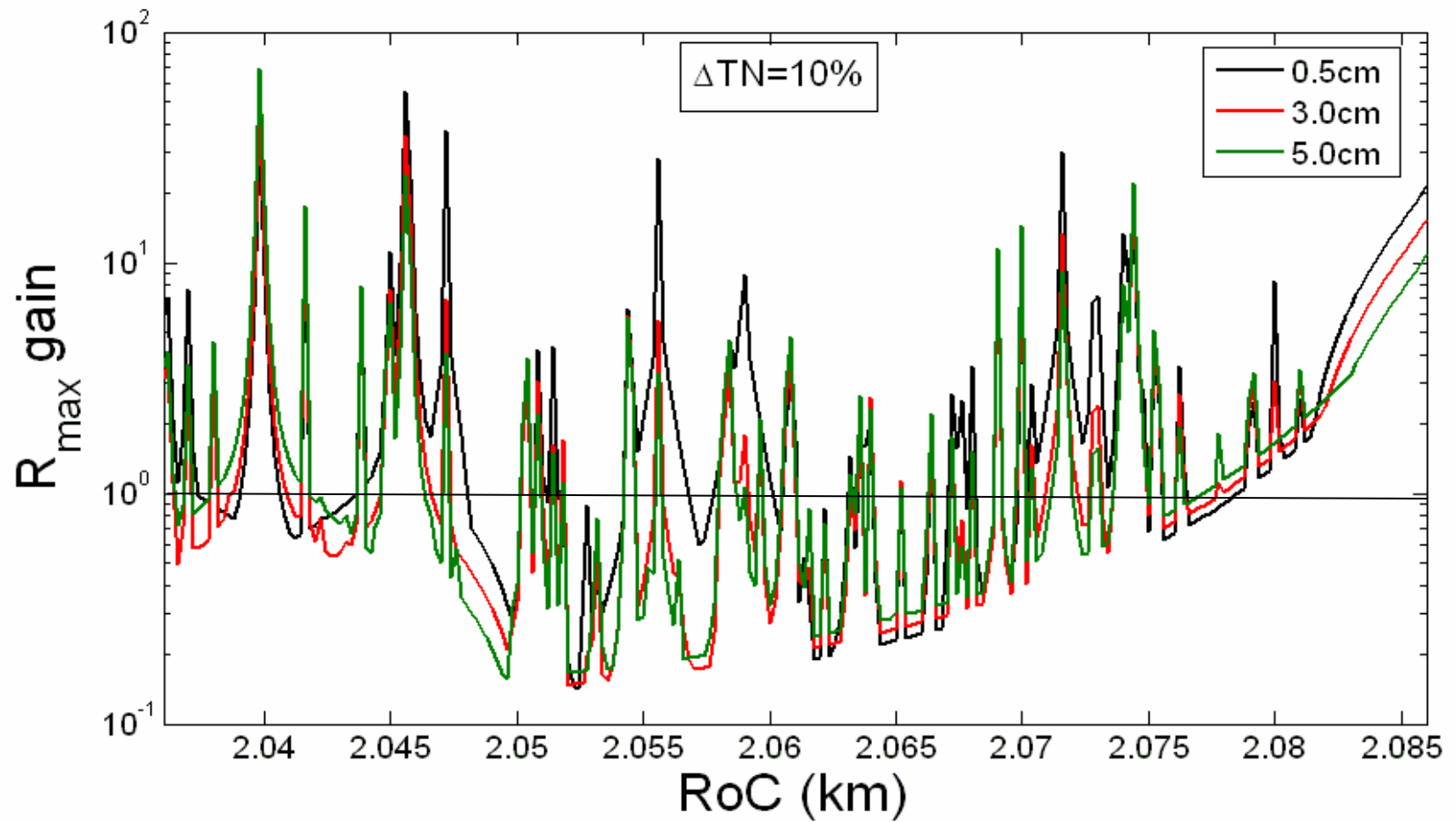
Thickness: 20μm

10% TN degradation



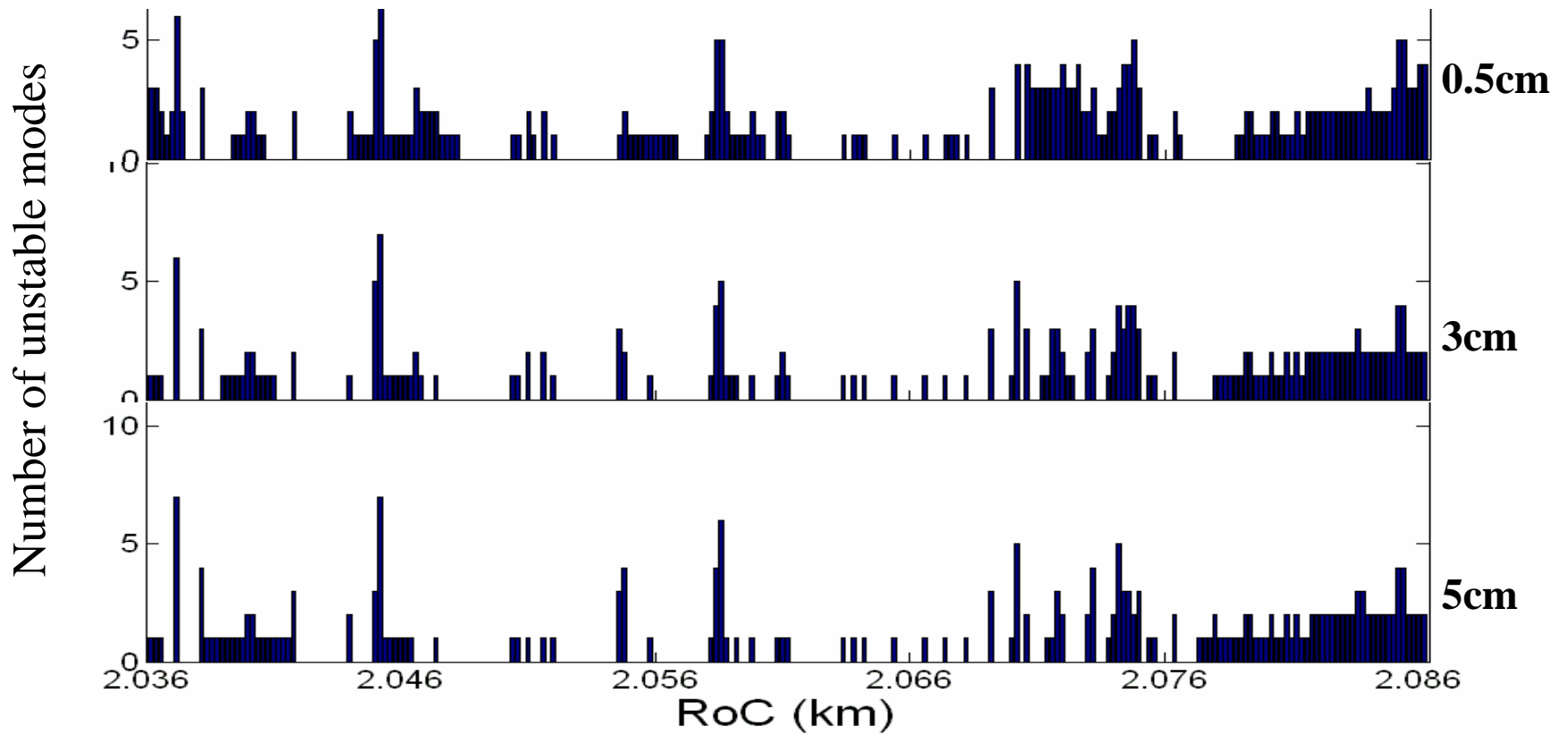
s—substrate; c—coating; r—ring damper

Effect of different strip width (fixed ΔTN)

