

Viewgraphs on Test Mass Suspensions and Suspension Noise

10 DECEMBER 1995

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LIGO-G960005



TEST MASS SUSPENSIONS AND SUSPENSION NOISE

ANNUAL REPORT OF MSU GROUP

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THREE DIRECTIONS OF RESEARCH IN 1995:

1. STUDY OF DISSIPATION OF ENERGY IN PENDULUM MODES OF THE INTERMEDIATE MASS $M \approx 2\text{kg}$, SUSPENDED BY FUSED SILICA FIBERS.
2. MEASUREMENT OF EXCESS NOISE IN TUNGSTEN WIRES
 - i) EXCESS NOISE IN VIOLIN MODES,
 - ii) TORSION JUMPS IN THE LOADED WIRES.
3. MEASUREMENT OF MAGNETIC FIELD AND ITS FLUCTUATIONS IN THE LABORATORY.

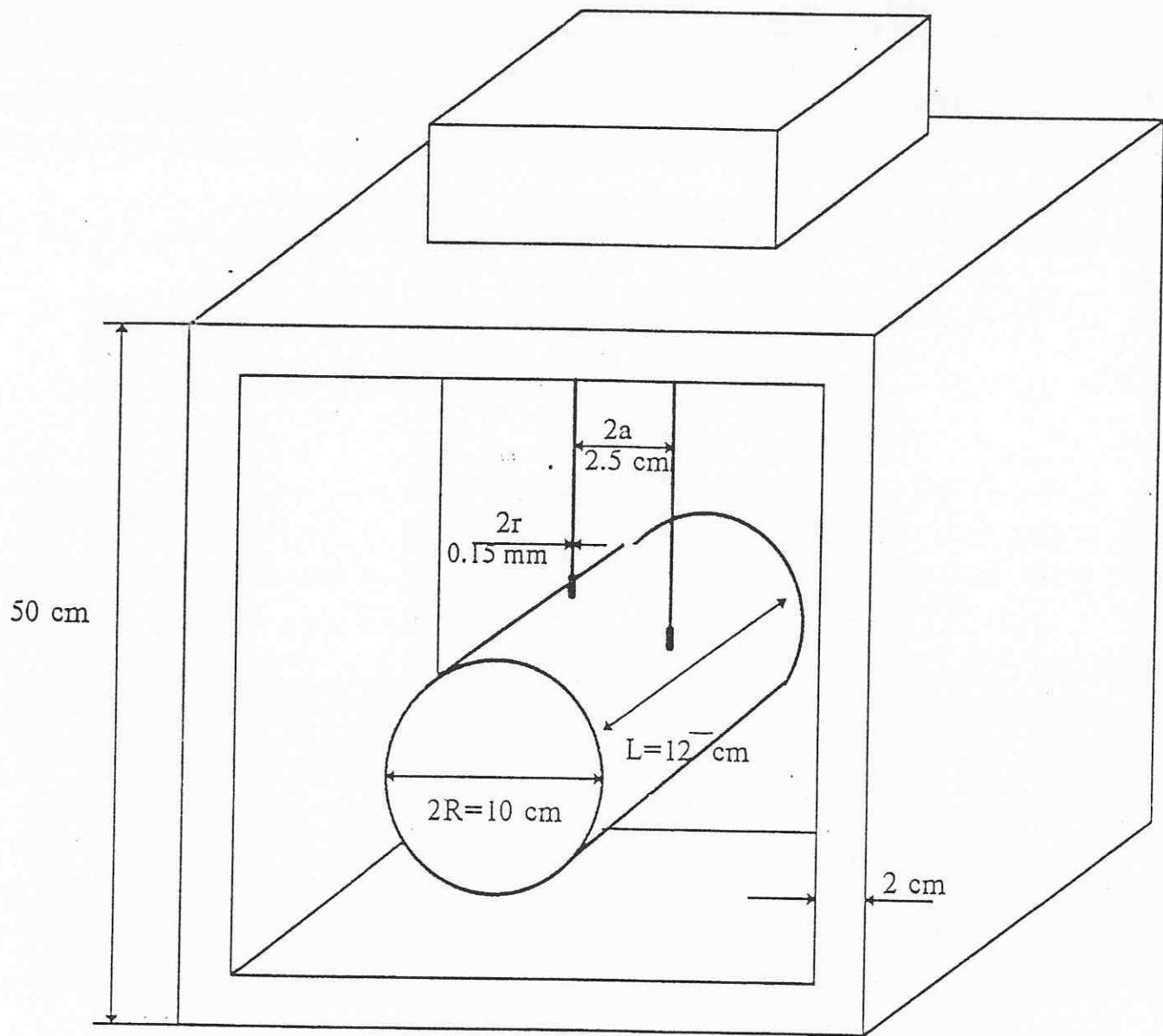


Fig. 1. Design of the double fiber pendulum and its support.

Q OF THE TORSIONAL-PENDULUM MODE OF THE TEST MASS SUSPENDED BY TWO FUSED SILICA FIBERS.

THIS MODE HAS SMALLER RECOIL LOSSES IN SUPPORT THAN THE ORDINARY PENDULUM MODE

$$f_{T-P} = 0.35 \text{ Hz}$$

THEORETICAL ESTIMATION OF THE LOSSES:

$$Q_{\text{MATERIAL}}^{-1} \approx \frac{\pi G r^4}{M g a^2} \phi_g + \frac{\sqrt{T Y I}}{2 M g L} \phi_y < 3 \cdot 10^{-9}$$

$$Q_{\text{SUPPORT}}^{-1} \underset{(\text{SIMPLE MODEL})}{\approx} \frac{J_{\text{T.M.}}}{J_{\text{SUP}}} \left(\frac{\omega_{\text{T.M.}}}{\omega_{\text{SUP}}} \right)^2 Q_{\text{OWN SUP}}^{-1} < 2 \cdot 10^{-9}$$

THE MEASURED $Q = (0.5 \div 1) \cdot 10^8$.

POSSIBLE ADDITIONAL LOSSES MECHANISMS
REQUIRED FURTHER DETAILED STUDY

- a) CONTACT LOSSES IN THE SUPPORT STRUCTURE,
- b) SILICA VAPOUR SEDIMENTS ON THE FIBER SURFACE,
- c) ?!

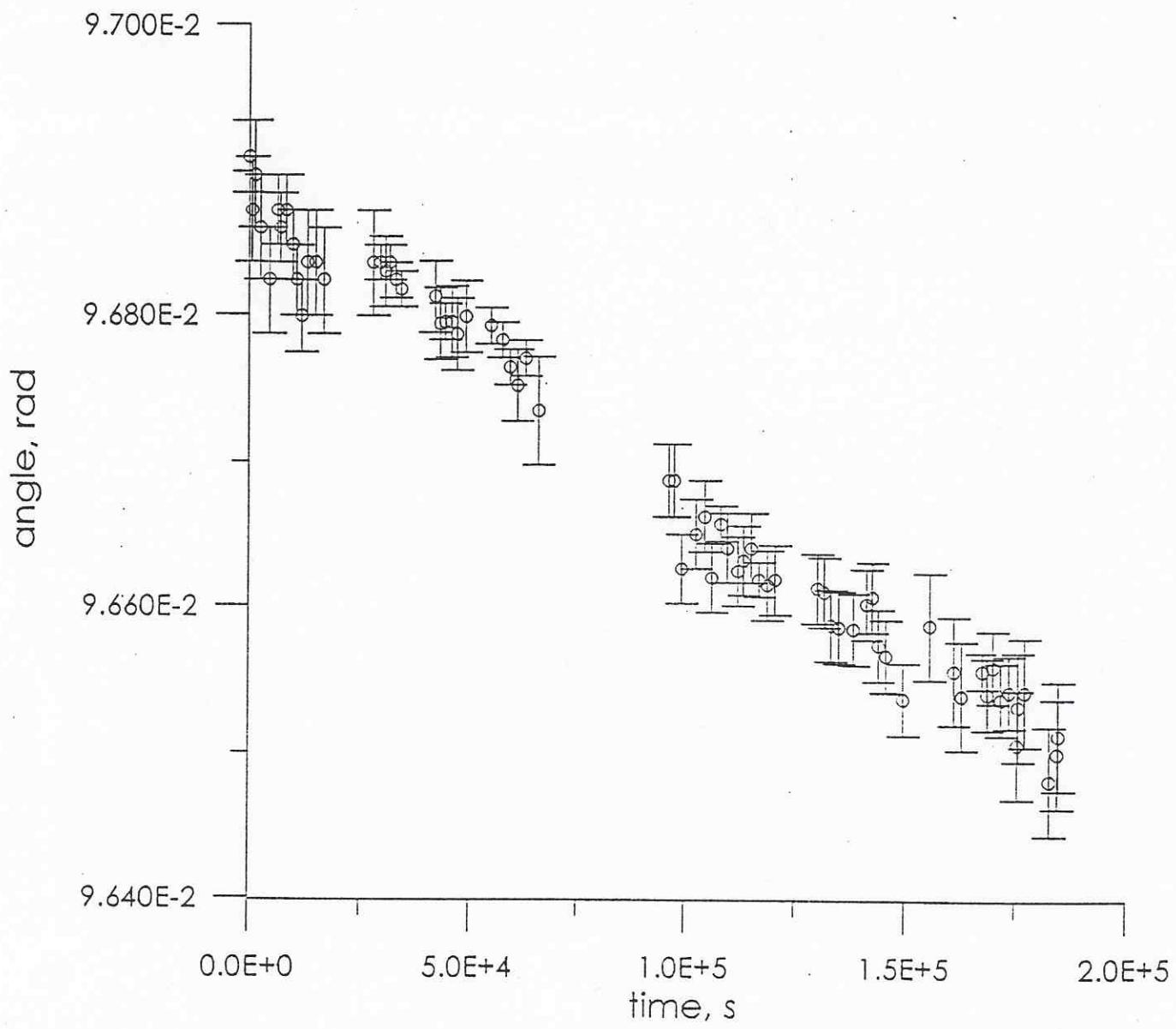


Fig. 2 . Time dependence of amplitude of the double fiber pendulum № 1
 $(M = 2 \text{ kg.}, d_{\text{fiber}} = 150\text{-}200 \text{ mkm}, T=2.85 \text{ s})$.