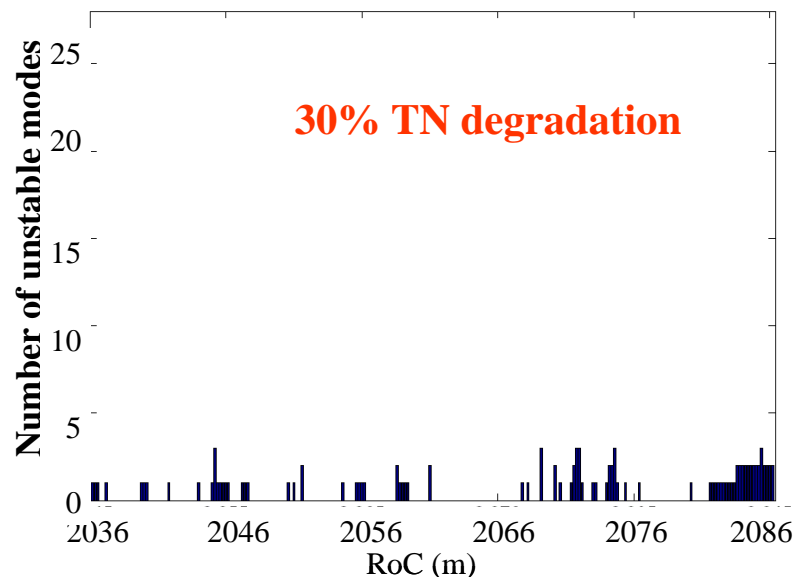
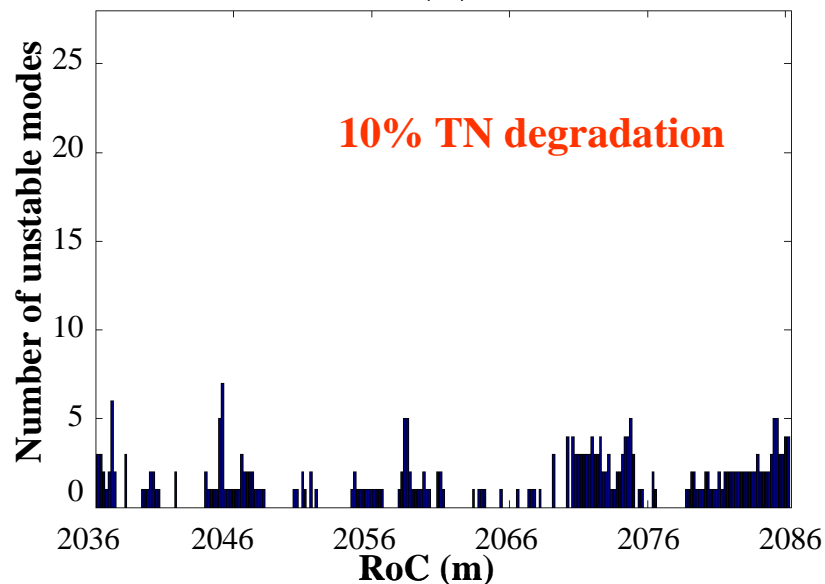
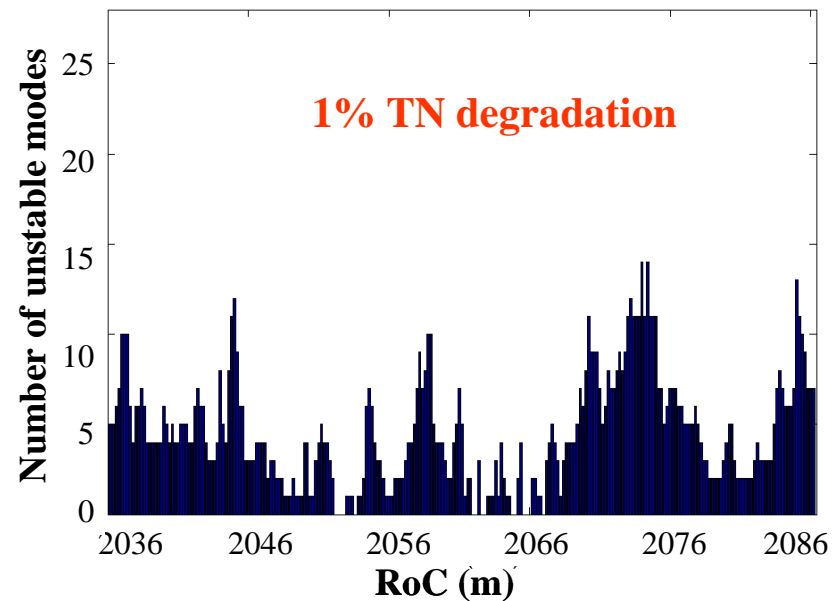
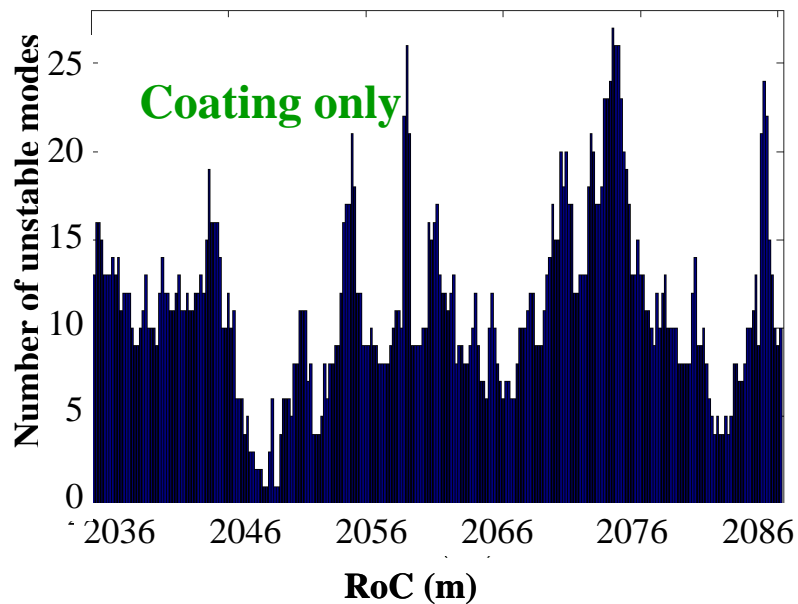


$\Delta TN=10\%$

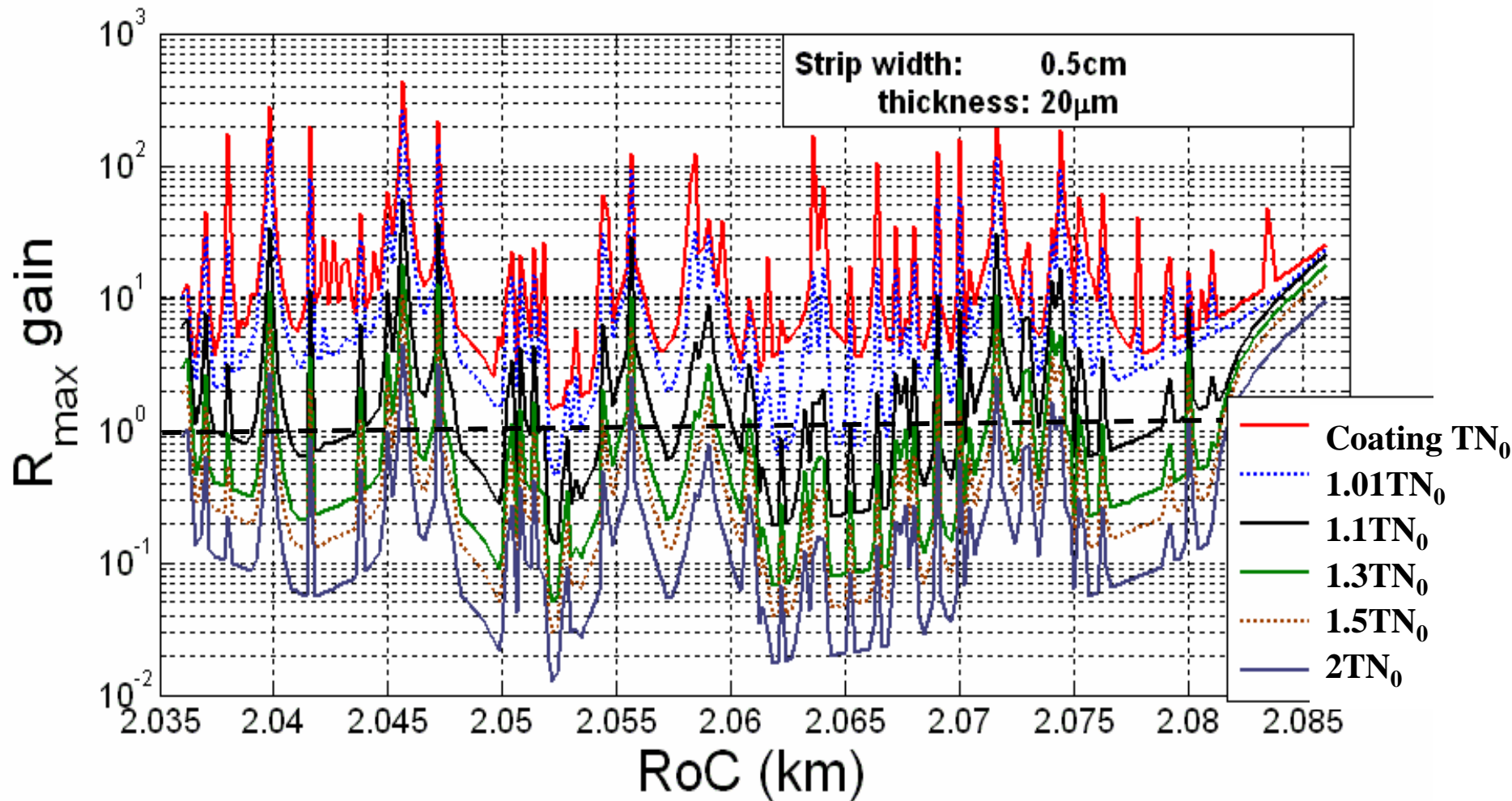
Strip width

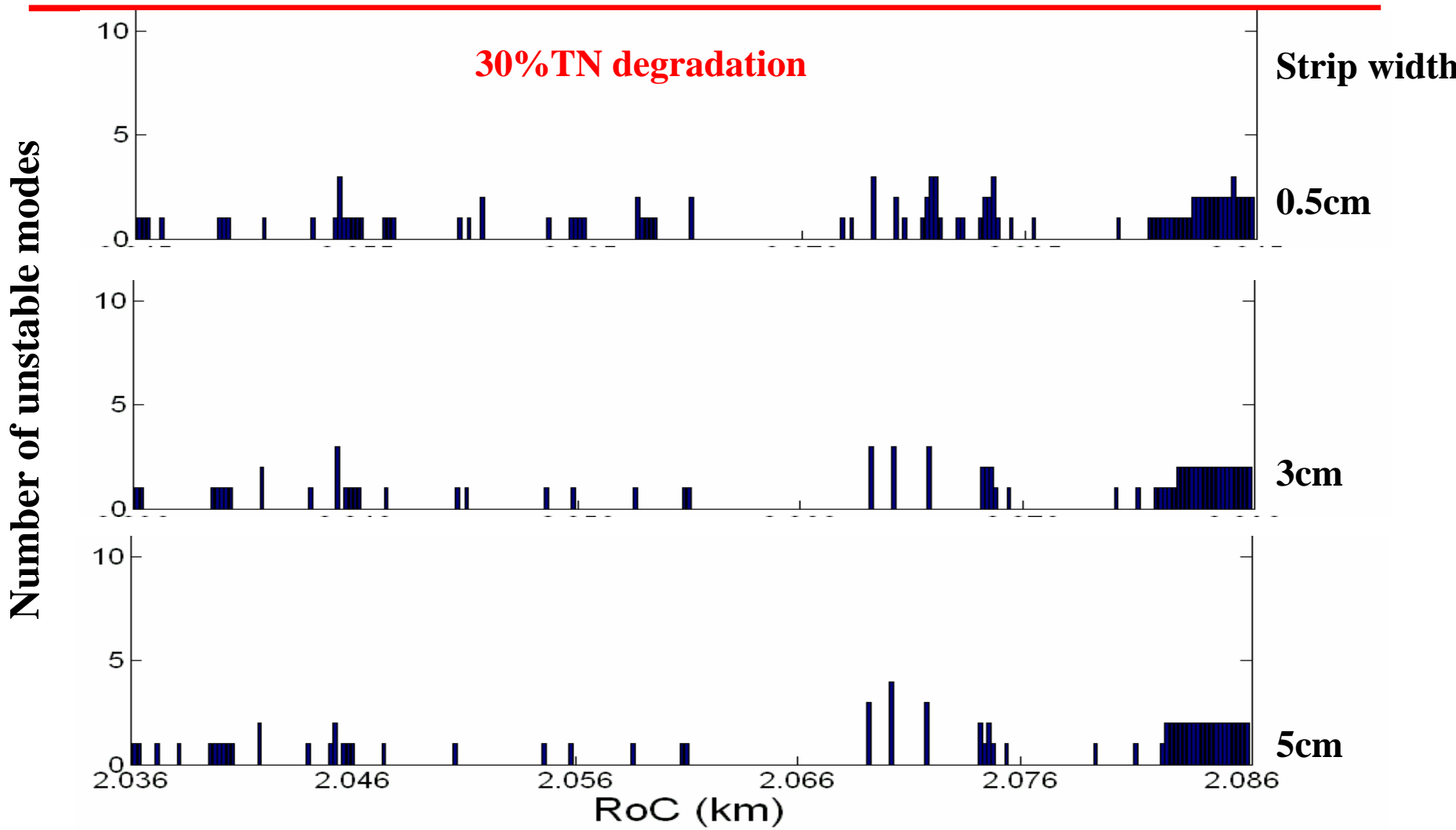


Unstable mode ($R>1$) with different thermal noise degradation (5mm strip width with different material loss angle)