The measured time of relaxation $\tau^* = 4.8 \times 10^7 \text{ s} (\pm 2\%)$.
 With extraction of residual gas damping ($p = 2 \times 10^{-6} \text{ Torr}$)
 $\tau^* = 1 \times 10^8 \text{ s. } (\pm 25\%)$

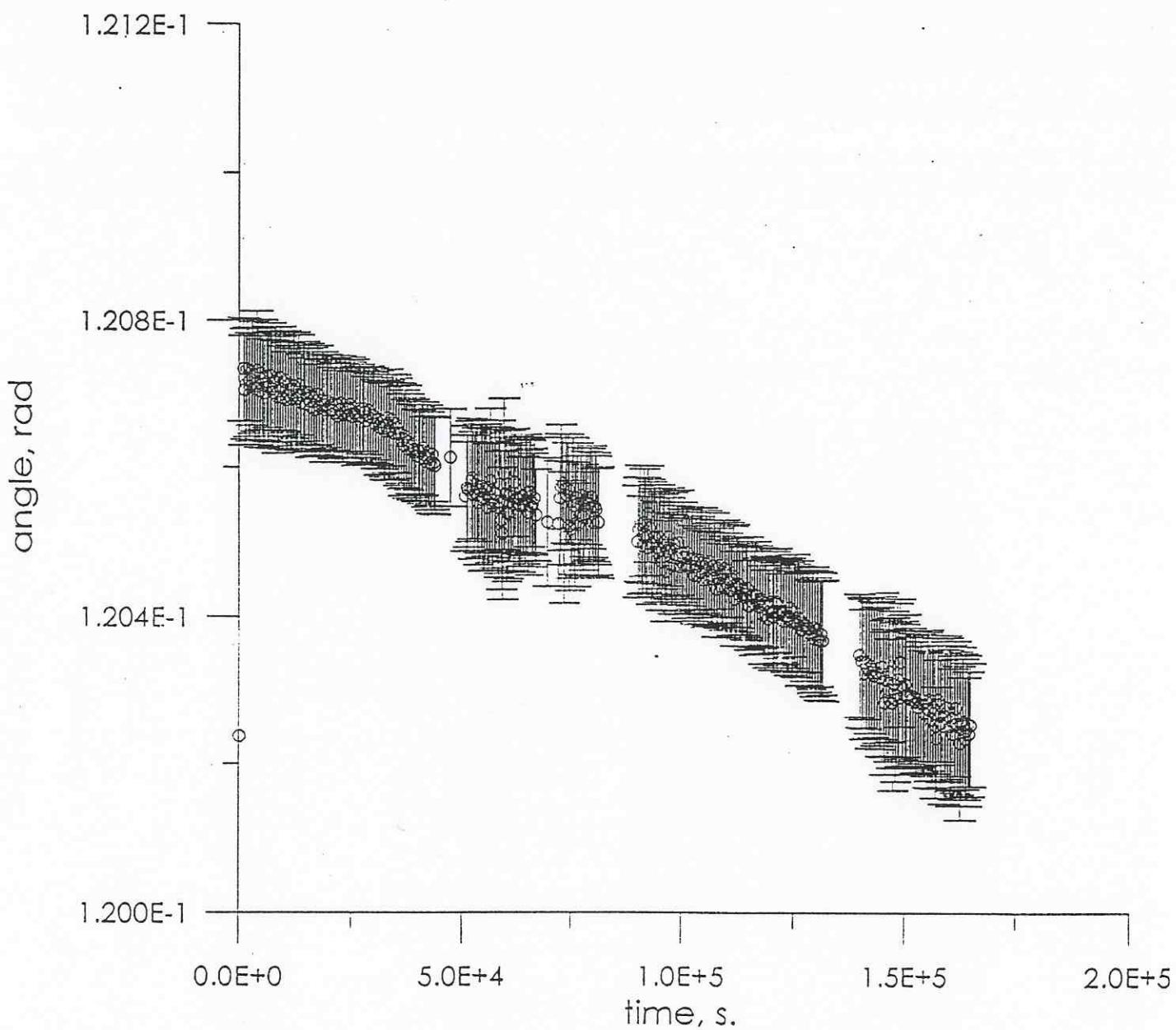


Fig. 3 . Time dependence of amplitude of the double fiber pendulum № 2
 $(M = 2 \text{ kg.}, d_{\text{fiber}} = 150\text{-}200 \text{ mkm}, T=2.95 \text{ s})$.

The measured time of relaxation $\tau^* = 4.6 * 10^7 \text{ s} (\pm 2 \%)$.
 With extraction of residual gas damping ($p = 1 * 10^{-6} \text{ Torr}$)
 $\tau^* = 6.5 * 10^7 \text{ s. } (\pm 25\%)$

EXCESS NOISE IN THE VIOLIN MODES OF TUNGSTEN WIRES



$$L_w = 15 \text{ cm}$$

$$d_w = 20 \mu\text{m}$$

$$\omega_w(\eta) < 2\pi \cdot 1.7 \text{ kHz}$$

$$\langle A \rangle_{\text{THERM}} = \frac{1}{\omega d} \sqrt{\frac{8k_B T}{\pi g L}} \approx 8 \cdot 10^{-10} \text{ cm}$$

RAYLEIGH DISTRIBUTION OF AMPLITUDE

$$\Delta f_{\text{MEASUR}} = 10 \text{ Hz}$$

$$\Delta A_{\text{RES}} \approx 4 \cdot 10^{-11} \text{ cm}$$

TIME OF OBSERVATION AFTER MOUNTING
OF THE WIRE - 5 DAYS.

NET TIME OF MEASUREMENT - 3.5 HOURS.

THE FIRST RUN - $\eta \approx (0.4 \div 0.5) \eta_{\text{BREAK}}$

THE SECOND RUN - $\eta \approx (0.8 \div 0.9) \eta_{\text{BREAK}}$

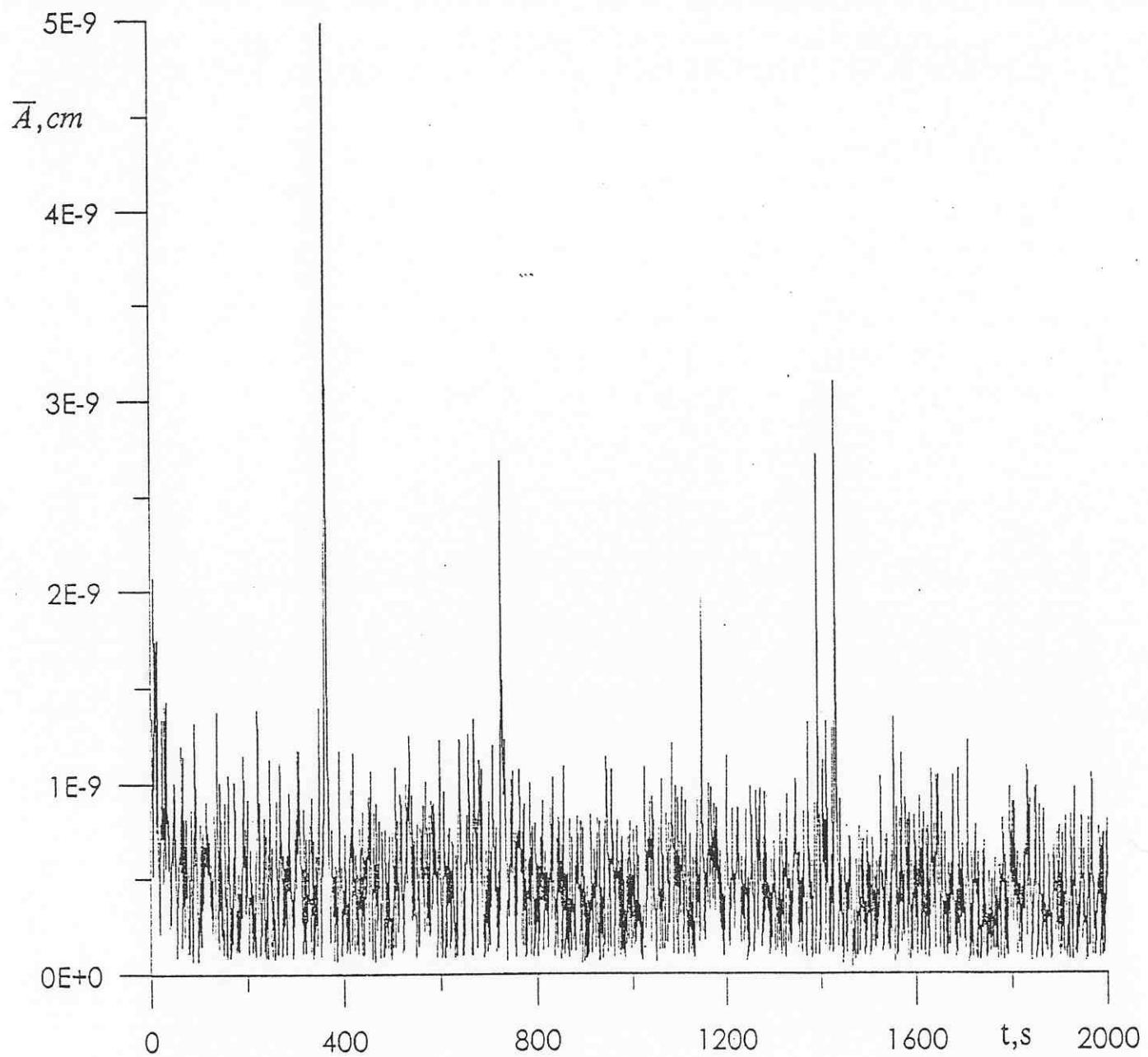


Fig. 5. Fragment of the record of noise oscillation of tungsten wire

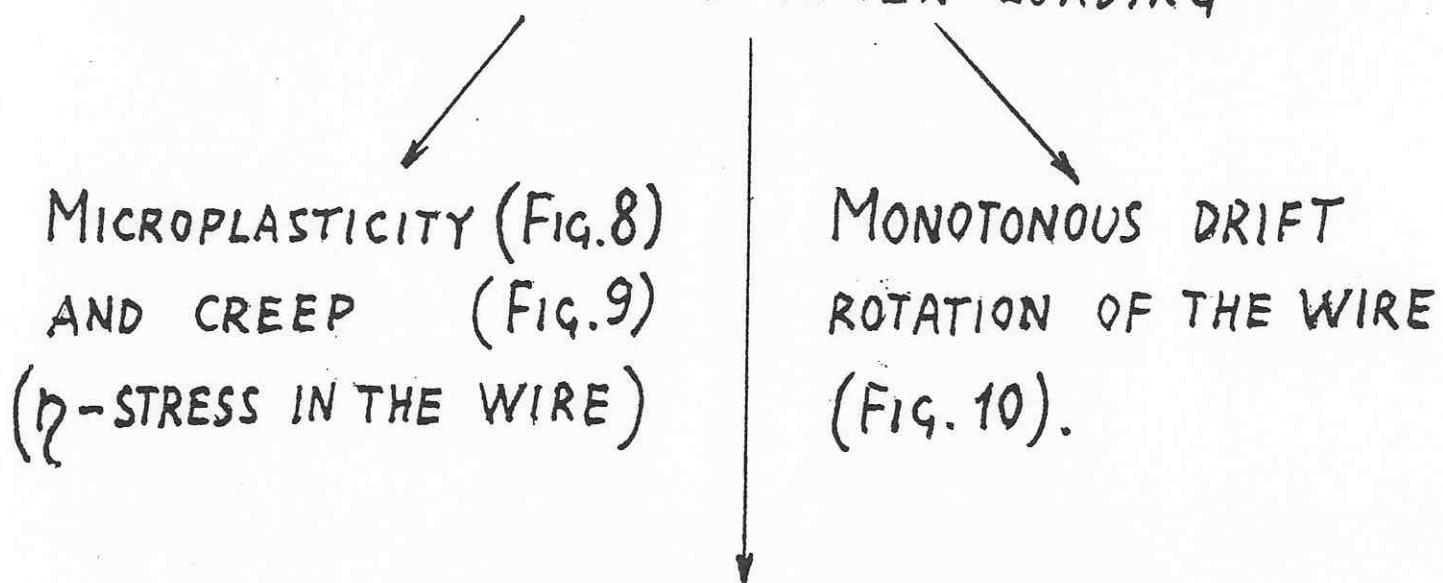
TABLE 1

Number of amplitude peaks exceeded the given level

	Low stressed sample $\eta=0.5\eta_B$	Well stressed sample $\eta=0.9\eta_B$	Theory
$> 10 \sigma$	0	298	$\ll 1$
$> 9 \sigma$	0	341	$\ll 1$
$> 8 \sigma$	4	427	0.19
$> 7 \sigma$	11	563	3.2
$> 6 \sigma$	38	800	38
$> 5 \sigma$	270	1478	320
$> 4 \sigma$	1748	3848	1900

DISTORTIONAL MECHANICAL EFFECTS IN THE WIRE SUSPENSION

THE CHANGE OF DEFORMATION IN THE WIRE
DURING TIME PASSED AFTER LOADING



JUMPS OF THE ANGLE OF TORSION OF THE WIRE

$$\Delta\varphi \approx (5 \cdot 10^{-5} \div 3 \cdot 10^{-4}) \text{ RAD}$$

AT STRESSES $\eta \geq 0.7 \eta_{\text{BREAK}}$

THE MEAN RATE OF JUMPS $\sim 1(\text{HOUR})^{-1}$

POSSIBLE CONSEQUENCE: FOR $\Delta\varphi = 10^{-6} \text{ RAD}$ ($d = 10^{-2} \text{ CM}$)

THE CHANGE OF THE ANGLE OF THE TEST MASS $\sim 6 \cdot 10^{-14} \text{ RAD}$ FOR 10^{-2} S. ; IF THE LIGHT BEAM DISPLACEMENT $\Delta r \approx 5 \cdot 10^{-3} \text{ CM}$, THEN $\Delta L \approx 3 \cdot 10^{-16} \text{ CM}$