Parametric gain reduction with ring damper

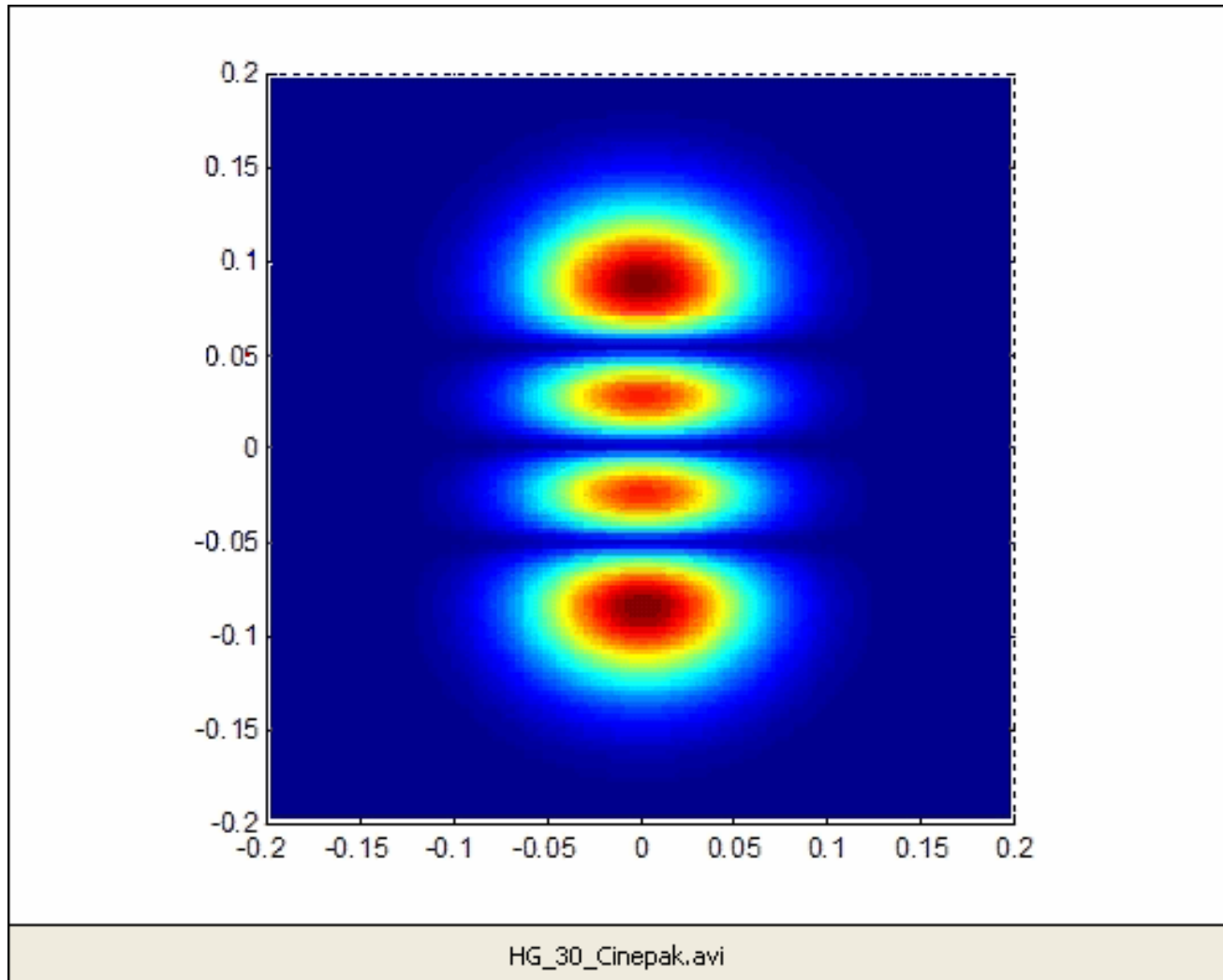




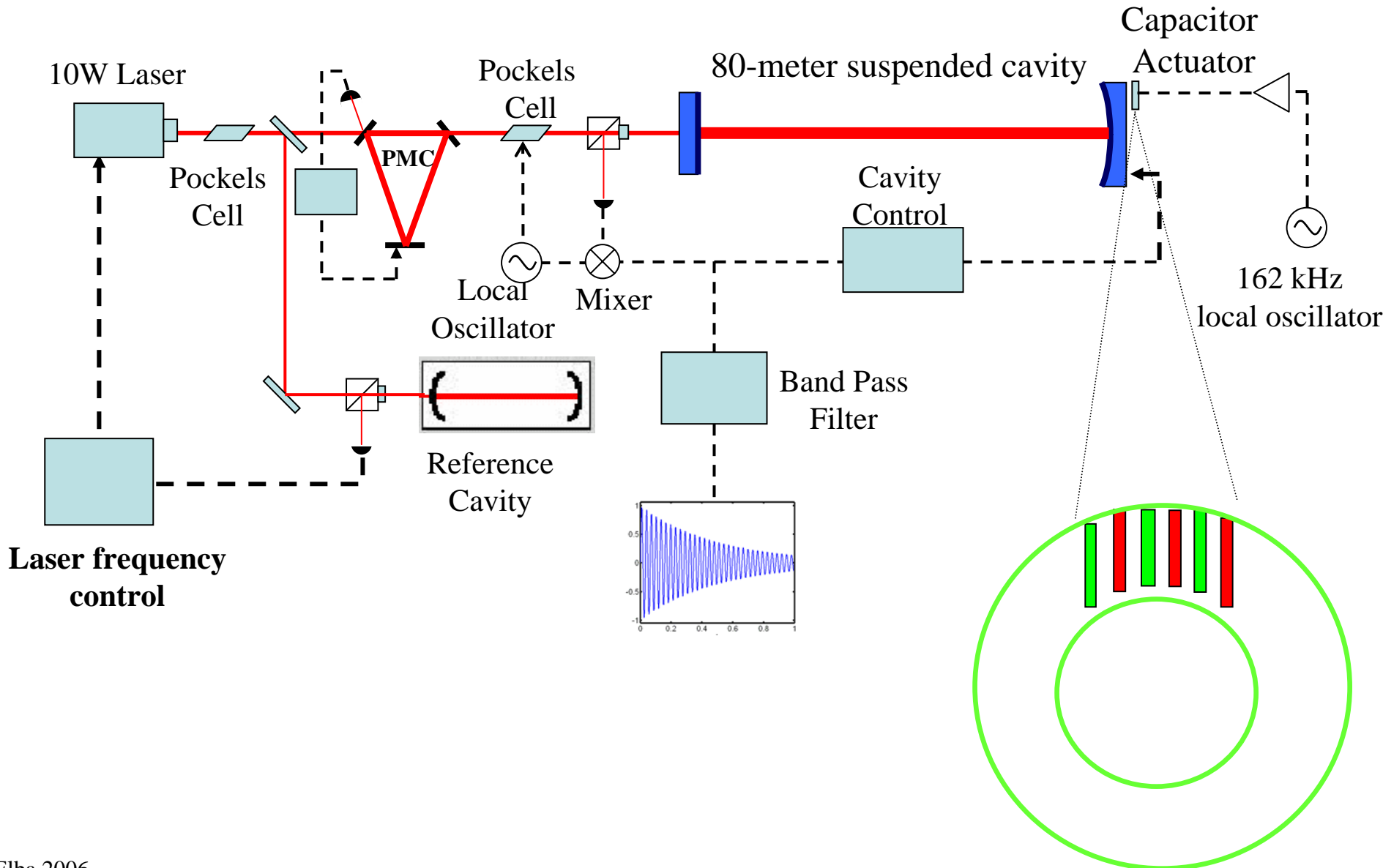
Conclusion

- Ring damper is an effective method to open up stable operation window, at a price of increased thermal noise
 - How much thermal noise increase is tolerable?
- Other possible solution
 - Active feedback (complicated)
- Gingin high power facility will investigate PI experimentally

Mode



Active feedback to suppress parametric instability



FEM analysis

Substrate: Al_2O_3

- solid95, 21725 elements