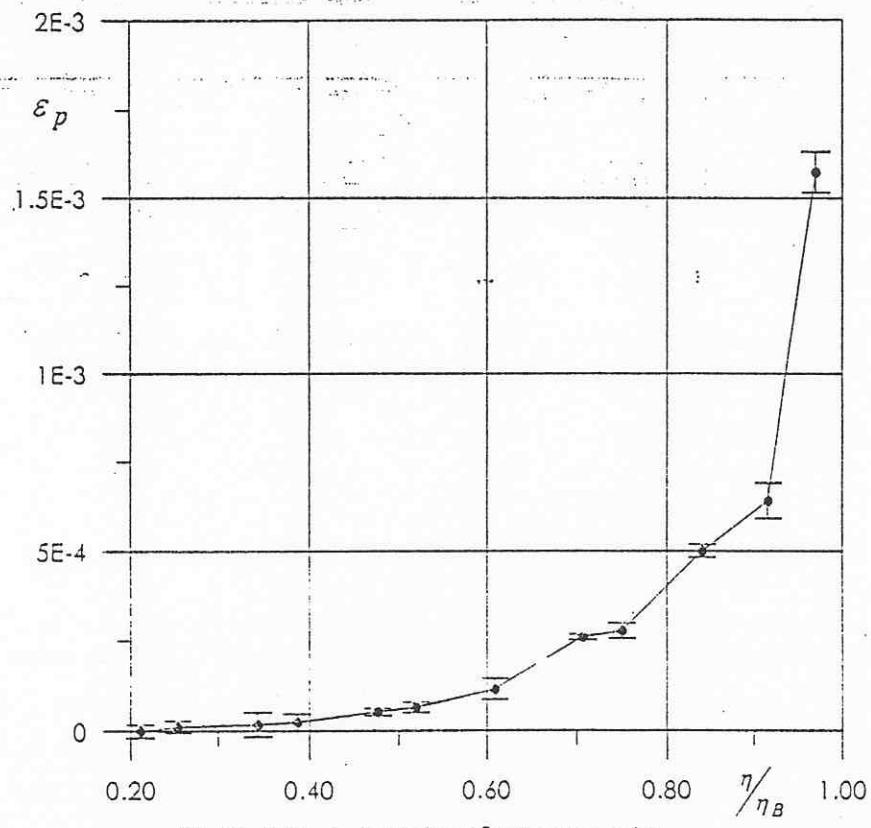


Fig.7 . Microplasticity of tungsten wire.

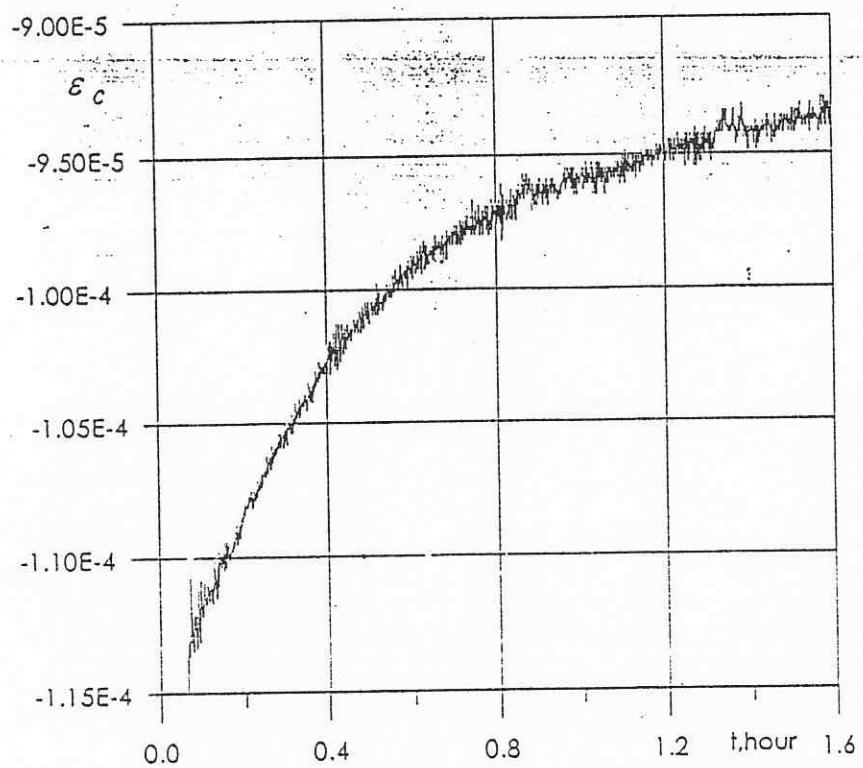


Fig. 8. Creep of tungsten wire (stress $\sigma = 0.35\sigma_b$).

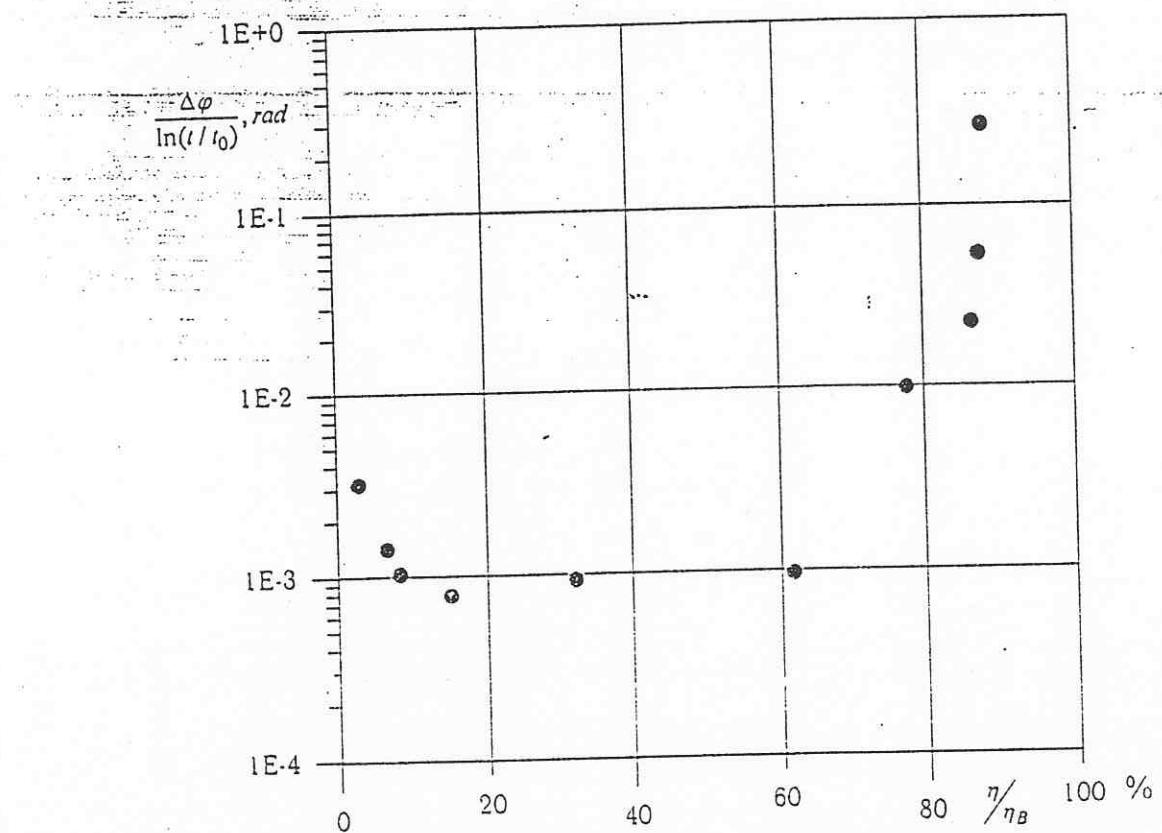
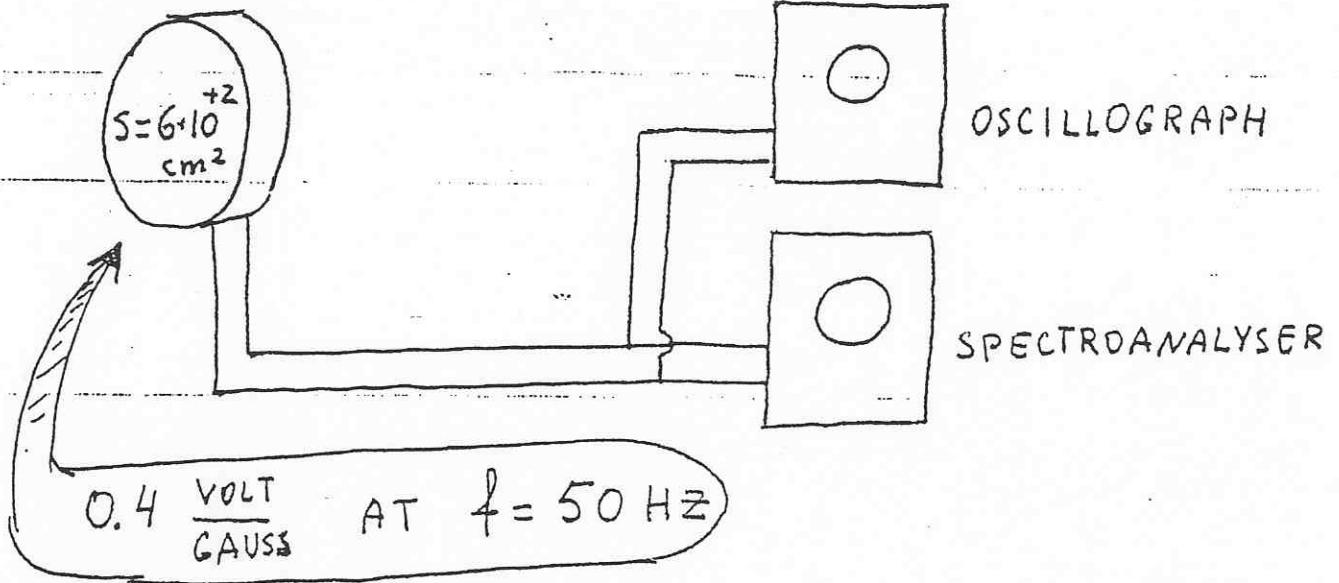


Fig. 9. Stress dependence of the angular drift rate of the wire.