- assumed isotropy

- loss angle $\varphi = 1\text{E-}08$

Young modulus $E = 400\text{E}09$

Poisson ratio $\rho = 0.23$

Density $\rho = 3983 \text{ kg/m}^3$

Coating: $\text{SiO}_2/\text{Ta}_2\text{O}_5$

- solid46, 869 elements

- 30 layers of $\text{SiO}_2/\text{Ta}_2\text{O}_5$

- thickness $30(\lambda/4 + \lambda/4) = 15\mu\text{m}$

- assumed loss isotropy $\varphi_{\parallel} = \varphi_{\perp}$

- loss frequency dependent (*)

$$\varphi = 4.0\text{E-}05 + f \cdot 2.7\text{E-}09$$

$$\varphi = 4.2\text{E-}04 + f \cdot 0.4\text{E-}09$$

SiO_2 : Young modulus $E = 70\text{E}09$

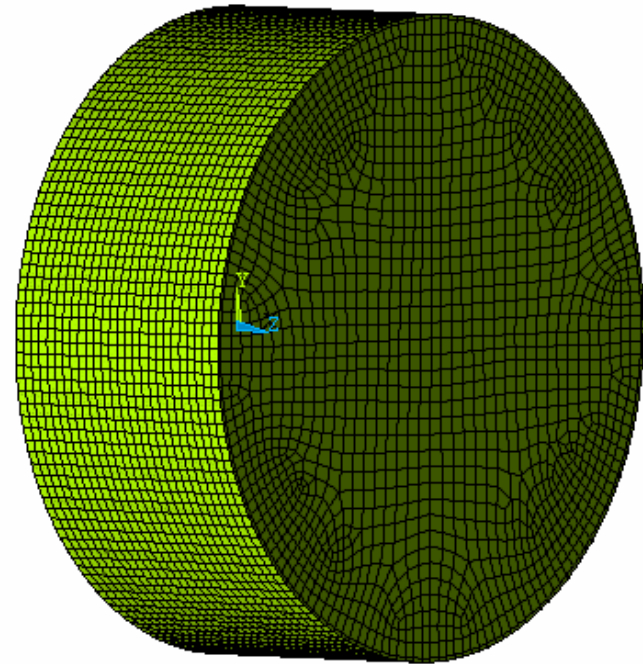
Poisson ratio $\rho = 0.17$

Density $\rho = 2200 \text{ kg/m}^3$

Ta_2O_5 : Young modulus $E = 140\text{E}09$

Poisson ratio $\rho = 0.23$

Density $\rho = 8200 \text{ kg/m}^3$



FEM model of the test mass

Strip:

- Material properties like for Al_2O_3 (still good approximation)
- Various loss angles, various thickness and width for desired thermal noise level

* Harr G M, ... et al. 2004 Proceedings of the SPIE 5527 33