THE FLUCTUATIONS AND THE CHANGES
OF THE MAGNETIC FIELD IN THE LABORATORY

$N = 250$



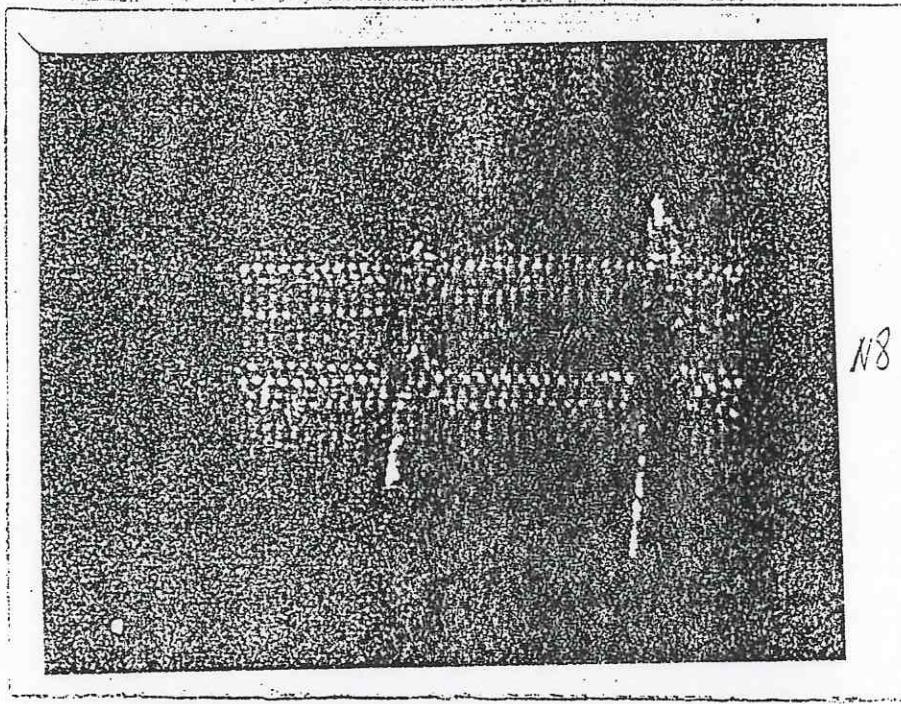
$$B_{\text{PICK-TO-PICK}} \simeq 3 \times 10^{-2} - 1 \times 10^{-4} \text{ GAUSS}$$

$$l_{\text{CHARACT}} \simeq 50 - 100 \text{ cm}$$

$$\Delta B_{\text{ON-OFF}} \simeq (1 - 2) \times 10^{-3} \text{ GAUSS} \Rightarrow \frac{400 \text{ WATT}}{3 \times 10^{+2} \text{ cm}}$$

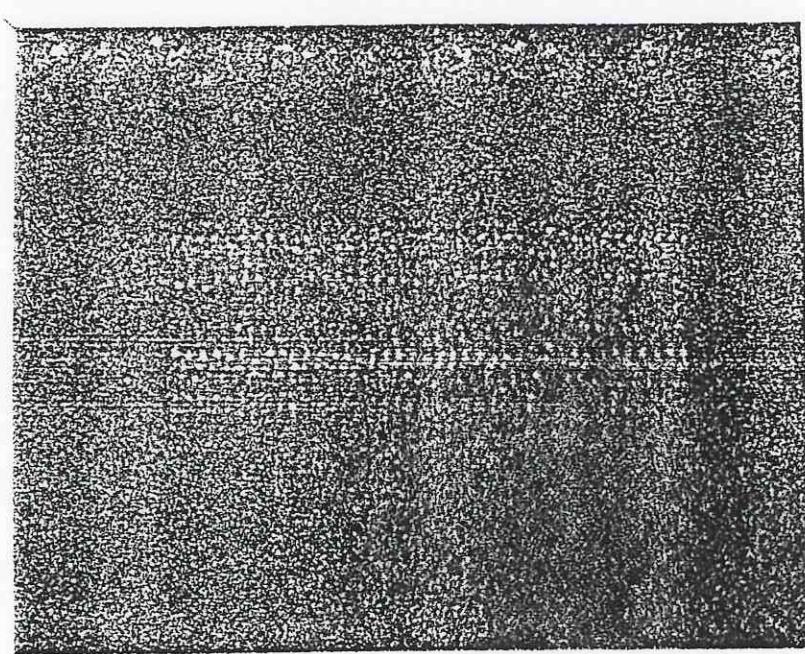
$$\Delta B_{\text{ON-OFF}} \simeq 3 \times 10^{-4} \text{ GAUSS} \Rightarrow \frac{100 \text{ WATT}}{3 \times 10^{+2} \text{ cm}}$$

$$\Delta T \simeq 10^{-2} \text{ sec.} - 10^{-3} \text{ sec}$$



N8

1.0 m gauss
0.1 m gauss



N9

1.0 m gauss
0.1 m gauss

60 - 70 Hz

$$B_f \simeq 1 \times 10^{-6} \frac{\text{GAUSS}}{\sqrt{\text{Hz}}}$$

80 - 90 Hz

$$B_f \simeq 4 \times 10^{-7} \frac{\text{GAUSS}}{\sqrt{\text{Hz}}}$$

110 - 120 Hz

$$B_f \simeq 2 \times 10^{-7} \frac{\text{GAUSS}}{\sqrt{\text{Hz}}}$$

$$\Delta X_f \simeq V_{\text{mAGN}} \times B_{\text{D.C.}} \times \frac{B_f}{l_{\text{CHAR}}} \times \frac{1}{m \cdot \omega^2}$$

$$\begin{array}{c} \uparrow \\ 3 \times 10^{-2} \text{ cm}^3 \end{array} \quad \begin{array}{c} \uparrow \\ 10^{+3} \text{ GAUSS} \end{array} \quad \begin{array}{c} \uparrow \\ 1.6 \times 10^{+3} \text{ GRAM} \end{array}$$

$$\Delta X_{65} \simeq 2 \times 10^{-15} \frac{\text{cm}}{\sqrt{\text{Hz}}}$$

$$\Delta X_{85} \simeq 5 \times 10^{-16} \frac{\text{cm}}{\sqrt{\text{Hz}}}$$

$$\Delta X_{115} \simeq 2 \times 10^{-16} \frac{\text{cm}}{\sqrt{\text{Hz}}}$$